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# **Study on the efect of porosity OPEN of hollow fber membrane on humidifcation performance**

**Runping Niu**\***, Xiaoting Jia & LizhiGeng**

**Hollow fber membranes are used in industrial processes widely. Porosity is one of the important parameters afecting the humidifcation performance of hollow fber membrane components. The aim of this study was to analyze the efect of porosity of hollow fber membrane on humidifcation performance. In order to perform this analysis, a model based on the fnite element method was used to simulate numerically the heat and mass transfer under 6 porosity conditions. Five working conditions with diferent air fow was considered in order to get more data. The results show that when the porosity increases from 0.35 to 0.8, the humidifcation performance is greatly improved. However, when it increases from 0.8 to 0.9, the humidifcation performance is almost unchanged. Considering the humidifcation performance and support strength of hollow fber membrane, it is suggested to control the porosity of hollow fber membrane between 0.65 and 0.8.**

Indoor air humidity is not only directly related to human comfort but also closely related to human health. Air with too little or too much humidity can lead to decreased comfort, and even cause mouth or eye dryness, res-piratory tract infection and other diseases<sup>[1](#page-8-0)</sup>. In an industrial production environment, the scientific regulation of humidity control in a reasonable range of humidity also has a vital position. If humidity is not be regulated, it will seriously afect the product quality and cause unnecessary economic losses, such as in electronic components, food processing, wood furniture, agricultural production and other industries.

There are cooling dehumidification, liquid absorption dehumidification, solid adsorption dehumidification, membrane dehumidification, membrane liquid dehumidification and so on<sup>2-4</sup>. Dehumidification using hollow fber membrane belongs to the category of membrane liquid dehumidifcation. It combines membrane separation technology with liquid dehumidifcation technology, which can efectively prevent direct constant between high humidity air and desiccant and eliminate the possibility of mutual pollution between desiccant and air<sup>5[,6](#page-8-4)</sup>. Hollow fiber membrane material is an important part to determine the dehumidification efficiency. Its performance is mainly reflected in selectivity, permeability and its own structure<sup>[7](#page-8-5)-9</sup>. So it is of practical significance to study the characteristics of hollow fber membrane material to promote the development of this technology.

Porosity refers to the size of microspore volume (or area) contained per unit membrane volume (or area) $10$ , which is one of the significant structural parameters of hollow fiber membrane<sup>11</sup>. Porosity leads to mass transfer resistance<sup>12,[13](#page-8-10)</sup> and mass transfer area<sup>14–[17](#page-8-12)</sup>, which further affects membrane flux. And then some scholars took it a step further. Xiang et al.[18](#page-8-13) found by using experimental methods that porosity increased by 35.3% and pure water flux increased by 286.9%. Peng et al.<sup>19</sup> studied the performance of porous of ceramic membrane elements, and they found that porosity has a great relationship with pure water fux, and the numerical simulation is in good agreement with experimental results. Liu et al.<sup>20</sup> showed the humidification efficiency of the membrane liquid dehumidifer increased signifcantly when the porosity of the fber membrane changes from 0.1 to 0.5. Wang<sup>21</sup> used numerical simulation methods to found that the dehumidification efficiency would increase with the increase of porosity. To sum up, previous studies showed that porosity is an important parameter afecting the humidifcation performance of hollow fber membrane, but they did not point out the specifc law of porosity and humidifcation amount.

In order to explore the specifc law of porosity and humidifcation performance, a numerical calculation model of countercurrent hollow fiber membrane humidification component was established in this paper. The humidifcation performance of polypropylene hollow fber membrane with porosity of 0.45 was analyzed experimentally and the numerical results were verified. Then, by changing the porosity of hollow fiber membrane material in Fluent, the moisture content of the humidified air was analyzed. The relationship between the porosity of hollow fiber membrane material and humidification efficiency was obtained, and a reasonable porosity

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<span id="page-1-0"></span>**Figure 1.** Schematic diagram of countercurrent hollow fber membrane humidifer component.



Figure 2. Schematic diagram of countercurrent hollow fiber membrane humidification process.

<span id="page-1-1"></span>

range of hollow fber membrane was proposed, which laid a theoretical foundation for the future development of polypropylene hollow fber membrane material.

#### **Theoretical basis**

In this paper, countercurrent hollow fber membrane module was used for humidifcation research. When this module works, water vapor molecules reach gas measurement from solution side through the gap between fbers, which is a particularly complex heat and mass transfer process. Figures [1](#page-1-0) and [2](#page-1-1) are schematic diagrams of humidifcation of countercurrent hollow fber membrane assembly.

Afer years of research by researchers, the heat and mass transfer process is simplifed to the exchange of heat and water vapor in air–membrane, membrane–membrane and membrane–solution three regions<sup>22–25</sup>. The heat on both sides of the gas and liquid is mainly transmitted by thermal convection<sup>26-[28](#page-9-5)</sup>. Heat transfer in the film is composed of heat conduction and latent heat of vaporization<sup>[29](#page-9-6)</sup>. The calculation formula of the total heat transfer coefficient is shown in Eq.  $(1)$ .

$$
\frac{1}{h} = \frac{1}{h_1} + \frac{1}{h_m} + \frac{1}{h_2} \tag{1}
$$

where  $h_1$  is the heat transfer coefficient of the solution side (W/(m<sup>2</sup> K)),  $h_2$  is the heat transfer coefficient of the air side (W/(m<sup>2</sup> K)),  $h_{\rm m}$  is the heat transfer coefficient of hollow fiber membrane (W/(m<sup>2</sup> K)).

Heat transfer can be expressed as Eq. [\(2\)](#page-1-3)

$$
Q = hA\Delta T \tag{2}
$$

where *A* is heat transfer area (m<sup>2</sup>),  $\Delta T$  is the logarithmic average temperature difference between air side and solution side (K).

<span id="page-1-3"></span><span id="page-1-2"></span>2



<span id="page-2-0"></span>**Table 1.** Parameters of the hollow fber membrane humidifer component.

In the process of humidification, mass transfer and heat transfer occur simultaneously in three areas. The humidifcation amount was mainly discussed in this paper, which refers to the amount of water vapor absorbed by the air in unit time. Its calculation formula $30$  is as follows:

$$
M = GC(d_{\text{out}} - d_{\text{in}}) \tag{3}
$$

where *G* is air flow (kg/h), *C* is the insurance coefficient, which is 1.1 in this paper,  $d_{out}$  is the moisture content of air outlet (g/kg),  $d_{\text{in}}$  is moisture content of air inlet (g/kg).

The calculation formula of humidification efficiency<sup>[30](#page-9-7)</sup> is as follows:

$$
\eta = \frac{d_{\text{out}} - d_{\text{in}}}{d_{\text{e}} - d_{\text{out}}}
$$
\n(4)

$$
d_{\rm e} = \frac{622P_{\rm q}}{\rm B - P_{\rm q}}\tag{5}
$$

where  $d_e$  is equivalent moisture content (g/kg),  $P_q$  is the partial pressure of water vapor at gas–liquid equilibrium (Pa), B is atmospheric pressure (Pa).

#### **Numerical simulation**

Assumptions. The heat and mass transfer process of countercurrent hollow fiber membrane humidifier component is complex and easily afected by the external environment. In order to ensure the accuracy of the experiment and simulation, the following hypotheses were made in this paper:

- $(1)$  All flows are laminar flows, and the liquid is Newtonian fluid;
- (2) Air is an ideal gaseous mixture of water vapor and dry air;
- (3) Hollow fber membrane is homogeneous material and porous medium;
- (4) Air and solution are evenly distributed in their respective channels and are fully developed fuids;
- (5) The whole humidifier component is adiabatic with the surrounding environment, and heat transfer only occurs inside;
- (6) Ignore the heat and mass transfer generated along the fow direction;
- (7) Heat and mass transfer are steady-state.

**Physical model.** Porosity directly affects the flux of the membrane material in the humidification process and the support strength of the membrane material. Membrane flux is proportional to porosity. The support strength is inversely proportional to porosity<sup>[31](#page-9-8)</sup>. Excessive porosity will reduce the support of the fiber membrane and shorten its service life. Generally, the porosity of hollow fiber membrane is between 0.35 and 0.9<sup>[32](#page-9-9)</sup>. In this paper, countercurrent hollow fber membrane humidifer components with porosity of 0.35, 0.45, 0.65, 0.8, 0.85 and 0.9 were set for simulation. Table [1](#page-2-0) lists component parameters. Countercurrent hollow fber membrane assembly is mainly composed of shell and hollow fber tube. When it works, the air enters from the lower side of the shell and exits from the upper side, and the solution enters through the right entrance of the fber tube and exits through the left exit, as shown in Fig. [3.](#page-3-0)

**Boundary conditions.** In order to explore the relationship between porosity and humidifcation performance, the humidifcation process of hollow fber membrane with six porosity values was simulated under fve air flow conditions (Table [2](#page-3-1)). The solution was completed in Fluent16.0, and related parameter Settings are shown in Table [3.](#page-3-2)



<span id="page-3-0"></span>**Figure 3.** Physical model of counter-fow hollow fber membrane humidifcation module (ANSYS 16.0).



<span id="page-3-1"></span>**Table 2.** Air condition.



<span id="page-3-2"></span>**Table 3.** Parameter settings of the numerical simulation.

## **Results and discussion**

**Experimental verification.** In order to ensure the correctness of numerical simulation, an experimental system of polypropylene hollow fber membrane humidifcation component was built according to the relevant parameters shown in Table [1,](#page-2-0) and the schematic diagram is shown in Fig. [4.](#page-4-0) The experimental system consisted of an air loop and a solution loop. The air loop comprised air compressor, air duct, hot-wires anemometer, temperature and humidity tester, and shell of hollow fiber membrane humidifier component. The temperature and humidity of the air in the experiment were regulated by the air conditioning equipment in the room. The air compressor adjusted the air fow rate by controlling the pneumatic valve, and sended the air into the hollow fber membrane humidification component to provide power for the air circulation system. The solution loop consisted of constant temperature water tank, solution pump, rotameter, solution pipeline, and valve. The solution pump feeded the solution in the constant temperature tank into the hollow fber membrane humidifer, and the solution flow rate was controlled by the solution pump. The solution that is sent into the hollow fiber membrane

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<span id="page-4-0"></span>**Figure 4.** Experiment service schematic diagram (1. Air pump, 2. Temperature tester, 3. Hygrometer, 4. Hot-wire anemometer, 5. Hollow fber membrane humidifer component, 6. Rotameter, 7. Solution pump, 8. Constant temperature water tank).



#### <span id="page-4-1"></span>**Table 4.** Data statistics.

tube completed the heat and mass exchange with the air and then returns to the constant temperature fume again after treatment. The circulation of aqueous solution was realized.

Due to the limitation of experimental conditions, only hollow fber membrane with porosity of 0.45 was verified. The numerical simulation results and experimental results are shown in the Table [4.](#page-4-1) The results have a small margin of error, within 5%.

The simulation results were consistent with the experimental results as the air flow increases. As the air flow gradually increased, the temperature at the air outlet side increased (Fig. [5](#page-5-0)) and the moisture content decreased (Fig. [6](#page-5-1)). To sum up, the numerical simulation results obtained in this paper are reliable.

**Results and analysis.** In order to study the effect of porosity on humidification performance of countercurrent hollow fber membrane assembly, the numerical simulation results were analyzed as follows. Figure [7](#page-6-0) shows the temperature distribution cloud diagram at the air outlet side of the 6 porosity of the hollow fber membranes under working condition 1. It can be seen from the fgure that with the increase of porosity, the temperature at the air outlet side gradually decreased. This is because the increase of porosity enhances the heat transfer efficiency on both sides of the film, so that more heat is transferred from the air to the solution, resulting in lower and lower temperature at the air outlet side.

Figure [8](#page-6-1) shows the variation of air outlet temperature of the 6 porosity of hollow fber membranes with air fow under 5 working conditions. It can be seen that under the condition of constant porosity, the air outlet side temperature increased with the increase of air fow. Tis is because the increase of air fow reduces the air contact time with the fber membrane and thus shorten the heat transfer time. In addition, under constant air fow condition, the air outlet temperature decreased with the increase of porosity. When the porosity changes from 0.35 to 0.8, there was an obvious temperature diference of about 2 K at the air outlet side. However, when the porosity changed from 0.8 to 0.9, the temperature difference at the air outlet side was only about 0.2 K. There was no obvious temperature change. Tis phenomenon indicates that when the porosity was between 0.35 and 0.8, the heat transfer efect of the hollow fber membrane humidifer component was signifcantly enhanced. However, when the porosity was between 0.8 and 0.9, the increase of porosity had no obvious efect on enhancing the heat transfer efect of humidifer components.

Figure [9](#page-7-0) shows the water vapor mass fraction distribution cloud diagram at the air outlet side of the 6 porosity of the hollow fber membranes under working condition 1. It can be seen that as the porosity gradually increased, there were more and more water vapor molecules on the side of the air outlet, which also indicated that the



<span id="page-5-0"></span>**Figure 5.** Comparison of experimental and simulated air outlet temperature under diferent working conditions (Origin 2018).



<span id="page-5-1"></span>**Figure 6.** Comparison between experimental value and simulation value of air outlet moisture content under diferent working conditions (Origin 2018).

moisture content of the air was getting higher and higher. This is because the increase of porosity enhanced the mass transfer process and increases the transmembrane fux, thus increasing the moisture content of the air outlet side.

Figure [10](#page-7-1) shows the variation of air outlet moisture content of the 6 porosity of hollow fber membranes with air fow under 5 working conditions. It can be seen that under the condition of constant porosity, the moisture content at the air outlet side was decrease with the increase of air fow. Tis is because the increase of air fow reduced the mass transfer driving force between air and solution, leading to the decrease of air outlet moisture content. In addition, under the condition of constant air fow rate, the moisture content of air outlet increases with the increase of porosity. When the porosity varied from 0.35 to 0.8, the moisture content at the air outlet side increased obviously about 1.5 g/kg. However, when the porosity changes from 0.8 to 0.9, the moisture content at the air outlet side didn't increase signifcantly, and even does not increase when the porosity changes from 0.8 to 0.85 at the frst and ffh working conditions. Tis indicates that when the porosity was 0.35–0.8, the mass transfer efect of hollow fber membrane is signifcantly improved, while when the porosity is 0.8–0.9, the mass transfer efect is not signifcantly enhanced.

Figure [11](#page-8-15) shows the variation of humidification amount and efficiency with porosity under condition 1. It can be seen that when the porosity increases from 0.35 to 0.8, the humidifcation amount increased from 0.0193 to 0.0242 kg/h, and the humidification efficiency increased from 53.3% to 66.7%. The humidification amount and efficiency increase significantly with the increase of porosity. However, when the porosity increased from 0.8 to 0.85, the humidification amount and efficiency hardly change, indicating that the increase of porosity has no obvious effect on improving the humidification amount and efficiency. When the porosity increased from 0.8 to 0.9, the humidification capacity increased from 0.0242 to 0.0248 kg/h, and the humidification efficiency increased



<span id="page-6-0"></span>Figure 7. Air outlet temperature distribution under different membrane porosity (ANSYS 16.0).



<span id="page-6-1"></span>Figure 8. Change of air outlet temperature with air flow rate under different porosity (Origin 2018).

from 66.7 to 67.6%. Although the increase of porosity also increases the humidification capacity and efficiency, compared with the increase of porosity from 0.35 to 0.8, the increase of porosity has no signifcant efect on the improvement of humidification capacity and efficiency. This result is obtained through the analysis of the right amount and humidification efficiency under all air flow conditions, so it will not be described too much here.

### **Conclusion**

Trough the study of the performance of hollow fber membrane humidifcation system made of porous materials, the following conclusions are drawn:

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<span id="page-7-0"></span>**Figure 9.** Distribution of water vapor mass fraction at air outlet with diferent membrane porosity (ANSYS 16.0).



<span id="page-7-1"></span>Figure 10. Change of air outlet moisture content with air flow rate under different porosity (Origin 2018).

- (1) Under the condition of constant porosity, the increase of air fow reduced the contact time between air and solution per unit volume, resulting in the decrease of heat transfer efect, and the air outlet temperature increases with the increase of air fow. Similarly, the increase of air fow leaded to the decrease of moisture content at the air outlet side, which reduced the humidification efficiency.
- (2) Under constant fow condition, the air outlet temperature decreases with the increase of porosity, indicating that porosity strengthens the heat transfer effect of hollow fiber membrane humidifier. The moisture



<span id="page-8-15"></span>

content at the air outlet side increases with the increase of porosity, indicating that the greater the porosity, the greater the membrane fux and the greater the humidifcation.

Heat and mass transfer performance of hollow fiber membrane humidifier component shows an upward trend with porosity. When the porosity increases from 0.35 to 0.8, the increment of heat and mass transfer is obvious, but when the porosity is greater than 0.8, the increment of heat and mass transfer performance is not obvious. The increase of porosity will reduce the mechanical strength of hollow fiber membrane humidifer components to a certain extent and afect the service life. Taking the above factors into consideration. When the porosity is between 0.65 and 0.8, the hollow fber membrane humidifer can not only ensure high heat and transfer quality, but also ensure its mechanical strength to a certain extent.

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#### **Author contributions**

N.R. is mainly responsible for writing papers, J.X. is responsible for numerical simulation, G.L. is responsible for experiments.

#### **Competing interests**

The authors declare no competing interests.

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