Increase in the Wear Resistance of Polytetrafluoroethylene on Filling with Disperse Particles

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Abstract—The reduction in the wear of polytetrafluoroethylene (PTFE) in the presence of disperse filler is analyzed. It is found that decrease in the load on the PTFE matrix increases the mechanical component in the wear of PTFE composites. The calculations are confirmed by experiments on the wear of F4K15M5 com posite.

Keywords: wear resistance, wear rate, linear wear, polytetrafluoroethylene, composites, friction, polymers **DOI:** 10.3103/S1068798X15050214

Polytetrafluoroethylene (PTFE) is not generally used in frictional systems in its pure form, on account of its low wear resistance [1]. As a rule, polymers form films in friction at a metallic body, as a result of adhe sion [2]. In the wear of PTFE, transfer of the polymer is due to the destruction of its strip structure [3]. Indi vidual particles (length around 0.3 μm and width 20– 30 nm) fill the microdepressions on the metal surface and are partially transferred to the microprojections [4]. That reduces the frictional coefficient in the polymer–steel pair. The small adhering particles eventually form a layer, which, on reaching a specific size, are removed from the contact surface in the form of a wear particle. To reduce the wear, fillers are added to PTFE, with considerable increase in its wear resistance: the fillers slow the propagation of subsurface cracks and form a thin protective film [3, 5].

No theoretical explanation has been proposed for the sharp decrease in the wear of PTFE in the presence of disperse filler. In the present work, we analyze pos sible wear mechanisms with a view to developing a new approach to the creation of PTFE composites. On the basis of an empirical wear law, we propose an explana tion for the reduced wear of the PTFE in the presence of disperse filler.

We may write an empirical wear law in terms of the linear wear rate I_h [6]

$$
I_h = I_{hm} + I_{hm} = ap^2 v e^{-bp v} + cp(e^{dp v} - 1), \qquad (1)
$$

where I_{hm} and I_{hm} are the mechanochemical and mechanical components of the wear rate; *p* is the con tact pressure; v is the slip velocity; *a*, *b*, *c*, and *d* are constants.

A physical model of the wear of polymer compos ites was proposed in [6]. The model takes account of the structural parameters that affect the load distribution at the polymer–filler contact surfaces. For the sake of convenience, we write the linear wear rate of the PTFE matrix in the form $I_h^m = I_{hm}^m + I_{hm}^m$, where I_{hm}^m and I_{hm}^m are the mechanochemical and mechanical components of the wear rate for the matrix. For the filler, correspondingly, $I_h^f = I_{hmc}^f + I_{hm}^f$, where I_{hm}^f and I_{hm}^f are the mechanochemical and mechanical components of the wear rate for the filler. The dimensional coefficients in the empirical wear law are a_1, b_1, c_1, d_1 for the matrix and a_2, b_2, c_2, d_2 for the filler. We may now write the linear wear rate of the composite in the form

$$
I_{h\Sigma} = I_{hm}^m + I_{hm}^m + I_{hm}^f + I_{hm}^f
$$

= $\beta^2 p^2 v a_1 e^{-b_1 p v} + \beta p c_1 (e^{d_1 p v} - 1)$ (2)
+ $(1 - \beta)^2 p^2 v a_2 e^{-b_2 p v} + (1 - \beta) p c_2 (e^{d_2 p v} - 1),$

where β is the contact pressure on the matrix.

Polymer composites based on PTFE have the spa tial structure of the filler [7]. This may be attributed to the manufacturing process employed, which includes the mixing of the large granulated PTFE particles and smaller filler particles. The micronic filler particles form single-layer structures around the polymer parti cles in most cases. On introducing nanoparticles in the matrix, we obtain multilayer filler structures [8]. In the present work, we analyze the wear resistance of a com posite with a single-layer filler structure.

Consider a model composite with a PTFE matrix containing micronic particles of AG-600 SO5 carbon filler [9]. To simplify the calculation of the load distri bution, we assume that the polymer and filler particles

Fig. 1. Dependence of the wear rate *Ih* of PTFE on the pressure *p* when $β_1 = 1, β_2 = 0.0165$, and $ν = 1$ m/s. In Eq. (2), $a_1 = 1.1 \times 10^{-11}$, $b_1 = 6.2$, $c_1 = 3 \times 10^{-12}$, and $d_1 = 0.2$.

are cubic (with faces r_p and r_f , respectively). In that case [7]

$$
\beta = \frac{(1 - 0.7 \varphi_f)}{(1 - 0.7 \varphi_f) + 0.7 \chi \varphi_{SK} \varphi_f},
$$
(3)

where φ_f is the filler concentration; $\chi = E_m/E_{f0}$; E_m is the elastic modulus of the matrix; E_{f0} is the resistance to deformation for the wall of the structure formed by the filler; φ_{SK} is the defect content of the wall.

For the wall of the structure formed by the filler, the defect content is [7]

$$
\varphi_{SK} = \frac{\varphi_f r_p}{3(1 - \varphi_f) r_f}.\tag{4}
$$

We assume the following dimensions: $r_p = 63 \text{ }\mu\text{m}$; r_f = 7 μ m. We also assume that E_m = 0.45 GPa and \dot{E}_{f0} = 220 GPa, in accordance with [10].

When using micronic filler particles, as a rule, the optimal value of φ_f is around 20 vol % [11]. In that case, according to Eqs. (4) and (3), $\beta = 0.0165$.

To better understand the relative importance of the wear resistance of the filler particles and PTFE matrix in the composite and to determine the coefficients in the empirical wear law, we test PTFE and AG-600 SO5 carbon filler samples by the method in [12]. We select a steel 45 counterbody (State Standard GOST 1050–88). In Fig. 1, we show the pressure dependence of the lin ear wear rate according to the sum of the first two terms in Eq. (2) for pure PTFE (with $\beta = 1$) and for PTFE in the model composite (with $\beta = 0.165$).

Fig. 2. Dependence of components I_{hm}^m and I_{hm}^m of the wear rate of PTFE on the pressure *at various values of* $β$ *,* when $v = 1$ m/s, according to the first and second terms in Eq. (2). As in Fig. 1, $a_1 = 1.1 \times 10^{-11}$, $b_1 = 6.2$, $c_1 = 3 \times$ 10^{-12} , and $d_1 = 0.2$.

Analysis shows that the linear wear rate of PTFE in the composite is about two orders of magnitude less than for pure PTFE. In addition, the dependence of *Ih* on *p* increases monotonically when $\beta = 0.0165$ but has a section with relatively stable I_h when $\beta = 1$ [7, 9]. For detailed analysis of this discrepancy, we plot the first and second terms in Eq. (2) individually.

When PTFE forms part of the composite, the amplitude I_{hm}^m is sharply less (by more than three orders of magnitude) than for pure PTFE. This is probably associated with the low rate of film formation on the counterbody, on account of the low load on the PTFE in the composite. In addition, it follows from Fig. 2 that the mechanical component of wear signifi cantly predominates in the working load range for PTFE in the composite, while the mechanochemical component predominates for pure PTFE.

In Fig. 3, we show the pressure dependence of the linear wear rate for AG-600 SO5 carbon filler accord ing to the sum of the third and fourth terms in Eq. (2).

To obtain the dependence of *Ih*Σ on the pressure *p* on the basis of Eq. (2), it is incorrect to use curve *1* in Fig. 3, since the structure formed by the filler is not in contact with the counterbody during friction but with the transfer layer. We need to correct the coefficients of the mechanochemical wear component in the empiri cal wear law obtained from experimental data for AG-600 SO5 carbon filler so as to take account that it is present in the composite, where $\beta = 0.0165$.

The literature indicates that, in the wear of PTFE composites on steel, material is transferred to the counterbody from both the matrix and the filler [4].

Fig. 3. Dependence of components I_{hm}^f and I_{hm}^f of the wear rate of AG-600 SO5 carbon filler on the pressure *p*, when $\beta = 0$ (1, 2) and 0.0165(3) and $v = 1$ m/s, according to the third and fourth terms in Eq. (2): (1, 2) $a_2 = 1.3 \times$ 10^{-13} , $b_2 = 0.65$, $c_2 = 5.4 \times 10^{-15}$, and $d_2 = 0.12$; (3) $a_2 =$ 1.3×10^{-13} , $b_2 = b_1 = 6.2$.

This has been attributed to the remote action of the surfaces. Thus, the interaction force of the surfaces is unchanged over gaps of a few hundred nm, according to [13]. When a hard surface is covered with a screen ing polymer film, action over a distance of 100– 150 nm is possible [14]. Since the thickness of the pri mary particles transferred to the steel is 20–30 nm, the presence of the film of transferred material may be dis regarded in considering the transfer of filler to the counterbody, on account of remote action. However, in the presence of screening polymer films at the solid surface, the influence of the temperature of such films must be taken into account. We know that the adhesive strength of PTFE at steel depends considerably on the temperature [4]. In certain conditions, this may present the transfer of filler to the counterbody. There-

fore, in the I_{hm}^f term in Eq. (2), the coefficient b_2 , which takes account of the influence of temperature on the wear, must be assumed equal to b_1 . Then, the refined pressure dependence of the mechanochemical wear component for filler in the composite, corre sponding to the third term in Eq. (2), takes the form of curve *3* in Fig. 3. Thus, the resulting curve is shifted to the *p*v range where wear of the matrix is also observed.

To verify our analysis of the contributions of the filler and matrix to the total wear of the PTFE com posite, we compare the results based on Eq. (2) for the model composite consisting of PTFE and AG-600 SO5 carbon filler (20 vol %) with the experimental pressure dependence of the linear wear rate for the wear of industrial F4K15M5 composite, in which the

Fig. 4. Dependence of the total wear rate *Ih*Σ on the pres sure p , when $v = 1$ m/s: (1) for the model composite consisting of PTFE and AG-600 SO5 carbon filler ($a_2 = 1.3 \times$ 10^{-13} , $b_2 = b_1 = 6.2$, $c_2 = 5.4 \times 10^{-15}$, and $d_2 = 0.12$; (2) F4K15M5 industrial composite ($a = 1.5 \times 10^{-13}$, $b = 1.5 \times 10^{-13}$ 5.5, $c = 4 \times 10^{-14}$, $d = 0.125$).

filler is foundry coke. For F4K15M5 composite, $a =$ 150×10^{-15} , $b = 515$, $c = 4 \times 10^{-14}$, and $d = 0.125$. Note that b is close to the b_1 value obtained from experimental data for the wear of PTFE at steel 45.

In Fig. 4, we plot the dependence of *Ih*Σ on the pres sure *p* when $v = 1$ m/s according to Eqs. (1) and (2) for F4K15M5 composite and the model composite. Curves *1* and *2* in Fig. 4 are close. The difference is due to the tribological properties of the foundry coke and AG-600 SO5 carbon and the discrepancy between the particle sizes r_p and r_f assumed for the industrial composite and their actual values.

Note that the satisfactory agreement of curves *1* and *2* in Fig. 4 validates the proposed approach to assessing the contributions of the filler and matrix to the total wear resistance of the composite. That allows us to predict the wear resistance of PTFE composites at the design stage on the basis of analytical formulas.

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Translated by Bernard Gilbert