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Dynamic modeling and infinite-dimensional observer-based control for manipulation of flexible beam by a multi-link robot

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Abstract

This paper concerns an infinite-dimensional observer for manipulation of flexible beam by a rigid arm robot. The complex dynamic of the system is described by distributed parameter model in terms of ordinary differential equations and partial differential equation. A novel infinite-dimensional observer is proposed to estimate the vibration information of the flexible object. In addition, an observer-based independent joint controller is designed to achieve the position control and vibration suppression, which do not need end-point boundary control. The semigroup theory and LaSalle's invariance principle are adopted to prove the asymptotic stability of the robot system. The efficiency of the observers and the proposed control strategy are demonstrated by numerical simulations.

Keywords Infinite-dimensional observer · Flexible beam · Independent joint control · Distributed parameter model

Introduction

In industry, the research of robotic manipulators handling rigid bodies has received extensive attentions [\[1](#page-10-0)[,2](#page-10-1)]. However, flexible objects such as solar panels, flexible metal plates and spring components are often used in the automotive, aerospace and medical fields [\[3](#page-10-2)[,4\]](#page-10-3). For example, space robotic arms are used to maintain the aircraft and replace failed batteries, and the panels are mostly made of flexible materials. Different from rigid parts, flexible parts have the characteristics of light weight, high flexibility, man– machine interaction and low energy consumption, which also bring vibration. In the field of high accuracy and safety requirements, the vibration is necessary to be solved [\[5](#page-10-4)[,6](#page-10-5)]. Therefore, it is of theoretical and practical significance to research the trajectory and vibration control of manipulation for flexible beam during industry operation and assembly tasks.

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In the literature, research on manipulation of flexible structures is always based on simplified lumped parameter model $[7,8]$ $[7,8]$. In $[9,10]$ $[9,10]$, the rigid control method has been used in moving flexible object by rigid robot based on assumed modes model. [\[11](#page-10-10)] researched two rigid robot manipulating flexible beam by finite element method. However, the simplified model may bring control or observer overflow problem, which may bring instability to the robot system. To avoid the above drawbacks, great attention has been paid to study the control design based upon distributed parameter model [\[12](#page-10-11)– [14](#page-10-12)]. Distributed parameter model is an infinite-dimensional model, which refers to the relationship between the system state and the change of space coordinates and time variables. The PDE–ODE model is established for dual-arm coordinated operation of large spatial flexible structures in [\[15](#page-10-13)]. [\[16\]](#page-10-14) studied the position/force control of flexible beams based on PDE model. [\[17](#page-10-15)[,18\]](#page-10-16) address robust control for flexible system based on distributed parameter system. However, most of the above achievements are about the flexible beam, research about the system of manipulation for flexible object based on distributed parameter model are relatively few, which exists many problems to be solved.

Furthermore, flexible beam operating system is a complex dynamically coupled system, which includes not only the rigid motion of large range, but also the local elastic deformation. The most important problem in this system is the vibration of the flexible structure in motion. Due

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to the requirements of high accuracy and high flexibility, the resulting vibration cannot be ignored [\[19](#page-10-17)[,20](#page-10-18)]. At the same time, the characteristics of flexible object to be handled vary with different manipulation tasks. So this makes it impractical to outfit the flexible object with sensors and actuators. Therefore, it is necessary to propose an observer to estimate the vibration states [\[21](#page-10-19)]. At present, the results of PDE observers for infinite-dimensional systems are limited. In [\[22](#page-10-20)], the PDE observer for flexible single-link robot was proposed to estimate the infinite-dimensional states in task space. Feng et al. [\[23](#page-10-21)] designed an exponentially converging observer to estimates the state for the heat system [\[24](#page-10-22)] develops PDE observer to estimate the freeway traffic states. In [\[25\]](#page-10-23), the Luenberger-like observers is proposed for an infinite-dimensional rotating body-beam system. Therefore, the research on infinite dimensional observer design for manipulation of flexible object is necessary and meaningful, and has not been reported yet.

Besides, the boundary control is always used for the control design of flexible system [\[26](#page-10-24)[–28](#page-10-25)]. It not only needs joint input actuator to adjust the position of the robot, but also needs end-point input force to suppress the vibration. For the single-link manipulator or multi-link manipulator, boundary control is relatively easy to achieve, and is also effective to suppress the vibration for the manipulator [\[29](#page-10-26)[,30\]](#page-10-27). The actuator installed at the end of the manipulator will not vary with the change of the task when the manipulator systems perform different operations. However, different from the manipulator system, the flexible object operated by the manipulator is the actuated mechanism, the actuator must be reinstalled when the operation object is changed, it is not appropriate to install the actuator at the end of the flexible object system [\[31\]](#page-10-28). In this paper, we only use the independent joint input to control the system, which do not need the end-point input force. The vibration can be suppressed by adding the root vibration observer signals of flexible object into the joint input controller. The contributions are summarized below:

- 1. The complex dynamic is expressed as original infinitedimensional model without any simplification or discretization, which is very effective to reduce the vibration of the system.
- 2. An infinite-dimensional observer is designed for the manipulation of flexible object by a rigid arm manipulator, it prevents the installation of sensors on flexible objects.
- 3. The independent joint observer-based controller is proposed based on distributed parameter model. The stability is proved by LaSalle's Invariance Principle.

This paper is structured as follows. ["System description"](#page-1-0) section describes the system dynamic. The infinite-dimensional observer is designed in ["Infinite-dimensional observer"](#page-3-0) sec-

Fig. 1 The structure diagram of manipulator operation

tion. ["Observer-based controller design"](#page-5-0) section proposes the observer-based controller and the asymptotic stability. The ["Simulation"](#page-6-0) section shows validation results by simulation, and the conclusion is given in ["Conclusion"](#page-8-0) section.

System description

Distributed parameter model

In the following, we consider a three-link robot moving a flexible object as shown in Fig. [1.](#page-1-1) XOY is the inertial coordinate. The flexible object is supposed to be an Euler–Bernoulli beam due to its own characteristics. The beam has uniform mass density ρK , length *l* and uniform flexural rigidity *EI*, $u(t, r)$ stands for the elastic deformation at length r of flexible beam at time *t*. The rigid link $i(i = 1, 2, 3)$ has length L_i , mass center length L_i _g, mass m_i , and moment of inertia I_i . θ_i denotes the rotation angle of link *i*, and τ_i is the joint torque at the motor *i*. $S_2 = [S_{2x} \ S_{2y}]^T$, $S_3 = [S_{3x} \ S_{3y}]^T$ and $S = [S_x \ S_y]^T$ express the position of the second link, the third link and the mass center of the flexible object in the reference coordinate system, respectively.

$$
S_{2x} = L_1 \cos \theta_1 + L_{2g} \cos(\theta_1 + \theta_2),\tag{1}
$$

$$
S_{2y} = L_1 \sin \theta_1 + L_{2g} \sin(\theta_1 + \theta_2),\tag{2}
$$

$$
S_{3x} = L_1 \cos \theta_1 + L_2 \cos(\theta_1 + \theta_2) + L_{3g} \cos(\theta_1 + \theta_2 + \theta_3),
$$
\n(3)

$$
S_{3y} = L_1 \sin \theta_1 + L_2 \sin(\theta_1 + \theta_2) + L_{3g} \sin(\theta_1 + \theta_2 + \theta_3),
$$
\n(4)

$$
S_x = L_1 \cos \theta_1 + L_2 \cos(\theta_1 + \theta_2) + L_3 \cos(\theta_1 + \theta_2 + \theta_3)
$$

+
$$
r \cos(\theta_1 + \theta_2 + \theta_3) - u \sin(\theta_1 + \theta_2 + \theta_3),
$$
 (5)

$$
S_y = L_1 \sin \theta_1 + L_2 \sin(\theta_1 + \theta_2) + L_3 \sin(\theta_1 + \theta_2 + \theta_3) + r \sin(\theta_1 + \theta_2 + \theta_3) - u \cos(\theta_1 + \theta_2 + \theta_3).
$$
 (6)

Two assumptions are introduced [\[9](#page-10-8)]:

Assumption 1 Assumed the longitudinal deflection of flexible object is ignored, and only transverse deflection and deflection angle are taken into consideration.

Assumption 2 The flexible beam is grasped rigidly, no deformation occurs between the manipulator and the contacted beam.

Remark 1 The superscript "." is denoted as the derivative of time t and superscript "" is the derivative of length r . The subscript " E " of $u(t, r)$ express variable r equal to l and subscript "0" represent *r* equal to "0", that is $u_0 =$ $u(t, r)|_{r=0}, u_E = u(t, r)|_{r=l}$

The total kinetic and potential energy are expressed as

$$
T = \frac{1}{2} \left[m_1 L_{1g}^2 \dot{\theta}_1^2 + I_1 \dot{\theta}_1^2 + m_2 \dot{S}_2^T \dot{S}_2 + I_2 (\dot{\theta}_1 + \dot{\theta}_2)^2 + m_3 \dot{S}_3^T \dot{S}_3 + I_3 (\dot{\theta}_1 + \dot{\theta}_2 + \dot{\theta}_3)^2 \right] + \frac{1}{2} \rho K \int_0^l \dot{S}^T \dot{S} dr,
$$
\n(7)

 $U = m_1 g L_{1g} \sin \theta_1 + m_2 g L_1 \sin \theta_1 + m_2 g L_{2g} \sin(\theta_1 + \theta_2)$ $+m_3gL_1 \sin \theta_1 + m_3gL_2 \sin(\theta_1 + \theta_2) + m_3gL_3{}_g \sin(\theta_1$

$$
+\theta_2 + \theta_3) + \rho K g \int_0^l S_y dr + \frac{1}{2} \int_0^l E I(u'')^2 dr. \quad (8)
$$

The virtual work of the system is

$$
\delta W = \sum_{i=1}^{3} \tau_i \delta \theta_i.
$$
 (9)

By Hamilton's principle, we have

$$
\int_{t_0}^{t_1} (\delta T - \delta U + \delta W) dt = 0.
$$
 (10)

The system dynamic model is expressed as

$$
M(\theta)\ddot{\theta} + [C(\theta, \dot{\theta}) + f(\theta, \dot{u})]\dot{\theta} + F(\theta, u_0'', u_0''')+ G = \tau,
$$
 (11)

$$
\ddot{z} + \frac{EIu^{(4)}}{\rho K} = 0,\t(12)
$$

 $u_0 = 0, \quad u'_0 = 0, \quad u'''_E = 0, \quad u''_E = 0.$ (13)

Parameters of (11) and (12) are as following

$$
z(r, t) = (r + L_3)(\theta_1 + \theta_2 + \theta_3) + u(t, r)
$$

+
$$
\int_0^t [\dot{\theta}_1 L_1 \cos(\theta_2 + \theta_3) + (\dot{\theta}_1 + \dot{\theta}_2) L_2 \cos \theta_3] dt,
$$
(14)

 $\theta = [\theta_1 \ \theta_2 \ \theta_3 \]^T, F (\theta, u_0'', u_0'') = [F_1 \ F_2 \ F_3 \]^T,$ $\tau = [\tau_1 \ \tau_2 \ \tau_3]^{T}$, $G = [G_1 \ G_2 \ G_3]^{T}$. $M(\theta)$, $C(\theta, \dot{\theta})$ and $f(\theta, \dot{u})$ are 3×3 matrix; θ , *G* and τ are 3×1 vec-tor (see Appendix [A\)](#page-8-1). The Eq. (11) also have the following property:

Property 1 *M* (θ) is symmetrical and positive definite, $M(\theta)$, $C(\theta, \dot{\theta})$ and $f(\theta, \dot{u})$ satisfy $\dot{M} - 2(C + f) = -(\dot{M} - 2)$ $(C + f)^{T}$.

Energy analysis of flexible beam

First, the elastic deformation of Euler–Bernoulli beam is assumed to be much smaller than the beam length $(|u| \ll l)$, so the square velocity $\dot{S}^T \dot{S}$ of flexible beam in (7) is simplified as follows

$$
\dot{S}^{\mathrm{T}}\dot{S} = [\dot{\theta}_{1}L_{1}\cos(\theta_{2} + \theta_{3}) + (L_{3} + r)(\dot{\theta}_{1} + \dot{\theta}_{2} + \dot{\theta}_{3}) + (\dot{\theta}_{1} + \dot{\theta}_{2})L_{2}\cos\theta_{3} + \dot{\mu}^{2} + [\dot{\theta}_{1}L_{1}\sin(\theta_{2} + \theta_{3}) + (\dot{\theta}_{1} + \dot{\theta}_{2})L_{2}\sin\theta_{3}]^{2}.
$$
\n(15)

The transversal velocity S_t is only considered since the vibration of beam is hardly affected by the kinetic energy due to the longitudinal velocity. The transversal component $\dot{S}^T \dot{S}$ of the beam is

$$
\dot{S}_t^{\mathrm{T}} \dot{S}_t = [L_1 \dot{\theta}_1 \cos(\theta_2 + \theta_3) + (L_3 + r)(\dot{\theta}_1 + \dot{\theta}_2 + \dot{\theta}_3) + (\dot{\theta}_1 + \dot{\theta}_2)L_2 \cos \theta_3 + \dot{u}^2.
$$
 (16)

Then, the kinetic energy T_f and the potential energy U_f of flexible beam due to the transversal velocity are computed as

$$
T_f + U_f = \frac{1}{2} \int_0^l \rho K \dot{S}_t^{\mathrm{T}} \dot{S}_t \mathrm{d}r + \frac{1}{2} \int_0^l E I(u'')^2 \mathrm{d}r. \tag{17}
$$

Next, the time derivative of the total flexible energy T_f + U_f can be derived as

$$
\dot{T}_f + \dot{U}_f
$$
\n
$$
= \rho K \int_0^l [L_1 \dot{\theta}_1 \cos(\theta_2 + \theta_3) + (\dot{\theta}_1 + \dot{\theta}_2 + \dot{\theta}_3)(L_3 + r)
$$
\n
$$
+ \cos \theta_3 L_2 (\dot{\theta}_1 + \dot{\theta}_2) + \dot{u}][-\dot{\theta}_1 L_1 \sin(\theta_2 + \theta_3)(\dot{\theta}_2 + \dot{\theta}_3)
$$
\n
$$
+ L_1 \ddot{\theta}_1 \cos(\theta_2 + \theta_3) + (\ddot{\theta}_1 + \ddot{\theta}_2) L_2 \cos \theta_3
$$
\n
$$
- (\dot{\theta}_1 + \dot{\theta}_2) L_2 \sin \theta_3 \dot{\theta}_3
$$
\n
$$
+ (L_3 + r)(\ddot{\theta}_1 + \ddot{\theta}_2 + \ddot{\theta}_3) + \ddot{u}] dr + \int_0^l E I u^{(4)} \dot{u} dr,
$$
\n(18)

where \ddot{u} can be get from the vibration Eq. [\(6\)](#page-1-2).

Finally, according to the boundary condition [\(7\)](#page-2-2), we can get

$$
\dot{T}_f + \dot{U}_f = [L_1 \dot{\theta}_1 \cos(\theta_2 + \theta_3) + (\dot{\theta}_1 + \dot{\theta}_2) L_2 \cos \theta_3
$$

3252 Complex & Intelligent Systems (2023) 9:3249–3260

+
$$
(\dot{\theta}_1 + \dot{\theta}_2 + \dot{\theta}_3)L_3[EIu_0'''
$$

- $EIu_0''(\dot{\theta}_1 + \dot{\theta}_2 + \dot{\theta}_3) = -q^T\dot{\theta},$ (19)

where $\dot{\theta} = [\dot{\theta}_1 \quad \dot{\theta}_2 \quad \dot{\theta}_3]^T$, *q* is defined as

$$
q = \begin{pmatrix} [-\cos(\theta_2 + \theta_3)L_1 - \cos\theta_3 L_2 - L_3]E I u_0''' + E I u_0'' \\ (-\cos\theta_3 L_2 - L_3)E I u_0''' + E I u_0'' \\ -E I L_3 u_0''' + E I u_0'' \end{pmatrix}.
$$
\n(20)

Infinite-dimensional observer

Observer design

For avoid installing sensors in the flexible beam, a nonlinear infinite-dimensional observer is designed to estimate the root strain and shear force of flexible beam based on Eqs. [\(11\)](#page-2-0)–[\(13\)](#page-2-3). Assume that the angular θ_i , angular velocity θ_i are available for measurement, and the estimates of the angular and the flexible deformation are defined as θ_i and $\hat{u}(r, t)$. The estimate errors are defined as $\theta_i = \theta_i - \theta_i$, $\tilde{u}(t, r) = u(t, r) - \hat{u}(t, r)$ and $\tilde{z}(r, t) = z(r, t) - \hat{z}(r, t)$. The observer is proposed to satisfy the following relations as $t \rightarrow \infty$:

$$
\hat{\theta}_i \to \theta_i, \quad \dot{\hat{\theta}}_i \to \dot{\theta}_i, \quad \hat{u}(t,r) \to u(t,r), \quad \dot{\hat{u}}(t,r) \to \dot{u}(t,r). \tag{21}
$$

Define the estimate $\hat{z}(r, t)$ of $z(r, t)$ as

$$
\hat{z}(r, t) = (r + L_3)(\hat{\theta}_1 + \hat{\theta}_2 + \hat{\theta}_3) + \hat{u}(t, r) + \int_0^t [\dot{\hat{\theta}}_1 L_1 \cos(\theta_2 + \theta_3) + (\dot{\hat{\theta}}_1 + \dot{\hat{\theta}}_2) L_2 \cos \theta_3] dt.
$$
\n(22)

An observer is designed to reconstruct the states in the domain as follows

$$
M(\theta)\hat{\dot{\theta}} + C(\theta, \dot{\theta})\hat{\dot{\theta}} + f(\dot{\theta}, \hat{u})\dot{\theta} + F(\theta, \hat{u}_0'', \hat{u}_0''')
$$

$$
-k_d(\dot{\hat{\theta}} - \dot{\theta}) - k_p(\hat{\theta} - \theta) + G = \tau,
$$
(23)

$$
\ddot{\tilde{z}}(r,t) + \frac{EI}{\rho K} \hat{u}^{(4)}(t,r) = 0,
$$
\n(24)

$$
\hat{u}_0 = 0, \quad \hat{u}'_0 = 0, \quad \hat{u}''_E = 0, \quad \hat{u}'''_E = 0,
$$
\n(25)

where $k_d = diag(k_{d1}, k_{d2}, k_{d3}), k_p = diag(k_{p1}, k_{p2}, k_{p3}),$ k_{di} , $k_{pi} \in R^+$ (*i* = 1, 2, 3). Then, we subtract system model (11) – (13) by (23) – (25) and get the model of estimate errors.

$$
M_{11}(\theta)\ddot{\tilde{\theta}}_1 + M_{12}(\theta)\ddot{\tilde{\theta}}_2 + M_{13}(\theta)\ddot{\tilde{\theta}}_3 + C_{11}(\theta,\dot{\theta})\dot{\tilde{\theta}}_1
$$

+ C₁₂(
$$
\theta
$$
, $\dot{\theta}$) $\tilde{\theta}_2$ + C₁₃(θ , $\dot{\theta}$) $\tilde{\theta}_3$
\n- $\rho K[L_1 \sin(\theta_2 + \theta_3)] \int_0^l \dot{\tilde{u}} d\tilde{r} d\tilde{r}_2 - \rho K[\sin(\theta_2 + \theta_3)L_1$
\n+ $\sin \theta_3 L_2 \int_0^l \dot{\tilde{u}} d\tilde{r} d\tilde{r}_3$
\n- $E I \tilde{u}_0^{"} + [\cos(\theta_2 + \theta_3)L_1 + \cos \theta_3 L_2 + L_3] E I \tilde{u}_0^{"}$
\n+ $k_{d1} \tilde{\theta}_1 + k_{p1} \tilde{\theta}_1 = 0$, (26)
\n $M_{21}(\theta) \ddot{\theta}_1 + M_{22}(\theta) \ddot{\theta}_2 + M_{23}(\theta) \ddot{\theta}_3 + C_{21}(\theta, \dot{\theta}) \dot{\theta}_1$
\n+ $C_{22}(\theta, \dot{\theta}) \dot{\theta}_2$
\n+ $C_{23}(\theta, \dot{\theta}) \dot{\theta}_3$
\n+ $\rho K \sin(\theta_2 + \theta_3) L_1 \int_0^l \dot{\tilde{u}} d\tilde{r} d\tilde{r}_1 - \rho K \sin \theta_3 L_2$
\n $\int_0^l \dot{\tilde{u}} d\tilde{r} d\tilde{r}_3 - E I \tilde{u}_0^{"}$
\n+ $(L_2 \cos \theta_3 + L_3) E I \tilde{u}_0^{"} + k_{d2} \dot{\tilde{\theta}}_2 + k_{p2} \tilde{\theta}_2 = 0$, (27)
\n $M_{31}(\theta) \ddot{\theta}_1 + M_{32}(\theta) \ddot{\theta}_2 + M_{33}(\theta) \ddot{\theta}_3 + C_{31}(\theta, \dot{\theta}) \dot{\tilde{\theta}}_1$
\n+ $C_{32}(\theta, \dot{\theta}) \dot{\tilde{\theta}}_2 + C_{33}(\theta, \dot{\theta}) \dot{\tilde{\theta}}_3$
\n+ ρK

$$
\ddot{\tilde{z}}(r,t) + \frac{EI}{\rho K} \tilde{u}^{(4)}(t,r) = 0,
$$
\n(29)

$$
\tilde{u}_0 = 0, \quad \tilde{u}'_0 = 0, \quad \tilde{u}''_E = 0, \quad \tilde{u}'''_E = 0.
$$
\n(30)

The asymptotic stability of the observer

Define a generalized coordinate vector as

$$
w = [w_1 \t w_2 \t w_3 \t w_4 \t w_5 \t w_6 \t w_7 \t w_8]^T
$$

= $[\tilde{\theta}_1 \t \dot{\tilde{\theta}}_1 \t \tilde{\theta}_2 \t \dot{\tilde{\theta}}_2 \t \tilde{\theta}_3 \t \dot{\tilde{\theta}}_3 \t \tilde{\tilde{u}} \t \tilde{\tilde{u}}]^T$. (31)

The estimate error model (26) – (30) can be rewritten as

$$
\dot{w} = Aw + Y(w), \quad w(0) \in H,
$$
\n(32)

where *A* is infinite-dimensional linear operator, $Y(w)$ is finite dimensional nonlinear operator.

$$
Aw = [w_2 \varphi_1 \ w_4 \ \varphi_2 \ w_6 \ \varphi_3 \ w_8 \ \varphi_4]^T, \ \forall w \in D(A),
$$
\n(33)

$$
Y(w) = [0 \ Y_1 \ 0 \ Y_2 \ 0 \ Y_3 \ 0 \ Y_4]^T, \tag{34}
$$

where

$$
\varphi_1 = (ELIw_7''(0) - ELI_3w_7'''(0) - k_{d1}w_2 - k_{p1}w_1)/M_{11},
$$

\n
$$
\varphi_2 = (ELIw_7''(0) - ELI_3w_7'''(0) - k_{d2}w_4 - k_{p2}w_3)/M_{22},
$$

\n
$$
\varphi_3 = (ELIw_7''(0) - ELI_3w_7'''(0) - k_{d3}w_6 - k_{p3}w_5)/M_{33},
$$

\n
$$
\varphi_4 = -ELIw_7^{(4)}/\rho K,
$$

\n
$$
Y_1 = \left\{-\hat{M}_{11}(\theta)\dot{w}_2 - M_{12}(\theta)\dot{w}_4 - M_{13}(\theta)\dot{w}_6 - C_{11}w_2 - C_{12}w_4 - C_{13}w_6\right\}
$$

\n
$$
+ [L_1 \sin(\theta_2 + \theta_3)\dot{\theta_2} + [\sin(\theta_2 + \theta_3)L_2]
$$

\n
$$
+ \sin \theta_3L_2]\dot{\theta_3}]\rho K \int_0^l w_8 dr
$$

\n
$$
- [\cos(\theta_2 + \theta_3)L_1 + \cos \theta_3L_2]ELIw_7'''(0) \} / M_{11},
$$

\n
$$
Y_2 = \left\{-\hat{M}_{22}(\theta)\dot{w}_4 - M_{21}(\theta)\dot{w}_2 - M_{23}(\theta)\dot{w}_6 - C_{21}w_2 - C_{22}w_4 - C_{23}w_6\right\}
$$

\n
$$
- [L_1 \sin(\theta_2 + \theta_3)\dot{\theta_1} - L_2 \sin \theta_3\dot{\theta_3}] \rho K
$$

\n
$$
\int_0^l w_8 dr - L_2 \cos \theta_3 ELw_7'''(0) \} / M_{22},
$$

\n
$$
Y_3 = \left\{-\hat{M}_{33}(\theta)\dot{w}_6 - M_{31}(\theta)\dot{w}_2 - M_{32}(\theta)\dot{w}_4 - C_{31}w_2 - C_{32}w_4 - C_{33}w_6\right\}
$$

\n
$$
- [\sin(\theta_2 +
$$

in which $M_{ij}(\theta)$ (*i*, *j* = 1, 2, 3) in [\(23\)](#page-3-1) contains θ form as $M_{ij}(\theta)$, otherwise form as M_{ij} . The defined Hilbert space is shown below

$$
H = R^{6} \times H^{2}(\Omega) \times L^{2}(\Omega),
$$

$$
D(A) = R^{6} \times H^{4}(\Omega) \times H^{2}(\Omega),
$$

where

$$
\Omega = [0, l], L^{2}(\Omega) = \{f : \Omega \to R | \int_{0}^{l} |f|^{2} dr < \infty \},
$$

$$
H^{k}(\Omega) = \{f : \Omega \to R | f, f', f'', \cdots, f^{(k)} \in L^{2}(\Omega) \}.
$$

Define the energy of estimate error model as follows

$$
E_a = \frac{1}{2} \int_0^l [\rho K \dot{\tilde{z}}^2(r, t) + EI \tilde{u}''^2(t, r)] dr
$$

$$
+\frac{1}{2}\dot{\tilde{\theta}}^{\mathrm{T}}M(\theta)\dot{\tilde{\theta}}+\frac{1}{2}k_{p}\tilde{\theta}^{\mathrm{T}}\tilde{\theta}.
$$
 (35)

Then the time derivative of (35) is computed as

$$
\dot{E}_a = \int_0^l [\rho K \dot{\tilde{z}}(r, t) \ddot{\tilde{z}}(r, t) + EI \tilde{u}''(t, r) \dot{\tilde{u}}''(t, r)] dr \n+ \dot{\tilde{\theta}}^{\mathrm{T}} M(\theta) \ddot{\tilde{\theta}} + \frac{1}{2} \dot{\tilde{\theta}}^{\mathrm{T}} M(\theta) \dot{\tilde{\theta}} + k_p \dot{\tilde{\theta}}^{\mathrm{T}} \tilde{\theta}.
$$
\n(36)

Substituting the observer error dynamics (26) – (30) into the above equation, then further obtain(see Appendix \bf{B})

$$
\dot{E}_a = -k_d \dot{\tilde{\theta}}^\mathrm{T} \dot{\tilde{\theta}} \le 0. \tag{37}
$$

From Eq. [\(37\)](#page-4-1), it can be easily shown that operator *A* is dissipated, and *A* generates a C_0 semigroup in *H*. Since $Y(0) =$ 0, *Y*(*w*) is differentiable, we can easily get $(\lambda I - A)^{-1}$ is a compact operator for sufficiently large $\lambda > 0$ (e.g., for $\lambda > \tilde{\alpha}$) [\[32](#page-11-0)[,33\]](#page-11-1), then the solutions of [\(32\)](#page-3-5) locally exist in, moreover the bounded solutions are also precompact in $D \subset H$; Based on the LaSalle's Invariance Principle, it then follows that the solution of the system $w(t) \rightarrow \hat{C}$ asymptotically as $t \to \infty$, where $\hat{C} = \{w \in D | \vec{E}(w) = 0\}$, and $\hat{\Omega}$ is the largest invariant set in \hat{C} . However, Eq. [\(37\)](#page-4-1) is not sufficient to prove the stability of the designed observer. To strictly prove the observer is asymptotically stable, we should apply the extended LaSalle's Invariance Principle to show $\dot{E}_a = 0$ implies $w = 0$ and then verify the observer stability.

So from $\dot{E}_a = 0$, yields

$$
\dot{\tilde{\theta}}_i = 0, \quad \tilde{\theta}_i = \theta_i - \hat{\theta}_i = \text{const.}
$$
\n(38)

Putting (38) into (26) – (30) yields

$$
\rho K \int_0^l [-L_1 \sin(\theta_2 + \theta_3)] \dot{\vec{u}} dr \dot{\theta}_2 - \rho K [\sin(\theta_2 + \theta_3) L_1 + \sin \theta_3 L_2] \int_0^l \dot{\vec{u}} dr \dot{\theta}_3 - EI \tilde{u}_0'' + [\cos(\theta_2 + \theta_3) L_1 + \cos \theta_3 L_2 + L_3] EI \tilde{u}_0''' + k_{p1} \tilde{\theta}_1 = 0,
$$
\n(39)

$$
\rho KL_1 \sin(\theta_2 + \theta_3) \int_0 \dot{\tilde{u}} dr \dot{\theta}_1 - \rho KL_2 \sin \theta_3
$$

$$
\int_0^l \dot{\tilde{u}} dr \dot{\theta}_3 - EI \tilde{u}_0''
$$

$$
+ (L_2 \cos \theta_3 + L_3) EI \tilde{u}_0''' + k_{p2} \tilde{\theta}_2 = 0,
$$
 (40)

$$
\rho K[\sin(\theta_2 + \theta_3)L_1 + \sin \theta_3 L_2] \int_0^l \dot{\tilde{u}} dr \dot{\theta}_1 + \rho K L_2 \sin \theta_3
$$

$$
\int_0^l \dot{\tilde{u}} dr \dot{\theta}_2 - EI \tilde{u}_0'' + L_3 EI \tilde{u}_0''' + k_{p3} \tilde{\theta}_3 = 0, \tag{41}
$$

$$
\ddot{\tilde{u}}(t,r) + \frac{EI}{\rho K} \tilde{u}^4(t,r) = 0,
$$
\n(42)

² Springer

$$
\tilde{u}_0 = 0, \tilde{u}'_0 = 0, \tilde{u}''_E = 0, \tilde{u}'''_E = 0.
$$
\n(43)

By solving [\(42\)](#page-4-3) and [\(43\)](#page-4-3), the solutions $\tilde{u}(t, r)$ can be get according to the method of separating variables.

$$
\tilde{u}(t,r) = \varphi(r)e^{\lambda t},\tag{44}
$$

where $\varphi(r)$ are eigenfunctions for Eq. [\(44\)](#page-5-1) and λ are nonzero complex eigenvalues. Substituting [\(44\)](#page-5-1) into [\(42\)](#page-4-3), we have

$$
\varphi^4(r) = -\frac{\lambda^2 \rho K}{EI} \varphi(r). \tag{45}
$$

The solution of (45) is given by

$$
\varphi(r) = \gamma_1 e^{\beta r} + \gamma_2 e^{-\beta r} + \gamma_3 e^{j\beta r} + \gamma_4 e^{-j\beta r},
$$
\n(46)

where $\gamma_i (i = 1, 2, 3, 4) \in R$, $\beta = \sqrt[4]{-\lambda^2 \rho K / EI} \in C$; *j* is the imaginary unit, $j^2 = -1$.

Applying Eq. (46) to the boundary condition (43) , we obtain

$$
\varphi(0) = \varphi'(0) = \varphi''(l) = \varphi'''(l) = 0.
$$
\n(47)

Using (46) and (47) , we have

$$
\begin{cases}\n\gamma_1 + \gamma_2 + \gamma_3 + \gamma_4 = 0 \\
\gamma_1 - \gamma_2 + \gamma_3 - \gamma_4 = 0 \\
\gamma_1 + \gamma_2 - \gamma_3 - \gamma_4 = 0 \\
\gamma_1 - \gamma_2 - \gamma_3 + \gamma_4 = 0.\n\end{cases}
$$
\n(48)

The solution of Eq. [\(48\)](#page-5-5) is $\gamma_1 = \gamma_2 = \gamma_3 = \gamma_4 = 0$, hence $\varphi(r) = 0$, then we have

$$
\tilde{u}(t,r) = \varphi(r)e^{\lambda t} = 0.
$$
\n(49)

From Eqs. [\(39\)](#page-4-3)–[\(41\)](#page-4-3), we obtain that $\theta_i = 0$ and thus get

$$
\hat{\theta}_i \to \theta_i, \ \hat{\theta}_i \to \dot{\theta}_i, \ \hat{u}(t,r) \to u(t,r), \ \hat{u}(t,r) \to \dot{u}(t,r). \tag{50}
$$

Therefore, we proved $w = 0$ from $E_a = 0$, the designed observer is asymptotic stability.

Observer-based controller design

Furthermore, we propose the observer-based controller that requires only the independent joint input, and the controller can ensure the asymptotic stability.

Theorem 1 *If we set the control law as follows based on the estimate of observer* [\(23\)](#page-3-1)–[\(25\)](#page-3-2)*, the closed-loop system is asymptotically stability.*

$$
\tau = -k_1 e - k_2 \dot{e} - k_3 \dot{e} F(\theta, \hat{u}_0'', \hat{u}_0''')^{\mathrm{T}} F(\theta, \hat{u}_0'', \hat{u}_0''') + G,
$$
 (51)

 $in which k_1 = diag(k_{11}, k_{12}, k_{13}), k_2 = diag(k_{21}, k_{22}, k_{23}),$ $k_3 = diag(k_{31}, k_{32}, k_{33})$ *are* 3×3 *positive-definite diagonal matrix.*

Proof First, define vector *p* as

$$
p = [p_1 \ p_2 \ p_3 \ p_4 \ p_5 \ p_6 \ p_7 \ p_8 \ w^T]^T
$$

= $[e_1 \ \dot{e}_1 \ e_2 \ \dot{e}_2 \ e_3 \ \dot{e}_3 \ u \ \dot{u} \ w^T]^T$. (52)

The observer error dynamics [\(26\)](#page-3-3)–[\(30\)](#page-3-4), closed-loop system model (11) – (13) and (51) are rewritten in the matrix operator form as

$$
\dot{p} = Bp + N(p), \quad p(0) \in H,\tag{53}
$$

where *B* represents an infinite-dimensional linear operator as follows

$$
Bp = [p_2 \ s_1 \ p_4 \ s_2 \ p_6 \ s_3 \ p_8 \ s_4 \ Aw^T]^T,
$$

$$
N(p) = [0 \ N_1 \ 0 \ N_2 \ 0 \ N_3 \ 0 \ N_4 \ Y(w)^T]^T,
$$

in which

$$
s_1 = (E1p''_7(0) - EL1_3p'''_7(0) - k_{11}p_1 - k_{21}p_2)/M_{11},
$$

\n
$$
s_2 = (E1p''_7(0) - EL1_3p'''_7(0) - k_{12}p_3 - k_{22}p_4)/M_{22},
$$

\n
$$
s_3 = (E1p''_7(0) - EL1_3p'''_7(0) - k_{13}p_5 - k_{23}p_6)/M_{33},
$$

\n
$$
s_4 = -\frac{EI}{\rho K}p_7^{(4)},
$$

\n
$$
N_1 = \left\{-\hat{M}_{11}(\theta)\hat{p}_2 - M_{12}(\theta)\hat{p}_4 - M_{13}(\theta)\hat{p}_6 - C_{11}p_2 - C_{12}p_4 - C_{13}p_6\right.
$$

\n
$$
+ [\sin(\theta_2 + \theta_3)L_1p_4 + [\sin(\theta_2 + \theta_3)L_1]
$$

\n
$$
+ \sin \theta_3L_2]p_6]p_2\rho K \int_0^l p_8 dr
$$

\n
$$
- [\cos(\theta_2 + \theta_3)L_1 + \cos \theta_3L_2]EIp'''_7(0) - k_{31}p_2\hat{F}_1^T\hat{F}_1 \right\}/M_{11},
$$

\n
$$
N_2 = \left\{-\hat{M}_{22}(\theta)\hat{p}_4 - M_{21}(\theta)\hat{p}_2 - M_{23}(\theta)\hat{p}_6 - C_{21}p_2 - C_{22}p_4 - C_{23}p_6 - [L_1 \sin(\theta_2 + \theta_3)p_2 - L_2 \sin \theta_3p_6]p_4\rho K \int_0^l p_8 dr - L_2 \cos \theta_3 EIp'''_7(0) - k_{32}p_4\hat{F}_2^T\hat{F}_2)/M_{22},
$$

\n
$$
N_3 = \left\{-\hat{M}_{33}(\theta)\hat{p}_6 - M_{31}(\theta)\hat{p}_2 - M_{32}(\theta)\hat{p}_4 - C_{31}p_2 - C_{32}p_4 - C_{33}p_6 - [\sin(\theta_2 + \theta_3)L_1
$$

 $-(L_3 + r)(\dot{p_2} + \dot{p_4} + \dot{p_6}) - \dot{p_4}[\theta_1 L_1 \sin(\theta_2 + \theta_3) + \sin \theta_3 L_2 \theta_3]$

Table 1 Physical parameters of

Table 1 Physical parameters of the system	Parameter	Value	Parameter	Value	Parameter	Value
	L_1	0.7 _m	L_{1g}	0.35 m	m ₁	8 kg
	L_2	0.6 _m	L_{2g}	0.35 m	m ₂	8 kg
	L_3	0.2 _m	L_{3g}	0.17 m	m ₃	4 kg
	I_i	37×10^{-5} kg m	EΙ	20 Nm^2		1 m
	ρK	0.5 kg/m				

$$
- p_6[\theta_1 L_1 \sin(\theta_2 + \theta_3) + (\theta_1 + \theta_2)L_2 \sin \theta_3].
$$

The state space H and the domain $D(B)$ of the operator *B* are defined as

$$
H = R^6 \times H^2(\Omega) \times L^2(\Omega) \times R^6 \times H^2(\Omega) \times L^2(\Omega),
$$

$$
D(\mathsf{B}) = R^6 \times H^4(\Omega) \times H^2(\Omega) \times R^6 \times H^4(\Omega) \times H^2(\Omega).
$$

Then, choose the energy function of the close-loop system as

$$
E = \frac{1}{2} \int_0^l [\rho K \dot{z}^2(r, t) + EI u''^2(t, r)] dr
$$

$$
+ \frac{1}{2} e^{T} M(\theta) \dot{e} + \frac{1}{2} e^{T} k_1 e + E_a.
$$
 (54)

Differentiating [\(54\)](#page-6-1) with respect to time by using [\(19\)](#page-3-6) and [\(37\)](#page-4-1) yields

$$
\dot{E} = -q^T \dot{e} + \dot{e}^T [M(\theta)\ddot{e} + \frac{1}{2}M(\theta)\dot{e}] + \dot{e}^T k_1 e - \dot{\tilde{\theta}}^T k_d \dot{\tilde{\theta}}.
$$
\n(55)

We further have

$$
\dot{E} = -q^T \dot{e} + \dot{e}^T [\tau - F(\theta, u_0'', u_0''') - G] - \dot{\tilde{\theta}}^T k_d \dot{\tilde{\theta}}.
$$
 (56)

From [\(20\)](#page-3-7), we can get $F(\theta, u_0'', u_0''') - q = 0$, then substitute controller (51) into (56) and have

$$
\dot{E} = -\dot{e}^{\mathrm{T}}k_2\dot{e} - k_3\dot{e}^{\mathrm{T}}\dot{e}F(\theta, \hat{u}_0'', \hat{u}_0'')^{\mathrm{T}}F(\theta, \hat{u}_0'', \hat{u}_0''') - \dot{\tilde{\theta}}^{\mathrm{T}}k_d\dot{\tilde{\theta}} \le 0.
$$
\n(57)

According to [\(57\)](#page-6-3), we can know that operator *B* is dissipated. It can be also verified operator *B* generates a *C*0-semigroup in *H* based on Lumer–Phillips theorem that used in ["Infinite-dimensional observer"](#page-3-0) section. Then similar to the application of LaSalle's Invariance Principle in ["Infinite-dimensional observer"](#page-3-0) section, from $\dot{E} = 0$ we can also get

$$
e = \dot{e} = u(t, r) = \dot{u}(t, r) = \tilde{\theta} = \dot{\tilde{\theta}} = \tilde{u}(t, r) = \dot{\tilde{u}}(t, r) = 0.
$$
 (58)

Table 2 Control parameters of the system

Value Parameter		Parameter	Value	Parameter	Value
k_{11}	50	k_{12}	20	k_{13}	12
k_{21}	45	k_{22}	20	k_{23}	9
k_{31}	10	k_{32}	10	k_{33}	10
k_{d1}	37	k_{d2}	10	k_d 3	
k_{p1}	22	k_{p2}	17	k_{p3}	8

This means $p = 0$, it can be proved that the solution of $\dot{E} = 0$ is $p = 0$, and therefore, we get the asymptotically stable of the proposed observer-based control. \Box

Simulation

The designed observer and controller strategy are tested with the simulation implemented in MATLAB. System physical parameters are given in Table [1.](#page-6-4)

The initial and desired trajectory are as follows:

$$
\theta_{10} = 2.2 \text{ rad}, \quad \theta_{20} = 1.7 \text{ rad}, \quad \theta_{30} = 0.2 \text{ rad},
$$

\n $\theta_{1d} = 2.3 \text{ rad}, \quad \theta_{2d} = 1.8 \text{ rad}, \quad \theta_{3d} = 0.1 \text{ rad}.$

The controller and observer parameters of the system are given in Table [2.](#page-6-5) The observer-based controller is given to guarantee the asymptotic stability, which make the robot track the desired joint angular and its speed, and suppress the vibration. The simulation result of the proposed controller is provided by Figs. [2,](#page-7-0) [3,](#page-7-1) [4,](#page-7-2) [5,](#page-7-3) [6,](#page-8-2) [7](#page-8-3) and [8.](#page-8-4) Note that the simulation time is 10 s. Figure [2](#page-7-0) displays the three angles, angular velocities and the corresponding estimates, which can all track the desired trajectory. It is clear that the estimate angles and angle velocities could converge to their true values within 4 s. Figure [3](#page-7-1) shows the deflection at link end of the flexible beam and its estimation value, we can conclude that the vibration of the beam is suppressed and the estimate can converge to the real value. Figures [4](#page-7-2) and [5](#page-7-3) represent the strain force, the shear force and their estimate values, which indicate the vibration informations can be exactly observed. Figure [6](#page-8-2) indicates the three-dimensional deformation and its estimation of flexible beam. From Fig. [7,](#page-8-3) we can see the deformation error

Fig. 2 Angle θ_i , angle velocity $\dot{\theta}_i$ and estimations $\hat{\theta}_i$, $\hat{\theta}_i$

Fig. 3 End-point deflection u_E and estimation \hat{u}_E

Fig. 4 Strain force u_0'' and estimation \hat{u}_0''

Fig. 5 Shear force $u_0^{\prime\prime\prime}$ and estimation $\hat{u}_0^{\prime\prime\prime}$

Fig. 6 Deformation $u(t, r)$ and estimation $\hat{u}(t, r)$ of flexible beam

Fig. 7 Deformation error $\tilde{u}(t, r)$ of flexible beam

Fig. 8 Control input τ_i ($i = 1, 2, 3$)

can be effectively suppressed to asymptotically approach 0, which demonstrates the observer could accurately estimate the vibration deformation states. As shown in Fig. [8,](#page-8-4) the result is the controller input of the proposed independent joint control. Therefore, the simulations demonstrate the good performance of the proposed infinite-dimensional observer and controller.

Conclusion

In this paper, a non-linear infinite-dimensional observer is proposed for manipulator operating a flexible beam based on distributed parameter model. We further design an observerbased independent joint control to regulate the angles to follow the desired states and suppress the vibration of the beam simultaneously, which can avoid setting sensors and actuators on the beam. The asymptotic convergence of the observer and the controller is validated through theoretical proof. Numerical simulations have demonstrated the performance of the proposed observer and the control strategy. The next problem to be tackled is to research the cooperative force control of manipulation for flexible object by multi-arm robot based on infinite-dimensional model.

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Declarations

Conflict of interest The authors declared no potential conflict of interest with respect to the research, authorship, and/or publication of this article.

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Appendix A: Modeling parameters

The parameters of joint Eq. (11) are as follows.

$$
M_{11} = I_1 + I_2 + I_3 + m_1 L_{1g}^2 + m_2 L_{2g}^2 + m_3 L_{3g}^2
$$

⁺ (*m*² ⁺ *^m*3)*L*² 1+ *^m*3*L*² ² + 2*L*1(*m*2*L*2*^g* + *m*3*L*2) cos θ² + 2*m*3*L*1*L*3*^g* cos(θ² + θ3) + 2*m*3*L*2*L*3*^g* cos θ³ + ρ*Kl*[sin(θ² + θ3)*L*¹ + sin θ3*L*2] ², (A1) *^M*¹² ⁼ *^M*²¹ ⁼ *^I*² ⁺ *^I*³ ⁺ *^m*2*L*² ²*^g* ⁺ *^m*3*L*² 3*g* ⁺ *^m*3*L*² ² + (*m*2*L*2*^g* + *m*3*L*2) cos θ2*L*¹ + *m*3*L*1*L*3*^g* cos(θ² + θ3) + 2*m*3*L*2*L*3*^g* cos θ³ ⁺ ^ρ*KlL*2[sin(θ² ⁺ ^θ3)*L*¹ ⁺ sin ^θ3*L*2]sin ^θ3, (A2) *^M*¹³ ⁼ *^M*³¹ ⁼ *^I*³ ⁺ *^m*3*L*² ³*^g* + *m*3*L*1*L*3*^g* cos(θ² + θ3) ⁺ *^m*3*L*2*L*3*^g* cos ^θ3, (A3) *^M*²² ⁼ *^I*² ⁺ *^I*³ ⁺ *^m*2*L*² ²*^g* ⁺ *^m*3*L*² ³*^g* ⁺ *^m*3*L*² 2 + 2*m*3*L*2*L*3*^g* cos θ³ + ρ*K L*² ²*^l* sin ^θ³ sin ^θ3, (A4) *^M*²³ ⁼ *^M*³² ⁼ *^I*³ ⁺ *^m*3*L*² ³*^g* ⁺ *^m*3*L*2*L*3*^g* cos θ , (A5) *^M*³³ ⁼ *^I*³ ⁺ *^m*3*L*² ³*g*, (A6) *C*¹¹ = −(*m*2*L*2*^g* + *m*3*L*2)*L*1θ˙ 2 sin θ2 − *m*3*L*1*L*3*^g* sin(θ² + θ3)(θ˙ ²+θ˙ 3) − *m*3*L*2*L*3*g*θ˙ ³ sin θ³ + ρ*Kl*[sin(θ² + θ3)*L*¹ + sin θ3*L*2][*L*1(θ˙ 2 + θ˙ ³) cos(θ² + θ3) + *L*2θ˙ ³ cos ^θ3], (A7) *C*¹² = −(*m*2*L*2*^g* + *m*3*L*2)(θ˙ ¹ + θ˙ 2)*L*1 sin θ2 − *m*3(θ˙ ¹ + θ˙ ² + θ˙ 3)*L*1*L*3*g* sin(θ2 + θ3) − *m*3*L*2*L*2*g*θ˙ ³ sin θ³ − (*L*3*l* + 1 2 *l* 2) × (θ˙ ¹ + θ˙ ² + θ˙ ³)[*L*¹ sin(θ² + θ3)] + ρ*K L*2*l* sin θ3[*L*1(θ˙ ² + θ˙ ³) cos(θ² + θ3) + *L*2θ˙ ³ cos θ3] − ρ*K L*2(θ˙ 1 + θ˙ ²) cos θ3[*L*¹ sin(θ² + θ3)] + ρ*K L*1*L*2*l*θ˙ ¹ cos(θ² ⁺ ^θ3)sin ^θ3, (A8) *C*¹³ = −*m*3*L*1*L*3*g*(θ˙ ¹ + θ˙ ² + θ˙ ³)sin(θ² + θ3) − *m*3*L*2*L*3*g*(θ˙ ¹ + θ˙ ² + θ˙ 3)sin θ3 − ρ*K*(*L*3*l* + 1 2 *l* ²)(θ˙ ¹ + θ˙ ² + θ˙ 3) × [*L*¹ sin(θ² ⁺ ^θ3) ⁺ *^L*² sin ^θ3], (A9) *C*²¹ = (*m*2*L*2*^g* + *m*3*L*2)*L*1θ˙ 1 sin θ2 + *m*3*L*1*L*3*g*θ˙ ¹ sin(θ² + θ3) − *m*3*L*2*L*2*g*θ˙ ³ sin θ³ + ρ*K*(*L*3*l* + 1 2 *l* ²)(θ˙ ¹ + θ˙ ² + θ˙ 3) × sin(θ² + θ3) + ρ*K L*2*l* cos θ3[*L*1(θ˙ ¹ + θ˙ ² + θ˙ ³)sin(θ² + θ3) + *L*2θ˙ ³ sin θ3] − ρ*K L*1*L*2*l*θ˙ ¹ cos(θ² ⁺ ^θ3)sin ^θ3, (A10) *C*²² = −(*m*3*L*2*^g* + ρ*K L*² cos θ3)*L*2θ˙ ³ sin ^θ3, (A11) *C*²³ = −*m*3*L*2*L*3*g*(θ˙ ¹ + θ˙ ² + θ˙ 3)sin θ3 − ρ*K*(*L*3*l* + 1 2 *l* ²)(θ˙ ¹ + θ˙ ² + θ˙ ³)*L*² sin ^θ3, (A12) *C*³¹ = *m*3*L*1*L*3*g*θ˙ ¹ sin(θ² + θ3) + *m*3(θ˙ ¹ + θ˙ 2) *L*2*L*3*g* sin θ3 + ρ*K*(*L*3*l* + 1 2 *l* ²)(θ˙ ¹ + θ˙ ² + θ˙ ³)[sin(θ² + θ3) *^L*¹ ⁺ sin ^θ3*L*2], (A13) *C*³² = *m*3*L*2*L*3*g*(θ˙ ¹ + θ˙ ²)sin θ³ + ρ*K*(*L*3*l*

where $A = \rho K L_1 \sin(\theta_2 + \theta_3)$.

Appendix B: Calculation of the derivative of Lyapunov function

Substituting the observer error dynamics [\(26\)](#page-3-3)–[\(30\)](#page-3-4) into Eq. [\(36\)](#page-4-4), then further obtain

$$
\vec{E}_a = \rho K \int_0^l \dot{\tilde{z}} \left(-\frac{EI}{\rho K} \tilde{u}^{(4)} \right) dr + EI u'' \dot{u}' \Big|_0^l - EI u''' \dot{u} \Big|_0^l \n+ \int_0^l EI u^{(4)} \dot{u} dr \n+ \dot{\tilde{\theta}}^{\mathrm{T}} [M(\theta) \ddot{\tilde{\theta}} + \frac{1}{2} M(\theta) \dot{\tilde{\theta}}] + k_p \dot{\tilde{\theta}}^{\mathrm{T}} \tilde{\theta} \n= \rho K \int_0^l [\dot{\tilde{z}} \left(-\frac{EI}{\rho K} \tilde{u}^{(4)} \right) + EI \tilde{u}^{(4)} \dot{\tilde{u}}] dr + \dot{\tilde{\theta}}_1 \{ EI \tilde{u}_0'' \} \n- [L_1 cos(\theta_2 + \theta_3) \n+ L_2 cos \theta_3 + L_3] EI \tilde{u}_0''' - k_{d1} \dot{\tilde{\theta}}_1 - k_{p1} \tilde{\theta}_1 \} \n+ \dot{\tilde{\theta}}_2 \{ EI \tilde{u}_0''' \} \n- (L_2 cos \theta_3 + L_3) EI \tilde{u}_0''' - k_{d2} \dot{\tilde{\theta}}_2 - k_{p2} \tilde{\theta}_2 \} \n+ \dot{\tilde{\theta}}_3 \{ EI \tilde{u}_0''' - L_3 EI \tilde{u}_0''' \} \n- k_{d3} \dot{\tilde{\theta}}_3 - k_{p3} \tilde{\theta}_3 \} + k_p \dot{\tilde{\theta}}^{\mathrm{T}} \tilde{\theta},
$$
\n(B1)

where $EIu''\dot{u}'\vert_0^l = 0$, $EIu'''\dot{u}\vert_0^l = 0$. Based on the energy dynamic of flexible beam, we can obtain

$$
\rho K \int_0^l \dot{\tilde{z}} \left(-\frac{EI}{\rho K} \tilde{u}^{(4)} \right) dr + \int_0^l EI \tilde{u}^{(4)} \dot{\tilde{u}} dr \n= \rho K \int_0^l [\dot{\tilde{\theta}}_1 L_1 \cos(\theta_2 + \theta_3) + (\dot{\tilde{\theta}}_1 + \dot{\tilde{\theta}}_2) L_2 \cos \theta_3 \n+ (L_3 + r)(\dot{\tilde{\theta}}_1 + \dot{\tilde{\theta}}_2 + \dot{\tilde{\theta}}_3)
$$

+
$$
\ddot{u}][L_1\ddot{\theta}_1\cos(\theta_2+\theta_3)-\dot{\tilde{\theta}}_1\sin(\theta_2+\theta_3)L_1(\dot{\theta}_2+\dot{\theta}_3)
$$

+
$$
L_2(\ddot{\tilde{\theta}}_1+\ddot{\tilde{\theta}}_2)\cos\theta_3
$$

-
$$
(\dot{\tilde{\theta}}_1+\dot{\tilde{\theta}}_2)L_2\dot{\theta}_3\sin\theta_3+(L_3+r)(\ddot{\tilde{\theta}}_1+\ddot{\tilde{\theta}}_2+\ddot{\tilde{\theta}}_3)+\ddot{\tilde{u}}]dr
$$

+
$$
\int_0^l EI\tilde{u}^{(4)}\dot{\tilde{u}}dr
$$

=
$$
\rho K \int_0^l \dot{\tilde{r}}_t^T \ddot{\tilde{r}}_t dr + \int_0^l EI\tilde{u}^{(4)}\dot{\tilde{u}} dr.
$$
 (B2)

From Eq. (19) , we can get

 $\boldsymbol{0}$

$$
\rho K \int_0^l \dot{\tilde{z}} \left(-\frac{EI}{\rho K} \tilde{u}^{(4)} \right) dr + \int_0^l EI \tilde{u}^{(4)} \dot{\tilde{u}} dr \n= [L_1 \dot{\tilde{\theta}}_1 \cos(\theta_2 + \theta_3) + (\dot{\tilde{\theta}}_1 + \dot{\tilde{\theta}}_2) L_2 \cos \theta_3 + (\dot{\tilde{\theta}}_1 + \dot{\tilde{\theta}}_2) L_3] EI \tilde{u}_0''' - EI \tilde{u}_0'' (\dot{\tilde{\theta}}_1 + \dot{\tilde{\theta}}_2 + \dot{\tilde{\theta}}_3) \n= \dot{\tilde{\theta}}_1 \{ EI \tilde{u}_0'' - [L_1 \cos(\theta_2 + \theta_3) + L_2 \cos \theta_3 + L_3] EI \tilde{u}_0''' \} \n+ \dot{\tilde{\theta}}_2 \{ EI \tilde{u}_0'' - (L_2 \cos \theta_3 + L_3) EI \tilde{u}_0''' \} \n+ \dot{\tilde{\theta}}_3 \{ EI \tilde{u}_0'' - L_3 EI \tilde{u}_0''' \}. \tag{B3}
$$

Thus, we get

$$
\dot{E}_a = -k_d \dot{\tilde{\theta}}^\mathrm{T} \dot{\tilde{\theta}} \le 0. \tag{B4}
$$

References

- 1. Zhao B, Liu DR (2019) Event-triggered decentralized tracking control of modular reconfigurable robots through adaptive dynamic programming. IEEE Trans Ind Electron 67(4):3054–3064
- 2. Ren XL, Li HW (2022) Adaptive dynamic programming-based feature tracking control of visual serving manipulators with unknown dynamics. Complex Intell Syst 8(1):255–269
- 3. He W, Tang XY, Wang TT, Liu ZJ (2022) Trajectory tracking control for a three-dimensional flexible wing. IEEE Trans Control Syst Technol. <https://doi.org/10.1109/TCST.2021.3139087>
- 4. Han ZJ, Liu ZJ, Kang W, He W (2022) Boundary feedback control of a nonhomogeneous wind turbine tower with exogenous disturbances. IEEE Trans Automat Control 67(4):1952–1959
- 5. Zhao ZJ, Liu ZJ, He W, Hong KS, Li HX (2021) Boundary adaptive fault-tolerant control for a flexible Timoshenko arm with backlashlike hysteresis. Automatica 130(8):109690
- 6. Liu Y, Chen XB, Mei YF, Wu YL (2022) Observer-based boundary control for an asymmetric output-constrained flexible robotic manipulator. Sci China Inf Sci 65(3):139203:1-139203:3
- 7. Sun C, He W, Hong J (2017) Neural network control of a flexible robotic manipulator using the lumped spring-mass model. IEEE Trans Syst Man Cybern Syst 47(8):1863–1874
- 8. Gao HJ, He W, Zhou C, Sun CY (2019) Neural network control of a two-link flexible robotic manipulator using assumed mode method. IEEE Trans Ind Inform 15(2):755–765
- 9. Liu YH, Sun D (2000) Stabilizing a flexible beam handled by two manipulators via PD feedback. IEEE Trans Automat Control 45(11):2159–2164
- 10. Zhang Q, Mills JK, Cleghorn WL, Jin J, Zhao CS (2015) Trajectory tracking and vibration suppression of a 3-PRR parallel manipulator with flexible links. Multibody Syst Dyn 33(1):27–60
- 11. Zhang P, Li YC (2007) Position/force control of two manipulators handling a flexible payload based on finite-element model. In: IEEE international conference on robotics and biomimetics, pp 2178– 2182
- 12. Meng TT, He W, He XY (2021) Tracking control of a flexible string system based on iterative learning control. IEEE Trans Control Syst Technol 29(1):436–443
- 13. Liu SY, Langari R, Li YC (2019) Nonlinear direct joint control for manipulator handling a flexible payload with input constraints. Int J Robot Autom 34(6):645–653
- 14. Zhao ZJ, He XY, Ahn CK (2021) Boundary disturbance obsserverbased control of a vibrating single-link exible manipulator. IEEE Trans Syst Man Cybern Syst 51(4):2382–2390
- 15. Kawai Y, Endo T, Matsuno F (2020) Cooperative control of large flexible space structure by two planar robots. IET Control Theory A 12:1–13
- 16. Endo T, Sasaki M, Matsuno F (2017) Contact-force control of a flexible Timoshenko arm. IEEE Trans Automat Control 62(2):1004–1009
- 17. Liu ZJ, Han ZJ, Zhao ZJ, He W (2021) Modeling and adaptive control for a spatial flexible spacecraft with unknown actuator failures. Sci China Inform Sci 64(5):1–16
- 18. He W, Kang FS, Kong LH, Feng YH, Cheng GQ, Sun CY (2022) Vibration control of a constrained two-link flexible robotic manipulator with fixed-time convergence. IEEE Trans Cybern 52(7):5973–5983
- 19. Ren Y, Zhao ZJ, Zhang CL, Yang QM, Hong KS (2020) Adaptive neural-network boundary control for a flexible manipulator with input constraints and model uncertainties. IEEE Trans Cybern 51(10):4796–4807
- 20. Liu ZJ, He XY, Zhao ZJ, Ahn CK, Li HX (2021) Vibration control for spatial aerial refueling hoses with bounded actuators. IEEE Trans Ind Electron 68(5):4209–4217
- 21. Jiang TT, Liu JK, He W (2017) A robust observer design for a flexible manipulator based on a PDE model. J Vib Control 23(6):871–882
- 22. Yang HJ, Liu JK, Lan X (2015) Observer design for a flexible-link manipulator with PDE model. J Sound Vib 341(4):237–245
- 23. Feng H, Xu CZ, Yao PF (2020) Observers and disturbance rejection control for a heat equation. IEEE Trans Automat Control 65(131):4957–4964
- 24. Yu H, Gan G, Bayen A, Krstic M (2020) PDE trac observer validated on freeway data. IEEE Trans Control Syst Technol 29(3):1048–1060
- 25. Li XD, Xu CZ (2011) Infinite-dimensional Luenberger-like observers for a rotating body-beam system. Syst Control Lett 60(2):138–145
- 26. Cao FF, Liu JK (2017) An adaptive iterative learning algorithm for boundary control of a coupled ODE–PDE two-link rigid-flexible manipulator. J Franklin Inst 354(1):277–297
- 27. He W, Wang TT, He XY, Yang LJ, Kaynak O (2020) Dynamical modeling and boundary vibration control of a rigid-flexible wing system. IEEE/ASME Trans Mech 25(6):2711–2721
- 28. Cao FF, Liu JK (2019) Partial differential equation modeling and vibration control for a nonlinear 3D rigid-flexible manipulator system with actuator faults. Int J Robust Nonlinear 29(11):3793–3807
- 29. Xing XY, Liu JK (2018) LMI-based boundary and distributed control design for a flexible string subject to disturbance. Int J Control $92(8):1-11$
- 30. Wang JW, Liu YQ, Sun CY (2019) Adaptive neural boundary control design for nonlinear flexible distributed parameter systems. IEEE Trans Control Syst Technol 27(5):2085–2099
- 31. Liu SY, Liu ZJ, Li YC, He W (2022) Nonlinear disturbance observer-based direct joint control for manipulation of a flexible payload with output constraints. Int J Control. [https://doi.org/10.](https://doi.org/10.1080/00207179.2022.2046858) [1080/00207179.2022.2046858](https://doi.org/10.1080/00207179.2022.2046858)
- 32. Pazy A (1983) Semigroups of linear operators and applications to partial differential equations. Appl Math Sci 44:13–17
- 33. Luo ZH, Guo BZ, Morgul O (1999) Stability and stabilization of infinite dimensional systems with applications. Springer, London, pp 157–161

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