



OPEN

Reliable smart models for estimating frictional pressure drop in two-phase condensation through smooth channels of varying sizes

M. A. Moradkhani¹, S. H. Hosseini¹✉, Mengjie Song^{2,3}✉ & A. Abbaszadeh⁴

Reliable and comprehensive predictive tools for the frictional pressure drop (FPD) are of particular importance for systems involving two-phase flow condensation. However, the available models are only applicable to specific operating conditions and channel sizes. Thus, this study aims at developing universal models to estimate the FPD during condensation inside smooth mini/micro and conventional (macro) channels. An extensive databank, comprising 8037 experimental samples and 23 working fluids from 50 reliable sources, was prepared to achieve this target. A comprehensive investigation on the literature models reflected the fact that all of them are associated with high deviations, and their average absolute relative errors (AAREs) exceed 26%. Hence, after identifying the most effective input variables through the Spearman's correlation analysis, three soft-computing paradigms, i.e., multilayer perceptron (MLP), gaussian process regression (GPR) and radial basis function (RBF) were employed to establish intelligent and dimensionless predictive tools for the FPD based on the separated model suggested by Lockhart and Martinelli. Among them, the most accurate results were presented by the GPR approach with AARE and R^2 values of 4.10%, 99.23% respectively, in the testing step. The truthfulness and applicability of the models were explored through an array of statistical and visual analyses, and the results affirmed the obvious superiority of the newly proposed approaches over the literature correlations. Furthermore, the novel predictive tools excellently described the physical variations of the condensation FPD versus the operating parameters. Ultimately, the order of importance of factors in controlling the condensation FPD was clarified by a sensitivity analysis.

Keywords Frictional pressure drop, Condensation, Machine learning algorithms, Modeling, Mini/micro and conventional channels

List of symbols

Bo	Bond number, $Bo = gD^2(\rho_l - \rho_v)/\sigma$
C	Chisholm parameter
D	Channel hydraulic diameter (m)
Fr_v	Vapor Froude number, $Fr_v = G^2/\rho_v^2gD$
f	Friction factor, defined by Eq. (7)
G	Total mass flux ($\text{kg/m}^2 \text{ s}$)
g	Acceleration due to gravity (m/s^2)
k	Thermal conductivity (W/m k)
La	Laplace number
P_c	Critical pressure (Pa)
P_s	Saturated pressure (Pa)

¹Department of Chemical Engineering, Ilam University, Ilam 69315-516, Iran. ²Department of Energy and Power Engineering, School of Mechanical Engineering, Beijing Institute of Technology, Beijing 100081, China. ³School of Mechanical Engineering, Hanyang University, Seoul 04763, South Korea. ⁴Department of Civil Engineering, Ilam University, Ilam 69315-516, Iran. ✉email: s.h.hosseini@ilam.ac.ir; mengjie.song@bit.edu.cn

P_{red}	Reduced pressure, $P_{red} = P_s/P_c$
R	Phases' density ratio, $R = (\rho_l - \rho_v)/\rho_l$
Re_l	Liquid Reynolds number, $Re_l = G(1-x)D/\mu_l$
Re_{lo}	Liquid only Reynolds number, $Re_{lo} = GD/\mu_l$
Re_v	Vapor Reynolds number, $Re_v = GxD/\mu_v$
Re_{vo}	Vapor only Reynolds number, $Re_{vo} = GD/\mu_v (-)$
Su_{vo}	Vapor only Suratman number, $Su_{vo} = \rho\sigma D/\mu^2$
T	Temperature (°C)
V	Velocity (m/s)
We_{vo}	Vapor only Weber number, $We = G^2D_h/\rho_v\sigma$
X	Lockhart–Martinelli parameter, defined by Eq. (9)
x	Vapor quality

Greek

μ	Dynamic viscosity (Pa s)
ρ	Density (kg/m ³)
\sum	Mathematical symbol for summation
σ	Surface tension (N/m)
ϕ	Two-phase multiplier
ν	Kinematic viscosity (m ² /s)

Subscripts

A	Acceleration
F	Frictional
G	Gravitation
l	Liquid
lo	Liquid only
red	Reduced
s	Saturated
v	Vapor
vo	Vapor only
tp	Two-phase

Heat exchangers involving two-phase flow condensation are widely utilized in numerous industries, such as nuclear, food processing, refrigeration, air conditioning, etc^{1–6}. During the last years, compact heat exchangers (mini/micro channels) have attracted special attention, since they provide much higher energy efficiency, require lower amounts of refrigerant, and take up less space compared to conventional ones^{7–9}. However, the enhanced flow area per mass velocity in these channels leads to higher pressure drop, which is followed by several negative impacts on the system¹⁰. Increasing the energy consumption in pumps and obstructing the two-phase flow system are well-known examples regarding the destructive influences of high pressure drop inside channels^{11–13}. Consequently, the optimal design of heat exchangers necessitates comprehensive predictive tools for pressure drop in channels of various sizes.

The total pressure drop during two-phase flow inside channels is defined as the sum of three different terms, including frictional, gravitational and accelerational pressure drops,

$$\left(\frac{dP}{dz}\right)_{tp,T} = \left(\frac{dP}{dz}\right)_{tp,F} + \left(\frac{dP}{dz}\right)_{tp,G} + \left(\frac{dP}{dz}\right)_{tp,A} \quad (1)$$

The gravitational and acceleration terms are given by,

$$\left(\frac{dP}{dz}\right)_{tp,A} = G^2 \frac{d}{dz} \left[\frac{(1-x)^2}{\rho_l(1-\alpha)} + \frac{x^2}{\rho_v\alpha} \right] \quad (2)$$

$$\left(\frac{dP}{dz}\right)_{tp,G} = g[\rho_l(1-\alpha) + \rho_v\alpha] \sin(\omega) \quad (3)$$

where the void fraction, α is calculated by Eq. (4) proposed by Zivi¹⁴.

$$\alpha = \left[\left(\frac{\rho_v}{\rho_l}\right)^{\frac{2}{3}} \left(\frac{1-x}{x}\right) + 1 \right]^{-1} \quad (4)$$

According to earlier experimental investigations, the frictional pressure drop (FPD) envelopes more than 90% of total pressure drop^{15–18}. Thereupon, it is vital to derive precise and reliable approaches for estimating the two-phase FPD during condensation, covering both mini/micro and macro channels.

Several theoretical and empirical correlations can be found in the literature for predicting the FPD during condensation inside channels^{19–29}. Most of these correlations have been established by inspiring the homogeneous and separated models. The homogenous approach supposes the flow as a pseudo single-phase flow, in which both phases have the same velocity. However, the earlier studies have implied that this method is not reliable for low pressures and mass fluxes^{30,31}. In the separated model presented by Lockhart and Martinelli³², FPD is determined by modification of the single-phase pressure drop by applying a two-phase multiplier,

$$\left(\frac{dP}{dz}\right)_{tp,F} = \left(\frac{dP}{dz}\right)_{vo} \phi_{vo}^2 = \left(\frac{dP}{dz}\right)_{lo} \phi_{lo}^2 = \left(\frac{dP}{dz}\right)_v \phi_v^2 = \left(\frac{dP}{dz}\right)_l \phi_l^2 \quad (5)$$

where the pressure drop of various phases are defined as follows,

$$\left(\frac{dp}{dz}\right)_l = \frac{2(1-x)^2 G^2 f_l}{D \rho_l} \quad (6a)$$

$$\left(\frac{dp}{dz}\right)_v = \frac{2x^2 G^2 f_v}{D \rho_v} \quad (6b)$$

$$\left(\frac{dp}{dz}\right)_{lo} = \frac{2G^2 f_{lo}}{D \rho_l} \quad (6c)$$

$$\left(\frac{dp}{dz}\right)_{vo} = \frac{2G^2 f_{vo}}{D \rho_v} \quad (6d)$$

where the friction factor, f for a given phase (k) can be calculated by,

$$f_k = \frac{16}{Re_k} \text{ for } Re_k < 2000 \quad (7a)$$

$$f_k = \frac{0.079}{Re_k^{0.25}} \text{ for } 2000 \leq Re_k < 20,000 \quad (7b)$$

$$f_k = \frac{0.046}{Re_k^{0.2}} \text{ for } Re_k \geq 20,000 \quad (7c)$$

According to Chisholm³³ theory, the two-phase multiplier is defined as function of Lockhart and Martinelli parameter, i.e., X ,

$$\phi_l^2 = \frac{1}{X^2} + \frac{C}{X} + 1 \quad (8)$$

$$X = \sqrt{\left(\frac{dp}{dz}\right)_l / \left(\frac{dp}{dz}\right)_v} \quad (9)$$

According to this methodology, the Chisholm parameter, C may experience alternations from 5 to 20, depending on the flow regime. Sun and Mishima³⁴ developed a correlation for FPD in mini/micro channels based on Chisholm method, in which Re_l and $Re_v = 2000$ were defined as the transition point for Chisholm parameter. An AARE value of 30.6% for all analyzed data was yielded by this model. Hossain et al³⁵. utilized their own experimental data for correlating the Chisholm parameter with Froude and Bond Numbers. The reasonable consistencies between measured data and those predicted by the correlation were testified with AARE of 9.51%. Jige et al³⁶. estimated the two-phase multiplier using their experimental data for condensation of four different refrigerants in multiport mini-channels, and obtained a reasonable accuracy with AARE of 9.6%.

Various FPD models have also been proposed based on extensive sets of experimental data. However, most of these models are applicable for specific channel sizes. The Friedel³⁷ model was developed by employing 25,000 data for macro channels with diameters higher than 4 mm and working fluids of air-oil, R12, and air-water. Another comprehensive model was suggested by Muller-Steinhagen and Heck³⁸ utilizing around 9300 data points for channel diameters exceeding 4 mm. Although the database encompassed various types of working fluids, the number of data corresponding to air-water was remarkably larger than the rests. Kim and Mudawar³⁹ proposed an universal predictive tool for condensation FPD in mini/micro channels with diameters up to 6.25 mm. In order to attain more accurate predictions, the analyzed data were allocated to four subdomains, depending on the vapor and liquid Reynolds numbers. A total number of 7115 data points analyzed for the condensation FPD were predicted with an AARE of 23.3% by this model. There is just one correlation in the literature applicable for both mini/micro and conventional channels, which has been presented by Moradkhani et al⁴⁰. This correlation was derived by implementing the genetic programming approach over a widespread databank, containing 7328 FPD samples, and exhibited satisfactory outcomes for different sizes of channels.

Recently, intelligent techniques, such as machine learning algorithms have been broadly employed to solve the engineering problems^{41–48}. There some limited studies on the application of the foregoing approaches to model the two-phase pressure drop in heat exchangers of various configuration. Zendehboudi and Li⁴⁹ studied the application of four intelligent methods in estimating the condensation FPD in inclined tubes based on 312 data from just one source. All established models provided reliable estimations with R^2 values exceeding 95% for the test data. López-belchí et al⁵⁰. employed the group method of data handling (GMDH) approach to estimate the condensation FPD inside micro-channels based on their own measured data, and yielded a total AARE of 10.59%. Longo et al⁵¹. developed a predictive method for two-phase pressure drop in plate heat exchangers through the gradient boosting machines (GBM). The modeling phase was performed by utilizing 925 data for flow condensation and 1624 data for flow boiling. It was demonstrated that the intelligent approaches perform much better than the empirical correlations. In another relevant work, Qiu et al¹³. suggested several intelligent models for boiling FPD in mini/micro channels using 2787 data from 21 studies. Among them, the MLP based model, which included 12 hidden layers and 23 dimensionless groups as inputs provided the superior outcomes. Moradkhani et al⁵². assessed some neural network-based approaches for estimating the two-phase FPD inside helically coiled tubes. Their analyzed dataset included 1267 experimental samples adopted from 12 studies. The most precise outcomes were obtained by the radial basis function (RBF) method with AARE of 4.73% during the testing phase. More recently, Montañez-barrera et al⁵³. studied the application of correlated-informed neural networks (CoINN) for modeling the FPD of zeotropic mixtures during two-phase flow in micro-channels. The model overcame the available correlations with a total AARE of 6% from the actual data.

This study addresses critical limitations in two-phase FPD estimation during condensation. Existing correlations are often limited to specific channel sizes, and the predictive tools applicable to both mini/micro and conventional channels are rare. On the other hand, the machine learning approaches haven't been explored for developing generalizable models for condensation FPD inside smooth channels of various sizes. Furthermore, identifying the key factors influencing FPD is crucial for engineers. To address these gaps, this study analyzes a comprehensive dataset of 8037 samples encompassing mini/micro and conventional channels from 50 published studies. For the first time, three important soft computing approaches, namely, MLP, GPR and RBF are implemented to establish novel models for FPD estimation. The correctness and validity of the proposed models are statistically assessed for estimating the FPD in various channel sizes, flow patterns, and flow regimes. Furthermore, the outputs of the models are utilized to examine the changes in condensation FPD with respect to different operational factors. A sensitivity analysis is then used to identify the most effective factors on FPD.

Materials and methods

Machine learning algorithms

To address the need for robust predictive tools in two-phase FPD estimation during condensation in channels, this study employed three widely recognized machine learning algorithms: MLP, GPR, and RBF. These approaches were chosen due to their success in describing complex two-phase flow behavior, as demonstrated in previous studies^{52,54–56}.

MLP

The capable machine learning approach of MLP follows a process similar to that observed in the nervous system of humans, and it is mainly implemented to solve the complicated mathematical problems, including approximation, classification and pattern recognition⁵⁷. This is done through a parallel algorithm, in which a set of data is utilized to train the network, and the artificial neurons are responsible for transferring the information⁵⁸. Figure 1 shows the structure of an artificial neuron included in the MLP network. Mathematically, this neuron is explained as follow,

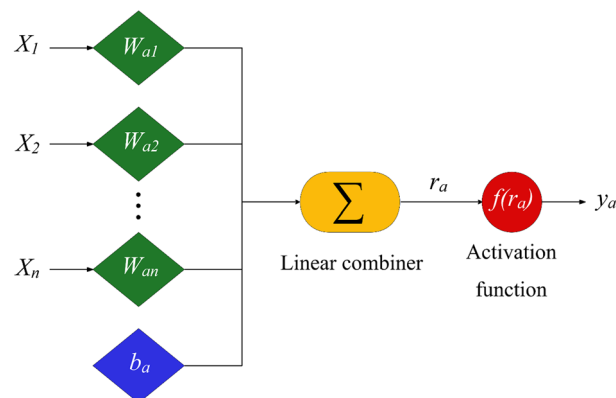


Figure 1. Description of an artificial neuron.

$$r_a = \sum_{i=1}^n x_i W_{ai} + b_a \quad (10)$$

$$y_a = f(r_a) \quad (11)$$

where r_a , x_i , W_{ai} , b_a and stand for linear combiner, i th input factor, synaptic weight, neuron bias and activation function, respectively.

MLP is recognized as feed-forward network, meaning it processes the information only in one direction. The graphical description regarding the MLP network designed to model the condensation FPD has been illustrated in Fig. 2. It is clear that it entails three linked layers, including some artificial neurons. The input, hidden and output layers are responsible for introducing the information to the network, specify network parameters and display the outcomes, respectively. It should be emphasized that the architecture of hidden layer depends on the complexity of problems, and may include some independent layers with different numbers of neurons in each of them. While, the number of neurons in the first and last layers equal to the number of input and output variables, respectively. The MLP network detects the system nonlinearity via the activation functions included in the hidden layer's neurons.

This study identified a four-hidden layer MLP network with (40, 30, 20, 10) neurons per layer as the optimal architecture for modeling condensation FPD. The number of neurons in input and output layers are 8 and 1, respectively. Moreover, the neurons benefited from the tan-sigmoid function as activation function⁵⁹.

A vital stage in designing the MLP network is optimizing its weights and biases, as these parameters significantly influence the network's performance. For this purpose, the back propagation (BP) algorithm specifies the weights and biases to minimize a deviation function⁶⁰,

$$D = (y_{pre} - y_{exp})^2 \quad (12)$$

After applying each training data, the value of deviation function is spread in the network, followed by re-adjusting the parameters. Herein, the Bayesian regularization approach was used to train the BP algorithm.

GPR

Recently, due to robustness and capabilities of non-parametric machine learning approaches, trends toward them in engineering application have been increased. Among the foregoing approaches, GPR is a widely used machine learning algorithm, by which a gaussian joint probability distribution is provided^{61,62}. The main preponderances of GPR are high accuracy and strength to modulate the hyper-parameters. This method is also capable to catch the uncertainty of analyzed samples.

The GPR-based learning process is accomplished through a probabilistic framework, in which a training dataset is provided. It should be noted that and stand for the input variables vector and target function, respectively. Thus, the predictive model provides the output function in any point through the following approximation⁵⁴,

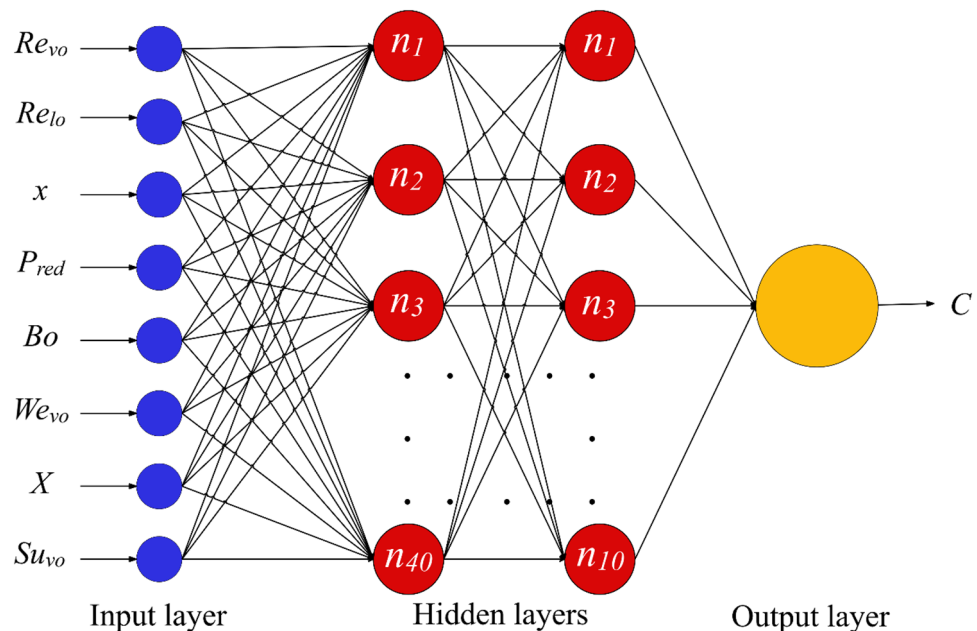


Figure 2. The MLP network designed to estimate the condensation FPD.

$$y_i = L(x_i) + \varepsilon_i \quad (13)$$

where $L(x_i)$ is the latent function corresponding to the input variables (x_i), and its values constitutes a random variable. Furthermore, ε_i represents the gaussian noise, which its mean and variance are 0 and σ_n^2 , respectively⁶³,

$$\varepsilon_i = N(0, \sigma_n^2) \quad (14)$$

Consequently, the target function can be easily approximated by defining a mean function $m(x)$ and a covariance function $cov(x, x')$. The predictive probability distribution for the input variables, x^* can be defined as,

$$\hat{y}^* = m(x^*) + k_*^T (K + \sigma_n^2 I)^{-1} (y - m(x^*)) \quad (15)$$

$$\sigma_{y^*}^2 = k_* + \sigma_n^2 - k_*^T (K + \sigma_n^2 I)^{-1} k_* \quad (16)$$

where k_* may be defined as $[k_*]_i = cov(x_i, x^*)$, K stands for a covariance matrix, which its elements are $[K]_{i,j} = cov(x_i, x_j)$, and I shows the identity matrix.

As the predictive probability distribution is specified by the hyperparameters, an optimization process should be performed in order to determine these factors^{64,65}. In the GPR approach, the log-likelihood function is maximized during the training step in order to calculate the hyperparameters,

$$\log p(y|X) = -\frac{1}{2} y^T (K + \sigma_n^2 I)^{-1} y - \frac{1}{2} \log (|(K + \sigma_n^2 I)|) - \frac{n}{2} \log(2\pi) \quad (17)$$

where n represents the number of training data points.

RBF

Resolving the drawbacks of MLP networks, such as their empirical structure and under or over-fitting possibility is the motivation behind the development of RBF networks. The high potency for interpolation, prompt convergence, simple structure and sublime reliability are the main advantages of these networks⁶⁶. The structure of this network is similar to that of a single-layer MLP network, which includes a number of neurons equal to the number of data points used for training. It should be noted that, the activation functions employed in these networks are radial basis functions, among which the most well-known function is gaussian,

$$H(x) = \exp\left(-\frac{(x-c)^2}{r^2}\right) \quad (18)$$

where c and r denote the center and radius of the gaussian function, respectively.

After receiving the information from the input layer, the hidden layer exert the nonlinear functions and sends the results to the final layer, which is responsible to combine the weights and the activation functions, linearly^{67,68},

$$f(x) = \sum_{i=1}^n W_i H_i(x) \quad (19)$$

The gradient descent method is employed in the RBF network to optimize the network center and synaptic weights, minimizing the mean squared error (MSE),

$$MSE = \frac{1}{n} \sum_{i=1}^n (y_{pre} - y_{exp})^2 \quad (20)$$

The RBF network used in this study has an 8-6430-1 architecture, signifying 8 neurons in the input layer, 6430 neurons in the hidden layer, and 1 neuron in the output layer. Additionally, the hidden layer employs a Gaussian function as its activation function.

Experimental data gathering

Since the availability of credible data plays a major role in designing the data-driven models, the collection of as much data as possible regarding the condensation FPD inside channels built the primary cornerstone of the present communication. Hence, a comprehensive dataset, encompassing 8037 FPD samples was adopted from 50 published studies. The foregoing data envelop the FPD of 23 fluids, such as chemicals, halocarbons, natural refrigerants, water, hydrocarbons, cryogenes, etc., condensing inside both mini/micro and conventional channels. A detailed description of the operating conditions of the analyzed sources has been provided in Table 1. It should be expressed that the REFPROP v.9.1 software⁶⁹ (<https://www.nist.gov/programs-projects/reference-fluid-thermodynamic-and-transport-properties-database-refprop>) has been employed to determine the thermophysical characteristics of the fluids under saturation condition.

Error analysis

In this study, the correctness of various models for estimating the condensation FPD was assayed based on three statistical metrics, including average absolute relative error (AARE), standard deviation (SD), and coefficient of determination,

Sources	Fluid	Channel geometry	Hydraulic diameter (mm)	Reduced pressure (-)	Mass flux(k)	Number of points
70	R717, R744, R245fa	Circular	1.02	0.07–0.69	100–440	238
71	R410A	Circular	0.76–3.05	0.8–0.9	200–800	291
72	R134a	Circular	8.1	0.25–0.32	300–500	40
73,74	R32, R245fa	Circular	0.96	0.07–0.43	200–1000	63
75,76	R134a, R1234yf	Circular	2.14	0.14–0.23	50–200	61
77	R134a, R1234yf	Square	1.16	0.25–0.49	470–710	81
78	R32, R410A	Square	1.16	0.33–0.63	350–800	250
79	R290	Square	1.16	0.25–0.40	175–350	109
80	R125, R22, R32, R134a, R236ea, R410A	Circular	8	0.1–0.56	100–750	151
81,82	R245fa	Circular	3	0.13–0.53	100–1000	174
83,84	R134a	Circular, Rectangular, Triangular, Square, Barrel	0.424–1.524	0.30	150–750	561
85	R1234yf, R134a	Circular	0.96–2	0.17–0.32	200–800	286
86	R290, R600a	Circular	1	0.10–0.22	240–480	147
87	R744	Rectangular	0.1–0.16	0.69–0.87	400–800	175
88	R1234yf	Circular	3.2–8	0.17–0.23	200–400	162
89,90	R744	Circular	1.5	0.41–0.54	400–1000	95
36	R32, R1234ze(E), R134a, R410A	Circular	0.85–1.1	0.35–0.78	100–500	284
91	R744	Circular	5.15	0.36–0.54	600–1000	52
92	R404A	Circular	0.508–3.05	0.38–0.77	200–800	392
93	R410A	Circular	6.3	0.56	100–250	12
15–18	R152a, R290, R1234ze(E), R22,	Circular, Square	0.952–1.152	0.2–0.4	200–800	293
94–96	R290, R1270, R404A, R32, R410A, R1234yf, R1234ze(E), R134a, R152a,	Circular	4	0.15–0.49	75–800	707
97,98	R290	Circular	7.75–14.45	0.25–0.95	150–450	276
99,100	R601, R245fa	Circular	7.75–14.45	0.03–0.17	100–600	274
101,102	R410A	Circular	6.2–9.4	0.80–0.90	200–800	450
103	R718	Circular	19	0.18–0.45	400–1000	113
104	R134a, R410A	Circular	1.02–1.54	0.10–0.30	50–300	407
105	R22, R290, R32, R410A	Rectangular	0.83	0.37–0.60	50–500	112
106	R728	Circular	1–2	0.31	32.7–262	58
107	R134a	Circular	3.8297–8.91	0.22–0.29	450–650	113
108,109	R14, R170	Circular	4	0.27–0.8	200–650	155
110	R1234yf, R1234ze(E), R134a, R600a	Square, Circular, Triangular	0.835–1.1	0.11–0.31	100–1400	746
111	R1234ze(E), R134a	Circular	1.88	0.17–0.22	450–900	281
112	R290	Circular	1.22	0.09–0.14	40–90	30
113,114	R50, R170	Circular	4	0.21–0.65	99–251	398
Total			0.1–19	0.03–0.95	32.7–1400	8037

Table 1. Geometrical and operating conditions of analyzed sources for condensation FPD.

$$AARE(\%) = \frac{1}{N} \sum \left| \frac{FPD_{exp} - FPD_{pre}}{FPD_{exp}} \right| \times 100 \quad (21)$$

$$SD(\%) = \sqrt{\frac{\sum (E_i - \bar{E}_i)^2}{N - 1}} \times 100 \quad (22)$$

$$R^2(\%) = \left(1 - \frac{\sum (FPD_{exp} - FPD_{pre})^2}{\sum (FPD_{exp} - \overline{FPD_{exp}})^2} \right) \times 100 \quad (23)$$

where E_i is relative error that can be defined as the following equation,

$$E_i = \frac{FPD_{exp} - FPD_{pre}}{FPD_{exp}} \quad (24)$$

Results and discussion

Accuracy of literature models

As discussed earlier, several predictive tools can be found in the open literature concerning the estimation of condensation FPD inside channels. A statistical investigation on the accuracy of eleven well-known FPD models based on the collected databank has been presented in Table 2. It is clear that the highest precision among these models belongs to the correlation suggested by Moradkhani et al.⁴⁰ with AARE, SD and R^2 values of 26.23%, 40.23% and 89.05%, respectively from the experimental data. Since the foregoing model has been established based on a widespread FPD dataset enveloping a broad range of conditions in both conventional and mini/micro channels, it is reasonable to provide the highest level of precision. The models presented by Muller-Steinhagen and Heck³⁸, Jige et al.³⁶, Kim and Mudawar³⁹, and Sun and Mishima³⁴ present almost same results with AAREs of 30.10%, 31.75%, 31.85% and 32.31%, respectively. Such deviations stem from the fact that the mentioned correlations have been recommended for limited geometrical or operating conditions. Nevertheless, their estimation capabilities are superior to those of Friedel³⁷, Gu et al.¹¹⁵, Hossain et al.³⁵, Zhang et al.¹¹⁶, and Koyama et al.¹¹⁷, models, which show AARE values between 34.85 and 61.74%. The weakest performance in predicting the condensation FPD can be attributed to correlation presented by Chisholm³³ with AARE and SD values of 95.28% and 114.65%, respectively. Overall, the current analysis testifies the requirement for designing comprehensive and precise models to estimate the two-phase FPD during flow condensation in mini/micro and conventional channels.

The novel predictive tools for condensation FPD

Optimization of the input variables defined in the models

This study introduces novel predictive approaches for estimating condensation FPD based on the separated model proposed by Lockhart and Martinelli³² and the dimensionless form suggested by Chisholm³³. These intelligent approaches are utilized to estimate the Chisholm parameter in Eq. (8). However, before developing the new models, it is crucial to identify the most important input factors. To meet this requirement, the relevancies between Chisholm parameter and 16 common dimensionless groups were measured through the Spearman's correlation coefficients¹¹⁸, and the corresponding results have been plotted in the heatmap of Fig. 3. As it is clear, the vapor and liquid-only Reynolds number, i.e., Re_{vo} and Re_{lo} have remarkable impacts on Chisholm parameter. However, there is no obvious correlation between liquid-phase Reynolds number, Re_l and C . While the vapor-phase Reynolds number, Re_v exhibits relatively high influence of Chisholm parameter, its correlations with Re_{vo} and Re_{lo} are extremely great. A similar argument can be made for the liquid and vapor-phases friction factors, f_l and f_v , which have significant correlations with Re_{vo} and Re_{lo} . Almost all foregone experimental investigations have demonstrated that the vapor quality, x , and reduced pressure, P_{red} , play substantial roles in controlling the condensation FPD. This fact is also affirmed by the Spearman's correlation coefficient. Therefore, these factors should be considered as the models' inputs. Since the collected data include the FPD during condensation inside both mini/micro and conventional channels, the surface tension can noticeably affect the FPD. This is why the Bond number, Bo shows a fairly high correlation with Chisholm parameter. The Suratman number, Su_{vo} , Webber number, We_{vo} and Lockhart and Martinelli parameter, X also can be included among the input factors, as they make unique and evident influences on C . In contrast, the impact of liquid-phase Prandtl number on Chisholm parameter is negligible. Furthermore, since the phases relative density, R , and the Prandtl number of the vapor phase show strong correlations with reduced pressure, they should not be incorporated as inputs to the models.

According to the above discussions, the following eight dimensionless groups were taken into account as the optimized input parameters in order to model the condensation FPD in mini/micro and conventional channels,

$$C = f(Re_{lo}, Re_{vo}, x, P_{red}, Bo, We_{vo}, X, Su_{vo}) \quad (25)$$

The newly proposed models are applicable within a wide range of conditions, as shown by the minimum and maximum values of dimensionless groups in Table 3. This broad applicability reflects the extensive data bank

Models	AARE, (%)	SD, (%)	R^2 , (%)
Chisholm ³³	95.28	114.65	84.62
Hossain et al. ³⁵	51.18	26.49	31.36
Sun and Mishima ³⁴	32.31	35.19	67.11
Zhang et al. ¹¹⁶	60.88	91.15	42.36
Koyama et al. ¹¹⁷	61.74	91.17	52.17
Muller-Steinhagen and Heck ³⁸	30.10	38.48	59.42
Jige et al. ³⁶	31.75	43.68	59.33
Friedel ³⁷	34.85	50.42	58.10
Kim and Mudawar ³⁹	31.85	41.42	52.17
Gu et al. ¹¹⁵	38.15	44.19	43.28
Moradkhani et al. ⁴⁰	26.23	40.23	89.05

Table 2. Assessment of the literature FPD models based on the collected experimental databank.

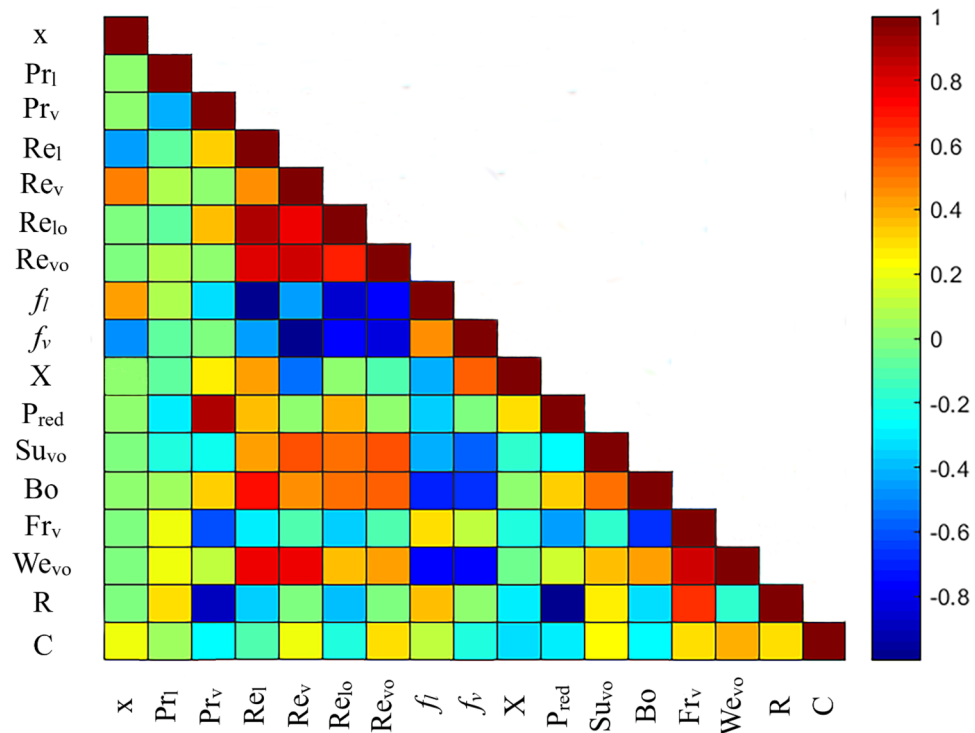


Figure 3. The Spearman's correlation coefficient between different dimensionless groups and Chisholm parameter.

Dimensionless factor	Minimum value	Maximum value
Re_{lo}	327.84	232,506.9
Re_{vo}	1984.42	1,089,325
x	0.01	0.994
P_{red}	0.033	0.952
Bo	0.03	2732.708
We_{vo}	5.09	175,880.70
X	0.01	19.84
Su_{vo}	33,022.37	34,485,016

Table 3. Analyzed ranges of dimensionless groups included in the novel models.

used in this study. Since machine learning models are often sensitive to input conditions, these models offer reliable predictions for condensation FPD within the ranges summarized in Table 3.

Development of the novel models

After determining the input variables, the soft computing approaches of GPR, MLP and RBF were enforced to derive the novel dimensionless models for predicting the condensation FPD based on the Chisholm method. This study utilized the Model-Based Calibration 5.2.1 toolbox within MATLAB to achieve this objective. This toolbox provided the necessary functionalities for designing the predictive models. Notably, the predictive tools were first trained on 80% (6,430 samples) of the randomly chosen data bank. The remaining 20% (1,607 samples) were then used to rigorously evaluate the models' reliability. The precisions of the proposed smart models during the training and testing stages have been competitively assessed in Table 4. As can be seen, the GPR-based model presents much exact estimations with AAREs of 3.77% and 4.10%, R^2 values of 99.01% and 99.23%, and SD values of 7.25% and 6.04% for train and test data, respectively. Such excellent outcomes corroborate the strength and truthfulness of this approach to describe the condensation FPD in mini/micro and conventional channels. Furthermore, it can be concluded that the optimized input variables introduced in Eq. (25) have been appropriately chosen and satisfy the influences of various factors on the condensation FPD. The MLP model also provide satisfactory outcomes, and ranks second in terms of accuracy with AARE and R^2 values of 12.46% and 97.56%, respectively in the testing phase. In addition, its predictions capabilities are adequately better those

Statistical parameters	Train, (6430 data)			Test, (1607 data)			Total, (8037 data)		
	GPR	MLP	RBF	GPR	MLP	RBF	GPR	MLP	RBF
AARE (%)	3.77	6.70	0.03	4.10	12.46	21.04	3.83	7.86	4.23
R ² (%)	99.01	97.60	100	99.23	97.56	95.15	99.05	97.59	99.17
SD (%)	7.25	10.50	0.39	6.04	78.12	39.00	7.74	36.53	17.70

Table 4. Error metrics corresponding to the novel predictive tools for the condensation FPD.

that of the literature correlations. Despite giving the best precision in the training step, the RBF model exhibits fairly great deviations for the test data points with an AARE of 21.04%. Consequently, this model cannot be assumed as a capable predictive tool. Overall, although all newly proposed models perform much better than the literature correlations in describing the condensation FPD, the one designed based on the GPR method is the best choice for this purpose. This can be justified by the fact that the GPR approach is widely recognized as a non-parametric regression tool. Unlike methods that require a pre-defined form for the data, GPR builds the model's form based on information extracted from the training samples. This flexibility allows the GPR approach to adapt to the unique characteristics and complexities of the data, resulting in improved predictive accuracy. The performance of the novel correlations has been compared with the top earlier correlations in Fig. 4, by plotting the estimated values of FPD versus the experimental data. The figure reflects the fact that the outcomes of the intelligent approaches, especially GPR, are impressively closer to the diagonal (best fit) line, affirming the obvious superiority of this model over the literature correlations. Among the literature correlation, the results obtained by the correlation proposed by Moradkhani et al.⁴⁰ have the best consistencies with experimental values of FPD. The rest of the correlations have almost same predictions capabilities, and this fact was also asserted by the statistical assessments provided in Table 2. Altogether, a great number of data estimated by the previous models lie beyond the satisfactory range (± 30 error margin), and the proposed model provide considerable advances in this area.

Assessing the validity of the newly proposed models

To further authenticate the truthfulness of the models proposed in the present study for estimating the condensations FPD inside channels, two visual accuracy analyses are carried out in this section by employing the cumulative frequency and contour diagrams.

A comparison between the reliability of the predictive approaches designed in the current study and the top models among those presented in the previous studies from the perspective of cumulative frequency has been rendered in Fig. 5. It should be expressed that the cumulative frequency measures the percentage of samples, which have been predicted within a given extent of relative error. Thus, a sharp growth in the cumulative frequency curve of a predictive tool at low levels of relative error represents its high accuracy. This observation is true regarding the novel intelligent models, since they show very high cumulative frequencies at the beginning points of its curve. Among them, the GPR approach estimates 50.78%, 78.11%, 91.45%, 96.11% and 97.92% of the whole data within the error bounds of 2%, 5%, 10%, 15% and 20%, respectively. This is why it can be recognized as the most trustful predictive tools for the condensation FPD inside channels. While the remaining intelligent models also present better results compared to literature correlations, their precisions are fairly less than the GPR model. The correlation suggested by Moradkhani et al.⁴⁰ has the best performance in terms of cumulative frequency among those available in the literature, and it is the only case that predicts the majority of samples (56.59% of all data) with relative errors less than 20%. The cumulative frequencies of the Jige et al.³⁶, Muller-Steinhagen and Heck³⁸, Kim and Mudawar³⁹ and Sun and Mishima³⁴ models at the absolute relative error of 20% are 45.56%, 43.06%, 40.00% and 35.49%, respectively, which means that applying them to estimate the FPD is followed by relatively high deviations. As a results, the present analysis is another verification regarding the supremacy of the GPR model for describing the condensation FPD.

The dispersion of the relative deviations yielded by the GPR approach for describing the condensation FPD in diverse ranges of flow mass flux, channel diameter, vapor quality and reduced pressure has been represented in the contour plot of Fig. 6. The spectrum of colors differing between dark green and dark blue expresses the AARE values obtained by the model in a given range of condition. As it is obvious, the predominant portion of all diagrams have been encompassed by green colors, implying the fact that the majority of FPD data in various ranges of conditions have been estimated with relative errors up to 6%. On the other hand, several small blue regions are visible in the contour plots, which are generally corresponding to the condensation of near-critical fluids with mass fluxes of around 800 kg.m⁻².s⁻¹ inside small-sized channels. Some of these minor deviations arise not only from the performance of the model, but also from the errors occurring in the FPD experimental measurements under the foregoing situations. Overall, the current results reflect the fact that the GPR model benefits from very high confidence levels in predicting the FPD during condensation inside channels.

The prediction capabilities of the models

In order to better figure out the universality and sufficiency of the presented models, in this section, the precision of the new models to estimate the condensation FPD inside channels of different sizes as well as under various flow patterns and flow regimes is investigated, and the results are compared with the top literature correlation. Furthermore, the variations of condensation FPD versus the operating parameters are studied based on the outcomes of the model.

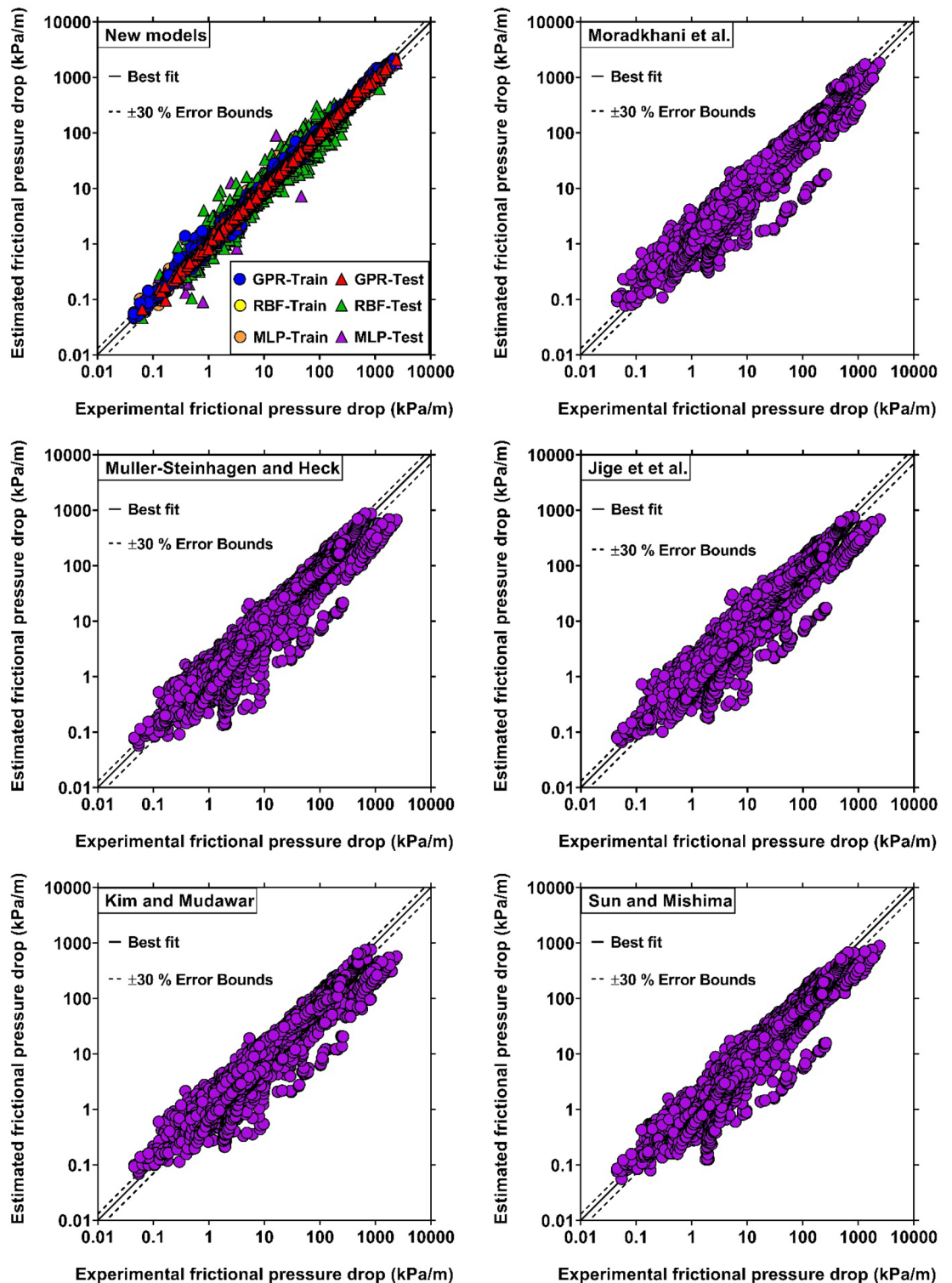


Figure 4. Comparison between the FPD values obtained by experimental studies and those predicted by the novel and literature models.

Predicting the condensation FPD inside channels of various sizes

Several criteria have been proposed to classify the channels of heat exchangers based on their size. The most prevalent one is that obtained by Kandlikar^{119,120}, in which the channels can be splitted into three main categories:

- Micro-channels with $D \leq 0.2$ mm
- Mini-channels with $0.2 < D \leq 3$ mm
- Conventional channels with $D > 3$ mm

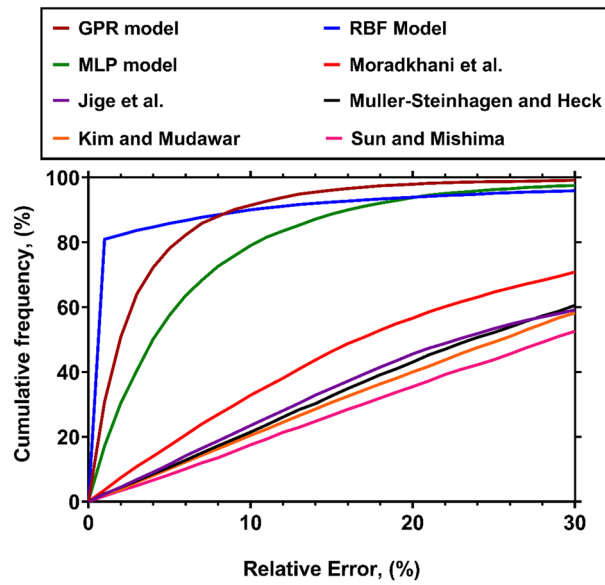


Figure 5. The cumulative frequency of data predicted by different FPD models at various levels of relative error.

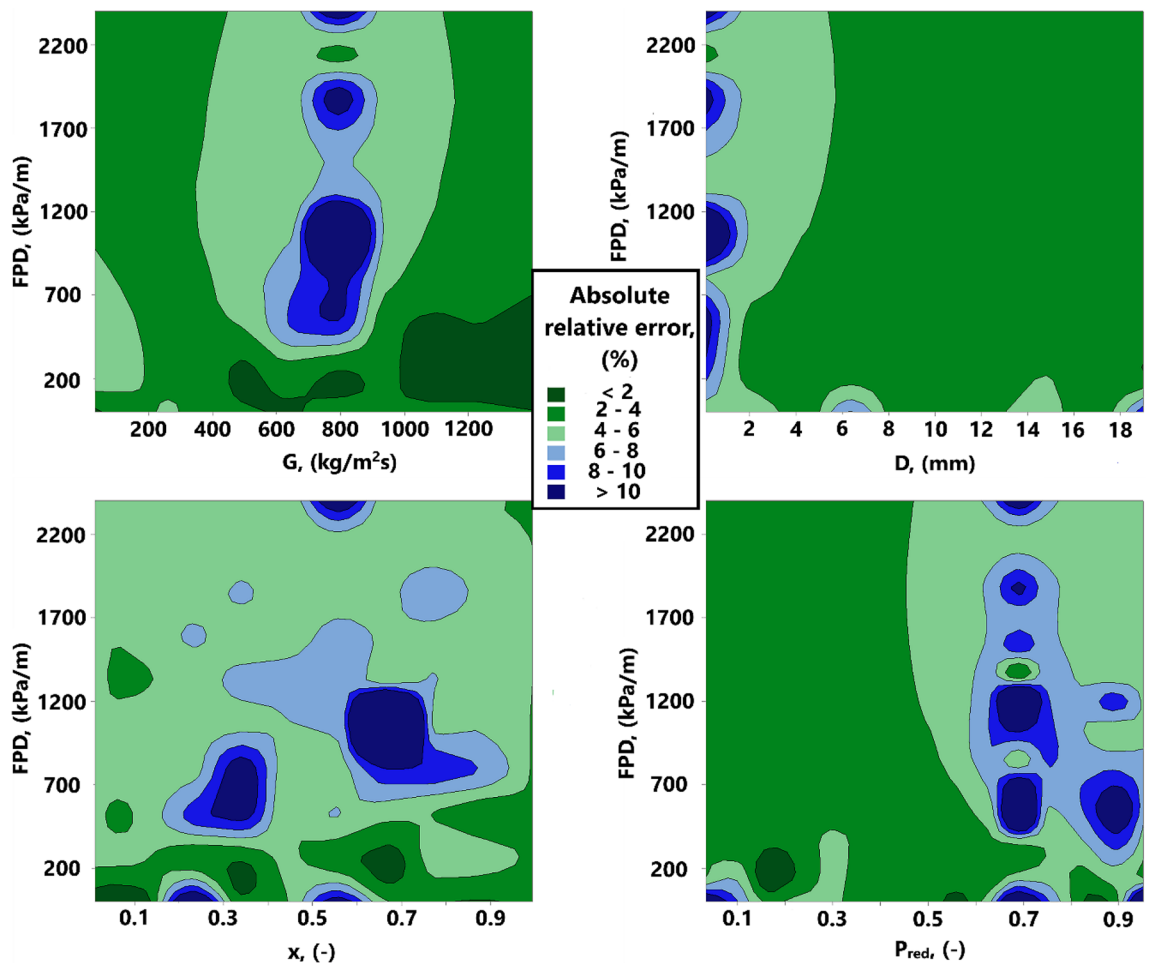


Figure 6. Dispersion of relative deviations given by the GPR approach for estimating the condensation FPD in diverse ranges of conditions.

By applying the foregoing classifications, the AAREs of the FPD models for the data corresponding to various sizes of channels have been illustrated in Fig. 7. The superiority of the GPR for all three cases, including micro, mini and conventional channels is obvious with AARE values of 9.74%, 4.09% and 3.13%, respectively. Such excellent results admit the applicability and reliability of this model to predict the condensation FPD in a broad range of channel sizes. The RBF and MLP models are ranked second and third from the standpoint of accuracy, since their AARE values for various channel sizes are less than 15%, and they can be used as proper alternatives to GPR approach. In contrast, the literature correlations exhibit fairly larger deviations for all types of channels, and this fact is more evident in the cases of micro and conventional channels, where all of them give AARE values exceeding 30%. Nevertheless, all of them provide relatively proper estimations for the condensation FPD inside mini-channels with AARE values of 22.56%, 24.44%, 24.69%, 24.69% and 28.35 obtained by Moradkhani et al⁴⁰, Muller-Steinhagen and Heck³⁸, Kim and Mudawar³⁹, Jige et al³⁶. and Sun and Mishima³⁴ correlations, respectively. Accordingly, it can be concluded that the newly proposed models, especially GPR, are unique predictive tools with applicability for all sizes of channels.

Applicability for various flow patterns

Another perceptible way to make a fair judgement regarding the performance of the models is to ascertain their accuracy level for estimating the condensation FPD at various flow patterns. Herein, the foregoing scrutiny was performed by inspiring the method proposed by Kim et al¹²¹, according to which four diverse flow patterns may be observed during condensation within channels. These flow patterns are distinguished from each other by three boundary lines defined based on the values of the Lockhart and Martinelli parameter, X_{tt} and the modified Weber number, W^* . Figure 8 depicts the dispersion of the FPD data at all four flow patterns by determining the relevant boundary lines. As it is evident, although the wavy-annular and transition flows embody the majority of data, there are adequate numbers of samples at all flow patterns to make a comprehensive assessment on the accuracy of the models.

Figure 9 portrays the AARE values yielded by the novel and earlier models for estimating the experimental data pertinent to different flow patterns. The figure clearly expresses that the highest levels of accuracy for all cases are belonged to the recently proposed GPR model. It should be noted that the AARE values obtained by this model range from 2.67% to 7.23%, which corroborates its truthfulness and potential for estimating the condensation FPD estimation in various flow patterns. The RBF model also exhibits good consistencies with the data for all flow patterns, and has AARE values below 10% for all cases. While the performance of the MLP model is superior to the literature models, the uncertainties of this model, especially for slug and bubbly flow, are higher than the other intelligent models. The predictions performed by the literature correlations are fraught with deviations, especially when the flow pattern is slug and bubbly or transitions. However, for the remaining two cases, i.e., wavy and smooth annular flows, their outcomes have slightly better consistencies with the experimental data, and the corresponding AARE values are below 30%. The most accurate predictions in the

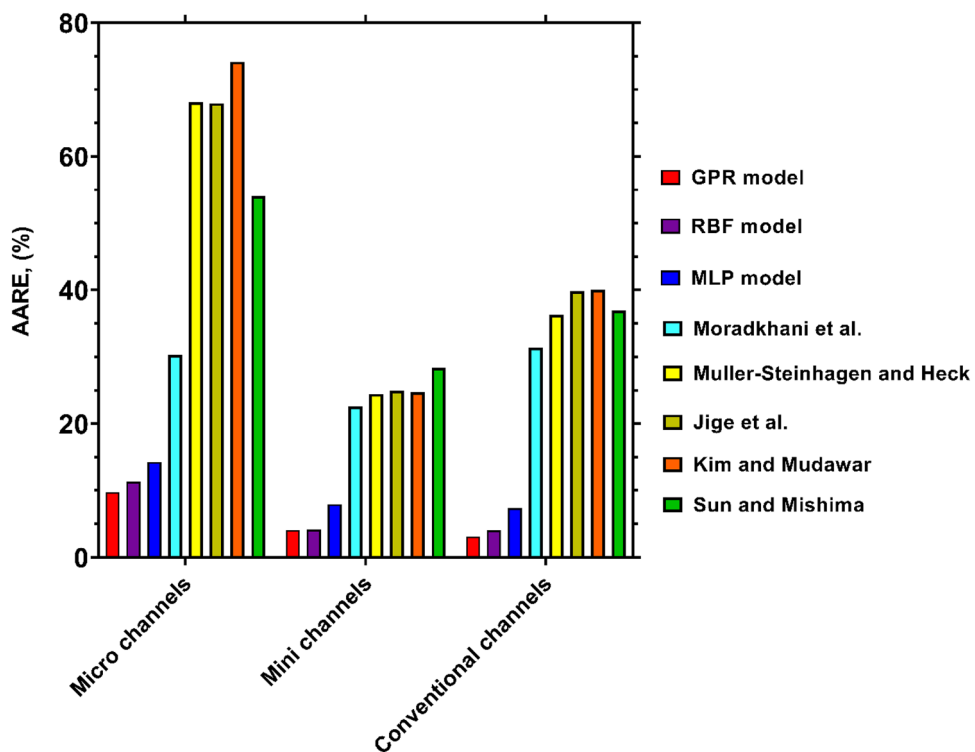


Figure 7. Accuracy of the novel and literature models for predicting the condensation FPD inside channels of various sizes.

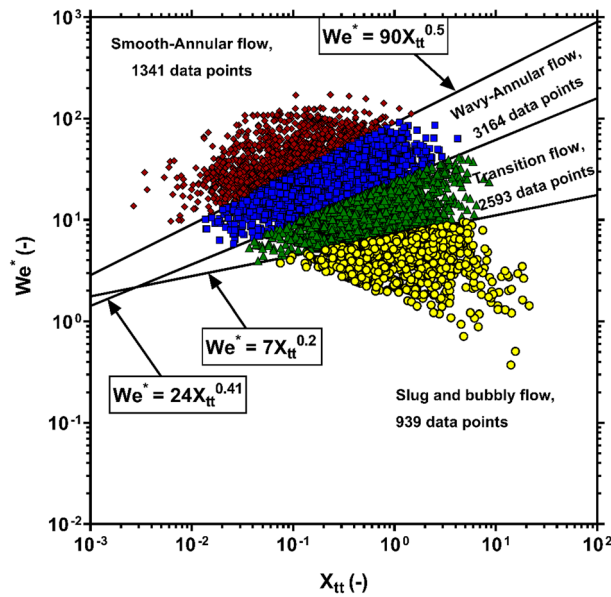


Figure 8. Dispersion of the analyzed FPD data in different flow patterns.

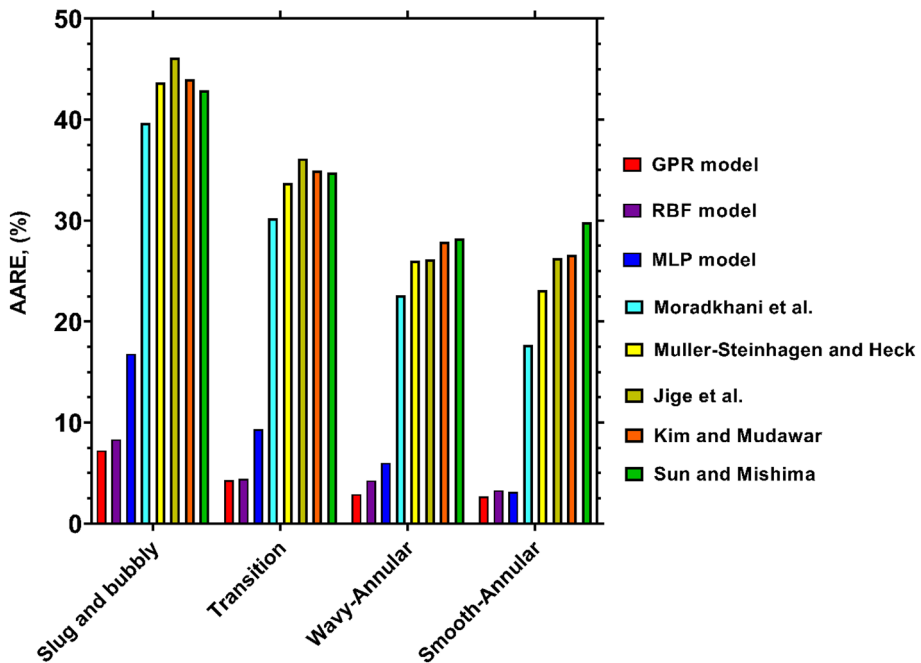


Figure 9. Accuracy of the novel and literature models for predicting the condensation FPD at various flow patterns.

foregoing flow patterns have been presented by the Moradkhani et al⁴⁰, correlation with AAREs of 22.59% and 17.68%, respectively. Hence, employing the novel models, especially GPR, results in noticeable improvements in predicting the condensation FPD at all flow patterns.

Applicability for various flow regimes

Several benchmarks have been proposed to diagnose the flow regime during condensation in channels¹²²⁻¹²⁴. Some methods consider the values of single-phase Reynolds numbers, i.e., Re_l and Re_v , as the basis of regime classification. While, in the rest of approaches, the flow regime is specified based on the liquid-film Reynolds number, which is defined as follow,

$$Re_{IF} = \frac{GD(1-x)(1-e)}{\mu_l} = Re_l(1-e) \quad (20)$$

where e stands for the entrained liquid fraction that can be calculated by the empirical correlation developed by Cioncolini and Thome¹²⁵. Based on the above definition, Cioncolini et al¹²⁶, presented the following criteria for distinguishing between laminar, transitional and turbulent flow regimes,

- $Re_{IF} \leq 160$: Laminar flow
- $160 < Re_{IF} < 2785$: Transitional flow
- $Re_{IF} \geq 2785$: Turbulent flow

Figure 10 demonstrates the scattering of the collected data at different flow regimes based on the aforementioned criteria. It is clear that the bulk of the FPD data points are included in the transitional and turbulent regimes, while the laminar regime embodies just around 3% of the entire data.

Figure 11 compares the AARE values of different model in estimating the condensation FPD at different flow regimes. A glance at the results brings into view the decisive superiority of the GPR model over the literature correlations. The AARE values achieved by this predictive tool for various flow regimes are in the range of 3.06% and 5.06%, which is another confirmation of its universality and wide applicability. The RBF and MLP models are the next reliable predictive tool in this regard with AARE values below 10% for different flow regimes. Accordingly, the recent models provide enough accurate estimations at all two-phase flow regimes. As opposed to these approaches, the values calculated by the literature correlations are associated with relatively great errors, particularly for the turbulent flow. Although the Moradkhani et al⁴⁰, correlation provides satisfactory outcomes for transitional flow, its AARE values for the other cases exceed 30%. It is worthy to note that with AARE values below 30%, the predictions of the models proposed by Jige et al³⁶, Muller-Steinhagen and Heck³⁸, and Kim and Mudawar³⁹ satisfactorily match the experimental data pertinent to transitional flow. The same explanation applies to the predictions of Jige et al³⁶ for the transitional flow. Thereupon, the foregoing correlations can be regarded as substitutes for the novel model in the corresponding flow regimes. Overall, except for the recently proposed models, none of the existing predictive tools are capable to precisely describe the condensation FPD in all flow regimes.

Capturing the physical attitudes of the condensation FPD

For exploring the potency of the recently established model, i.e., GPR to capture the physical trends, in the following, its outcomes are employed to study the variations of the condensation FPD versus the operating parameters. In order to further highlight the excellence this approach over the literature correlations, the predictions of the Moradkhani et al⁴⁰, correlation are also included in the foregoing assessments.

Figure 12 delineates the variations of the condensation FPD with vapor quality and mass flux, during R1270 flowing within a 4 mm tube, when the reduced pressure is held constant at of 0.32. Obviously, the growth of vapor quality and mass flux lead to notable enhancements in the condensation FPD. This trend can be justified by amplifying the vapor velocity and shear stress arisen from the foregoing changes. While both predictive tools properly capture the overall trends, the recently proposed model are in closer conformities with the real data.

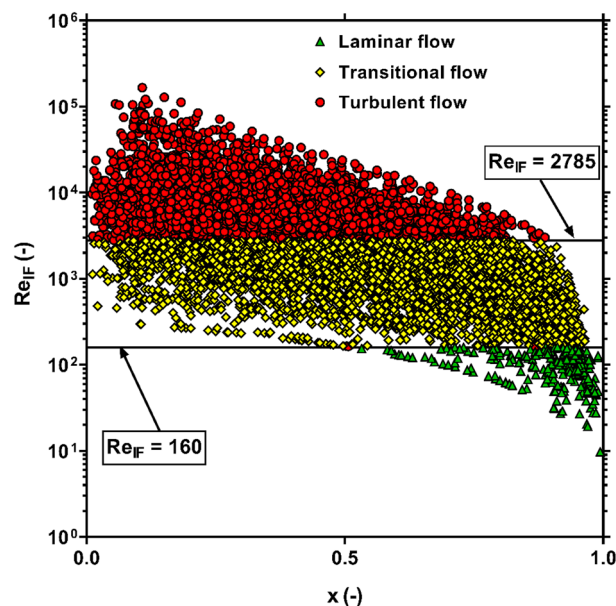


Figure 10. The scattering of the FPD data in various flow regimes.

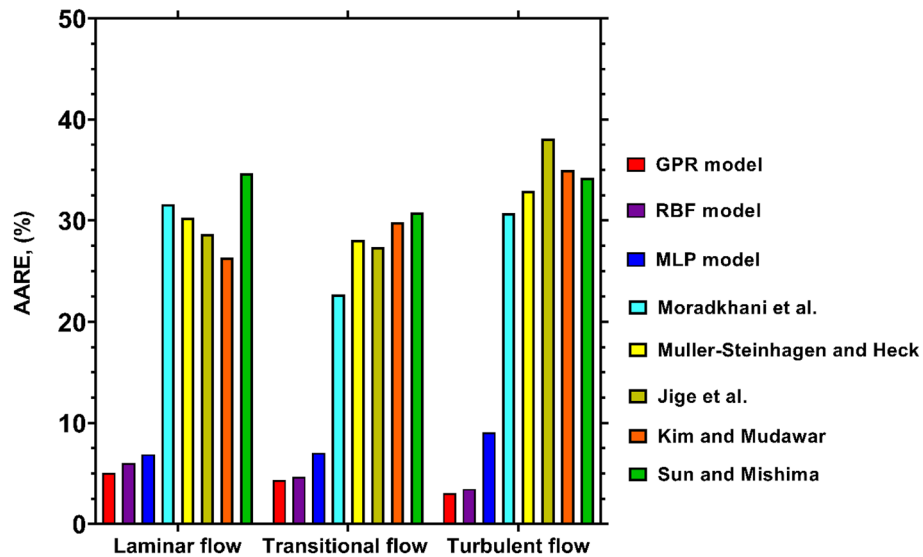


Figure 11. Accuracy of the novel and literature models for predicting the condensation FPD at various flow regimes.

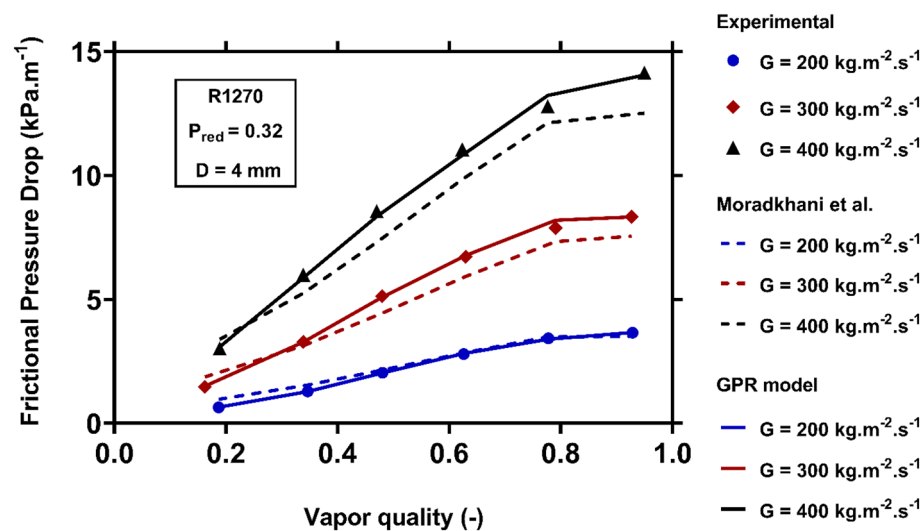


Figure 12. Examining the capability of the models to capture the variations of the condensation FPD versus vapor quality and mass flux.

Figure 13 compares the FPD variations of R290, R32 and R410A during condensation inside a channel of 1.16 mm inner diameter. As it is evident, R290 has the greatest condensation FPD among the evaluated working fluids, which is followed by R32 and R410A, respectively. Since the experiments have been carried out under constant mass flux and saturation temperature, the difference between the FPD values can be attributed to the nature of fluids. The physical characteristics of the refrigerants at the saturation temperature of 50 °C have been summarized in Table 5. As shown in the table, R290 has the lowest liquid and vapor densities, so it flows with higher liquid and vapor velocities inside the channel. On the other hand, the vapor and liquid kinematic viscosities of R290 are noticeably higher than the remaining fluids. These are the reasons why highest shear stress and FPD values are observed during R290 condensation. A similar argument can be presented to justify the higher condensation FPD of R32 compared to R410A. The novel model present correct physical trends, and perfectly matches the actual data. While, the Moradkhani et al.⁴⁰ correlation over-predicts the corresponding values.

The role of channel diameter in controlling the condensation FPD of R134 under fixed operating conditions has been demonstrated in Fig. 14. The results connote the enhancement of the condensation FPD by reducing the size of channel. In fact, the impression of surface tension on two-phase flow becomes more fundamental in small-diameter channels, and can boost the velocity gradient on the wall. Consequently, the shear rate and condensation FPD are increased. Although both models present appropriate results to describe the foregoing attitudes, the predictions of the novel one have better consistencies with the measure values.

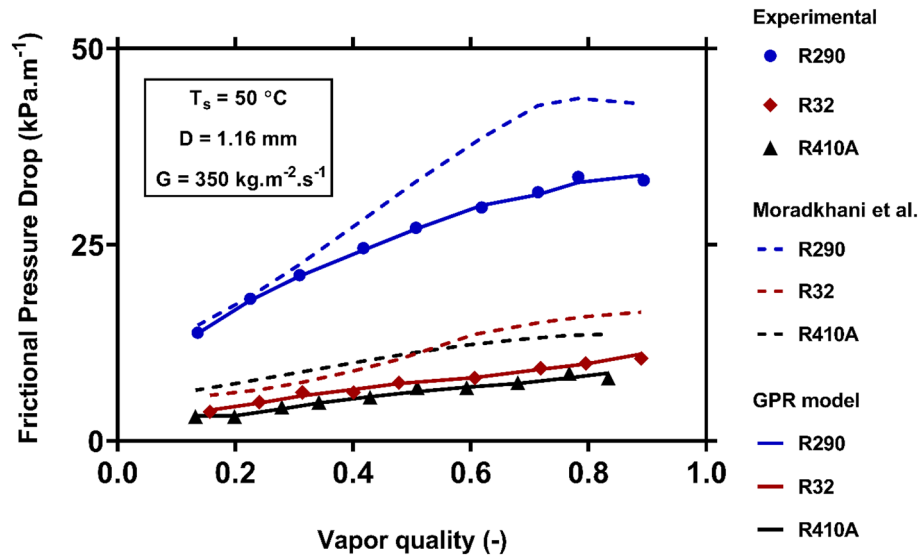


Figure 13. Examining the capability of the models to capture the FPD during condensation of various fluids.

Fluids	T_s (°C)	ρ_l (kgm ⁻³)	ρ_v (kgm ⁻³)	v_l (cm ² s ⁻¹)	v_v (cm ² s ⁻¹)
R290	50	448.87	38.63	0.0016501	0.0024336
R32	50	839.26	98.55	0.00099156	0.0014935
R410A	50	906.80	141.14	0.00090202	0.0014406

Table 5. Physical characteristics of R290, R32 and R410A at the saturation temperature of 50 °C.

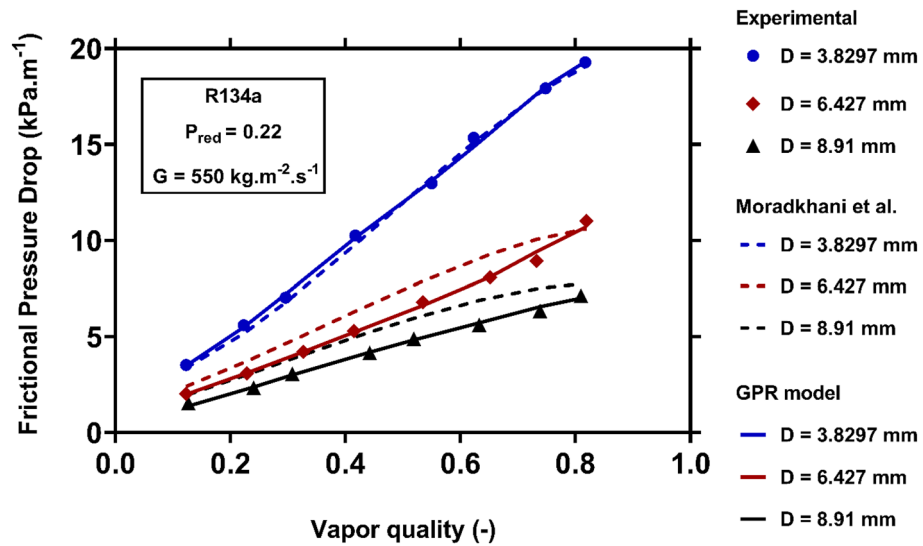


Figure 14. Examining the capability of the models to capture the variations of the condensation FPD versus channel diameter.

Figure 15 sketches the alternations of the condensation FPD with the reduced pressure during R14 flowing inside a channel with an inner diameter of 4 mm at the mass flux of $350 \text{ kg.m}^{-2}.\text{s}^{-1}$. A glance at the results brings it to view that the condensation FPD experiences a dramatical reduction by increasing the pressure. The principal cause of this behavior is that the phases' velocity difference at the vapor-liquid interface is amplified due to the foregoing change. While the predictions provided by the Moradkhani et al.⁴⁰ correlation have some deviations from the real samples, the GPR model excellently estimates the corresponding value.

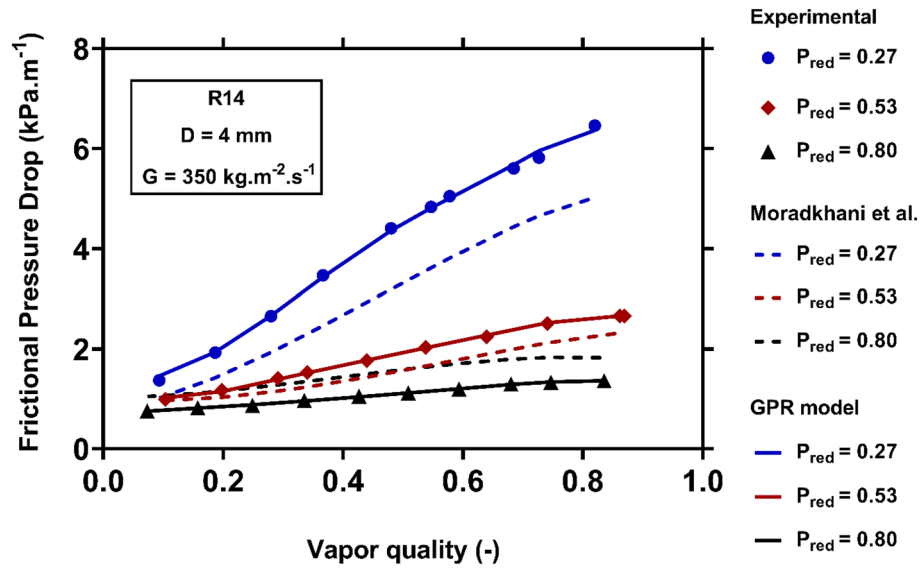


Figure 15. Examining the capability of the models to capture the variations of the condensation FPD versus reduced pressure.

Sensitivity analysis

Identifying the most effective factors on the condensation FPD is another indispensable requirement that should be fulfilled for the optimal design of the heat exchangers. Accordingly, the Spearman's correlation factors between the values of the condensation FPD estimated by the GPR model and various operating factors were determined, and the corresponding results have been illustrated in Fig. 16. The figure clearly reveals that the mass flux and vapor quality have direct relationships with the condensation FPD, while the remaining factors, i.e., channel diameter, reduced pressure, and fluid vapor density inversely affect this parameter. These findings are in complete consistency with those presented in Section "The prediction capabilities of the models". Another result draw-able from Fig. 13 is that the channel diameter and mass flux are the most substantial factors in controlling the condensation FPD. Furthermore, the reduced pressure, vapor quality and the vapor density of condensing fluid are ranked third to fifth from the standpoint of importance.

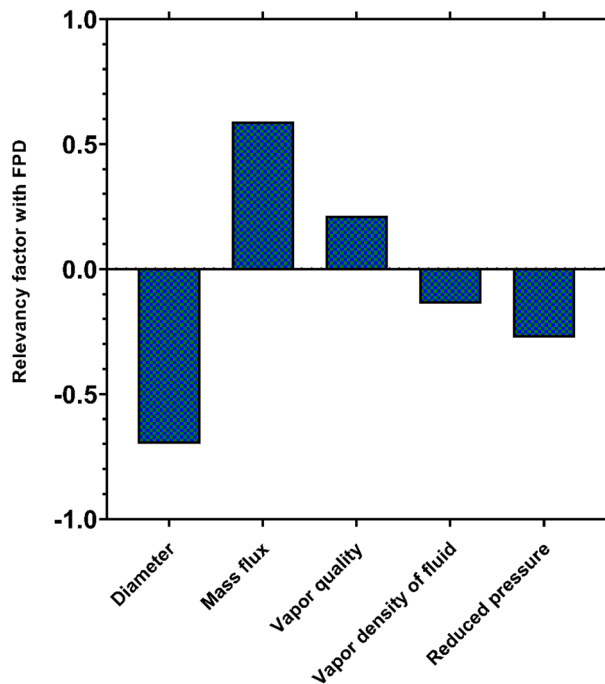


Figure 16. Spearman's correlation coefficient between the condensation FPD and various operating factors.

Concluding remarks

This study aimed to develop robust and comprehensive predictive tools for predicting the two-phase frictional pressure drop (FPD) during condensation inside channels of various sizes. A vast dataset of 8,037 samples was collected from 50 well-regarded sources. These samples encompassed the FPD of 23 fluids, such as chemicals, halocarbons, natural refrigerants, water, hydrocarbons, cryogenics, and more, over a broad range of conditions inside both mini/micro and conventional channels.

Evaluating the correctness of the literature correlations based on the gathered data denoted the point that more accurate FPD models are highly required, since all available ones gave the AARE values exceeding 26%. Hence, after choosing the optimized input variables (Re_{lo} , Re_{vo} , x , P_{red} , Bo , We_{vo} , X and Su_{vo}) according to Spearman's correlation analysis, the experimental data were employed for training and testing the soft computing approaches of MLP, GPR and RBF based on the theoretical method suggested by Lockhart and Martinelli³² and the dimensionless form suggested by Chisholm³³.

While all the novel intelligent models outperformed the literature correlations, the GPR-based model emerged as the superior predictive tool during the testing process. It achieved remarkable performance with an AARE of 4.10% and an R^2 value of 99.23%. Additionally, the visual representations, including the cumulative frequency and contour map, confirmed that the majority of the analyzed data (over 78% of all data) were predicted with relative deviations below 5% by the aforementioned model. The MLP and RBF models also presented satisfactory results with AAREs values of 12.56% and 21.04%, respectively for the test data. The proposed models have been recognized as reliable predictive tools with excellent accuracy for estimating the condensation frictional pressure drop (FPD) inside channels of various sizes, as well as in diverse flow patterns and regimes. Furthermore, these models effectively capture the physical variations of the condensation FPD in relation to the operating factors. In contrast, the literature correlations exhibited shortcomings in most of the aforementioned analyses. A sensitivity analysis conducted using the newly developed models demonstrated that the channel diameter and mass flux play fundamental roles in controlling the condensation FPD.

In contrast to conventional correlations, which often exhibit deviations exceeding 26%, the proposed machine learning models (MLP, GPR, RBF) offer a simple, robust, and well-verified approach for FPD prediction. These models serve as accurate tools for calculating two-phase FPD during condensation in channels of various sizes. Since FPD significantly impacts the design of heat exchangers and heat pumps, the results presented here provide valuable insights for relevant engineers and designers, enabling them to optimize these systems.

For future studies, these smart approaches can be implemented to develop reliable predictive models for other crucial design parameters of heat exchangers, such as heat transfer coefficient, condensation rate, and critical heat flux. Additionally, exploring the application of deep learning algorithms for modeling two-phase flow parameters presents a promising avenue for further research.

Data availability

The datasets used and/or analyzed during the current study are available from the corresponding author on reasonable request.

Received: 11 January 2024; Accepted: 29 April 2024

Published online: 07 May 2024

References

1. Ma, T. *et al.* Study on heat transfer and pressure drop performances of ribbed channel in the high temperature heat exchanger. *Appl. Energy* **99**, 393–401 (2012).
2. Kong, Y., Wang, W., Yang, L. & Du, X. Energy efficient strategies for anti-freezing of air-cooled heat exchanger. *Appl. Energy* **1(261)**, 114468 (2020).
3. Aktershev, S. P. & Alekseenko, S. V. Nonlinear waves and heat transfer in a falling film of condensate. *Phys. Fluids* **25(8)**, 1 (2013).
4. Hardik, B. K. & Prabhu, S. V. Boiling pressure drop and local heat transfer distribution of water in horizontal straight tubes at low pressure. *Int. J. Therm. Sci.* **110**, 65–82 (2016).
5. Lillo, G., Mastrullo, R., Mauro, A. W. & Viscito, L. Flow boiling heat transfer, dry-out vapor quality and pressure drop of propane (R290): Experiments and assessment of predictive methods. *Int. J. Heat Mass Transf.* **126**, 1236–1252 (2018).
6. Escobar, R. F. *et al.* Sensor fault detection and isolation via high-gain observers: Application to a double-pipe heat exchanger. *ISA Trans.* **50**, 480–486 (2011).
7. Zhang, Y., Wang, J., Liu, W. & Liu, Z. Heat transfer and pressure drop characteristics of R134a flow boiling in the parallel/tandem microchannel heat sinks. *Energy Convers. Manag.* **148**, 1082–1095 (2017).
8. Shen, B., Li, Z. & Gluesenkamp, K. R. Experimental study of R452B and R454B as drop-in replacement for R410A in split heat pumps having tube-fin and microchannel heat exchangers. *Appl. Therm. Eng.* **5(204)**, 117930 (2022).
9. Chen, Y., Shen, C., Shi, M. & Peterson, G. P. Visualization study of flow condensation in hydrophobic microchannels. *AIChE J.* **60**, 1182–1192 (2014).
10. Thiangtham, P., Mondal, P. K. & Wongwises, S. Flow boiling pressure drop characteristics in a multi-microchannel heat sink. *Phys. Fluids* **33(1)**, 1 (2021).
11. Dalkilic, A. S. & Wongwises, S. New experimental approach on the determination of condensation heat transfer coefficient using frictional pressure drop and void fraction models in a vertical tube. *Energy Convers. Manag.* **51**, 2535–2547 (2010).
12. Monte Verde, W. *et al.* Experimental investigation of pressure drop in failed electrical submersible pump (ESP) under liquid single-phase and gas-liquid two-phase flow. *J. Petrol. Sci. Eng.* **1(198)**, 108127 (2021).
13. Qiu, Y., Garg, D., Kim, S., Mudawar, I. & Kharangate, C. R. Machine learning algorithms to predict flow boiling pressure drop in mini / micro-channels based on universal consolidated data. *Int. J. Heat Mass Transf.* **178**, 121607 (2021).
14. Zivi, S. M. Estimation of steady-state steam void-fraction by means of the principle of minimum entropy production. *J. Heat Transf.* **86**, 247–251 (1964).
15. Liu, N. & Li, J. Experimental study on pressure drop of R32, R152a and R22 during condensation in horizontal minichannels. *Exp. Therm. Fluid Sci.* **71**, 14–24 (2016).
16. Liu, N., Li, J. M., Sun, J. & Wang, H. S. Heat transfer and pressure drop during condensation of R152a in circular and square microchannels. *Exp. Therm. Fluid Sci.* **47**, 60–67 (2013).

17. Liu, N., Xiao, H. & Li, J. Experimental investigation of condensation heat transfer and pressure drop of propane, R1234ze(E) and R22 in minichannels. *Appl. Therm. Eng.* **102**, 63–72 (2016).
18. Liu, N. & Li, J. Experimental study on condensation heat transfer of R32, R152a and R22 in horizontal minichannels. *Appl. Therm. Eng.* **90**, 763–773 (2015).
19. Jung, D. S. & Radermacher, R. Prediction of pressure drop during horizontal annular flow boiling of pure and mixed refrigerants. *Int. J. Heat Mass Transf.* **32**, 2435–2446 (1989).
20. Wang, C. C., Chiang, C. S. & Lu, D. C. Visual observation of two-phase flow pattern of R-22, R-134a, and R-407C in a 6.5-mm smooth tube. *Exp. Therm. Fluid Sci.* **15**(4), 395–405 (1997).
21. Li, W. & Wu, Z. Generalized adiabatic pressure drop correlations in evaporative micro/mini-channels. *Exp. Therm. Fluid Sci.* **35**, 866–872 (2011).
22. Mishima, K. & Hibiki, T. Some characteristics of air-water two-phase flow in small diameter vertical tubes. *Int. J. Multiph. Flow* **22**, 703–712 (1996).
23. Yan, Y. Y. & Lin, T. F. Condensation heat transfer and pressure drop of refrigerant R-134a in a small pipe. *Int. J. Heat Mass Transf.* **42**, 697–708 (1999).
24. Yam, Y. Y., Lin, T. F. & Yang, B. C. Evaporation heat transfer and pressure drop of refrigerant R134a in a plate heat exchanger. *Proc. ASME Turbo Expo* <https://doi.org/10.1115/97-AA-048> (1997).
25. Tran, T. N., Chyu, M. C., Wambsganss, M. W. & France, D. M. Two-phase pressure drop of refrigerants during flow boiling in small channels: An experimental investigation and correlation development. *Int. J. Multiph. Flow* **26**, 1739–1754 (2000).
26. Chen, I. Y., Yang, K. S., Chang, Y. J. & Wang, C. C. Two-phase pressure drop of air-water and R-410A in small horizontal tubes. *Int. J. Multiph. Flow* **27**, 1293–1299 (2001).
27. Lee, H. J. & Lee, S. Y. Pressure drop correlations for two-phase flow within horizontal rectangular channels with small heights. *Int. J. Multiph. Flow* **27**(5), 783–796 (2001).
28. Yu, W., France, D. M., Wambsganss, M. W. & Hull, J. R. Two-phase pressure drop, boiling heat transfer, and critical heat flux to water in a small-diameter horizontal tube. *Int. J. Multiph. Flow* **28**, 927–941 (2002).
29. Li, W. & Wu, Z. A general correlation for adiabatic two-phase pressure drop in micro/mini-channels. *Int. J. Heat Mass Transf.* **53**, 2732–2739 (2010).
30. Rahman, M. M., Kariya, K. & Miyara, A. Comparison and development of new correlation for adiabatic two-phase pressure drop of refrigerant flowing inside a multiport minichannel with and without fins Comparaison et développement d'une nouvelle corrélation pour la chute de pression diphasique. *Int. J. Refrig.* **82**, 119–129 (2017).
31. Thome, J. R. Wolverine Engineering Data Book III. *WielandWerke AG, Ger.* (2006).
32. Lockhart, R. W. & Martinelli, R. C. Proposed correlation of data for isothermal two-phase, two-component flow in pipes. *Chem. Eng. Prog.* **45**, 39–48 (1949).
33. Chisholm, D. A theoretical basis for the Lockhart-Martinelli correlation for two-phase flow. *Int. J. Heat Mass Transf.* **10**, 1767–1778 (1967).
34. Sun, L. & Mishima, K. Evaluation analysis of prediction methods for two-phase flow pressure drop in mini-channels. *Int. J. Multiph. Flow* **35**, 47–54 (2009).
35. Hossain, M. A., Afroz, H. M. M. & Miyara, A. Two-phase frictional multiplier correlation for the prediction of condensation pressure drop inside smooth horizontal tube. *Proced. Eng.* **105**, 64–72 (2015).
36. Jige, D., Inoue, N. & Koyama, S. Condensation of refrigerants in a multiport tube with rectangular minichannels. *Int. J. Refrig.* **67**, 202–213 (2016).
37. Friedel, L. Improved friction pressure drop correlations for horizontal and vertical two-phase pipe flow. in *European Two-phase Group Meeting* (1979).
38. Müller-Steinhagen, H. & Heck, K. A simple friction pressure drop correlation for two-phase flow in pipes. *Chem. Eng. Process.* **20**, 297–308 (1986).
39. Kim, S. M. & Mudawar, I. Universal approach to predicting two-phase frictional pressure drop for adiabatic and condensing mini/micro-channel flows. *Int. J. Heat Mass Transf.* **55**, 3246–3261 (2012).
40. Moradkhani, M. A., Hosseini, S. H., Valizadeh, M., Zendejboudi, A. & Ahmadi, G. A general correlation for the frictional pressure drop during condensation in mini/micro and macro channels. *Int. J. Heat Mass Transf.* **1**(163), 120475 (2020).
41. Parveen, N., Zaidi, S. & Danish, M. Artificial intelligence (AI)-based friction factor models for large piping networks. *Chem. Eng. Commun.* **207**, 213–230 (2020).
42. Parveen, N., Zaidi, S. & Danish, M. Development and analyses of data-driven models for predicting the bed depth profile of solids flowing in a rotary kiln. *Adv. Powder Technol.* **31**, 678–694 (2020).
43. Parveen, N., Zaidi, S. & Danish, M. Comparative analysis for the prediction of boiling heat transfer coefficient of R134a in micro/mini channels using artificial intelligence (AI)-based techniques. *Int. J. Model. Simul.* **40**, 114–129 (2020).
44. Parveen, N., Zaidi, S. & Danish, M. Support vector regression: A novel soft computing technique for predicting the removal of cadmium from wastewater. *Indian J. Chem. Technol.* **27**, 43–50 (2020).
45. Parveen, N., Zaidi, S. & Danish, M. Support vector regression (SVR)-based adsorption model for Ni (II) ions removal. *Groundw. Sustain. Dev.* **9**, 100232 (2019).
46. Parveen, N., Zaidi, S. & Danish, M. Modeling of flow boiling heat transfer coefficient of R11 in mini-channels using support vector machines and its comparative analysis with the existing correlations. *Heat Mass Transf. Und Stoffuebertragung.* **55**, 151–164 (2019).
47. Parveen, N., Zaidi, S. & Danish, M. Development of SVR-based model and comparative analysis with MLR and ANN models for predicting the sorption capacity of Cr(VI). *Process Saf. Environ. Prot.* **107**, 428–437 (2017).
48. Parveen, N., Zaidi, S. & Danish, M. Support vector regression prediction and analysis of the copper (II) biosorption efficiency. *Indian Chem. Eng.* **59**, 295–311 (2017).
49. Zendejboudi, A. & Li, X. A robust predictive technique for the pressure drop during condensation in inclined smooth tubes. *Int. Commun. Heat Mass Transf.* **86**, 166–173 (2017).
50. López-belchí, A., Illán-gómez, F., Cano-izquierdo José, M. & García-cascales José, R. Optimise model development: Prediction of the pressure drop and the heat gmdh ann to optimise model development: Prediction of the pressure drop and the heat transfer coefficient abstract. *Appl Therm. Eng.* <https://doi.org/10.1016/j.applthermaleng.2018.07.140> (2018).
51. Longo, G. A. *et al.* Machine learning approach for predicting refrigerant two-phase pressure drop inside Braze Plate Heat Exchangers (BPHE). *Int. J. Heat Mass Transf.* **163**, 120450 (2020).
52. Moradkhani, M. A., Hosseini, S. H., Mansouri, M., Ahmadi, G. & Song, M. Robust and universal predictive models for frictional pressure drop during two-phase flow in smooth helically coiled tube heat exchangers. *Sci. Rep.* **11**(1), 20068 (2021).
53. Barroso-Maldonado, J. M., Montañez-Barrera, J. A., Belman-Flores, J. M. & Aceves, S. M. ANN-based correlation for frictional pressure drop of non-azeotropic mixtures during cryogenic forced boiling. *Appl. Therm. Eng.* **149**, 492–501 (2019).
54. Moradkhani, M. A. *et al.* New general models for condensation heat transfer coefficient of carbon dioxide in smooth tubes by intelligent and least square fitting approaches. *J. Clean. Prod.* **330**, 129762 (2022).
55. Moradkhani, M. A., Hosseini, S. H. & Song, M. Robust and general predictive models for condensation heat transfer inside conventional and mini/micro channel heat exchangers. *Appl. Therm. Eng.* **201**, 117737 (2022).

56. Moradkhani, M. A., Hosseini, S. H. & Karami, M. Forecasting of saturated boiling heat transfer inside smooth helically coiled tubes using conventional and machine learning techniques. *Int. J. Refrig.* **143**, 78–93 (2022).
57. Liu, W., Qi, H., Shi, H., Yu, C. & Li, X. Helical model based on artificial neural network for large eddy simulation of compressible wall-bounded turbulent flows. *Phys. Fluids* <https://doi.org/10.1063/5.0137607> (2023).
58. Wang, Z. & Zhang, W. A unified method of data assimilation and turbulence modeling for separated flows at high Reynolds numbers. *Phys. Fluids*. **35**(2), 1 (2023).
59. Hosseini, S. H. *et al.* Estimation of the minimum spouting velocity in shallow spouted beds by intelligent approaches: Study of fine and coarse particles. *Powder Technol.* **354**, 456–465 (2019).
60. Moradkhani, M. A., Hosseini, S. H., Shangwen, L. & Mengjie, S. Intelligent computing approaches to forecast thickness and surface roughness of frost layer on horizontal plates under natural convection. *Appl. Therm. Eng.* **217**, 119258 (2022).
61. Hosseini, S. H., Moradkhani, M. A., Rasteh, M. & Rahimi, M. New smart models for minimum fluidization velocity forecasting in the tapered fluidized beds based on particle size distribution. *Ind. Eng. Chem. Res.* **60**, 15289–15300 (2021).
62. Kumar, P. S., Kumaraswamidhas, L. A. & Laha, S. K. Selection of efficient degradation features for rolling element bearing prognosis using Gaussian process regression method. *ISA Trans.* **112**, 386–401 (2021).
63. Moradkhani, M. A., Hosseini, S. H., Karami, M., Olazar, M. & Saldarriaga, J. F. Applying conventional and intelligent approaches to model the minimum spouting velocity of vegetable biomasses in conical spouted beds. *Powder Technol.* **418**, 118300 (2023).
64. Moradkhani, M. A., Hosseini, S. H., Valizadeh, M. & Mengjie, S. O. Machine learning based models to predict frost characteristics on cryogenic surfaces under forced convection conditions. *Int. Commun. Heat Mass Transf.* **1**(129), 105667 (2021).
65. He, N., Qian, C., Shen, C. & Huangfu, Y. A fusion framework for lithium-ion batteries state of health estimation using compressed sensing and entropy weight method. *ISA Trans.* **135**, 585–604 (2023).
66. Moradkhani, M. A., Hosseini, S. H. & Ranjbar, K. Universal intelligent models for liquid density of CO₂ + hydrocarbon mixtures. *Fuel* **334**, 126642 (2023).
67. Moradkhani, M. A., Hosseini, S. H., Ranjbar, K. & Moradi, M. Intelligent modeling of hydrogen sulfide solubility in various types of single and multicomponent solvents. *Sci. Rep.* **13**, 3777 (2023).
68. Lin, X., Zhu, K., Zhou, J. & Fuh, J. Y. H. Intelligent modeling and monitoring of micro-droplet profiles in 3D printing. *ISA Trans.* **105**, 367–376 (2020).
69. Lemmon, E. W., Huber, M. L. & McLinden, M. O. Reference fluid thermodynamic and transport properties—REFPROP[®], NIST. (2013).
70. Adams, D. C., Hrnjak, P. S. & Newell, T. A. Pressure drop and void fraction in microchannels using Carbon Dioxide, Ammonia, and R245fa as refrigerants. **61801**, (2003).
71. Andresen, U. Supercritical gas cooling and near-critical-pressure condensation of refrigerant blends in microchannels. *PhD Thesis* (2007).
72. Aroonrat, K. & Wongwises, S. Experimental study on two-phase condensation heat transfer and pressure drop of R-134a flowing in a dimpled tube. *Int. J. Heat Mass Transf.* **106**, 437–448 (2017).
73. Bortolin, S. Two-Phase Heat Transfer Inside Minichannels. *PhD Thesis* (2010).
74. Cavallini, A., Bortolin, S., Del Col, D., Matkovic, M. & Rossetto, L. Condensation heat transfer and pressure losses of high- and low-pressure refrigerants flowing in a single circular minichannel. *Heat Transf. Eng.* **32**, 90–98 (2011).
75. Bashar, M. K., Nakamura, K. & Kariya, K. Experimental study of condensation heat transfer and pressure drop inside a small diameter microfin and smooth tube at low mass flux condition. *Appl. Sci.* <https://doi.org/10.3390/app8112146> (2018).
76. Bashar, M. K., Nakamura, K., Kariya, K. & Miyara, A. Condensation heat transfer of R1234yf in a small diameter smooth and microfin tube and development of correlation. *Int. J. Refrig.* **120**, 331–339 (2020).
77. López-Belchí, A. Assessment of a mini-channel condenser at high ambient temperatures based on experimental measurements working with R134a, R513A and R1234yf. *Appl. Therm. Eng.* **155**, 341–353 (2019).
78. López-Belchí, A., Illán-Gómez, F., Cascales, J. R. & Garcia, F. V. R32 and R410A condensation heat transfer coefficient and pressure drop within minichannel multiport tube. Experimental technique and measurements. *Appl. Therm. Eng.* **105**, 118–131 (2016).
79. López-Belchí, A., Illán-Gómez, F., García-Cascales, J. R. & Vera-García, F. Condensing two-phase pressure drop and heat transfer coefficient of propane in a horizontal multiport mini-channel tube: Experimental measurements. *Int. J. Refrig.* **68**, 59–75 (2016).
80. Cavallini, A. *et al.* Condensation of halogenated refrigerants inside smooth tubes. *HVAC R Res.* **8**, 429–451 (2002).
81. Charnay, R. Experimental study of flow boiling in horizontal minichannels at high saturation temperature. (2014).
82. Charnay, R., Revellin, R. & Bonjour, J. Discussion on the validity of prediction tools for two-phase flow pressure drops from experimental data obtained at high saturation temperatures. *Int. J. Refrig.* **54**, 98–125 (2015).
83. Coleman, J. W. Flow visualization and pressure drop for refrigerant phase change and air-water flow in small hydraulic diameter geometries. (2000).
84. Coleman, J. W. & Garimella, S. Two-phase flow regimes in round, square and rectangular tubes during condensation of refrigerant R 134a. *Int. J. Refrig.* **26**, 117–128 (2003).
85. Del Col, D., Bisetto, A., Bortolato, M., Torresin, D. & Rossetto, L. Experiments and updated model for two phase frictional pressure drop inside minichannels. *Int. J. Heat Mass Transf.* **67**, 326–337 (2013).
86. de Oliveira, J. D., Copetti, J. B. & Passos, J. C. Experimental investigation on flow boiling pressure drop of R-290 and R-600a in a horizontal small tube. *Int. J. Refrig.* **84**, 165–180 (2017).
87. Fronk, B. M. & Garimella, S. Condensation of carbon dioxide in microchannels. *Int. J. Heat Mass Transf.* **100**, 150–164 (2016).
88. Pabon, J. G., Khosravi, A., Nunes, R. & Machado, L. Experimental investigation of pressure drop during two-phase flow of R1234yf in smooth horizontal tubes with internal diameters of 3.2 mm to 8.0 mm. *Int. J. Refrigeration.* **1**(104), 426–436 (2019).
89. Heo, J., Park, H. & Yun, R. Condensation heat transfer and pressure drop characteristics of CO₂ in a microchannel. *Int. J. Refrig.* **36**, 1657–1668 (2013).
90. Heo, J., Park, H. & Yun, R. Comparison of condensation heat transfer and pressure drop of CO₂ in rectangular microchannels. *Int. J. Heat Mass Transf.* **65**, 719–726 (2013).
91. Kang, P., Heo, J. & Yun, R. Condensation heat transfer characteristics of CO₂ in a horizontal smooth tube. *Int. J. Refrig.* **36**, 1090–1097 (2013).
92. Keinath, B. L. & Garimella, S. High-pressure condensing refrigerant flows through microchannels, part ii: Heat transfer models. *Heat Trans. Eng.* **40**(9–10), 830–843 (2019).
93. Kim, N. H. Condensation heat transfer and pressure drop of R-410A in a 7.0 mm O.D microfin tube at low mass fluxes. *Heat Mass Transf.* <https://doi.org/10.1007/s00231-016-1789-2> (2016).
94. Longo, G. A., Mancin, S., Righetti, G. & Zilio, C. Saturated vapour condensation of HFC404A inside a 4 mm ID horizontal smooth tube: Comparison with the long-term low GWP substitutes HC290 (Propane) and HC1270 (Propylene). *Int. J. Heat Mass Transf.* **108**, 2088–2099 (2017).
95. Longo, G. A., Mancin, S., Righetti, G. & Zilio, C. Saturated vapour condensation of R410A inside a 4 mm ID horizontal smooth tube: Comparison with the low GWP substitute R32. *Int. J. Heat Mass Transf.* **125**, 702–709 (2018).
96. Longo, G. A., Mancin, S., Righetti, G. & Zilio, C. Saturated vapour condensation of R134a inside a 4 mm ID horizontal smooth tube: Comparison with the low GWP substitutes R152a, R1234yf and R1234ze(E). *Int. J. Heat Mass Transf.* **133**, 461–473 (2019).
97. Macdonald, M. Condensation of pure and zeotropic hydrocarbons in smooth horizontal tubes. (2015).

98. Macdonald, M. & Garimella, S. Hydrocarbon condensation in horizontal smooth tubes: Part i—Measurements. *Int. J. Heat Mass Transf.* **93**, 75–85 (2016).
99. Milkie, J. A. Condensation of hydrocarbons and zeotropic hydrocarbon/refrigerant mixtures in horizontal tubes. (2014).
100. Garimella, S., Milkie, J. & Macdonald, M. Condensation heat transfer and pressure drop of low-pressure hydrocarbons and synthetic refrigerants. *Int. J. Heat Mass Transf.* **161**, 120295 (2020).
101. Mitra, B. Supercritical gas cooling and condensation of refrigerant R410A AT near-critical pressures. (2005)
102. Garimella, S., Mitra, B., Andresen, U. C., Jiang, Y. & Fronk, B. M. Heat transfer and pressure drop during supercritical cooling of HFC refrigerant blends. *Int. J. Heat Mass Transf.* **91**, 477–493 (2015).
103. Nie, Z., Bi, Q., Lei, S., Lv, H. & Guo, Y. Experimental study on condensation of high-pressure steam in a horizontal tube with pool boiling outside. *Int. J. Heat Mass Transf.* **108**, 2523–2533 (2017).
104. Nino, V. G., Hrnjak, P. S. & Newell, T. a. Characterization of two-phase flow in microchannels. *Ph.D. thesis* (2002).
105. Pham, Q. V., Choi, K. I. & Oh, J. T. Condensation heat transfer characteristics and pressure drops of R410A, R22, R32, and R290 in a multiport rectangular channel. *Sci. Technol. Built Environ.* **25**(10), 1325–1336 (2019).
106. Qi, C., Chen, X., Wang, W., Miao, J. & Zhang, H. Experimental investigation on flow condensation heat transfer and pressure drop of nitrogen in horizontal tubes. *Int. J. Heat Mass Transf.* **132**, 985–996 (2019).
107. Solanki, A. K. & Kumar, R. Condensation heat transfer and pressure drop characteristics of R-134a inside the flattened tubes at high mass flux and different saturation temperature. *Exp. Heat Transf.* **32**, 69–84 (2019).
108. Song, Q., Chen, G., Guo, H., Shen, J. & Gong, M. Two-phase flow condensation pressure drop of R14 in a horizontal tube: Experimental investigation and correlation development. *Int. J. Heat Mass Transf.* **139**, 330–342 (2019).
109. Song, Q., Wang, D., Shen, J., Zhao, Y. & Gong, M. Flow condensation pressure drop characteristics of zeotropic mixtures of tetrafluoromethane/ethane: Experimental and analytical investigation. *Int. J. Heat Mass Trans.* **1**(182), 122045 (2022).
110. Sempértegui-Tapia, D. F. & Ribatski, G. Two-phase frictional pressure drop in horizontal micro-scale channels: Experimental data analysis and prediction method development. *Int. J. Refrig.* **79**, 143–163 (2017).
111. Xu, Y., Yan, Z. & Li, L. Experimental investigation on adiabatic two-phase frictional pressure drop of R1234ze (E) and R134a in a horizontal minichannel. *J. Therm. Sci. Eng. Appl.* **14**(2), 021003 (2022).
112. Yoo, J. W., Nam, C. W. & Yoon, S. H. Experimental study of propane condensation heat transfer and pressure drop in semicircular channel printed circuit heat exchanger. *Int. J. Heat Mass Transf.* **182**, 121939 (2022).
113. Zhuang, X. R., Gong, M. Q., Zou, X., Chen, G. F. & Wu, J. F. Experimental investigation on flow condensation heat transfer and pressure drop of R170 in a horizontal tube. *Int. J. Refrig.* **66**, 105–120 (2016).
114. Zhuang, X. R., Chen, G. F., Zou, X., Song, Q. L. & Gong, M. Q. Experimental investigation on flow condensation of methane in a horizontal. *Int. J. Refrig.* **78**, 193–214 (2017).
115. Gu, X. *et al.* Condensation flow patterns and model assessment for R1234ze(E) in horizontal mini/macro-channels. *Int. J. Therm. Sci.* **134**, 140–159 (2018).
116. Zhang, W., Hibiki, T. & Mishima, K. Correlations of two-phase frictional pressure drop and void fraction in mini-channel. *Int. J. Heat Mass Transf.* **53**, 453–465 (2010).
117. Koyama, S., Kuwahara, K. & Nakashita, K. Condensation of Refrigerant in a Multi-Port Channel. in *1st International Conference on Microchannels and Minichannels* (2003).
118. Bolboaca, S.-D. & Jäntschi, L. Pearson versus Spearman, Kendall's tau correlation analysis on structure-activity relationships of biologic active compounds. *Leonardo J. Sci.* **5**, 179–200 (2006).
119. Kandlikar, S. G. Fundamental issues related to flow boiling in minichannels and microchannels. *Exp. Therm. Fluid Sci.* **26**, 389–407 (2002).
120. Lee, H. J. & Lee, S. Y. Heat transfer correlation for boiling flows in small rectangular horizontal channels with low aspect ratios. *Int. J. Multiph. Flow* **27**, 2043–2062 (2001).
121. Kim, S. M., Kim, J. & Mudawar, I. Flow condensation in parallel micro-channels—Part 1: Experimental results and assessment of pressure drop correlations. *Int. J. Heat Mass Transf.* **55**, 971–983 (2012).
122. Berto, A. *et al.* Liquid film thickness and heat transfer measurements during downflow condensation inside a small diameter tube. *Int. J. Multiph. Flow* **140**, 103649 (2021).
123. Lakehal, D., Fulgosi, M., Banerjee, S. & Yadigaroglu, G. Turbulence and heat exchange in condensing vapor-liquid flow. *Phys. Fluids* **20**, 110906 (2008).
124. Chen, B. L., Yang, T. F., Sajjad, U., Ali, H. M. & Yan, W. M. Deep learning-based assessment of saturated flow boiling heat transfer and two-phase pressure drop for evaporating flow. *Eng. Anal. Bound. Elem.* **151**, 519–537 (2023).
125. Cioncolini, A. & Thome, J. R. Entrained liquid fraction prediction in adiabatic and evaporating annular two-phase flow. *Nucl. Eng. Des.* **243**, 200–213 (2017).
126. Cioncolini, A., Del Col, D. & Thome, J. R. An indirect criterion for the laminar to turbulent flow transition in shear-driven annular liquid films. *Int. J. Multiph. Flow* **75**, 26–38 (2015).

Acknowledgements

This research was funded by the National Natural Science Foundation of China (Grant No. 52076013), the Beijing Municipal Science & Technology Commission (Grant No. 3212024), the China-South Korea Youth Scientists Exchange Program in 2023 supported by the Ministry of Science and Technology of China and Ministry of Science and ICT of the Republic of Korea, and the National Research Foundation of Korea in 2023.

Author contributions

M.A.M.: Methodology, Software, Writing- Original draft. S.H.H.: Conceptualization, Methodology, Writing- Reviewing and Editing. M.S.: Conceptualization, Supervision. A.A.: Supervision, Methodology.

Competing interests

The authors declare no competing interests.

Additional information

Correspondence and requests for materials should be addressed to S.H.H. or M.S.

Reprints and permissions information is available at www.nature.com/reprints.

Publisher's note Springer Nature remains neutral with regard to jurisdictional claims in published maps and institutional affiliations.



Open Access This article is licensed under a Creative Commons Attribution 4.0 International License, which permits use, sharing, adaptation, distribution and reproduction in any medium or format, as long as you give appropriate credit to the original author(s) and the source, provide a link to the Creative Commons licence, and indicate if changes were made. The images or other third party material in this article are included in the article's Creative Commons licence, unless indicated otherwise in a credit line to the material. If material is not included in the article's Creative Commons licence and your intended use is not permitted by statutory regulation or exceeds the permitted use, you will need to obtain permission directly from the copyright holder. To view a copy of this licence, visit <http://creativecommons.org/licenses/by/4.0/>.

© The Author(s) 2024