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Nonlinear buckling and postbuckling behavior of cylindrical shear deformable nanoshells subjected to radial compression including surface free energy effects

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ABSTRACT

The objective of the present investigation is to predict the nonlinear buckling and postbuckling characteristics of cylindrical shear deformable nanoshells with and without initial imperfection under hydrostatic pressure load in the presence of surface free energy effects. To this end, Gurtin-Murdoch elasticity theory is implemented into the first-order shear deformation shell theory to develop a size-dependent shell model which has an excellent capability to take surface free energy effects into account. A linear variation through the shell thickness is assumed for the normal stress component of the bulk to satisfy the equilibrium conditions on the surfaces of nanoshell. On the basis of variational approach and using von Karman-Donnell-type of kinematic nonlinearity, the non-classical governing differential equations are derived. Then a boundary layer theory of shell buckling is employed incorporating the effects of surface free energy in conjunction with nonlinear prebuckling deformations, large deflections in the postbuckling domain and initial geometric imperfection. Finally, an efficient solution methodology based on a two-stepped singular perturbation technique is put into use in order to obtain the critical buckling loads and postbuckling equilibrium paths corresponding to various geometric parameters. It is demonstrated that the surface free energy effects cause increases in both the critical buckling pressure and critical end-shortening of a nanoshell made of silicon.

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1. Introduction

Several application potentials of nanotechnology that appeared for a short to medium-term time range could lead to significant improvements within several areas of space technology. Nanoengineering could produce surfaces that regulate spacecraft temperatures more efficiently than the

materials used today. It could also generate more efficient solar cells, rendering large panels redundant.

Therefore, nanostructures such as ultra thin plates (nanofilms), nanobeams, nanoshells, etc. have attracted much attention from the scientific community due to their considerably enormous physical, mechanical, and electrical properties [1–3]. Across the applications, when the thickness of the general structures reaches the order of submicron size, the size-dependent influence of material plays an essential role in the mechanical behavior of nanostructures. The classical continuum models are independent of the

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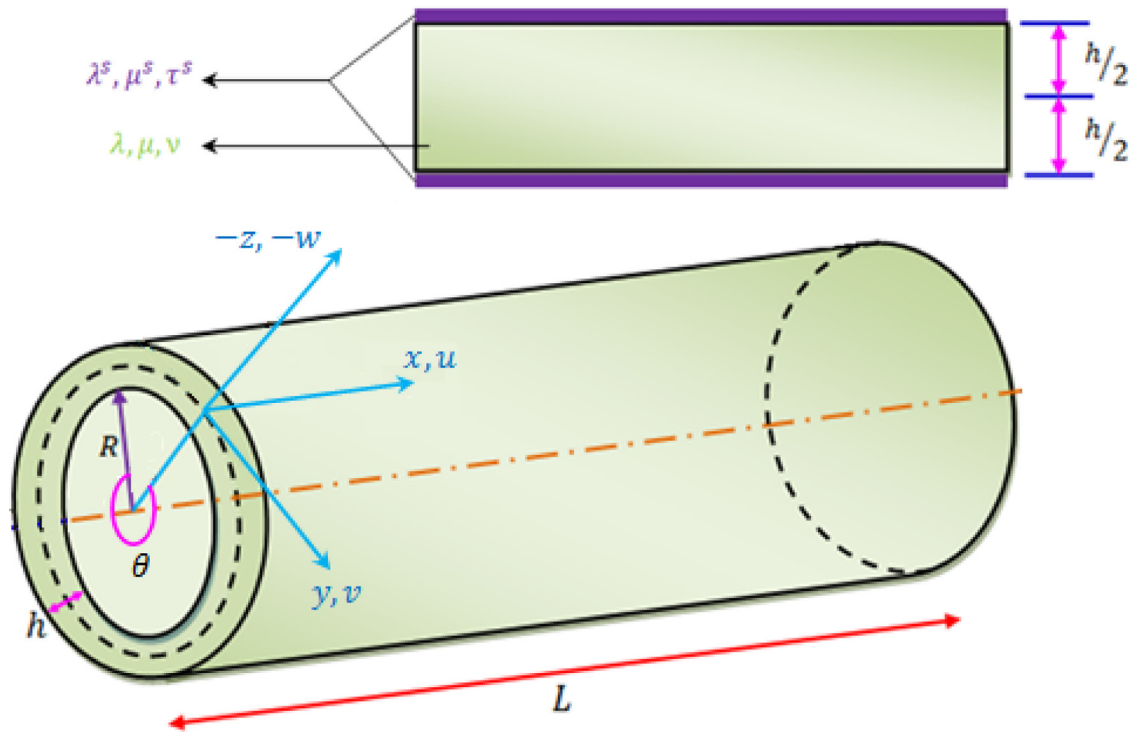


Fig. 1 – Schematic view of a cylindrical shear deformable nanoshell with surface layers.

scale of structures, making it controversial to implement them in nanomaterials for small scale effect. As reported in several studies, the length scales associated with material microstructures are often small enough to call the capability of the classical continuum models into question. Hence, to extend the continuum mechanics to accommodate the size dependence of microstructures becomes a topic of major concern. Modified continuum models are among the most applied theoretical approaches for the investigation of nanomechanics due to their computational efficiency and capability of producing accurate results which are comparable to those from the atomistic models [4–23].

Surface stress effect is one of these effects. The high surfacetobulk ratio causes nanostructures to behave differently compared with conventional structures. Gurtin and Murdoch [24,25] developed a theoretical framework based on continuum mechanics including surface stress effects, in which the surface was simulated as a mathematical layer of zero thickness with different material properties from the underlying bulk which was completely bonded by the membrane. They presented the following general and simple expression for the surface stress-strain relation

$$\begin{aligned} \sigma_{ij}^s &= \tau_s \delta_{ij} + (\tau_s + \lambda_s) \varepsilon_{kk} \delta_{ij} + 2(\mu_s - \tau_s) \varepsilon_{ij} + \tau_s u_{i,j}^s \quad (i, j = x, y) \\ \sigma_{iz}^s &= \tau_s u_{z,i}^s \end{aligned} \quad (1)$$

where λ_s and μ_s are the surface Lamé constants, and τ_s denotes the residual surface stress under unstrained conditions.

It should be noted that the surface elastic constants are quite different from the bulk elastic constants as the former have dimensions of N/m, which are different from the latter with dimensions of N/m². It has been shown that the surface

can be assumed to behave isotropically, and it is still meaningful to use appropriate average of surface stress [25–27].

The Gurtin-Murdoch model has the capability of incorporating the effects of surface free energy into the mechanical response of nanostructures efficiently. This fact has been examined in several studies conducted for different problems about the mechanical behavior of structures at nanoscale [28–52].

In the current study, a non-classical shell model is introduced in order to predict the effects of surface free energy on the nonlinear buckling and postbuckling behavior of cylindrical shear deformable nanoshells subjected to radial compressive load. For this purpose, novel size-dependent differential equations are derived on the basis of Gurtin-Murdoch elasticity theory in conjunction with the first-order shear deformation shell theory. Afterwards, the size-dependent governing differential equations are converted into the boundary layer-type equations. Subsequently, a two-stepped singular perturbation technique is utilized to obtain the size-dependent critical buckling pressures and the postbuckling equilibrium paths for nanoshells with different geometric parameters.

2. Mathematical formulations

As shown in Fig. 1, a cylindrical shear-deformable nanoshell with length L , thickness h , and mid-surface radius R is considered. The nanoshell includes a bulk part and two additional thin surface layers (the inner and outer layers). For the bulk part, the material properties are Young’s modulus E and Poisson’s ratio ν . The two surface layers are assumed to have

surface elasticity modulus E_s , Poisson's ratio ν_s and residual surface tension τ_s . According to a curvilinear coordinate system with its origin located on the middle surface of nanoshell, coordinates of a typical point in the axial, circumferential and radial directions are denoted by x, y and z , respectively. Now, in accordance with the classical shell theory, the displacement field can be expressed as

$$u_x(x, y, z) = u(x, y) + z\psi_x(x, y) \tag{2a}$$

$$u_y(x, y, z) = v(x, y) + z\psi_y(x, y) \tag{2b}$$

$$u_z(x, y, z) = w(x, y) \tag{2c}$$

in which u, v and w denote the middle surface displacements along the x, y and z axis, respectively, and ψ_x and ψ_y are respectively the rotations of middle surface normals about the y - and x - axis.

Based upon the von Karman-Donnell-type kinematics of nonlinearity [53], it is assumed that the thickness of the shell, h , is remarkably small in comparison with its radius of curvature R , and the kinematical strain-displacement relationship can be given as follows

$$\begin{aligned} \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} &= \begin{Bmatrix} \varepsilon_{xx}^0 \\ \varepsilon_{yy}^0 \\ \gamma_{xy}^0 \end{Bmatrix} + Z \begin{Bmatrix} \kappa_{xx} \\ \kappa_{yy} \\ \kappa_{xy} \end{Bmatrix} \\ &= \begin{Bmatrix} \frac{\partial u}{\partial x} + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 + \frac{\partial w}{\partial x} \frac{\partial w^*}{\partial x} \\ \frac{\partial v}{\partial y} - \frac{w+w^*}{R} + \frac{1}{2} \left(\frac{\partial w}{\partial y} \right)^2 + \frac{\partial w}{\partial y} \frac{\partial w^*}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + \frac{\partial w}{\partial x} \frac{\partial w}{\partial y} + \frac{\partial w}{\partial x} \frac{\partial w^*}{\partial y} + \frac{\partial w}{\partial y} \frac{\partial w^*}{\partial x} \end{Bmatrix} \\ &\quad + Z \begin{Bmatrix} \frac{\partial \psi_x}{\partial x} \\ \frac{\partial \psi_y}{\partial y} \\ \frac{\partial \psi_x}{\partial y} + \frac{\partial \psi_y}{\partial x} \end{Bmatrix} \end{aligned} \tag{3a}$$

$$\begin{Bmatrix} \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix} = \begin{Bmatrix} \psi_x + \frac{\partial w}{\partial x} \\ \psi_y + \frac{\partial w}{\partial y} \end{Bmatrix} \tag{3b}$$

where $\varepsilon_{xx}^0, \varepsilon_{yy}^0, \gamma_{xy}^0$ are the strain components of the middle surface, $\kappa_{xx}, \kappa_{yy}, \kappa_{xy}$ denote the curvature components of nanoshell, and w^* stands for the initial geometric imperfection.

Afterwards, the constitutive relations can be written as

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \\ \sigma_{xz} \\ \sigma_{yz} \end{Bmatrix} = \begin{bmatrix} \lambda + 2\mu & \lambda & 0 & 0 & 0 \\ \lambda & \lambda + 2\mu & 0 & 0 & 0 \\ 0 & 0 & \mu & 0 & 0 \\ 0 & 0 & 0 & \mu & 0 \\ 0 & 0 & 0 & 0 & \mu \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \\ \gamma_{xz} \\ \gamma_{yz} \end{Bmatrix} \tag{4}$$

in which $\Lambda = (E^*\nu)/((1+\nu)(1-2^*\nu))$ and $\mu = E/(2(1+\nu))$ are Lamé constants.

The classical concepts of continuum mechanics are incapable of considering the atomic features of nanostructures. However, for simplicity and computational efficiency of continuum mechanics, different modified continuum models have been developed to incorporate size-effects into the conventional continuum approach. In the present work, the modified continuum elasticity based on Gurtin-Murdoch theory is utilized to develop a size-dependent shell model for nonlinear

buckling and postbuckling analysis of nanoshells in the presence of surface free energy effects. On the basis of equation (1), the components of surface stress can be determined with respect to the displacement components as follow

$$\begin{aligned} \sigma_{xx}^s &= (\lambda_s + 2\mu_s)\varepsilon_{xx} + (\tau_s + \lambda_s)\varepsilon_{yy} + \tau_s - \frac{\tau_s}{2} \left(\frac{\partial(w+w^*)}{\partial x} \right)^2 \\ \sigma_{yy}^s &= (\lambda_s + 2\mu_s)\varepsilon_{yy} + (\tau_s + \lambda_s)\varepsilon_{xx} + \frac{\tau_s}{R}(w+w^*) \\ &\quad + \tau_s - \frac{\tau_s}{2} \left(\frac{\partial(w+w^*)}{\partial y} \right)^2 \\ \sigma_{xy}^s &= \mu_s\gamma_{xy} - \tau_s \left(\frac{\partial v}{\partial x} + \frac{\partial(w+w^*)}{\partial x} \frac{\partial(w+w^*)}{\partial y} + Z \frac{\partial \psi_y}{\partial x} \right) \\ \sigma_{yx}^s &= \mu_s\gamma_{xy} - \tau_s \left(\frac{\partial u}{\partial y} + \frac{\partial(w+w^*)}{\partial x} \frac{\partial(w+w^*)}{\partial y} + Z \frac{\partial \psi_x}{\partial y} \right) \\ \sigma_{xz}^s &= \tau_s \frac{\partial w}{\partial x}, \quad \sigma_{yz}^s = \tau_s \frac{\partial w}{\partial y} \end{aligned} \tag{5}$$

In the classical shell theories, because the stress component σ_{zz} is small in comparison with other components of stress, it is assumed that $\sigma_{zz} = 0$. But this assumption does not satisfy the surface conditions related to the Gurtin-Murdoch model. To solve this problem, it is assumed that the stress component σ_{zz} varies linearly through the shell thickness and satisfies the balance conditions on the surfaces [39]. According to this assumption, σ_{zz} can be obtained as

$$\begin{aligned} \sigma_{zz} &= \frac{\left(\frac{\partial \sigma_{xz}^{S+}}{\partial x} + \frac{\partial \sigma_{yz}^{S+}}{\partial y} \right) - \left(\frac{\partial \sigma_{xz}^{S-}}{\partial x} + \frac{\partial \sigma_{yz}^{S-}}{\partial y} \right)}{2} \\ &\quad + \frac{\left(\frac{\partial \sigma_{xz}^{S+}}{\partial x} + \frac{\partial \sigma_{yz}^{S+}}{\partial y} \right) + \left(\frac{\partial \sigma_{xz}^{S-}}{\partial x} + \frac{\partial \sigma_{yz}^{S-}}{\partial y} \right)}{h} z \end{aligned} \tag{6}$$

in which the superscripts S^+ and S^- refer to the inner and outer surfaces of nanoshell, respectively. Through inserting equation (5) into equation (6), σ_{zz} can be achieved as follows

$$\sigma_{zz} = \frac{2\tau_s z}{h} \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) \tag{7}$$

As a result, by substituting σ_{zz} into the constitutive equation (3) corresponding to the normal stresses (σ_{xx}, σ_{yy}) for the bulk of the nanoshell, one will have

$$\sigma_{xx} = (\lambda + 2\mu)\varepsilon_{xx} + \lambda\varepsilon_{yy} + \frac{\nu\sigma_{zz}}{(1-\nu)} \tag{8a}$$

$$\sigma_{yy} = (\lambda + 2\mu)\varepsilon_{yy} + \lambda\varepsilon_{xx} + \frac{\nu\sigma_{zz}}{(1-\nu)} \tag{8b}$$

Based on the continuum surface elasticity theory, the total strain energy of a cylindrical shear-deformable nanoshell in the presence of surface stress effects can be expressed as

$$\begin{aligned} \Pi_s &= \frac{1}{2} \int_S \int_{-\frac{h}{2}}^{\frac{h}{2}} \sigma_{ij}\varepsilon_{ij} dz dS + \frac{1}{2} \left(\int_{S^+} \sigma_{ij}^s \varepsilon_{ij} dS^+ + \int_{S^-} \sigma_{ij}^s \varepsilon_{ij} dS^- \right) \\ &= \frac{1}{2} \int_S \left\{ \bar{N}_{xx}\varepsilon_{xx}^0 + \bar{N}_{yy}\varepsilon_{yy}^0 + \bar{N}_{xy}\gamma_{xy}^0 + \bar{M}_{xx}\kappa_{xx} + \bar{M}_{yy}\kappa_{yy} \right. \\ &\quad \left. + \bar{M}_{xy}\kappa_{xy} + Q_x\gamma_{xz} + Q_y\gamma_{yz} + Q_x^s \frac{\partial w}{\partial x} + Q_y^s \frac{\partial w}{\partial y} \right\} dS \end{aligned} \tag{9}$$

where S is the area occupied by the middle surface of the nanoshell. In equation (9), the resultants including in-plane

forces, bending moments and shear forces can be expressed as

$$\begin{aligned}\bar{N}_{xx} &= N_{xx} + \sigma_{xx}^{s+} + \sigma_{xx}^{s-} = A_{11}^* \varepsilon_{xx}^0 + A_{12}^* \varepsilon_{yy}^0 + 2\tau_s - \tau_s \left(\frac{\partial(w+w^*)}{\partial x} \right)^2 \\ \bar{N}_{yy} &= N_{yy} + \sigma_{yy}^{s+} + \sigma_{yy}^{s-} = A_{11}^* \varepsilon_{yy}^0 + A_{12}^* \varepsilon_{xx}^0 + \frac{2\tau_s}{R}(w+w^*) \\ &\quad + 2\tau_s - \tau_s \left(\frac{\partial(w+w^*)}{\partial y} \right)^2 \\ \bar{N}_{xy} &= N_{xy} + \frac{1}{2} (\sigma_{xy}^{s+} + \sigma_{yx}^{s+} + \sigma_{xy}^{s-} + \sigma_{yx}^{s-}) \\ &= A_{55}^* \gamma_{xy}^0 - \tau_s \frac{\partial(w+w^*)}{\partial x} \frac{\partial(w+w^*)}{\partial y} \\ \bar{M}_{xx} &= M_{xx} + \frac{h}{2} (\sigma_{xx}^{s+} - \sigma_{xx}^{s-}) = D_{11}^* \kappa_{xx} + D_{12}^* \kappa_{yy} \\ &\quad + E_{11}^* \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) \\ \bar{M}_{yy} &= M_{yy} + \frac{h}{2} (\sigma_{yy}^{s+} - \sigma_{yy}^{s-}) = D_{11}^* \kappa_{yy} + D_{12}^* \kappa_{xx} \\ &\quad + E_{11}^* \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) \\ \bar{M}_{xy} &= M_{xy} + \frac{h}{4} (\sigma_{xy}^{s+} + \sigma_{yx}^{s+} - \sigma_{xy}^{s-} - \sigma_{yx}^{s-}) = D_{55}^* \kappa_{xy} \\ Q_x &= k_s A_{55}^* \gamma_{xz}, \quad Q_y = k_s A_{55}^* \gamma_{yz} \\ Q_x^s &= \sigma_{xz}^{s-} + \sigma_{xz}^{s+} = 2\tau_s \frac{\partial w}{\partial x}, \quad Q_y^s = \sigma_{yz}^{s-} + \sigma_{yz}^{s+} = 2\tau_s \frac{\partial w}{\partial y}\end{aligned}\quad (10)$$

in which k_s represents shear correction factor, and

$$\begin{aligned}\begin{Bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{Bmatrix} &= \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} dz, \quad \begin{Bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{Bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{Bmatrix} z dz, \\ \begin{Bmatrix} Q_x \\ Q_y \end{Bmatrix} &= k_s \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{Bmatrix} \sigma_{xz} \\ \sigma_{yz} \end{Bmatrix} dz\end{aligned}\quad (11)$$

and

$$\begin{aligned}A_{11}^* &= (\lambda + 2\mu)h + 2(\lambda_s + 2\mu_s), \quad A_{12}^* = \lambda h + 2\tau_s + 2\lambda_s, \\ A_{55}^* &= \mu h + 2\mu_s - \tau_s, \quad D_{11}^* = \frac{(\lambda + 2\mu)h^3}{12} + \frac{(\lambda_s + 2\mu_s)h^2}{2}, \\ D_{12}^* &= \frac{\lambda h^3}{12} + \frac{(\tau_s + \lambda_s)h^2}{2}, \quad E_{11}^* = \frac{\nu h^2 \tau_s}{6(1-\nu)}, \\ D_{55}^* &= \frac{\mu h^3}{12} + \frac{(2\mu_s - \tau_s)h^2}{4}\end{aligned}\quad (12)$$

Moreover, the work Π_P done by the external radial load, q can be expressed in the following form

$$\Pi_P = \int_S q w dS \quad (13)$$

Now, by using the below virtual work's principle

$$\delta \int_{t_1}^{t_2} (\Pi_s - \Pi_P) dt = 0 \quad (14)$$

and taking the variations of u , v , w , ψ_x and ψ_y integrating by parts, the non-classical governing differential equations can be derived as

$$\frac{\partial \bar{N}_{xx}}{\partial x} + \frac{\partial \bar{N}_{xy}}{\partial y} = 0 \quad (15a)$$

$$\frac{\partial \bar{N}_{xy}}{\partial x} + \frac{\partial \bar{N}_{yy}}{\partial y} = 0 \quad (15b)$$

$$\begin{aligned}\frac{\partial Q_x}{\partial x} + \frac{\partial Q_y}{\partial y} + \frac{\partial Q_x^s}{\partial x} + \frac{\partial Q_y^s}{\partial y} + \frac{\bar{N}_{yy}}{R} + \bar{N}_{xx} \frac{\partial^2(w+w^*)}{\partial x^2} \\ + 2\bar{N}_{xy} \frac{\partial^2(w+w^*)}{\partial x \partial y} + \bar{N}_{yy} \frac{\partial^2(w+w^*)}{\partial y^2} + q = 0\end{aligned}\quad (15c)$$

$$\frac{\partial \bar{M}_{xx}}{\partial x} + \frac{\partial \bar{M}_{xy}}{\partial y} - Q_x = 0 \quad (15d)$$

$$\frac{\partial \bar{M}_{xy}}{\partial x} + \frac{\partial \bar{M}_{yy}}{\partial y} - Q_y = 0 \quad (15e)$$

The Airy stress function $f(x, y)$ can be introduced as

$$\bar{N}_{xx} = \frac{\partial^2 f}{\partial y^2}, \quad \bar{N}_{yy} = \frac{\partial^2 f}{\partial x^2}, \quad \bar{N}_{xy} = -\frac{\partial^2 f}{\partial x \partial y} \quad (16)$$

As a result, the strain components can be expressed as

$$\begin{aligned}\varepsilon_{xx}^0 &= -\varphi_2 \frac{\partial^2 f}{\partial x^2} + \varphi_1 \frac{\partial^2 f}{\partial y^2} + 2\tau_s \varphi_2 \frac{w+w^*}{R} - \frac{2\tau_s}{A_{11}^* + A_{12}^*} \\ &\quad + \tau_s \varphi_1 \left(\frac{\partial(w+w^*)}{\partial x} \right)^2 - \tau_s \varphi_2 \left(\frac{\partial(w+w^*)}{\partial y} \right)^2\end{aligned}\quad (17a)$$

$$\begin{aligned}\varepsilon_{yy}^0 &= -\varphi_2 \frac{\partial^2 f}{\partial y^2} + \varphi_1 \frac{\partial^2 f}{\partial x^2} - 2\tau_s \varphi_1 \frac{w+w^*}{R} - \frac{2\tau_s}{A_{11}^* + A_{12}^*} \\ &\quad + \tau_s \varphi_1 \left(\frac{\partial(w+w^*)}{\partial y} \right)^2 - \tau_s \varphi_2 \left(\frac{\partial(w+w^*)}{\partial x} \right)^2\end{aligned}\quad (17b)$$

$$\gamma_{xy}^0 = -\varphi_3 \frac{\partial^2 f}{\partial x \partial y} + \tau_s \varphi_3 \frac{\partial(w+w^*)}{\partial x} \frac{\partial(w+w^*)}{\partial y} \quad (17c)$$

in which

$$\varphi_1 = \frac{A_{11}^*}{(A_{11}^*)^2 - (A_{12}^*)^2}, \quad \varphi_2 = \frac{A_{12}^*}{(A_{11}^*)^2 - (A_{12}^*)^2}, \quad \varphi_3 = \frac{1}{A_{55}^*} \quad (18)$$

Also, the geometrical compatibility equation for an imperfect cylindrical shell is written as

$$\begin{aligned}\frac{\partial^2 \varepsilon_{xx}^0}{\partial y^2} + \frac{\partial^2 \varepsilon_{yy}^0}{\partial x^2} - \frac{\partial^2 \gamma_{xy}^0}{\partial x \partial y} &= \left(\frac{\partial^2 w}{\partial x \partial y} \right)^2 - \frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w}{\partial y^2} + 2 \frac{\partial^2 w}{\partial x \partial y} \frac{\partial^2 w^*}{\partial x \partial y} \\ &\quad - \frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w^*}{\partial y^2} - \frac{\partial^2 w}{\partial y^2} \frac{\partial^2 w^*}{\partial x^2} - \frac{1}{R} \frac{\partial^2 w}{\partial x^2}\end{aligned}\quad (19)$$

From differential equations of (15c) and (19) and with the aid of equations (10) and (17), the nonlinear size-dependent governing differential equations in terms of f , w , ψ_x , and ψ_y can be expressed as

$$\begin{aligned}-D_{11}^* \frac{\partial^3 \psi_x}{\partial x^3} + \varphi_4 \frac{\partial^3 \psi_x}{\partial x \partial y^2} - D_{11}^* \frac{\partial^3 \psi_y}{\partial y^3} + \varphi_4 \frac{\partial^3 \psi_y}{\partial x^2 \partial y} \\ - E_{11}^* \left(\frac{\partial^4 w}{\partial x^4} + 2 \frac{\partial^4 w}{\partial x^2 \partial y^2} + \frac{\partial^4 w}{\partial y^4} \right) - 2\tau_s \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) - \frac{1}{R} \frac{\partial^2 f}{\partial x^2} \\ = \frac{\partial^2(w+w^*)}{\partial x^2} \frac{\partial^2 f}{\partial y^2} - 2 \frac{\partial^2(w+w^*)}{\partial x \partial y} \frac{\partial^2 f}{\partial x \partial y} \\ + \frac{\partial^2(w+w^*)}{\partial y^2} \frac{\partial^2 f}{\partial x^2} + q\end{aligned}\quad (20a)$$

$$\begin{aligned}\varphi_1 \frac{\partial^4 f}{\partial x^4} + (\varphi_3 - 2\varphi_2) \frac{\partial^4 f}{\partial x^2 \partial y^2} + \varphi_1 \frac{\partial^4 f}{\partial y^4} + \frac{2\tau_s}{R} \left(-\varphi_1 \frac{\partial^2 w}{\partial x^2} + \varphi_2 \frac{\partial^2 w}{\partial y^2} \right) \\ + \frac{1}{R} \frac{\partial^2 w}{\partial x^2} = \left(\frac{\partial^2(w+w^*)}{\partial x \partial y} \right)^2 - \frac{\partial^2(w+w^*)}{\partial x^2} \frac{\partial^2(w+w^*)}{\partial y^2}\end{aligned}$$

$$\begin{aligned}
 & -2\tau_s\varphi_1\left(\frac{\partial^3(w+w^*)}{\partial x\partial y^2}\frac{\partial(w+w^*)}{\partial x}+2\left(\frac{\partial^2(w+w^*)}{\partial x\partial y}\right)^2\right. \\
 & +\left.\frac{\partial^3(w+w^*)}{\partial x^2\partial y}\frac{\partial(w+w^*)}{\partial y}\right)+2\tau_s\varphi_2\left(\frac{\partial^3(w+w^*)}{\partial x^3}\frac{\partial(w+w^*)}{\partial x}\right. \\
 & +\left.\frac{\partial^3(w+w^*)}{\partial y^3}\frac{\partial(w+w^*)}{\partial y}+\left(\frac{\partial^2(w+w^*)}{\partial x^2}\right)^2+\left(\frac{\partial^2(w+w^*)}{\partial y^2}\right)^2\right) \\
 & +\tau_s\varphi_3\left(\frac{\partial^3(w+w^*)}{\partial x^2\partial y}\frac{\partial(w+w^*)}{\partial y}+\frac{\partial^2(w+w^*)}{\partial x^2}\frac{\partial^2(w+w^*)}{\partial y^2}\right. \\
 & \left.+\left(\frac{\partial^2(w+w^*)}{\partial x\partial y}\right)^2+\frac{\partial(w+w^*)}{\partial x}\frac{\partial^3(w+w^*)}{\partial x\partial y^2}\right) \quad (20b)
 \end{aligned}$$

$$\begin{aligned}
 D_{11}^*\frac{\partial^2\psi_x}{\partial x^2}+D_{55}^*\frac{\partial^2\psi_x}{\partial y^2}-k_sA_{55}^*\left(\psi_x+\frac{\partial w}{\partial x}\right)+\varphi_5\frac{\partial^2\psi_y}{\partial x\partial y} \\
 +E_{11}^*\left(\frac{\partial^3w}{\partial x^3}+\frac{\partial^3w}{\partial x\partial y^2}\right)=0 \quad (20c)
 \end{aligned}$$

$$\begin{aligned}
 D_{11}^*\frac{\partial^2\psi_y}{\partial y^2}+D_{55}^*\frac{\partial^2\psi_y}{\partial x^2}-k_sA_{55}^*\left(\psi_y+\frac{\partial w}{\partial y}\right)+\varphi_5\frac{\partial^2\psi_x}{\partial x\partial y} \\
 +E_{11}^*\left(\frac{\partial^3w}{\partial x^2\partial y}+\frac{\partial^3w}{\partial y^3}\right)=0 \quad (20d)
 \end{aligned}$$

where

$$\varphi_4=-D_{12}^*-2D_{55}^*, \quad \varphi_5=D_{12}^*+D_{55}^* \quad (21)$$

It is assumed that the end supports of nanoshell are simply supported or clamped. Therefore, the boundary conditions at $x=0, L$ can be expressed as:

For simply supported edge supports: $w=0, \bar{M}_{xx}=0$

For clamped edge supports: $w=0, \frac{\partial w}{\partial x}=0$

Additionally, it is clear that

$$\int_0^{2\pi R} \bar{N}_{xx}dy+\pi R^2q=0 \quad (22)$$

Moreover, the closed condition (periodicity) can be given as follows

$$\int_0^{2\pi R} \frac{\partial v}{\partial y}dy=0 \quad (23a)$$

which yields

$$\begin{aligned}
 \int_0^{2\pi R} \left(-\varphi_2\frac{\partial^2f}{\partial y^2}+\varphi_1\frac{\partial^2f}{\partial x^2}-2\tau_s\varphi_1\frac{w+w^*}{R}-\frac{2\tau_s}{A_{11}^*+A_{12}^*}\right. \\
 \left.+\tau_s\varphi_1\left(\frac{\partial(w+w^*)}{\partial y}\right)^2-\tau_s\varphi_2\left(\frac{\partial(w+w^*)}{\partial x}\right)^2+\frac{w+w^*}{R}-\frac{1}{2}\left(\frac{\partial w}{\partial y}\right)^2\right. \\
 \left.-\frac{\partial w}{\partial y}\frac{\partial w^*}{\partial y}\right)dy=0 \quad (23b)
 \end{aligned}$$

The average end shortening of nanoshell can be introduced as

$$\begin{aligned}
 \frac{\Delta_x}{L} & =-\frac{1}{2\pi RL}\int_0^{2\pi R}\int_0^L\frac{\partial u}{\partial x}dxdy \\
 & =-\frac{1}{2\pi RL}\int_0^{2\pi R}\int_0^L\left(-\varphi_2\frac{\partial^2f}{\partial x^2}+\varphi_1\frac{\partial^2f}{\partial y^2}+2\tau_s\varphi_2\frac{w+w^*}{R}\right. \\
 & \quad \left.-\frac{2\tau_s}{A_{11}^*+A_{12}^*}+\varphi_1\tau_s\left(\frac{\partial(w+w^*)}{\partial x}\right)^2-\varphi_2\tau_s\left(\frac{\partial(w+w^*)}{\partial y}\right)^2\right. \\
 & \quad \left.-\frac{1}{2}\left(\frac{\partial w}{\partial x}\right)^2-\frac{\partial w}{\partial x}\frac{\partial w^*}{\partial x}\right)dxdy \quad (24)
 \end{aligned}$$

3. Solution procedure

3.1. Boundary layer-type governing equations

In order to perform the solution methodology, the following dimensionless parameters are defined

$$\begin{aligned}
 X & =\frac{\pi x}{L}, \quad Y=\frac{y}{R}, \quad \beta=\frac{L}{\pi R}, \quad \eta=\frac{L^2}{\pi^2h^2}, \quad \epsilon=\frac{\pi^2Rh}{L^2} \\
 & \{a_{11}^*, a_{12}^*, a_{55}^*, d_{11}^*, d_{12}^*, d_{55}^*, e_{11}^*\} \\
 & =\left\{\frac{A_{11}^*}{A_{110}}, \frac{A_{12}^*}{A_{110}}, \frac{A_{55}^*}{A_{110}}, \frac{D_{11}^*}{A_{110}h^2}, \frac{D_{12}^*}{A_{110}h^2}, \frac{D_{55}^*}{A_{110}h^2}, \frac{E_{11}^*}{A_{110}h^2}\right\} \\
 \{W, W^*\} & =\frac{\epsilon}{h}\{w, w^*\}, \quad F=\frac{\epsilon^2f}{A_{110}h^2}, \quad \{\psi_x, \psi_y\} \\
 & =\frac{\epsilon^2L}{\pi h}\{\psi_x, \psi_y\}, \quad \bar{\tau}=\frac{\tau_s}{A_{110}} \\
 \{\mathcal{M}_{xx}, \mathcal{M}_{yy}\} & =\frac{\epsilon^2L^2\{\bar{M}_{xx}, \bar{M}_{yy}\}}{\pi^2A_{110}h^3}, \quad P_q=\frac{3^{3/4}qLR^{3/2}}{4\pi A_{110}h^{3/2}}, \\
 \delta_q & =\frac{3^{3/4}\Delta_x\sqrt{R}}{4\pi h^{3/2}} \quad (25)
 \end{aligned}$$

in which $A_{110}=(\lambda+2\mu)h$. Now, by introducing the below derivative operators

$$\begin{aligned}
 \mathcal{L}_{11}() & =-e_{11}^*\frac{\partial^4}{\partial X^4}-2e_{11}^*\beta^2\frac{\partial^4}{\partial X^2\partial Y^2}-e_{11}^*\beta^4\frac{\partial^4}{\partial Y^4} \quad (26) \\
 \mathcal{L}_{12}() & =-\eta\frac{\partial^2}{\partial X^2}-\eta\beta^2\frac{\partial^2}{\partial Y^2} \\
 \mathcal{L}_{13}() & =-d_{11}^*\frac{\partial^3}{\partial X^3}+\vartheta_4\beta^2\frac{\partial^3}{\partial X\partial Y^2} \\
 \mathcal{L}_{14}() & =-d_{11}^*\beta^3\frac{\partial^3}{\partial Y^3}+\vartheta_4\beta\frac{\partial^3}{\partial X^2\partial Y} \\
 \mathcal{L}_{21}() & =\vartheta_1\frac{\partial^4}{\partial X^4}+(\vartheta_3-2\vartheta_2)\beta^2\frac{\partial^4}{\partial X^2\partial Y^2}+\vartheta_1\beta^4\frac{\partial^4}{\partial Y^4} \\
 \mathcal{L}_{22}() & =-\vartheta_1\frac{\partial^2}{\partial X^2}+\vartheta_2\beta^2\frac{\partial^2}{\partial Y^2} \\
 \mathcal{L}_{31}() & =-k_s a_{55}^*\frac{\partial}{\partial X}+e_{11}^*\frac{\partial^3}{\partial X^3}+e_{11}^*\beta^2\frac{\partial^3}{\partial X\partial Y^2} \\
 \mathcal{L}_{32}() & =-k_s a_{55}^*+d_{11}^*\frac{\partial^2}{\partial X^2}+d_{55}^*\beta^2\frac{\partial^2}{\partial Y^2} \\
 \mathcal{L}_{41}() & =-k_s a_{55}^*\beta\frac{\partial}{\partial Y}+e_{11}^*\beta\frac{\partial^3}{\partial X^2\partial Y}+e_{11}^*\beta^3\frac{\partial^3}{\partial Y^3} \\
 \mathcal{L}_{42}() & =-k_s a_{55}^*+d_{55}^*\frac{\partial^2}{\partial X^2}+d_{11}^*\beta^2\frac{\partial^2}{\partial Y^2} \\
 \tilde{\mathcal{L}}_1() & =\frac{\partial^2}{\partial X^2}\frac{\partial^2}{\partial Y^2}-2\frac{\partial^2}{\partial X\partial Y}\frac{\partial^2}{\partial X\partial Y}+\frac{\partial^2}{\partial Y^2}\frac{\partial^2}{\partial X^2} \\
 \tilde{\mathcal{L}}_2() & =\frac{\partial^3}{\partial X\partial Y^2}\frac{\partial}{\partial X}+\frac{\partial}{\partial X}\frac{\partial^3}{\partial X\partial Y^2}+4\frac{\partial^2}{\partial X\partial Y}\frac{\partial^2}{\partial X\partial Y}+\frac{\partial}{\partial Y}\frac{\partial^3}{\partial X^2\partial Y} \\
 & \quad +\frac{\partial^3}{\partial X^2\partial Y}\frac{\partial}{\partial Y} \\
 \tilde{\mathcal{L}}_3() & =\frac{\partial^3}{\partial X^3}\frac{\partial}{\partial X}+\frac{\partial}{\partial X}\frac{\partial^3}{\partial X^3}+\beta^4\frac{\partial^3}{\partial Y^3}\frac{\partial}{\partial Y}+\beta^4\frac{\partial}{\partial Y}\frac{\partial^3}{\partial Y^3}+2\frac{\partial^2}{\partial X^2}\frac{\partial^2}{\partial X^2} \\
 & \quad +2\beta^4\frac{\partial^2}{\partial Y^2}\frac{\partial^2}{\partial Y^2} \\
 \tilde{\mathcal{L}}_4() & =\frac{\partial^2}{\partial X^2}\frac{\partial^2}{\partial Y^2}+2\frac{\partial^2}{\partial X\partial Y}\frac{\partial^2}{\partial X\partial Y}+\frac{\partial^2}{\partial Y^2}\frac{\partial^2}{\partial X^2}+\frac{\partial^3}{\partial X^2\partial Y}\frac{\partial}{\partial Y} \\
 & \quad +\frac{\partial}{\partial Y}\frac{\partial^3}{\partial X^2\partial Y}+\frac{\partial}{\partial X}\frac{\partial^3}{\partial X\partial Y^2}+\frac{\partial^3}{\partial X\partial Y^2}\frac{\partial}{\partial X}
 \end{aligned}$$

the dimensionless form of the size-dependent nonlinear governing differential equations can be expressed as

$$\begin{aligned} \epsilon^2 \mathcal{L}_{11}(W) + \epsilon^2 2\bar{\tau} \mathcal{L}_{12}(W) + \epsilon \mathcal{L}_{13}(\psi_x) + \epsilon \mathcal{L}_{14}(\psi_y) - \frac{\partial^2 F}{\partial X^2} \\ = \beta^2 \tilde{\mathcal{L}}_1(W + W^*, F) + \epsilon^{3/2} \frac{4}{3} 3^{1/4} \mathcal{P}_q \end{aligned} \quad (27a)$$

$$\begin{aligned} \mathcal{L}_{21}(F) + 2\bar{\tau} \mathcal{L}_{22}(W) + \frac{\partial^2 W}{\partial X^2} = & -\frac{\beta^2}{2} \tilde{\mathcal{L}}_1(W + 2W^*, W) \\ & -\bar{\tau} \beta^2 \tilde{\mathcal{L}}_2(W + 2W^*, W) \\ & + \bar{\tau} \vartheta_1 \tilde{\mathcal{L}}_3(W + 2W^*, W) \\ & + \frac{\bar{\tau} \vartheta_2 \beta^2}{2} \tilde{\mathcal{L}}_4(W + 2W^*, W) \end{aligned} \quad (27b)$$

$$\epsilon \mathcal{L}_{31}(W) + \mathcal{L}_{32}(\psi_x) + \vartheta_5 \beta \frac{\partial^2 \psi_y}{\partial X \partial Y} = 0 \quad (27c)$$

$$\epsilon \mathcal{L}_{41}(W) + \mathcal{L}_{42}(\psi_y) + \vartheta_5 \beta \frac{\partial^2 \psi_x}{\partial X \partial Y} = 0 \quad (27d)$$

where

$$\begin{aligned} \vartheta_1 = \frac{a_{11}^*}{(a_{11}^*)^2 - (a_{12}^*)^2}, \quad \vartheta_2 = \frac{a_{12}^*}{(a_{11}^*)^2 - (a_{12}^*)^2}, \quad \vartheta_3 = \frac{1}{a_{55}^*} \\ \vartheta_4 = -d_{12}^* - 2d_{55}^*, \quad \vartheta_5 = d_{12}^* + d_{55}^* \end{aligned} \quad (28)$$

Additionally, the boundary conditions in dimensionless form will be at $X = 0, \pi$

For simply supported edge supports: $W = 0, \quad \mathcal{M}_{xx} = 0$

For clamped edge supports: $W = 0, \quad \frac{\partial W}{\partial X} = 0$

Also, one will have

$$\frac{1}{2\pi} \int_0^{2\pi} \beta^2 \frac{\partial^2 F}{\partial Y^2} dY + \frac{2}{3} 3^{1/4} \epsilon^{3/2} \mathcal{P}_q P = 0 \quad (29)$$

and the closed condition becomes

$$\begin{aligned} \int_0^{2\pi} \left(-\vartheta_2 \frac{\partial^2 F}{\partial Y^2} + \vartheta_1 \frac{\partial^2 F}{\partial X^2} + (1 - 2\bar{\tau} \vartheta_1)(W + W^*) - \bar{\tau} \vartheta_2 \left(\frac{\partial(W + W^*)}{\partial X} \right)^2 \right. \\ \left. + (1 - 2\bar{\tau} \vartheta_1) \left(\frac{\partial(W + W^*)}{\partial Y} \right)^2 - 2\bar{\tau} \vartheta_6 (\vartheta_1 - \vartheta_2) \epsilon \right) dY = 0 \end{aligned} \quad (30)$$

In addition, the unit end shortening of nanoshell is achieved as

$$\begin{aligned} \vartheta_q = & -\frac{3^{3/4}}{8\pi^2 \epsilon^{3/2}} \int_0^{2\pi} \int_0^\pi \left(\vartheta_1 \beta^2 \frac{\partial^2 F}{\partial Y^2} - \vartheta_2 \frac{\partial^2 F}{\partial X^2} + 2\bar{\tau} \vartheta_2 (W + W^*) \right. \\ & - \frac{1}{2} (1 - 2\bar{\tau} \vartheta_1) \left(\frac{\partial(W + W^*)}{\partial X} \right)^2 - \bar{\tau} \vartheta_2 \beta^2 \left(\frac{\partial(W + W^*)}{\partial Y} \right)^2 \\ & \left. - 2\bar{\tau} \vartheta_6 (\vartheta_1 - \vartheta_2) \epsilon \right) dXdY \end{aligned} \quad (31)$$

where $\vartheta_6 = R/h$.

3.2. Singular perturbation technique

In the preceding subsection, the important parameter ϵ was introduced. It has been revealed that practically, for the shell structure, one will have $\frac{L^2}{Rh} \geq 20$, so $\epsilon < 1$ will always be true. Therefore, equation (27) represent the boundary layer type equations which consider both the nonlinear prebuckling deformations and large deflections in the postbuckling domain in conjunction with the effect of surface stress. Now, by

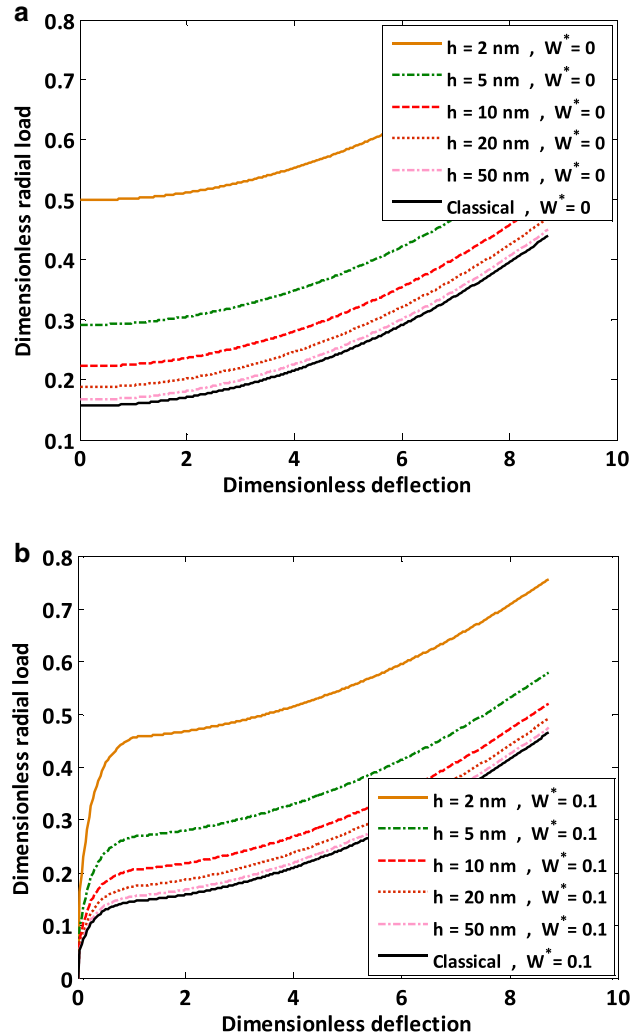


Fig. 2 – Dimensionless postbuckling load-deflection curves of nanoshells corresponding to various shell thicknesses: (a) perfect nanoshell, (b) imperfect nanoshell ($L^2/Rh = 200$, $R/h = 50$).

assuming ϵ as a small perturbation parameter, the singular perturbation technique can be put into use, which has been successfully applied to the nonlinear analyses of cylindrical shells at macroscale [54–58]. On the basis of this technique, it is assumed that

$$W = \bar{W}(X, Y, \epsilon) + \tilde{W}(X, Y, \epsilon, \xi) + \hat{W}(X, Y, \epsilon, \zeta) \quad (32a)$$

$$F = \bar{F}(X, Y, \epsilon) + \tilde{F}(X, Y, \epsilon, \xi) + \hat{F}(X, Y, \epsilon, \zeta) \quad (32b)$$

$$\psi_x = \bar{\psi}_x(X, Y, \epsilon) + \tilde{\psi}_x(X, Y, \epsilon, \xi) + \hat{\psi}_x(X, Y, \epsilon, \zeta) \quad (32c)$$

$$\psi_y = \bar{\psi}_y(X, Y, \epsilon) + \tilde{\psi}_y(X, Y, \epsilon, \xi) + \hat{\psi}_y(X, Y, \epsilon, \zeta) \quad (32d)$$

where $\bar{W}(X, Y, \epsilon)$, $\bar{F}(X, Y, \epsilon)$, $\bar{\psi}_x(X, Y, \epsilon)$, $\bar{\psi}_y(X, Y, \epsilon)$ denote regular solutions of the nanoshell, $\tilde{W}(X, Y, \epsilon, \xi)$, $\tilde{F}(X, Y, \epsilon, \xi)$, $\tilde{\psi}_x(X, Y, \epsilon, \xi)$, $\tilde{\psi}_y(X, Y, \epsilon, \xi)$ and $\hat{W}(X, Y, \epsilon, \zeta)$, $\hat{F}(X, Y, \epsilon, \zeta)$, $\hat{\psi}_x(X, Y, \epsilon, \zeta)$, $\hat{\psi}_y(X, Y, \epsilon, \zeta)$ are the boundary layer solutions corresponding to $X = 0$

Table 1 – Material properties of a cylindrical nanoshell made of silicon [59,60].

$E(\text{GPa})$	210
ν	0.24
$\mu_s(\text{N/m})$	-2.774
$\lambda_s(\text{N/m})$	-4.488
$\tau_s(\text{N/m})$	0.6048

Table 2 – Comparison of the critical buckling pressures of isotropic cylindrical shells with clamped edge supports subjected to lateral pressure ($\nu = 0.3, E = 200 \text{ GPa}$).

L/R	R/h	Present work (Pa)	Ref. [61] (Pa)
2	300	84991.09	85860
	3000	275.82	276.5
5	300	32897.22	32954
	3000	108.13	109

and $X = \pi$, respectively. These solutions can be expressed in the forms of perturbation expansions below

$$\begin{aligned}
 \bar{W}(X, Y, \epsilon) &= \sum_{i=0} \epsilon^{i/2} \bar{W}_{i/2}(X, Y), \quad \bar{F}(X, Y, \epsilon) = \sum_{i=0} \epsilon^{i/2} \bar{F}_{i/2}(X, Y) \\
 \bar{\psi}_x(X, Y, \epsilon) &= \sum_{i=1} \epsilon^{i/2} \bar{\psi}_{xi/2}(X, Y), \quad \bar{\psi}_y(X, Y, \epsilon) = \sum_{i=1} \epsilon^{i/2} \bar{\psi}_{yi/2}(X, Y) \\
 \bar{W}(X, Y, \epsilon, \xi) &= \sum_{i=0} \epsilon^{i/2+1} \bar{W}_{i/2+1}(X, Y, \xi), \\
 \bar{F}(X, Y, \epsilon, \xi) &= \sum_{i=0} \epsilon^{i/2+2} \bar{F}_{i/2+2}(X, Y, \xi) \\
 \bar{\psi}_x(X, Y, \epsilon, \xi) &= \sum_{i=0} \epsilon^{(i+3)/2} \bar{\psi}_{x(i+3)/2}(X, Y, \xi), \\
 \bar{\psi}_y(X, Y, \epsilon, \xi) &= \sum_{i=0} \epsilon^{i/2+2} \bar{\psi}_{yi/2+2}(X, Y, \xi) \\
 \hat{W}(X, Y, \epsilon, \varsigma) &= \sum_{i=0} \epsilon^{i/2+1} \hat{W}_{i/2+1}(X, Y, \varsigma), \\
 \hat{F}(X, Y, \epsilon, \varsigma) &= \sum_{i=0} \epsilon^{i/2+2} \hat{F}_{i/2+2}(X, Y, \varsigma) \\
 \hat{\psi}_x(X, Y, \epsilon, \varsigma) &= \sum_{i=0} \epsilon^{(i+3)/2} \hat{\psi}_{x(i+3)/2}(X, Y, \varsigma), \\
 \hat{\psi}_y(X, Y, \epsilon, \varsigma) &= \sum_{i=0} \epsilon^{i/2+2} \hat{\psi}_{yi/2+2}(X, Y, \varsigma) \tag{33}
 \end{aligned}$$

in which ξ and ς stand for boundary layer variables which are defined as

$$\xi = \frac{X}{\sqrt{\epsilon}}, \quad \varsigma = \frac{\pi - X}{\sqrt{\epsilon}} \tag{34}$$

Moreover, it is assumed that

$$\epsilon^{3/2} \frac{4}{3} 3^{1/4} q = \sum_{i=0} \epsilon^i Q_i \tag{35}$$

In order to determine the maximum order of ϵ which confirms the convergence of the solution procedure, an upper tolerance limit of 0.001 has been considered. It is observed that after the fourth order (ϵ^4), the error is smaller than the tolerance limit.

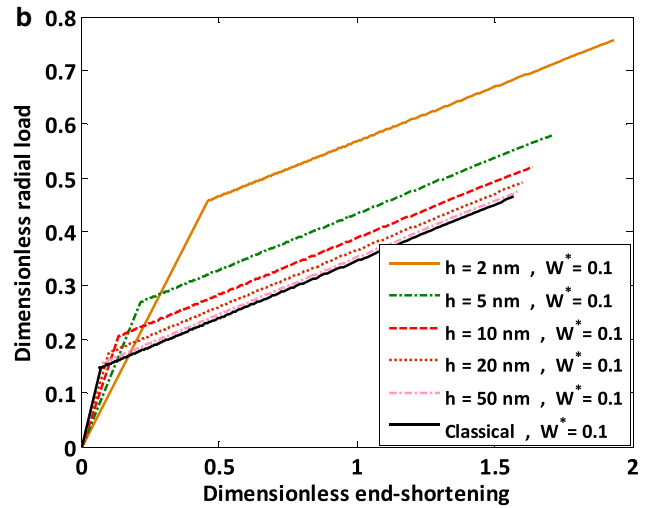
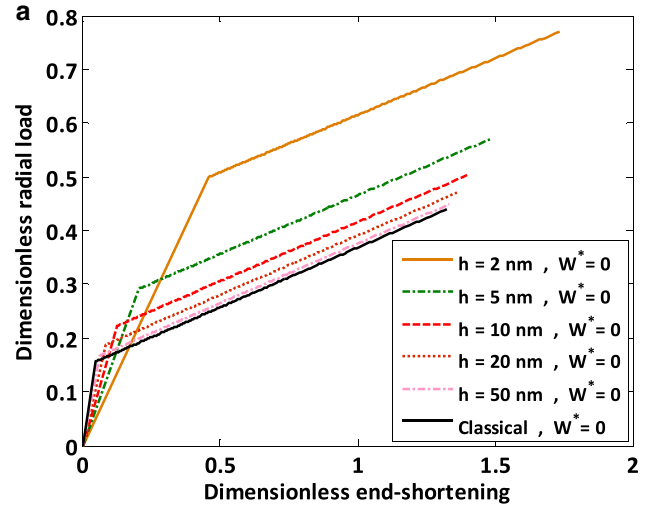


Fig. 3 – Dimensionless postbuckling load-shortening curves of nanoshells corresponding to various shell thicknesses: (a) perfect nanoshell, (b) imperfect nanoshell ($L^2/Rh = 200, R/h = 50$).

By inserting equations (32) and (33) into the size-dependent nonlinear governing differential equations (27) and collecting the expressions with the same order of ϵ , the sets of perturbation equations relevant to both the regular and boundary layer solutions will be derived, which are given in Appendix A. Afterwards, it is assumed that $\bar{W}_0(X, Y) = \mathcal{A}_{00}^{(0)}$, $\bar{W}_{1/2}(X, Y) = \bar{W}_1(X, Y) = 0$ and $\bar{W}_{3/2}(X, Y) = \mathcal{A}_{00}^{(3/2)}$, in addition to $\bar{F}_0(X, Y) = -\mathcal{B}_{00}^{(0)}(\beta^2 X^2 + \frac{Y^2}{2})$, $\bar{F}_{1/2}(X, Y) = \bar{F}_{3/2}(X, Y) = 0$ and $\bar{F}_1(X, Y) = -\mathcal{B}_{00}^{(1)}(\beta^2 X^2 + \frac{Y^2}{2})$. Moreover, the initial buckling mode and the initial geometric imperfection of the nanoshell are considered, respectively, as

$$\bar{W}_2(X, Y) = \mathcal{A}_{00}^{(2)} + \mathcal{A}_{11}^{(2)} \sin(mX) \sin(nY) \tag{36a}$$

$$\begin{aligned}
 W^*(X, Y, \epsilon) &= \epsilon^2 \mathcal{A}_{11}^* \sin(mX) \cos(nY) \\
 &= \epsilon^2 \ell \mathcal{A}_{11}^{(2)} \sin(mX) \cos(nY) \tag{36b}
 \end{aligned}$$

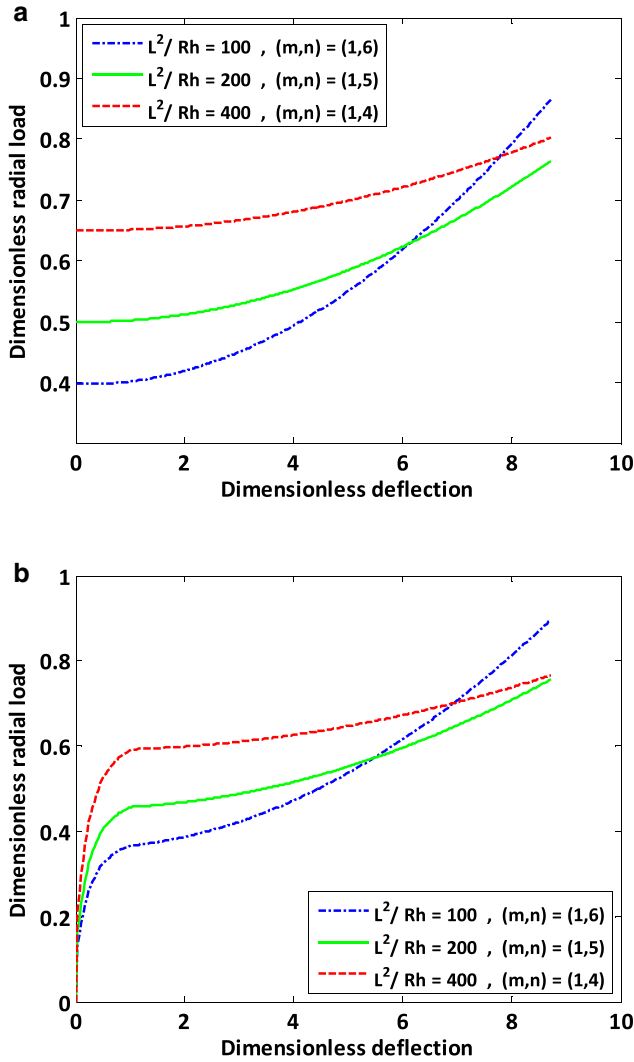


Fig. 4 – Dimensionless postbuckling load-deflection curves of nanoshells corresponding to various geometric parameters: (a) perfect nanoshell, (b) imperfect nanoshell ($R/h = 50$, $h = 2$ nm).

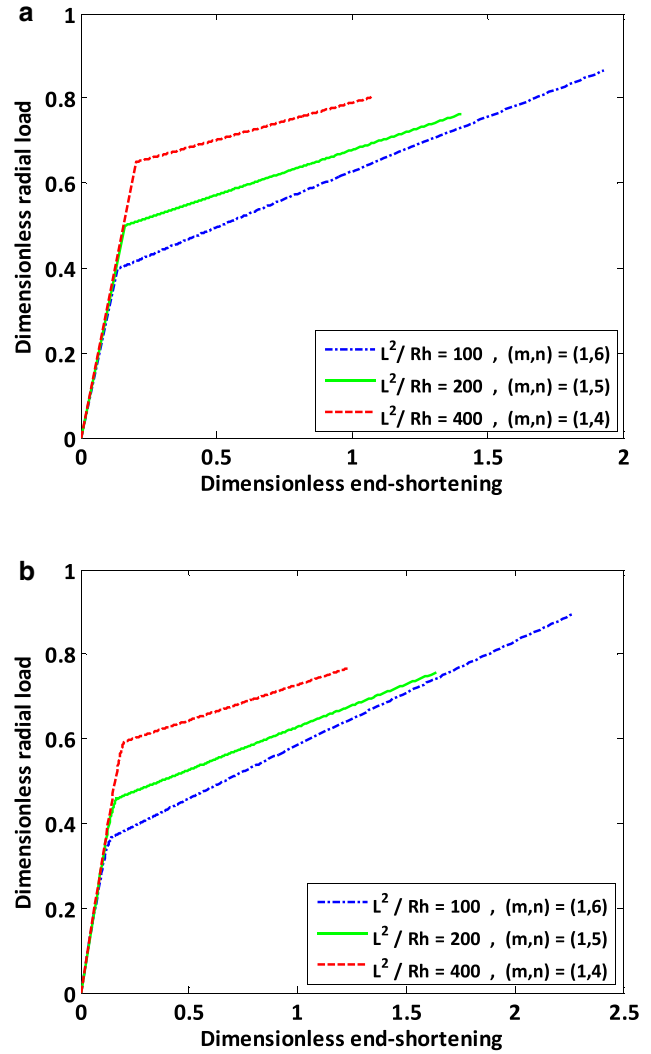


Fig. 5 – Dimensionless postbuckling load-shortening curves of nanoshells corresponding to various geometric parameters: (a) perfect nanoshell, (b) imperfect nanoshell ($R/h = 50$, $h = 2$ nm).

in which $\ell = \mathcal{A}_{11}^*/\mathcal{A}_{11}^{(2)}$ represents the imperfection parameter.

Now, by employing the given boundary, equation (29), closed conditions (30), and based on the unit end shortening (31), the postbuckling equilibrium paths can be derived as

$$\mathcal{P}_q = \frac{1}{2} 3^{3/4} \epsilon^{-3/2} \left\{ \mathcal{P}_q^{(0)} + \mathcal{P}_q^{(2)} \left(\mathcal{A}_{11}^{(2)} \epsilon^2 \right)^2 + \dots \right\} \quad (37)$$

and

$$\delta_q = \delta_q^{(0)} + \delta_q^{(2)} \left(\mathcal{A}_{11}^{(2)} \epsilon^2 \right)^2 + \dots \quad (38)$$

In accordance with the maximum dimensionless deflection of the nanoshell, $\mathcal{A}_{11}^{(2)} \epsilon^2$ is considered as the second perturbation parameter, which may be large in comparison with the first small perturbation parameter ϵ . If it is assumed that the maximum deflection occurs at the dimensionless point of $(X, Y) = (\pi/2m, \pi/2n)$, one will have

$$\mathcal{A}_{11}^{(2)} \epsilon^2 = \mathcal{W}_m + \mathcal{S}_1 \mathcal{W}_m^2 + \dots \quad (39)$$

in which \mathcal{W}_m denotes the maximum dimensionless deflection of the nanoshell as

$$\mathcal{W}_m = \epsilon \frac{w_m}{h} + \mathcal{S}_2 \quad (40)$$

where the symbols \mathcal{S}_1 and \mathcal{S}_2 are given in Appendix A.

In order to determine the correct values of m and n corresponding to the maximum deflection, the minimum value of buckling load obtained by equation (37) should be calculated by taking $W = 0$ (noting that $\mathcal{W}_m \neq 0$).

4. Numerical results and discussion

Here, the size-dependent postbuckling equilibrium paths of perfect and imperfect cylindrical shear deformable nanoshells subjected to axial compression are presented considering the surface stress effects. The results presented in this section contain two parts. In the first part including Figures 2–5,

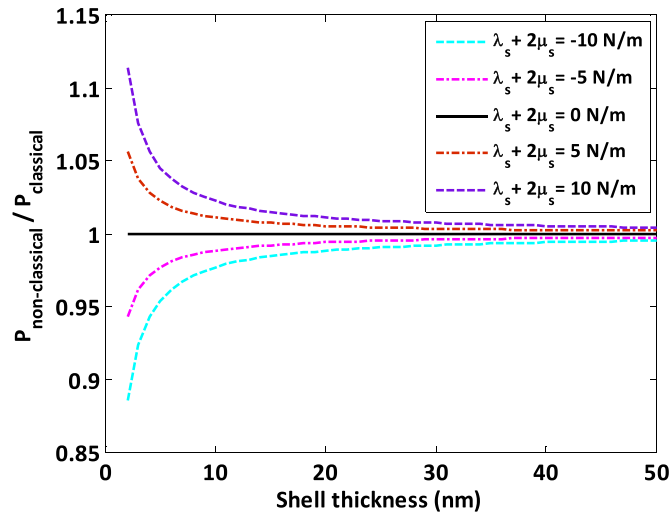


Fig. 6 – Variation of the critical buckling load ratio with nanoshell thickness corresponding to various surface elastic constants ($L^2/Rh = 200$, $R/h = 50$, $\tau_s = 0$ N/m).

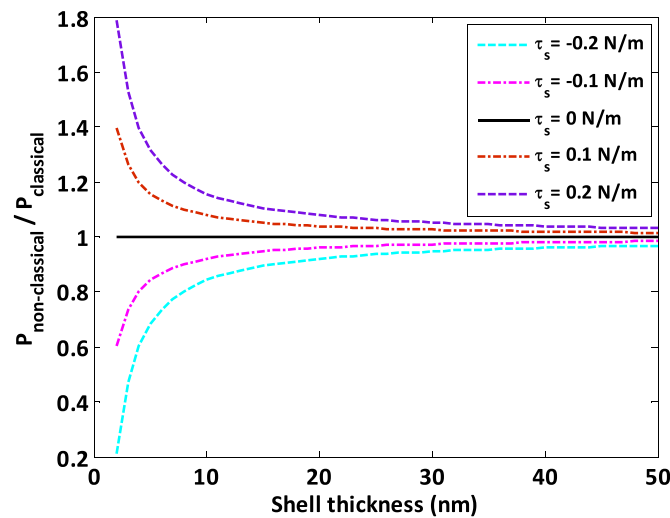


Fig. 7 – Variation of the critical buckling load ratio with nanoshell thickness corresponding to various residual surface stresses ($L^2/Rh = 200$, $R/h = 50$, $\lambda_s = \mu_s = 0$ N/m).

the results are obtained for a nanoshell made of silicon, the material properties of which are tabulated in Table 1. But in the second part of the numerical results including Figures 6 and 7, in order to demonstrate the influence of the value of surface elastic parameters on the critical buckling load of nanoshells, it is assumed that the nanoshells are made of different materials with various surface elastic parameters. Also, in all the preceding numerical results, it is assumed that the edge supports of nanoshells are clamped.

At first, the validity as well as the accuracy of the present solution methodology is checked. Because in accordance with the authors' knowledge, there is no investigation available in published literature in which the buckling or postbuckling behavior of nanoshells is studied in the presence of surface

stress effects, by ignoring the nonlinear and surface elasticity terms, the critical buckling load of a cylindrical shell at usual scale subjected to lateral pressure is calculated based on the present solution procedure and is compared with that of Mirfakhraei and Redekop [61] using differential quadrature numerical method. In Table 2, the critical buckling pressures of cylindrical shells with clamped edge supports obtained by the two different methods are compared corresponding to the same material and geometric properties. A very good agreement is found, which confirms the validity of the current analysis.

Plotted in Figure 2 are the classical and non-classical dimensionless postbuckling load-deflection equilibrium curves of perfect and imperfect cylindrical shear deformable

nanoshells made of silicon under hydrostatic pressure. It is revealed that through decreasing the value of shell thickness, the gap between classical and non-classical curves increases. It means that the surface free energy effects on the postbuckling characteristics of nanoshells with lower thickness are more prominent. This anticipation is the same for both the perfect and imperfect nanoshells.

The classical and non-classical dimensionless postbuckling load-shortening equilibrium curves of Silicon shear deformable nanoshells with and without initial geometric imperfection are depicted in Figure 3. It can be seen again that by decreasing the value of shell thickness, the effects of surface free energy on the postbuckling behavior of cylindrical nanoshells increase. Furthermore, it is found that the surface free energy effects cause increases in both the critical buckling pressure and critical end-shortening of nanoshell made of silicon. In addition, through a comparison of load-shortening curves related to the perfect and imperfect nanoshells, it can be observed that the initial geometric imperfection leads to decreases in both the critical buckling load and critical end-shortening of nanoshell subjected to radial compression.

Figure 4 illustrates the size-dependent dimensionless postbuckling load-deflection curves of cylindrical shear deformable nanoshells with various geometric parameters subjected to hydrostatic pressure. It is demonstrated that by increasing the value of L^2/Rh , the size-dependent dimensionless critical buckling pressure of nanoshell increases. Also, the slope of the variation of applied radial compressive load with deflection of nanoshell is lower in the postbuckling regime. Additionally, in the case of imperfect nanoshells, it is seen that the deflection of nanoshell corresponding to the prebuckling domain increases with higher values of L^2/Rh .

Presented in Figure 5 are the size-dependent dimensionless postbuckling load-shortening curves of cylindrical shear deformable nanoshells with and without initial geometric imperfection under radial compressive load relevant to various geometric parameters. It is observed that for both the perfect and imperfect nanoshells, increasing the value of L^2/Rh leads to higher dimensionless critical buckling pressure and higher dimensionless critical end-shortening. Also, it can be seen that by increasing the value of L^2/Rh , the slope of the postbuckling part of dimensionless load-shortening equilibrium path of nanoshell decreases.

Figures 6 and 7 show the variation of the ratio of non-classical buckling load to classical buckling load with the value of nanoshell thickness corresponding to different values of surface elastic constants and residual surface stress, respectively. It can be found that the effects of surface free energy including surface elasticity and residual surface stress may either increase or decrease the stiffness of cylindrical shear deformable nanoshell against compressive radial load, depending on the signs of surface properties. It is indicated that a positive value of surface elastic constant or residual surface stress causes to increase the size-dependent critical buckling pressure, whereas a negative one results in lower non-classical critical buckling load than that of the classical

one. However, by increasing the value of nanoshell thickness, it can be seen that these effects diminish.

5. Conclusion

The objective of the present study was to predict the effects of surface free energy as one of the most important size effects on the nonlinear buckling and postbuckling behavior of cylindrical shear deformable nanoshells with and without initial geometric imperfections subjected to hydrostatic pressure. To this end, Gurtin-Murdoch elasticity theory was implemented into the first-order shear deformation shell theory to develop an efficient size-dependent shell model. Based upon the variational approach and using virtual work's principle, the non-classical governing differential equations were derived. Subsequently, a boundary layer theory was employed which considered simultaneously the effects of surface free energy, nonlinear prebuckling deformations, large postbuckling deflections and initial geometric imperfection. Finally, an efficient solution methodology based on a two-stepped singular perturbation technique was utilized to obtain the size-dependent critical buckling pressures and postbuckling equilibrium paths.

It was seen that through decreasing the value of nanoshell thickness, the gap between classical and non-classical curves increases, meaning that the surface free energy effects on the postbuckling characteristics of nanoshells with lower thickness were more prominent. Moreover, it was observed that surface free energy effects caused increases in both the critical buckling load and critical end-shortening of nanoshell made of silicon. Additionally, it was found that by increasing the value of L^2/Rh , the slope of variation of the applied radial compressive load with deflection of nanoshell was lower in the postbuckling regime, indicating that for both perfect and imperfect nanoshells, increasing the value of L^2/Rh led to higher dimensionless critical buckling pressure, and higher dimensionless critical end-shortening, and additionally led to a decrease in the slope of the postbuckling part of the dimensionless load-shortening equilibrium path of nanoshell. Furthermore, it was revealed that the effects of surface free energy including surface elasticity and residual surface stress could either increase or decrease the stiffness of cylindrical shear deformable nanoshell, depending on the signs of surface properties.

Appendix A

The sets of perturbation equations for the regular solutions are

$$O(\epsilon^0): \begin{cases} -\frac{\partial^2 \bar{F}_0}{\partial X^2} = \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_0, \bar{F}_0) + Q_0 \\ \mathcal{L}_{21}(\bar{F}_0) + 2\bar{\tau} \mathcal{L}_{22}(\bar{W}_0) + \frac{\partial^2 \bar{W}_0}{\partial X^2} = -\frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_0, \bar{W}_0) \\ -\bar{\tau} \beta^2 \tilde{\mathcal{L}}_2(\bar{W}_0, \bar{W}_0) + \bar{\tau} \vartheta_1 \tilde{\mathcal{L}}_3(\bar{W}_0, \bar{W}_0) + \frac{\bar{\tau} \vartheta_2 \beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_0, \bar{W}_0) \\ 0 = 0 \\ 0 = 0 \end{cases}$$

$$O(\epsilon^4) : \begin{cases} \mathcal{L}_{11}(\bar{W}_2) + 2\bar{\tau}\mathcal{L}_{12}(\bar{W}_2) + \mathcal{L}_{13}(\bar{\psi}_{x_3}) + \mathcal{L}_{14}(\bar{\psi}_{y_3}) - \frac{\partial^2 \bar{F}_4}{\partial X^2} \\ = \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_0, \bar{F}_4) + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_{1/2}, \bar{F}_{7/2}) \\ + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_3, \bar{F}_3) + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_{3/2}, \bar{F}_{5/2}) + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_2 + W^*, \bar{F}_2) \\ + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_{5/2}, \bar{F}_{3/2}) \\ + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_3, \bar{F}_1) + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_{7/2}, \bar{F}_{1/2}) + \beta^2 \tilde{\mathcal{L}}_1(\bar{W}_4, \bar{F}_0) + \mathcal{Q}_4 \\ \mathcal{L}_{21}(\bar{F}_4) + 2\bar{\tau}\mathcal{L}_{22}(\bar{W}_4) + \frac{\partial^2 \bar{W}_4}{\partial X^2} = -\frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_0, \bar{W}_4) \\ - \frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_{1/2}, \bar{W}_{7/2}) - \frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_1, \bar{W}_3) \\ - \frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_{3/2}, \bar{W}_{5/2}) - \frac{\beta^2}{2} \tilde{\mathcal{L}}_1(\bar{W}_2, \bar{W}_2 + 2W^*) \\ - \bar{\tau}\vartheta_1\beta^2 \tilde{\mathcal{L}}_2(\bar{W}_0, \bar{W}_4) \\ - \bar{\tau}\vartheta_1\beta^2 \tilde{\mathcal{L}}_2(\bar{W}_{1/2}, \bar{W}_{7/2}) - \bar{\tau}\vartheta_1\beta^2 \tilde{\mathcal{L}}_2(\bar{W}_1, \bar{W}_3) \\ - \bar{\tau}\vartheta_1\beta^2 \tilde{\mathcal{L}}_2(\bar{W}_{3/2}, \bar{W}_{5/2}) \\ - \bar{\tau}\vartheta_1\beta^2 \tilde{\mathcal{L}}_2(\bar{W}_2, \bar{W}_2 + 2W^*) + \bar{\tau}\vartheta_2 \tilde{\mathcal{L}}_3(\bar{W}_0, \bar{W}_4) \\ + \bar{\tau}\vartheta_2 \tilde{\mathcal{L}}_3(\bar{W}_{1/2}, \bar{W}_{7/2}) + \bar{\tau}\vartheta_2 \tilde{\mathcal{L}}_3(\bar{W}_1, \bar{W}_3) \\ + \bar{\tau}\vartheta_2 \tilde{\mathcal{L}}_3(\bar{W}_{3/2}, \bar{W}_{5/2}) + \bar{\tau}\vartheta_2 \tilde{\mathcal{L}}_3(\bar{W}_2, \bar{W}_2 + 2W^*) \\ + \frac{\bar{\tau}\vartheta_3\beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_0, \bar{W}_4) \\ + \frac{\bar{\tau}\vartheta_3\beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_{1/2}, \bar{W}_{7/2}) + \frac{\bar{\tau}\vartheta_3\beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_1, \bar{W}_3) \\ + \frac{\bar{\tau}\vartheta_3\beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_{3/2}, \bar{W}_{5/2}) \\ + \frac{\bar{\tau}\vartheta_3\beta^2}{2} \tilde{\mathcal{L}}_4(\bar{W}_2, \bar{W}_2 + 2W^*) \\ \mathcal{L}_{31}(\bar{W}_3) + \mathcal{L}_{32}(\bar{\psi}_{x_4}) + \vartheta_5\beta \frac{\partial^2 \bar{\psi}_{y_4}}{\partial X\partial Y} = 0 \\ \mathcal{L}_{41}(\bar{W}_3) + \mathcal{L}_{42}(\bar{\psi}_{y_4}) + \vartheta_5\beta \frac{\partial^2 \bar{\psi}_{x_4}}{\partial X\partial Y} = 0 \end{cases} \quad (A1)$$

The sets of perturbation equations for the boundary layer solutions are:

$$O(\epsilon^{5/2}) : \begin{cases} -e_{11}^* \frac{\partial^4 \bar{W}_{3/2}}{\partial \xi^4} - d_{11}^* \frac{\partial^3 \bar{\psi}_{x_2}}{\partial \xi^3} - \frac{\partial^2 \bar{F}_{5/2}}{\partial \xi^2} = 0 \\ \vartheta_1 \frac{\partial^4 \bar{F}_{5/2}}{\partial \xi^4} + (1 - 2\bar{\tau}\vartheta_1) \frac{\partial^2 \bar{W}_{3/2}}{\partial \xi^2} = 0 \\ -k_s a_{55}^* \frac{\partial \bar{W}_{3/2}}{\partial \xi} - k_s a_{55} \bar{\psi}_{x_2} = 0 \\ -k_s a_{55}^* \bar{\psi}_{y_{5/2}} = 0 \end{cases}$$

$$O(\epsilon^3) : \begin{cases} -e_{11}^* \frac{\partial^4 \bar{W}_2}{\partial \xi^4} - d_{11}^* \frac{\partial^3 \bar{\psi}_{x_{5/2}}}{\partial \xi^3} - \frac{\partial^2 \bar{F}_3}{\partial \xi^2} = 0 \\ \vartheta_1 \frac{\partial^4 \bar{F}_3}{\partial \xi^4} + (1 - 2\bar{\tau}\vartheta_1) \frac{\partial^2 \bar{W}_2}{\partial \xi^2} = 0 \\ -k_s a_{55}^* \frac{\partial \bar{W}_2}{\partial \xi} - k_s a_{55} \bar{\psi}_{x_{5/2}} = 0 \\ -k_s a_{55}^* \bar{\psi}_{y_3} = 0 \end{cases}$$

$$O(\epsilon^3) : \begin{cases} -e_{11}^* \frac{\partial^4 \bar{W}_2}{\partial \xi^4} - d_{11}^* \frac{\partial^3 \bar{\psi}_{x_{5/2}}}{\partial \xi^3} - \frac{\partial^2 \bar{F}_3}{\partial \xi^2} = 0 \\ \vartheta_1 \frac{\partial^4 \bar{F}_3}{\partial \xi^4} + (1 - 2\bar{\tau}\vartheta_1) \frac{\partial^2 \bar{W}_2}{\partial \xi^2} = 0 \\ -k_s a_{55}^* \frac{\partial \bar{W}_2}{\partial \xi} - k_s a_{55} \bar{\psi}_{x_{5/2}} = 0 \\ -k_s a_{55}^* \bar{\psi}_{y_3} = 0 \end{cases}$$

$$O(\epsilon^{5/2}) : \begin{cases} -e_{11}^* \frac{\partial^4 \bar{W}_{3/2}}{\partial \zeta^4} + d_{11}^* \frac{\partial^3 \bar{\psi}_{x_2}}{\partial \zeta^3} - \frac{\partial^2 \bar{F}_{5/2}}{\partial \zeta^2} = 0 \\ \vartheta_1 \frac{\partial^4 \bar{F}_{5/2}}{\partial \zeta^4} + (1 - 2\bar{\tau}\vartheta_1) \frac{\partial^2 \bar{W}_{3/2}}{\partial \zeta^2} = 0 \\ k_s a_{55}^* \frac{\partial \bar{W}_{3/2}}{\partial \zeta} - k_s a_{55} \bar{\psi}_{x_2} = 0 \\ -k_s a_{55}^* \bar{\psi}_{y_{5/2}} = 0 \end{cases}$$

$$O(\epsilon^3) : \begin{cases} -e_{11}^* \frac{\partial^4 \bar{W}_2}{\partial \zeta^4} + d_{11}^* \frac{\partial^3 \bar{\psi}_{x_{5/2}}}{\partial \zeta^3} - \frac{\partial^2 \bar{F}_3}{\partial \zeta^2} = 0 \\ \vartheta_1 \frac{\partial^4 \bar{F}_3}{\partial \zeta^4} + (1 - 2\bar{\tau}\vartheta_1) \frac{\partial^2 \bar{W}_2}{\partial \zeta^2} = 0 \\ k_s a_{55}^* \frac{\partial \bar{W}_2}{\partial \zeta} - k_s a_{55} \bar{\psi}_{x_{5/2}} = 0 \\ -k_s a_{55}^* \bar{\psi}_{y_3} = 0 \end{cases} \quad (A2)$$

The obtained asymptotic solutions corresponding to clamped edge supports are:

$$W = \mathcal{A}_{00}^{(0)} + \epsilon^{3/2} \left[\mathcal{A}_{00}^{(3/2)} - \mathcal{A}_{00}^{(3/2)} \left(\sin\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) + \cos\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha X}{\sqrt{\epsilon}}} \right. \\ \left. - \mathcal{A}_{00}^{(3/2)} \left(\sin\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) + \cos\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}} \right] \\ + \epsilon^2 \left[\mathcal{A}_{00}^{(2)} + \mathcal{A}_{11}^{(2)} \sin(mX) \sin(nY) - \mathcal{A}_{00}^{(2)} \left(\sin\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right. \right. \\ \left. \left. + \cos\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha X}{\sqrt{\epsilon}}} - \mathcal{A}_{00}^{(2)} \left(\sin\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right. \right. \\ \left. \left. + \cos\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}} \right] \\ + \epsilon^4 \left[\mathcal{A}_{00}^{(4)} + \mathcal{A}_{11}^{(4)} \sin(mX) \sin(nY) + \mathcal{A}_{20}^{(4)} \cos(2mX) \right. \\ \left. + \mathcal{A}_{22}^{(4)} \cos(2mX) \cos(2nY) \right] + O(\epsilon^5) \quad (A3)$$

$$F = -\mathcal{B}_{00}^{(0)} \left(\beta^2 X^2 + \frac{Y^2}{2} \right) + \epsilon^2 \left[-\mathcal{B}_{00}^{(2)} \left(\beta^2 X^2 + \frac{Y^2}{2} \right) \right. \\ \left. + \mathcal{B}_{11}^{(2)} \sin(mX) \sin(nY) \right] \\ + \epsilon^{5/2} \left[\mathcal{A}_{00}^{(3/2)} \left(b_{10}^{(3/2)} \sin\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) + b_{01}^{(3/2)} \cos\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha X}{\sqrt{\epsilon}}} \right. \\ \left. + \mathcal{A}_{00}^{(3/2)} \left(b_{10}^{(5/2)} \sin\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) + b_{01}^{(5/2)} \cos\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}} \right] \\ + \epsilon^3 \left[\mathcal{A}_{00}^{(2)} \left(b_{10}^{(3)} \sin\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) + b_{01}^{(3)} \cos\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha X}{\sqrt{\epsilon}}} \right. \\ \left. + \mathcal{A}_{00}^{(2)} \left(b_{10}^{(3)} \sin\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) + b_{01}^{(3)} \cos\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}} \right] \\ + \epsilon^4 \left[-\mathcal{B}_{00}^{(4)} \left(\beta^2 X^2 + \frac{Y^2}{2} \right) + \mathcal{B}_{20}^{(4)} \cos(2mX) + \mathcal{B}_{02}^{(4)} \cos(2mY) \right. \\ \left. + \mathcal{B}_{22}^{(4)} \cos(2mX) \cos(2nY) \right] + O(\epsilon^5) \quad (A4)$$

$$\psi_x = \epsilon^2 \left[\left(c_{10}^{(2)} \sin\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) + c_{01}^{(2)} \cos\left(\frac{\alpha X}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha X}{\sqrt{\epsilon}}} \right. \\ \left. + \left(c_{10}^{(2)} \sin\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) + c_{01}^{(2)} \cos\left(\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}\right) \right) e^{-\frac{\alpha(\pi - X)}{\sqrt{\epsilon}}} \right] \\ + \epsilon^3 \left[\mathcal{C}_{11}^{(3)} \cos(mX) \sin(nY) \right] + O(\epsilon^5) \quad (A5)$$

$$\psi_y = \epsilon^3 \left[\mathcal{D}_{11}^{(3)} \sin(mX) \cos(nY) \right] + O(\epsilon^5) \quad (A6)$$

$$\mathcal{A}_{00}^{(0)} = 0 \quad (A7)$$

The periodicity condition yields

$$\mathcal{A}_{00}^{(3/2)} = \frac{2}{3} 3^{1/4} p_q \left(\frac{2\vartheta_1 - \vartheta_2}{1 - 2\bar{\tau}\vartheta_1} \right) + \frac{2\bar{\tau}\vartheta_6(\vartheta_1 - \vartheta_2)}{1 - 2\bar{\tau}\vartheta_1} \epsilon^{-1/2} \quad (A8)$$

$$\mathcal{A}_{00}^{(2)} = \mathcal{A}_{00}^{(3)} = 0 \quad (A9)$$

$$\mathcal{A}_{00}^{(4)} = \frac{2\bar{\tau}\vartheta_2 m^2 + \beta^2 n^2 (1 - 2\bar{\tau}\vartheta_1)(1 + 2\ell)}{8(1 - 2\bar{\tau}\vartheta_1)} (\mathcal{A}_{11}^{(2)})^2 \quad (A10)$$

The parameters in equations (37-40) are as follows

$$p_q^{(0)} = K_0 K_1 + \epsilon^2 \beta^2 K_4 \quad (A11)$$

$$\mathcal{P}_q^{(2)} = 8K_1K_9K_{14} + \frac{8K_1K_9(K_0K_1K_{13}\beta^2 + K_0K_9K_{12})}{K_0K_1\beta^2 - K_{12}} + \frac{4K_1K_9(K_0K_{13} + K_0^2K_9)}{K_0K_1\beta^2 - K_{12}} + 16K_0K_9K_{10} + \frac{4K_0K_1K_9K_{16}}{K_0K_{11} + K_{15}} \quad (A12)$$

$$\delta_q^{(0)} = \left[\frac{3^{3/4}}{2} \bar{\tau} \vartheta_6 (\vartheta_1 - \vartheta_2) \right] \epsilon^{-1/2} + \frac{\bar{\tau} \vartheta_2 \vartheta_6 (\vartheta_1 - \vartheta_2)}{\pi \alpha \vartheta_1 (1 - 2\bar{\tau} \vartheta_1)} + \left[\frac{\vartheta_1}{2} - \vartheta_2 - \bar{\tau} \vartheta_2 \left(\frac{2\vartheta_1 - \vartheta_2}{1 - 2\bar{\tau} \vartheta_1} \right) + \left(\frac{2\vartheta_1 \vartheta_2 - \vartheta_2^2}{\pi \alpha \vartheta_1} \right) \right] \epsilon^{1/2} \\ \mathcal{P}_q + \left[\left(\frac{3^{1/4} \alpha (2\vartheta_1 - \vartheta_2)^2}{6\pi} \right) \epsilon \right] \mathcal{P}_q^2 \quad (A13)$$

$$\delta_q^{(2)} = \left[\frac{3^{3/4}}{32} (m^2 (1 - 2\bar{\tau} \vartheta_1) + 2\bar{\tau} \vartheta_2 \beta^2 n^2) \right] \epsilon^{-3/2} \quad (A14)$$

where K_i ($i = 0, \dots, 17$) are the parameters in terms of $\vartheta_1, \vartheta_2, \vartheta_3, \vartheta_4, \vartheta_5, \ell, m, n, \beta$ obtained via the sets of perturbation equations.

$$\mathcal{S}_1 = - \left[\left(\frac{2\vartheta_1 - \vartheta_2}{1 - 2\bar{\tau} \vartheta_1} \right) (\mathcal{P}_q^{(2)}) \right] \quad (A15)$$

$$\mathcal{S}_2 = - \left(\frac{2\vartheta_1 - \vartheta_2}{1 - 2\bar{\tau} \vartheta_1} \right) (\mathcal{P}_q^{(0)}) - \frac{2\bar{\tau} \vartheta_6 (\vartheta_1 - \vartheta_2)}{1 - 2\bar{\tau} \vartheta_1} \epsilon \quad (A16)$$

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