TECHNICAL PAPER

An analytical solution for the free vibration of FG nanoplates

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Abstract

In the present work, an analytical solution for the free vibration of nanoplates made of functionally graded materials (FGMs) under various boundary conditions is provided. In this context, a new refned plate theory with four variables based on the theory of non-local elasticity including the small-scale infuences is adopted. Using the rule of mixture, the material properties of nanoplates are supposed to vary continuously across the thickness direction. Based on Eringen's non-local elasticity theory, the equations of motion of functionally graded (FG) nanoplate are derived using Hamilton's principle, and the obtained equations are solved analytically. Here, the number of unknowns and governing equations of the present model are reduced to four separating the vertical displacement into shear and bending components, and so the number of unknowns has become less than the other alternative theories. The infuences of the diferent parameters such as vibration mode, the aspect ratio, boundary conditions, power-law index, and non-local parameter on the natural frequencies of the FG nanoplate are discussed, in detail. Finally, it is decided that the considered parameters have major infuence on the natural frequencies of the FG nanoplates. Furthermore, the proposed solution method not only satisfactorily handled the present problem and yielded successful results but also it supplied ease in the examination of non-local free vibration of FG nanoplates.

Keywords Free vibration · Analytical solution · Non-local theory · FGMs · Nanoplate

1 Introduction

The new kind of composites with smoothly varying material properties changing the volume fraction of constituents gradually along with one or more directions is called FGMs. In comparison with the traditional composite materials, FGMs exhibit not only superior features such as extremely high stifness and strength combined with a very low density, resistance to chemicals, thermal and electrical insulation but also eliminate stress concentrations at the interfaces of composites [\[1,](#page-11-0) [2\]](#page-11-1). FGMs are regularly composed of two diferent

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kinds of structural components like ceramic and metal, and the volume fractions of these constituents change as a function of the certain dimensions of the structure to realize the desired functions. FGMs have a wide application range in modern technology including aerospace, medicine, defense, energy, optoelectronics, automotive, biotechnology, aviation, civil, and mechanical engineering [\[3](#page-11-2)[–10](#page-11-3)].

Since the introduction of carbon nanotubes (CNTs) by Iijima [\[11\]](#page-11-4), nanotechnology has become an important part of mankind as a result of the rapidly increasing use of nanosized structures in modern life as well as in defense technology for the miniaturization of information technology devices. Nanowires, nanotubes, nanobeams, nanomembranes, and nanoplates are some examples of nano-sized structures composed of nanomaterials with outstanding mechanical, chemical, electronic, electrical, and optical properties, and they have utilization in nano-/microelectromechanical systems (NEMS/MEMS), aerospace, biomedical and bioelectrical devices; some examples are photovoltaic cell, generators, micro-/nano-switches resonators, sensors, energy harvesting, transistors, and atomic force microscopy [[12–](#page-11-5)[21\]](#page-12-0).

Nanostructures have ridiculously small dimensions that are comparable to the size of their material microstructure, and therefore, it becomes essential to study the size efect as the mechanical behaviors of nanostructures. Several approaches have been developed to capture the size infuences as experimental, molecular dynamics simulations, and continuum mechanics. However, continuum mechanics is the most widely used method between these approaches because it is easy, cost-efficient, and effective. But the classical continuum mechanics cannot capture the size infuence since the constitutive model of it does not include the material length parameters. Thus, a variety of size-dependent continuum mechanics approaches are established such as the surface elasticity theory, non-local elasticity theory, the modifed couple stress theory, and strain gradient elasticity theory. Between these approaches, Eringen's non-local elasticity theory contains only one scale parameter of small length for the defnitions of the internal length scale of nanostructures and atomic forces, and hence, it has been widely employed [\[22](#page-12-1)[–24](#page-12-2)].

Currently, the investigations on the mechanical behaviors of nanoplates composed of FGMs have attracted many researchers due to the increasing demand for applications of these structures in modern technology because of their superior properties. In various engineering practices, structural elements are exposed to dynamic loads, which can excite the structural vibrations and induce durability concerns or discomfort due to the resulting noise and vibration. Besides, if the vibration exceeds particular limits, breakage or failure may occur. Since FG nanoplates have been started to be used in micro-/nanoelectromechanical systems, such as the components in the form of shape-memory alloy thin flms with a global thickness in micro- or nano-scale [\[25](#page-12-3), [26\]](#page-12-4), electrically actuated MEMS devices [[27](#page-12-5), [28](#page-12-6)], and atomic force microscopes [[29](#page-12-7)], as well as medicines, gas sensors, energy storage, feld emission, transportation of nano-cars and solar cells [\[30–](#page-12-8)[32\]](#page-12-9), the vibration analysis of nanoplates has been a vital task in their design for engineers and researchers for recent years. Gürses et al. [[33\]](#page-12-10) performed numerical computations using discrete singular convolution for the free vibration of nano-annular sector plates based on non-local continuum theory. The size-dependent free vibration of FG nanoplates is examined by Natarajan et al. [\[34\]](#page-12-11) using the isogeometric-based fnite element method. The three-dimensional non-local bending and vibration analyses of FG nanoplates are performed by Ansari et al. [\[35\]](#page-12-12) using the variational diferential quadrature method. Daneshmehr et al. [[36\]](#page-12-13) studied the free vibration of FG nanoplate using the generalized diferential quadrature method. Nami and Janghorban [\[37\]](#page-12-14) presented the free vibration of rectangular nanoplates with simply supported boundary conditions depending on refned plate theory with two variables employing strain gradient elasticity theory. Mechab et al. [[38\]](#page-12-15) examined the free vibration of FG nanoplate resting on Winkler–Pasternak elastic foundations based on refned plate theories with two variables including the efect of porosities. Phung-Van et al. [[39\]](#page-12-16) presented the size-dependent geometrically nonlinear transient analysis of FG nanoplates employing a solution procedure depending on isogeometric analysis combined with higher-order shear deformation theory. Arefi et al. [[40](#page-12-17)] analyzed the free vibration of FG polymer composite nanoplates reinforced with graphene nanoplatelets resting on a Pasternak foundation applying a two-variable sinusoidal shear deformation theory adopting the non-local elasticity theory. Barati and Shahverdi [\[41](#page-12-18)] dealt with the semi-analytical nonlinear thermal vibration analysis of FG nanoplates modeled by four-variable refned plate theory. Zargaripoor et al. [[42](#page-12-19)] examined the free vibration of FG nanoplate using the fnite element method. Baretta et al. [\[43](#page-12-20)] studied the sizedependent elastostatic responses of axisymmetric annular nano-plates using a stress-driven non-local integral methodology of elasticity. Ruocco and Mallardo [[44](#page-12-21)] considered the buckling and vibration of imperfect nanoplates via non-local Mindlin plate theory employing a coupling fnite strip–fnite element procedure. Sharif et al. [\[45](#page-12-22)] dealt with the vibration of FG piezoelectric nanoplates based on the non-local strain gradient theory, analytically. Daikh et al. [\[46](#page-12-23)] analyzed the free vibration of rectangular FGM sandwich nanoplates with simply supported boundary conditions. Tran et al. [\[47](#page-12-24)] analyzed the bending and free vibration of FG nanoplates resting on elastic foundations utilizing a fnite element model using four-unknown shear deformation theory integrated with the non-local theory. Zenkour et al. [[48](#page-12-25)] examined the bending of simply supported FG nanoplate utilizing the non-local mixed variational formula. Liu et al. [\[49](#page-12-26)] studied the thermo-electro-mechanical free vibration of piezoelectric nanoplates based on the non-local theory and Kirchhof theory. Malekzadeh and Shojaee [\[50\]](#page-13-0) investigated the free vibration of nanoplates utilizing a refned plate theory joint with the non-local theory. Kiani [[51\]](#page-13-1) examined the features of in-plane and out-of-plane free vibrations of a conducting nanoplate subjected to the unidirectional in plane steady magnetic feld utilizing diferent non-local shear deformable plate theories.

From the results of the search of open literature, it is observed that numerous studies are devoted to examine the mechanical behaviors of FG nanoplates. In most of these studies, numerical solution methods are employed, the number of studies based on the exact solution is quite limited, and in these studies, the exact solutions are developed for simple support boundary conditions, generally. Because the free vibration of the FG nanoplates has great importance in modern technology, it has become a necessity to derive a reliable exact solution for this behavior under diferent boundary conditions. Therefore, in the current study, an attempt is done to address this problem. For this aim, an exact analytical solution for the free vibration of nanoplates made of FGMs under four diferent boundary conditions is provided. The infuence of the diferent parameters such as vibration mode, the aspect ratio, boundary conditions, power-law index, and non-local parameter on the natural frequencies of the FG nanoplate is discussed, in detail.

2 The formulation of the problem

Figure [1](#page-2-0) displays an FG nanoplate of length *a*, width *b*, and height *h*. It is supposed that the top surface of the nanoplate $(z = +h/2)$ is ceramic-rich and it is graded to the metal-rich one at the bottom surface $(z = -h/2)$.

The material properties, *P*, are supposed to grade across the thickness of FG nanoplate depending on the rule of the mixture as follows:

$$
P(z) = P_c V_c + P_m V_m \tag{1}
$$

where P_c , P_m , V_c , and V_m represent the material properties and volume fractions of the ceramic and metal constituents, respectively.

The total volume fraction of constituents is as follows:

$$
V_m + V_c = 1\tag{2}
$$

The volume fraction of the ceramic constituent obeying the power-law distribution is as follows:

$$
V_c = \left(\frac{z}{h} + \frac{1}{2}\right)^k\tag{3}
$$

where *k* is a non-negative number and indicates the powerlaw index and *z* is the distance to the midplane of the nanoplate.

The efficient material properties of FG nanoplate could be expressed as:

$$
E(z) = (E_c - E_m)V_c + E_m
$$
\n(4a)

Fig. 1 The geometry of FG nanoplate

$$
\rho(z) = (\rho_c - \rho_m)V_c + \rho_m \tag{4b}
$$

where E and ρ are the Young's modulus and the mass density of FG nanoplate, respectively.

Adopting Eringen's [[22,](#page-12-1) [23](#page-12-27)] non-local theory of elasticity, the infuence of inter-atomic forces can be taken into account as the material parameters in the fundamental equation as follows:

$$
(1 - \tau^2 L^2 \nabla^2) \sigma_{ij}^{NL} = \sigma_{ij}^L; \quad \tau^2 = \frac{\mu}{L^2} = \left(\frac{e_0 a}{L}\right)^2 \tag{5}
$$

where superscripts (L, NL) denote the local and non-local, respectively, $\mu = (e_0 a)^2$ is the non-local parameter representing the small-scale influence, e_0 is a physical parameter that has been identifed by experimental results, and *a* and *L* are the internal and external characteristic lengths, respectively.

Non-local constitutive equations of FG nanoplate are as follows:

$$
\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{xz} \end{bmatrix} - \mu \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{xz} \end{Bmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 & 0 & 0 \\ Q_{12} & Q_{22} & 0 & 0 & 0 \\ 0 & 0 & Q_{44} & 0 & 0 \\ 0 & 0 & 0 & Q_{55} & 0 \\ 0 & 0 & 0 & 0 & Q_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{yz} \\ \gamma_{zx} \\ \gamma_{xy} \end{bmatrix}
$$
 (6)

where σ_x , σ_y , τ_{xy} , τ_{yz} , τ_{xz} and ϵ_x , ϵ_y , γ_{xy} , γ_{yz} , γ_{xz} are the components of stress and strain, respectively, and C_{ii} are the stiffness coefficients and described as follows:

$$
Q_{11} = Q_{22} = \frac{E(z)}{1 - v^2}, \quad Q_{12} = \frac{v E(z)}{1 - v^2},
$$

$$
Q_{44} = Q_{55} = Q_{66} = \frac{E(z)}{2(1 + v)}
$$
 (7)

Note that the transverse normal stress is negligible in comparison with in-plane stresses in *x*- and *y*-axes.

The displacements of FG nanoplate depending on the higher-order shear deformable plate theory are [[7,](#page-11-6) [52,](#page-13-2) [53\]](#page-13-3):

$$
u(x, y, z, t) = u_0(x, y, t) - z \frac{\partial w_b}{\partial x} - f(z) \frac{\partial w_s}{\partial x}
$$

$$
v(x, y, z, t) = v_0(x, y, t) - z \frac{\partial w_b}{\partial y} - f(z) \frac{\partial w_s}{\partial y}
$$
 (8)

 $w(x, y, z, t) = w_b(x, y, t) + w_s(x, y, t)$

where *u*, *v*, and *w* denote the displacements throughout *x*, *y*, and *z* directions, respectively, and u_0 , v_0 , w_b and w_s are the unknowns as well as $f(z)$ is the shape function and expressed as follows:

$$
f(z) = z \left[1 + \frac{3\pi}{2} \operatorname{sech}\left(\frac{1}{2}\right)^2 \right] - \frac{3\pi}{2} h \tanh\left(\frac{z}{h}\right) \tag{9}
$$

The nonzero strains related to the displacement feld are:

$$
\begin{cases}\n\varepsilon_x \\
\varepsilon_y \\
\gamma_{xy}\n\end{cases} = \begin{cases}\n\varepsilon_y^0 \\
\varepsilon_y^0 \\
\gamma_{xy}^0\n\end{cases} + z \begin{cases}\nk_x^b \\
k_y^b \\
k_y^b \\
k_y^b\n\end{cases} + f(z) \begin{cases}\nk_y^s \\
k_y^s \\
k_y^s \\
k_y^s\n\end{cases}
$$
\n(10)

where

$$
\begin{Bmatrix}\n\varepsilon_{x}^{0} \\
\varepsilon_{y}^{0} \\
\gamma_{xy}^{0}\n\end{Bmatrix} = \begin{Bmatrix}\n\frac{\partial u_{0}}{\partial x} \\
\frac{\partial v_{0}}{\partial x} \\
\frac{\partial u_{0}}{\partial y} + \frac{\partial v_{0}}{\partial x}\n\end{Bmatrix}, \quad\n\begin{Bmatrix}\nk_{x}^{b} \\
k_{y}^{b} \\
k_{xy}^{b}\n\end{Bmatrix} = \begin{Bmatrix}\n-\frac{\partial^{2} w_{b}}{\partial x^{2}} \\
-\frac{\partial^{2} w_{b}}{\partial y^{2}} \\
-2\frac{\partial^{2} w_{b}}{\partial x \partial y}\n\end{Bmatrix},
$$
\n
$$
\begin{Bmatrix}\nk_{x}^{s} \\
k_{y}^{s} \\
k_{xy}^{s}\n\end{Bmatrix} = \begin{Bmatrix}\n-\frac{\partial^{2} w_{s}}{\partial x^{2}} \\
-\frac{\partial^{2} w_{s}}{\partial x^{2}} \\
-2\frac{\partial^{2} w_{s}}{\partial x \partial y}\n\end{Bmatrix}, \quad\n\begin{Bmatrix}\n\gamma_{y}^{s} \\
\gamma_{xz}^{s}\n\end{Bmatrix} = g(z) \begin{Bmatrix}\n\frac{\partial w_{s}}{\partial y} \\
\frac{\partial w_{s}}{\partial x}\n\end{Bmatrix}
$$
\n(11a)

and then

$$
g(z) = 1 - \frac{df(z)}{dz}
$$

t

Utilizing Hamilton's principle, the equations of motion could be found as follows:

$$
0 = \int_{0}^{t} (\delta U - \delta K) dt
$$
 (12)

where δU and δK are the variations strain and kinetic energies, respectively.

The variation of strain energy of FG nanoplate can be specifed as follows:

$$
\delta U = \int_{V} \left[\sigma_x \delta \varepsilon_x + \sigma_y \delta \varepsilon_y + \tau_{xy} \delta \gamma_{xy} + \tau_{yz} \delta \gamma_{yz} + \tau_{xz} \delta \gamma_{xz} \right] dV
$$

\n
$$
= \int_{A} \left[N_x \delta \varepsilon_x^0 + N_y \delta \varepsilon_y^0 + N_{xy} \delta \gamma_{xy}^0 + M_x^b \delta k_x^b + M_y^b \delta k_y^b + M_{xy}^b \delta k_{xy}^b \right. \n+ M_x^s \delta k_x^s + M_y^s \delta k_y^s + M_{xy}^s \delta k_{xy}^s + S_{yz}^s \delta \gamma_{yz}^s + S_{xz}^s \delta \gamma_{xz}^s \right] dA = 0
$$
\n(13)

where *A* is the upper surface of FG nanoplate, and the stress resultants *N*, *M*, and *S* can be described as follows:

$$
(N_i, M_i^b, M_i^s) = \int_{-h/2}^{h/2} (1, z, f)\sigma_i dz, \quad (i = x, y, xy)
$$
 (14a)

$$
\left(S_{xz}^s, S_{yz}^s\right) = \int_{-h/2}^{h/2} g\left(\tau_{xz}, \tau_{yz}\right) dz \tag{14b}
$$

The variation of the kinetic energy of FG nanoplate can be specifed as follows:

$$
\delta T = \int_{-\frac{h}{2}}^{\frac{h}{2}} \int_{-\frac{h}{2}} \left[\dot{u}\delta\dot{u} + \dot{v}\delta\dot{v} + \dot{w}\delta\dot{w} \right] \rho(z) d\Omega dz
$$
\n
$$
= \int_{A} \left\{ I_0 \left[\dot{u}_0 \delta\dot{u}_0 + \dot{v}_0 \delta\dot{v}_0 + \left(\dot{w}_b + \dot{w}_s \right) \left(\delta\dot{w}_b + \delta\dot{w}_s \right) \right] \right\}
$$
\n
$$
- I_1 \left(\dot{u}_0 \frac{\partial \delta\dot{w}_b}{\partial x} + \frac{\partial \dot{w}_b}{\partial x} \delta\dot{u}_0 + \dot{v}_0 \frac{\partial \delta\dot{w}_b}{\partial y} + \frac{\partial \dot{w}_b}{\partial y} \delta\dot{v}_0 \right)
$$
\n
$$
- I_2 \left(\dot{u}_0 \frac{\partial \delta\dot{w}_s}{\partial x} + \frac{\partial \dot{w}_s}{\partial x} \delta\dot{u}_0 + \dot{v}_0 \frac{\partial \delta\dot{w}_s}{\partial y} + \frac{\partial \dot{w}_s}{\partial y} \delta\dot{v}_0 \right)
$$
\n
$$
+ J_1 \left(\frac{\partial \dot{w}_b}{\partial x} \frac{\partial \delta\dot{w}_b}{\partial x} + \frac{\partial \dot{w}_b}{\partial y} \frac{\partial \delta\dot{w}_b}{\partial y} \right) + K_2 \left(\frac{\partial \dot{w}_s}{\partial x} \frac{\partial \delta\dot{w}_s}{\partial x} + \frac{\partial \dot{w}_s}{\partial y} \frac{\partial \delta\dot{w}_s}{\partial y} \right)
$$
\n
$$
+ J_2 \left(\frac{\partial \dot{w}_b}{\partial x} \frac{\partial \delta\dot{w}_s}{\partial x} + \frac{\partial \dot{w}_s}{\partial x} \frac{\partial \delta\dot{w}_b}{\partial x} + \frac{\partial \dot{w}_b}{\partial y} \frac{\partial \delta\dot{w}_s}{\partial y} + \frac{\partial \dot{w}_s}{\partial y} \frac{\partial \delta\dot{w}_b}{\partial y} \right) \right\} dA \tag{15}
$$

where the dot superscript demonstrates diferentiation concerning the variable of time t , I_i , J_i , K_i terms define the mass moment of inertias of FG nanoplate as follows:

$$
(I_0, I_1, I_2) = \int_{-h/2}^{h/2} (1, z, z^2) \rho(z) dz
$$
 (16a)

$$
(J_1, J_2, K_2) = \int_{-h/2}^{h/2} (f, zf, f^2) \rho(z) dz
$$
 (16b)

Inserting Eqs. (13) and (15) (15) (15) in Eq. (12) (12) and after some mathematical operations and simplifcation, the subsequent equations are found

$$
\delta u_0 : \frac{\partial N_x}{\partial x} + \frac{\partial N_{xy}}{\partial y} = I_0 \ddot{u}_0 - I_1 \frac{\partial \ddot{w}_b}{\partial x} - J_1 \frac{\partial \ddot{w}_s}{\partial x}
$$

\n
$$
\delta v_0 : \frac{\partial N_{xy}}{\partial x} + \frac{\partial N_y}{\partial y} = I_{01} \ddot{v}_0 - I_1 \frac{\partial \ddot{w}_b}{\partial y} - J_1 \frac{\partial \ddot{w}_s}{\partial y}
$$

\n
$$
\delta w_b : \frac{\partial^2 M_x^b}{\partial x^2} + 2 \frac{\partial^2 M_y^b}{\partial x \partial y} + \frac{\partial^2 M_y^b}{\partial y^2}
$$

\n
$$
= I_0 (\ddot{w}_b + \ddot{w}_s) + I_1 \left(\frac{\partial \ddot{u}_0}{\partial x} + \frac{\partial \ddot{v}_0}{\partial y} \right) - I_2 \nabla^2 \ddot{w}_b - J_2 \nabla^2 \ddot{w}_s
$$

\n
$$
\delta w_s : \frac{\partial^2 M_x^s}{\partial x^2} + 2 \frac{\partial^2 M_{xy}^s}{\partial x \partial y} + \frac{\partial^2 M_y^s}{\partial y^2} + \frac{\partial S_x^s}{\partial x} + \frac{\partial S_y^s}{\partial y}
$$

\n
$$
= I_0 (\ddot{w}_b + \ddot{w}_s) + J_1 \left(\frac{\partial \ddot{u}_0}{\partial x} + \frac{\partial \ddot{v}_0}{\partial y} \right) - J_2 \nabla^2 \ddot{w}_b - K_2 \nabla^2 \ddot{w}_s
$$

\n(17)

Inserting Eqs. (6) (6) and (11) in Eq. (14) and after some mathematical operations, the stress resultants concerning generalized displacements (u_0, v_0, w_b, w_s) are as follows:

$$
\begin{Bmatrix} N \\ M^b \\ M^s \end{Bmatrix} - \mu \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) \begin{Bmatrix} N \\ M^b \\ M^s \end{Bmatrix} = \begin{bmatrix} A & B & B^s \\ B & D & D^s \\ B^s & D^s & H^s \end{bmatrix} \begin{Bmatrix} \varepsilon \\ k^b \\ k^s \end{Bmatrix},
$$
\n
$$
S - \mu \left(\frac{\partial^2 S}{\partial x^2} + \frac{\partial^2 S}{\partial y^2} \right) = A^s \gamma \tag{18}
$$

where

$$
N = \{N_x, N_y, N_{xy}\}^t, \quad M^b = \{M_x^b, M_y^b, M_{xy}^b\}^t,
$$

$$
M^s = \{M_x^s, M_y^s, M_{xy}^s\}^t
$$
 (19a)

$$
\varepsilon = \left\{ \varepsilon_x^0, \varepsilon_y^0, \gamma_{xy}^0 \right\}, \quad k^b = \left\{ k_x^b, k_y^b, k_{xy}^b \right\}, \quad k^s = \left\{ k_x^s, k_y^s, k_{xy}^s \right\}
$$
\n
$$
A = \begin{bmatrix} A_{11} & A_{12} & 0 \\ A_{12} & A_{22} & 0 \\ 0 & 0 & A_{66} \end{bmatrix}, \quad B = \begin{bmatrix} B_{11} & B_{12} & 0 \\ B_{12} & B_{22} & 0 \\ 0 & 0 & B_{66} \end{bmatrix}, \quad D = \begin{bmatrix} D_{11} & D_{12} & 0 \\ D_{12} & D_{22} & 0 \\ 0 & 0 & D_{66} \end{bmatrix}
$$
\n
$$
B^s = \begin{bmatrix} B_{11}^s & B_{12}^s & 0 \\ B_{12}^s & B_{22}^s & 0 \\ 0 & 0 & B_{66}^s \end{bmatrix}, \quad D^s = \begin{bmatrix} D_{11}^s & D_{12}^s & 0 \\ D_{12}^s & D_{22}^s & 0 \\ 0 & 0 & D_{66}^s \end{bmatrix}
$$
\n
$$
H^s = \begin{bmatrix} H_{11}^s & H_{12}^s & 0 \\ H_{12}^s & H_{22}^s & 0 \\ 0 & 0 & H_{66}^s \end{bmatrix}
$$
\n
$$
(19d)
$$

$$
S = \left\{ S_{xz}^s, S_{yz}^s \right\}^t, \quad \gamma = \left\{ \gamma_{xz}^0, \gamma_{yz}^0 \right\}^t, \quad A^s = \begin{bmatrix} A_{44}^s & 0 \\ 0 & A_{55}^s \end{bmatrix} \tag{19e}
$$

where A_{ij} , B_{ij} , D_{ij} are the material stiffness coefficients and defned as follows:

$$
\begin{cases}\nA_{11} B_{11} D_{11} B_{11}^{s} D_{11}^{s} H_{11}^{s} \\
A_{12} B_{12} D_{12} B_{12}^{s} D_{12}^{s} H_{12}^{s} \\
A_{66} B_{66} D_{66} B_{66}^{s} D_{66}^{s} H_{66}^{s}\n\end{cases}
$$
\n
$$
= \int_{-h/2}^{h/2} Q_{11}(1, z, z^{2}, f(z), z f(z), f^{2}(z)) \begin{cases}\n1 \\
\frac{1 - v}{2} \\
\frac{1 - v}{2}\n\end{cases} dz
$$
\n(20a)

(20b) $(A_{22}, B_{22}, D_{22}, B_{22}^s, D_{22}^s, H_{22}^s) = (A_{11}, B_{11}, D_{11}, B_{11}^s, D_{11}^s, H_{11}^s)$

$$
A_{44}^s = A_{55}^s = \int_{-h/2}^{h/2} Q_{44} [g(z)]^2 dz
$$
 (20c)

Inserting Eq. (18) (18) in Eq. (17) (17) (17) , the following equation is found:

$$
A_{11} \frac{\partial^2 u_0}{\partial x^2} + A_{66} \frac{\partial^2 u_0}{\partial y^2} + (A_{12} + A_{66}) \frac{\partial^2 v}{\partial x \partial y}
$$

\n
$$
- B_{11} \frac{\partial^3 w_b}{\partial x^3} - (B_{12} + 2B_{66}) \frac{\partial^3 w_b}{\partial x \partial y^2}
$$

\n
$$
- B_{11}^s \frac{\partial^3 w_s}{\partial x^3} - (B_{12}^s + 2B_{66}^s) \frac{\partial^3 w_s}{\partial x \partial y^2}
$$

\n
$$
= (1 - \mu \nabla^2) \left[I_0 \ddot{u}_0 - I_1 \frac{\partial \ddot{w}_b}{\partial x} - J_1 \frac{\partial \ddot{w}_s}{\partial x} \right],
$$
 (21a)

$$
(A_{12} + A_{66}) \frac{\partial^2 u_0}{\partial x \partial y} + A_{66} \frac{\partial^2 v_0}{\partial x^2} + A_{22} \frac{\partial^2 v_0}{\partial y^2} - (B_{12} + 2B_{66}) \frac{\partial^3 w_b}{\partial x^2 \partial y} - B_{22} \frac{\partial^3 w_b}{\partial y^3} - B_{22}^s \frac{\partial^3 w_s}{\partial y^3} - (B_{12}^s + 2B_{66}^s) \frac{\partial^3 w_s}{\partial x^2 \partial y} = (1 - \mu \nabla^2) \left[I_0 \ddot{v}_0 - I_1 \frac{\partial \ddot{w}_b}{\partial y} - J_1 \frac{\partial \ddot{w}_s}{\partial y} \right]
$$
(21b)

$$
B_{11} \frac{\partial^3 u_0}{\partial x^3} + (B_{12} + 2B_{66}) \frac{\partial^3 u_0}{\partial x \partial y^2} + (B_{12} + 2B_{66}) \frac{\partial^3 v_0}{\partial x^2 \partial y} + B_{22} \frac{\partial^3 v_0}{\partial y^3} - D_{11} \frac{\partial^4 w_b}{\partial x^4} - 2(D_{12} + 2D_{66}) \frac{\partial^4 w_b}{\partial x^2 \partial y^2} - D_{22} \frac{\partial^4 w_b}{\partial y^4} - D_{11}^s \frac{\partial^4 w_s}{\partial x^4} - 2(D_{12}^s + 2D_{66}^s) \frac{\partial^4 w_s}{\partial x^2 \partial y^2} - D_{22}^s \frac{\partial^4 w_s}{\partial y^4} = (1 - \mu \nabla^2) [I_0(\ddot{w}_b + \ddot{w}_s) + I_1 \left(\frac{\partial \ddot{u}_0}{\partial x} + \frac{\partial \ddot{v}_0}{\partial y} \right) - I_2 \nabla^2 \ddot{w}_b - J_2 \nabla^2 \ddot{w}_s]
$$
(21c)

$$
B_{11}^{s} \frac{\partial^{3} u_{0}}{\partial x^{3}} + (B_{12}^{s} + 2B_{66}^{s}) \frac{\partial^{3} u_{0}}{\partial x \partial y^{2}} + (B_{12}^{s} + 2B_{66}^{s}) \frac{\partial^{3} v_{0}}{\partial x^{2} \partial y} + B_{22}^{s} \frac{\partial^{3} v_{0}}{\partial y^{3}} - D_{11}^{s} \frac{\partial^{4} w_{b}}{\partial x^{4}} - 2(D_{12}^{s} + 2D_{66}^{s}) \frac{\partial^{4} w_{b}}{\partial x^{2} \partial y^{2}} - D_{22}^{s} \frac{\partial^{4} w_{b}}{\partial y^{4}} - H_{11}^{s} \frac{\partial^{4} w_{s}}{\partial x^{4}} - 2(H_{12}^{s} + 2H_{66}^{s}) \frac{\partial^{4} w_{s}}{\partial x^{2} \partial y^{2}} - H_{22}^{s} \frac{\partial^{4} w_{s}}{\partial y^{4}} + A_{55}^{s} \frac{\partial^{2} w_{s}}{\partial x^{2}} + A_{44}^{s} \frac{\partial^{2} w_{s}}{\partial y^{2}} = (1 - \mu \nabla^{2}) \left[I_{0} (\ddot{w}_{b} + \ddot{w}_{s}) + J_{1} \left(\frac{\partial \ddot{u}_{0}}{\partial x} + \frac{\partial \dddot{v}_{0}}{\partial y} \right) \right] - J_{2} \nabla^{2} \ddot{w}_{b} - K_{2} \nabla^{2} \ddot{w}_{s} \right]
$$
(21d)

3 The solution of the problem

In the present part, solutions for the free vibration of FG nanoplate under four different boundary conditions are found.

The boundary conditions of a random edge are defned as follows:

Clamped (C) edge boundary conditions

$$
u_0 = v_0 = w_b = w_s = \frac{\partial w_b}{\partial x} = \frac{\partial w_s}{\partial x} = 0 \text{ at } x = 0, a \qquad (22a)
$$

$$
u_0 = v_0 = w_b = w_s = \frac{\partial w_b}{\partial y} = \frac{\partial w_s}{\partial y} = 0 \text{ at } y = 0, b \qquad (22b)
$$

Simply supported (S) edge boundary conditions

$$
N_x = v_0 = w_b = w_s = M_x = 0 \quad \text{at} \quad x = 0, a \tag{23a}
$$

$$
u_0 = N_y = w_b = w_s = M_y = 0
$$
 at $y = 0, b$ (23b)

Free (F) edge boundary conditions

$$
N_x = N_{xy} = \frac{\partial M_x}{\partial x} + 2 \frac{\partial M_{xy}}{\partial y} = Q_x = M_x = 0 \text{ at } x = 0, a
$$
\n(24a)

$$
N_{xy} = N_y + 2\frac{\partial M_{xy}}{\partial y} + \frac{\partial M_y}{\partial y} = Q_y = M_y = 0 \text{ at } y = 0, b
$$
\n(24b)

The subsequent appropriate expressions are utilized for the related boundary conditions

$$
\begin{Bmatrix} u_0 \\ v_0 \\ w_b \\ w_s \end{Bmatrix} = \begin{Bmatrix} U_{mn} \frac{\partial X_m(x)}{\partial x} Y_n(y) e^{i\omega t} \\ V_{mn} X_n(x) \frac{\partial Y_n(y)}{\partial y} e^{i\omega t} \\ W_{bmn} X_n(x) Y_n(y) e^{i\omega t} \\ W_{smn} X_n(x) Y_n(y) e^{i\omega t} \end{Bmatrix}
$$
(25)

where U_{mn} , V_{mn} , W_{bmn} , and W_{smn} identify the random parameters and $\omega = \omega_{mn}$ shows the eigenfrequency related to (m, n) th eigenmode.

The following functions for $X_m(x)$ and $Y_n(y)$ which are suggested by Reddy [[54](#page-13-4), [55\]](#page-13-5) are utilized to satisfy various boundary conditions in Eqs. (22) (22) – (24) (24) and signify the defected surface of the nanoplate approximately:

SSSS

$$
X_m(x) = \sin(\lambda x), \ \lambda = \frac{m\pi}{a}
$$

\n
$$
Y_n(y) = \sin(\beta y), \ \beta = \frac{n\pi}{b}
$$
\n(26)

CCCC

$$
X_m(x) = [\sin(\lambda x) - \sinh(\lambda x)] - \left[\frac{\sin(\lambda a) - \sinh(\lambda a)}{\cos(\lambda a) - \cosh(\lambda a)}\right]
$$

\n
$$
[\cos(\lambda x) - \cosh(\lambda x)], \ \lambda = \frac{(m + 0.5)\pi}{a}
$$

\n
$$
Y_n(y) = [\sin(\beta y) - \sinh(\beta y)] - \left[\frac{\sin(\beta b) - \sinh(\beta b)}{\cos(\beta b) - \cosh(\beta b)}\right]
$$

\n
$$
[\cos(\beta y) - \cosh(\beta y)], \ \beta = \frac{(n + 0.5)\pi}{b}
$$
 (27)

CSCS

$$
X_m(x) = [\sin(\lambda x) - \sinh(\lambda x)] - \left[\frac{\sin(\lambda a) + \sinh(\lambda a)}{\cos(\lambda a) + \cosh(\lambda a)}\right]
$$

\n
$$
[\cos(\lambda x) - \cosh(\lambda x)], \ \lambda = \frac{(m + 0.25)\pi}{a}
$$

\n
$$
Y_n(y) = [\sin(\beta y) - \sinh(\beta y)] - \left[\frac{\sin(\beta b) + \sinh(\beta b)}{\cos(\beta b) + \cosh(\beta b)}\right]
$$

\n
$$
[\cos(\beta y) - \cosh(\beta y)], \ \beta = \frac{(n + 0.25)\pi}{b}
$$
\n(28)

FCFC

$$
X_m(x) = [\sin(\lambda x) - \sinh(\lambda x)] - \left[\frac{\sin(\lambda a) - \sinh(\lambda a)}{\cos(\lambda a) - \cosh(\lambda a)}\right]
$$

\n
$$
[\cos(\lambda x) - \cosh(\lambda x)], \ \lambda = \frac{(m - 0.25)\pi}{a}
$$

\n
$$
Y_n(y) = [\sin(\beta y) - \sinh(\beta y)] - \left[\frac{\sin(\beta b) - \sinh(\beta b)}{\cos(\beta b) - \cosh(\beta b)}\right]
$$

\n
$$
[\cos(\beta y) - \cosh(\beta y)], \ \beta = \frac{(n - 0.25)\pi}{b}
$$
 (29)

Substituting the expression [\(25](#page-5-0)) in Eqs. (21) and multiplying each eigenfunction with the corresponding equation and integrating throughout the solution domain, and after some mathematical operations, the following equation is found

$$
\begin{bmatrix} a_{11} & a_{12} & a_{13} & a_{14} \\ a_{21} & a_{22} & a_{23} & a_{24} \\ a_{31} & a_{32} & a_{33} & a_{34} \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} - \omega^2 \begin{bmatrix} m_{11} & 0 & m_{13} & m_{14} \\ 0 & m_{22} & m_{23} & m_{24} \\ m_{31} & m_{32} & m_{33} & m_{34} \\ m_{41} & m_{42} & m_{43} & m_{44} \end{bmatrix} \begin{bmatrix} U_{mn} \\ V_{mn} \\ W_{pm} \\ W_{pm} \end{bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}
$$
(30)

in which

$$
a_{11} = A_{11}\alpha_{12} + A_{66}\alpha_{8}, \ a_{12} = (A_{12} + A_{66})\alpha_{8},
$$

\n
$$
a_{13} = -B_{11}\alpha_{12} - (B_{12} + 2B_{66})\alpha_{8},
$$

\n
$$
a_{14} = -(B_{12}^{s} + 2B_{66}^{s})\alpha_{8} - B_{11}^{s}\alpha_{12},
$$

\n
$$
a_{21} = (A_{12} + A_{66})\alpha_{10}, \ a_{22} = A_{22}\alpha_{4} + A_{66}\alpha_{10},
$$

\n
$$
a_{23} = -B_{22}\alpha_{4} - (B_{12} + 2B_{66})\alpha_{10},
$$

\n
$$
a_{24} = -(B_{12}^{s} + 2B_{66})\alpha_{10} - B_{22}^{s}\alpha_{4},
$$

\n
$$
a_{31} = B_{11}\alpha_{13} + (B_{12} + 2B_{66})\alpha_{11},
$$

\n
$$
a_{32} = (B_{12} + 2B_{66})\alpha_{11} + B_{22}\alpha_{5},
$$

\n
$$
a_{33} = -D_{11}\alpha_{13} - 2(D_{12} + 2D_{66})\alpha_{11} - D_{22}\alpha_{5},
$$

\n
$$
a_{34} = -D_{11}^{s}\alpha_{13} - 2(D_{12}^{s} + 2D_{66}^{s})\alpha_{11} - D_{66}^{s}\alpha_{5},
$$

\n
$$
a_{41} = B_{11}^{s}\alpha_{13} + (B_{12}^{s} + 2B_{66}^{s})\alpha_{11},
$$

\n
$$
a_{42} = (B_{12}^{s} + 2B_{66}^{s})\alpha_{11} + B_{22}^{s}\alpha_{5},
$$

\n
$$
a_{43} = -D_{11}^{s}\alpha_{13} - 2(D_{12}^{s} + 2D_{66}^{s})\alpha_{11} - D_{22}^{s}\alpha_{5},
$$

\n
$$
a_{44} =
$$

and

$$
m_{11} = -I_0 [\alpha_6 - \mu (\alpha_{12} + \alpha_8)], m_{12} = 0, m_{13} = I_1 [\alpha_6 - \mu (\alpha_{12} + \alpha_8)],
$$

\n
$$
m_{14} = J_1 [\alpha_6 - \mu (\alpha_{12} + \alpha_8)],
$$

\n
$$
m_{21} = 0, m_{22} = -I_0 [\alpha_2 - \mu (\alpha_{10} + \alpha_4)],
$$

\n
$$
m_{23} = I_1 [\alpha_2 - \mu (\alpha_{10} + \alpha_4)], m_{24} = J_1 [\alpha_2 - \mu (\alpha_{10} + \alpha_4)],
$$

\n
$$
m_{31} = -I_1 [\alpha_9 - \mu (\alpha_{13} + \alpha_{11})], m_{32} = -I_1 [(\alpha_3 - \mu (\alpha_{11} + \alpha_5))],
$$

\n
$$
m_{33} = -I_0 [\alpha_1 - \mu (\alpha_9 + \alpha_3)] + I_2 [(\alpha_9 + \alpha_3) - \mu (\alpha_{13} + \alpha_5 + 2\alpha_{11})],
$$

\n
$$
m_{34} = -I_0 [\alpha_1 - \mu (\alpha_9 + \alpha_3)] + J_2 [(\alpha_9 + \alpha_3) - \mu (\alpha_{13} + \alpha_5 + 2\alpha_{11})],
$$

\n
$$
m_{41} = -J_1 [\alpha_9 - \mu (\alpha_{13} + \alpha_{11})], m_{42} = -J_1 [\alpha_3 - \mu (\alpha_{11} + \alpha_5)],
$$

\n
$$
m_{43} = -I_0 [\alpha_1 - \mu (\alpha_9 + \alpha_3)] + J_2 [(\alpha_9 + \alpha_3) - \mu (\alpha_{13} + \alpha_5 + 2\alpha_{11})],
$$

\n
$$
m_{44} = -I_0 [\alpha_1 - \mu (\alpha_9 + \alpha_3)] + K_2 [(\alpha_9 + \alpha_3) - \mu (\alpha_{13} + \alpha_5 + 2\alpha_{11})]
$$

\n(32)

with

Table 1 The non-dimensional fundamental frequencies of FG nanoplate against the varying non-local parameter

Boundary conditions	Results	μ (nm ²)								
		$\mathbf{0}$						2		
		$k=0$	$k=1$	$k=5$	$k=0$	$k=1$	$k=5$	$k=0$	$k=1$	$k=5$
SSSS	Present	0.0930	0.0547	0.0441	0.0850	0.0500	0.0403	0.0788	0.0463	0.0373
	Zargaripoor et al. $[42]$	0.0930	0.0504	0.0444	0.0850	0.0504	0.0405	0.0788	0.0467	0.0376
	Natarajan et al. [34]			0.0441	$\overline{}$		0.0403			0.0374
CCCC	Present	0.1618	0.0953	0.0764	0.1448	0.0853	0.0684	0.1322	0.0778	0.0625
	Zargaripoor et al. $[42]$	0.1597	0.0941	0.0753	0.1436	0.0846	0.0677	0.1315	0.0774	0.0677
	Natarajan et al. [34]			0.0758	$\overline{}$		0.0682	$\overline{}$	-	0.0624
SCSC	Present	0.1261	0.0742	0.0597	0.1136	0.0668	0.0538	0.1042	0.0613	0.0493
	Zargaripoor et al. $[42]$	0.1307	0.0771	0.0618	0.1185	0.0699	0.0560	0.1092	0.0644	0.0516

Table 2 The non-dimensional frequencies of the frst four modes of FG nanoplate with SSSS edges against the varying non-local parameter and powerlaw index

Table 3 The non-dimensional frequencies of the frst four modes of FG nanoplate with CCCC edges against the varying non-local parameter and powerlaw index

\boldsymbol{k}	μ (nm ²)	SSSS				CCCC				
		Ω_1	Ω_2	Ω_3	Ω_4	Ω_1	Ω_2	Ω_{3}	Ω_4	
$\mathbf{0}$	$\mathbf{0}$	0.0930	0.2219	0.2219	0.3407	0.1619	0.3147	0.3147	0.4454	
	2	0.0788	0.1575	0.1575	0.2121	0.1323	0.2129	0.2129	0.2628	
	4	0.0695	0.1287	0.1287	0.1671	0.1146	0.1714	0.1714	0.2045	
$\mathfrak{2}$	$\mathbf{0}$	0.0488	0.1162	0.1162	0.1782	0.0848	0.1646	0.1646	0.2327	
	\overline{c}	0.0413	0.0825	0.0825	0.1109	0.0693	0.1113	0.1113	0.1373	
	4	0.0365	0.0674	0.0674	0.0874	0.0600	0.0896	0.0896	0.1068	
4	$\boldsymbol{0}$	0.0450	0.1071	0.1071	0.1640	0.0781	0.1513	0.1513	0.2137	
	\overline{c}	0.0381	0.0759	0.0759	0.1021	0.0638	0.1024	0.1024	0.1261	
	$\overline{4}$	0.0337	0.0621	0.0621	0.8043	0.0553	0.0824	0.0824	0.0981	
\boldsymbol{k}	μ (nm ²)	CSCS				FCFC				
		Ω_1	Ω_2	Ω_3	Ω_4	Ω_1	Ω_2	Ω_3	Ω_4	
$\overline{0}$	Ω	0.1261	0.2681	0.2681	0.3920	0.1625	0.3152	0.3152	0.4453	
	\overline{c}	0.1042	0.1851	0.1851	0.2373	0.1325	0.2130	0.2130	0.2628	
	4	0.0908	0.1500	0.1500	0.1857	0.1146	0.1715	0.1715	0.2045	
$\mathfrak{2}$	$\mathbf{0}$	0.0661	0.1403	0.1403	0.2049	0.0850	0.1647	0.1647	0.2324	
	2	0.0546	0.0969	0.0969	0.1240	0.0694	0.1113	0.1113	0.1372	
	4	0.0476	0.0785	0.0785	0.0971	0.0600	0.0896	0.0896	0.1068	
4	$\boldsymbol{0}$	0.0609	0.1292	0.1292	0.1884	0.0783	0.1514	0.1514	0.2135	
	2	0.0504	0.0892	0.0892	0.1141	0.0639	0.1024	0.1024	0.1261	
	4	0.0439	0.0723	0.0723	0.0893	0.0553	0.0824	0.0824	0.0981	

Table 5 The change in the non-dimensional fundamental frequency of FG rectangular nanoplates under diferent boundary conditions against the varying non-local parameter and power-law index

μ (nm ²)	SSSS			CCCC					
	F_{r1}	F_{r2}	F_{r3}	F_{r4}	F_{r1}	F_{r2}	F_{r3}	F_{r4}	
$\mathbf{0}$			1	1				1	
$\mathbf{1}$	0.8183	0.6111	0.7475	0.5799	0.7820	0.5743	0.7111	0.5447	
2	0.7094	0.4792	0.6227	0.4496	0.6636	0.4444	0.5816	0.4173	
3	0.6349	0.4071	0.5448	0.3801	0.5866	0.3754	0.5042	0.3511	
$\overline{4}$	0.5799	0.3601	0.4904	0.3353	0.5314	0.3309	0.4512	0.3088	
5	0.5370	0.3264	0.4496	0.3033	0.4893	0.2993	0.4120	0.2789	
μ (nm ²)	CSCS				FCFC				
	F_{r1}	F_{r2}	F_{r3}	F_{r4}	F_{r1}	F_{r2}	F_{r3}	F_{r4}	
$\overline{0}$				1					
$\mathbf{1}$	0.7941	0.5908	0.7252	0.5609	0.7805	0.5753	0.7103	0.5458	
2	0.6786	0.4598	0.5974	0.4321	0.6618	0.4453	0.5809	0.4183	
3	0.6022	0.3894	0.5196	0.3643	0.5847	0.3762	0.5034	0.3519	
4	0.5469	0.3438	0.4660	0.3209	0.5295	0.3318	0.4505	0.3096	
5	0.5045	0.3112	0.4262	0.2899	0.4875	0.3000	0.4114	0.2796	

Table 6 The change in the frequency ratio of FG rectangular nanoplates under diferent boundary conditions against the non-local parameter

Fig. 2 The change in the non-dimensional fundamental frequencies of FG rectangular nanoplates under diferent boundary conditions against the varying aspect ratio

$$
(\alpha_1, \alpha_3, \alpha_5) = \int_{0}^{b} \int_{0}^{a} (X_m Y_n, X_m Y_n'', X_m Y_n''') X_m Y_n \, dx \, dy \tag{33}
$$

$$
(\alpha_2, \alpha_4, \alpha_{10}) = \int_{0}^{b} \int_{0}^{a} (X_m Y_n', X_m Y_n'', X_m'' Y_n') X_m Y_n' dx dy
$$

$$
(\alpha_6, \alpha_8, \alpha_{12}) = \int_{0}^{b} \int_{0}^{a} (X_m' Y_n, X_m' Y_n'', X_m''' Y_n) X_m' Y_n dx dy
$$

$$
(\alpha_7, \alpha_9, \alpha_{11}, \alpha_{13}) = \int_{0}^{b} \int_{0}^{a} (X_m' Y_n', X_m'' Y_n, X_n' Y_n'', X_m'''' Y_n) X_m Y_n dx dy
$$

The nontrivial solution of the present problem is found equating the determinant of Eq. [\(30](#page-6-0)) to zero.

4 The numerical results and discussion

Herein numerous examples are given for validating the accuracy of the current solution procedure as well as the investigation of the natural frequencies of FG nanoplate considering four diferent cases of boundary conditions. For this aim, in Tables [1](#page-6-1), [2](#page-7-0) and [3](#page-7-1) comparison studies are given to validate the accuracy of the present work. Then, detailed analyses are performed to study the infuence of diferent parameters such as vibration mode, the aspect ratio, boundary conditions, power-law index, and non-local parameter on natural frequencies of the FG nanoplate in Tables [4,](#page-8-0) [5](#page-8-1) and [6](#page-9-0) and Fig. [2.](#page-9-1)

In all numerical analyses, the free vibration analysis is performed considering that the top surface of the plate is $Si₃N₄$ (ceramic) and the bottom surface is SUS304 (metal). Young's modulus, E , and mass density, ρ , are taken to be $E_c = 348.43 \text{ GPa}$, $\rho_c = 2370 \text{ kg/m}^3$ for Si_3N_4 and $E_m = 201.01 \text{ GPa}$, $\rho_m = 8166 \text{ kg/m}^3$ for SUS304. Besides, the non-dimensional natural frequency, Ω , and frequency ratio, F_r , are expressed as follows:

$$
\Omega_i = \omega_i h \sqrt{\frac{\rho_c}{G_c}}, \ F_{\mathbf{r}_i} = \frac{\overline{\omega}_{i_{\text{NL}}}}{\overline{\omega}_{i_{\text{L}}}}, \ i = 1, 2, 3, 4
$$

where ρ_c and G_c are the mass density and shear modulus of the ceramic constituent, respectively, $\overline{\omega}_{i_{NL}}$ and $\overline{\omega}_{i_{L}}$ are the non-local and local ($\mu = 0$) non-dimensional frequencies, respectively.

4.1 Comparison examples

Example 1 The non-dimensional fundamental natural frequencies, Ω_1 , of FG square nanoplates versus the varying non-local parameter (μ) are checked against the findings of Natarajan et al. [\[34](#page-12-11)] and Zargaripoor et al. [[42\]](#page-12-19), in Table [1.](#page-6-1) Here, fully simply supported (SSSS), fully clamped (CCCC) and SCSC edge conditions, $a/b = 1$ and $a/h = 10$, are considered.

Example 2 The non-dimensional frequencies, Ω_i , of the first four modes of FG square nanoplates versus varying nonlocal parameter (μ) and power-law exponent (k) are checked against the fndings of Natarajan et al. [[34\]](#page-12-11) and Zargaripoor et al. $[42]$ $[42]$, in Table [2](#page-7-0). Here, SSSS edge conditions $a/b = 1$ and $a/h = 10$ are considered.

Example 3 The non-dimensional frequencies, Ω_i , of the first four modes of FG square nanoplates versus varying nonlocal parameter (μ) and power-law exponent (k) are checked against the fndings of Natarajan et al. [[34\]](#page-12-11) and Zargaripoor et al. $[42]$ $[42]$, in Table [2](#page-7-0). Here, CCCC edge conditions $a/b = 1$ and $a/h = 10$ are considered.

Consequently, it is realized that the obtained results are coinciding with the existing ones.

4.2 Illustrative examples

Example 4 The influence of varying non-local parameter, μ , and power-law index, k , on non-dimensional first four natural frequencies, $Ω_i$ ($i = 1, 2, 3, 4$), of FG nanoplates under diferent boundary conditions is examined for square and rectangular plates in Tables [5](#page-8-1) and [6,](#page-9-0) respectively. Here, $a/h = 10$, $a/b = 1$, and $a/b = 2$ are considered. The findings revealed that the values of Ω_i decrease with the increment of *k* since the percentage of metal phases that are weaker than ceramic phases become more prominent. The infuence of the variation of *k* on the values of Ω_i is independent of the variation of the non-local parameter, mode number, and edge conditions. Furthermore, the values of Ω_i decrease with the increase in the μ . Note that the influence of the variation of μ on the values of Ω_i is changing according to the boundary conditions, as well as the influence of the variation of μ on the values of Ω_i increases with the increment of the number of the mode, while it is independent of the variation of the *k*.

Example 5 The influence of varying non-local parameter, μ , on the frequency ratio, F_{ri} ($i = 1, 2, 3, 4$), of FG rectangular nanoplates versus four distinct boundary conditions is examined in Table [6](#page-9-0). Here, $k = 2$, $a/h = 10$, and $a/b = 2$ are considered. The results revealed that the values of F_{ri} decrease with the increment of μ . Note that the influence of the variation of μ on the values of is F_{ri} which is changing according to the boundary conditions, as well as the infuence of the variation of μ on the values of F_{ri} frequency ratio of FG nanoplates changes with the number of the mode.

Example 6 The infuence of varying aspect ratio, *a*∕*b* and non-local parameter, μ (nm^2), on the non-dimensional fundamental frequencies, Ω_1 , of FG rectangular nanoplates versus four distinct boundary conditions is examined in Fig. [2.](#page-9-1) Here, $k = 5$, $a/h = 10$ are considered. The results indicated that the values of Ω_1 increase with the increase in *a*/*b*. Note that the influence of the variation of a/b on the values of Ω_1 is changing according to the boundary conditions, as well as the influence of the variation of a/b on the values of Ω_1 decreases with the increment of μ .

5 Conclusions

In the present study, an analytical solution for the free vibration of nanoplates made of FGMs under four diferent boundary conditions was provided. For this aim, a new refned plate theory with four variables based on the theory of non-local elasticity including the small-scale infuence. Here, the number of unknowns and governing equations of the present model were reduced to four separating the vertical displacement into shear and bending components, and so the number of unknowns has become less than the other alternative theories. The infuence of the diferent parameters such as vibration mode, the aspect ratio, boundary conditions, power-law index, and non-local parameter on the natural frequencies of the FG nanoplate is discussed, in detail.

In sum, the following results were obtained:

- The non-dimensional natural frequencies reduce with the increment of the power-law index
- The non-dimensional natural frequencies reduce with the increment of the non-local parameter
- The influence of variation of the non-local parameter on non-dimensional natural frequencies increases with the increment of the mode number
- The frequency ratio reduces with the increment of nonlocal parameter
- The influence of non-local parameter on the frequency ratio changes with mode number
- The values of non-dimensional fundamental natural frequency increase with the increase in aspect ratio.
- The influence of aspect ratio on the non-dimensional fundamental natural frequency decreases with the increment of the non-local parameter
- All said infuence on non-dimensional natural frequencies changes depending on boundary conditions

Consequently, it is decided that considered factors have key infuences on the natural frequencies of FG nanoplates. Furthermore, the proposed exact solution method not only satisfactorily handled the present problem and yielded successful results but also it supplied ease for the non-local vibration analysis of FG nanoplates. In the future studies, the presented solution procedure will be extended for mechanical behaviors of other types of structures composed of varied materials with macro-/micro-dimensions.

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