

# A Review of Modeling Thermal Displacement Processes in Porous Media

Abiola David Obembe<sup>1</sup> · Sidqi A. Abu-Khamsin<sup>1</sup> · M. Enamul Hossain<sup>1</sup>

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**Abstract** The subject of heat transfer in oil reservoirs has gained huge attention, due to its diverse range of applications in petroleum reservoir management and thermal recovery for enhanced oil recovery. Thermal recovery methods entail the addition of heat energy into the reservoir through injection wells with the aim of reducing the in situ oil viscosity which is usually around several thousand centipoise cP (in S.I unit kg/ms) at reservoir conditions to very low values at steam temperatures. In addition, several other mechanisms are associated with thermal recovery methods. These include thermal expansion of oil, steam distillation, and relative permeability changes, which contribute to the ultimate recovery of the reservoir. In this article, a detailed review of non-isothermal modeling in an oil reservoir is presented. In addition, a few remarks regarding the momentum transport and the energy balance equations and its various modifications through the years are provided. Finally, a memory-based formulation is proposed to capture the alteration of rock and fluid properties with time as well as accounting for other phenomena not described by classic diffusion equations.

**Keywords** Heat transfer · Reservoir management · Enhanced oil recovery · Memory-based

## List of symbols

$a_{sf}$  Specific surface area (fluid to solid contact) ( $m^2$ )

✉ M. Enamul Hossain  
 menamul@kfupm.edu.sa; dr.mehossain@gmail.com

<sup>1</sup> Department of Petroleum Engineering, College of Petroleum Engineering and Geosciences, King Fahd University of Petroleum and Minerals, KFUPM Box 2020, Dhahran 31261, Saudi Arabia

$A(t)$	Cumulative heated area ( $m^2$ )
$c_F$	Non-dimensional form-drag constant
$c_p$	Specific heat (J/kg K)
$C$	Component
$C_c$	Coke concentration ( $gmol/m^3$ )
$d_p$	Spherical particle diameter (m)
$Da$	Darcy number, $\kappa/L^2$ , dimensionless
$E_H$	Heating efficiency, percentage
$g$	Acceleration due to gravity ( $m/s^2$ )
$h$	Pay thickness (m)
$h_{sf}$	Fluid to solid heat transfer coefficient ( $W/m^2 K$ )
$H$	Aquifer height (m)
$H_o$	Heat injection rate (J/s)
$k$	Thermal conductivity (W/m K)
$K$	Permeability ( $m^2$ )
$P$	Pressure (Pa)
$q$	Heat flux ( $W/m^2$ )
$Q(t)$	Heat stored in the pay zone (J)
$Ra$	Rayleigh number, dimensionless
$Re_{dp}$	Reynolds number based on particle diameter, $\rho u \frac{d_p}{\mu}$ dimensionless
$Re_{\kappa}$	Reynolds number based on permeability, $\rho \sqrt{\kappa} \frac{u}{\mu}$ , dimensionless
$r$	Radial distance (m)
$R$	Thermal retardation factor
$S$	Saturation, fraction
$t$	Time (s)
$T$	Temperature (K)
$\vec{u}$	Velocity vector (m/s)
$U_{hz}(x, y, z, t)$	Heat flux in the vertical direction (J/s)
$v$	Heat velocity (m/s)
$x_D$	Dimensionless distance
$z$	Vertical distance (m)

## Greek alphabets

$\alpha$	Thermal diffusivity ( $\text{m}^2/\text{s}$ )
$\alpha_L$	Longitudinal dispersivity (m)
$\alpha_t$	Transverse dispersivity (m)
$\alpha'$	Overburden thermal diffusivity ( $\text{m}^2/\text{s}$ )
$\gamma$	Fractional-order derivative
$\eta$	Pseudo-diffusivity ( $\text{m}^3 \text{s}^{2-\gamma}/\text{kg}$ )
$\mu$	Dynamic viscosity (kg/m s)
$\kappa$	Thermal dispersion tensor (W/m K)
$\nu$	Kinematic viscosity ( $\text{m}^2/\text{s}$ )
$\rho$	Fluid density ( $\text{kg}/\text{m}^3$ )
$\sigma_r$	Mean square variance
$\tau$	Dimensionless time
$\phi$	Porosity, fraction
$\Gamma$	Standard gamma function
$\Phi$	Fluid potential (Pa)

## Subscripts

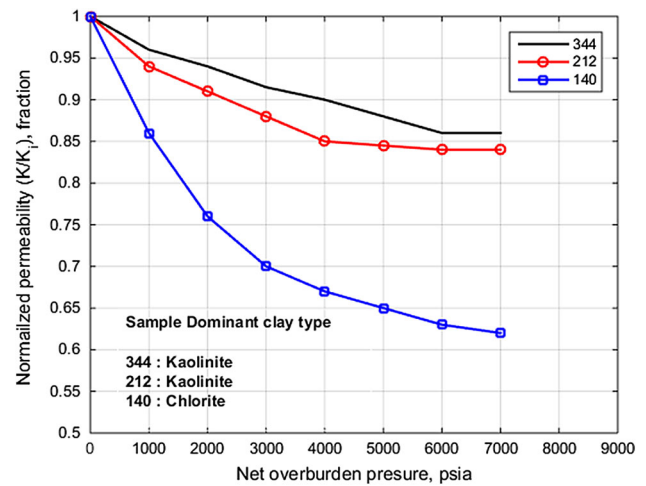
c	Coke
e	Effective
f	Fluid
g	Gas
o	Oil
p	Pore
r	Reservoir
s	Rock solid matrix
sf	Solid-to-fluid interface
w	Water
0	Initial
1	Reservoir

## 1 Introduction

Processes that involve the injection of a hot fluid into a cooler, fluid-saturated, subsurface rock or vice versa are well established and include thermal EOR, detection of water influx in production wells [1], groundwater transport, contaminant transport, heat scavenging, water re-injection in subsurface aquifers, and in geothermal reservoirs applications. Accurate prediction of the performance of any such process requires a model that addresses all heat transfer characteristics of the rock-to-fluid system. However, such mathematical models could be very complex to handle analytically.

Numerous formulations have been presented to model these thermal displacement processes, each focusing on one or more aspects of the problem and adopting different assumptions.

The classical equation describing the flow of single phase or multiple phases through an oil reservoir has been expressed



**Fig. 1** Effect of net confining stress on permeability. (Data source: Ref. [24])

in several publications [2–5] which are based on Darcy’s law. This empirical equation was developed based on the assumption of homogeneous and isotropic rock. However, different modifications have been proposed to improve the accuracy of Darcy equation. Each modification accounts for the effect of different observed phenomena, i.e., convective acceleration within the porous media [6], slip, desorption, and non-Darcy flow. Now you might ask are such modifications always necessary?

In order to macroscopically describe the flow through an oil reservoir, it is necessary to introduce variables that take into account the space left by the solid matrix to the fluid. One of them is the porosity represented either in percentage or in fraction, it is defined as the ratio of the pore volume to the bulk volume of the rock. Second and more important is the rock permeability (with a dimension of  $L^2$ ), described as a measure of the ability of the rock to transmit a fluid. The above rock properties have been established to be dependent on grain shape and size distribution, confining pressure, temperature (due to hot water injection or steam injection), stress, and reservoir process [7–35]. Due to the unconsolidated nature of heavy oil reservoirs, thermal operations usually result in particle mobilization [35]. The subsequent particle migration leads to pore throat plugging which is one of the root cause of permeability reduction. Figure 1 presents the results observed by Amaefule et al. [24], showing the permeability reduction observed on three core samples with increasing confining pressure.

In this survey, key issues in the literature have been grouped into categories where each category addresses one aspect of the problem pointing out the strengths and weaknesses associated with them. The main goal of this exposition into non-isothermal modeling in an oil reservoir is to enlighten researchers and scientists alike on an alternative means of addressing the effect of thermal alter-

ations in rock properties which may not be known priori in a more efficient manner. Secondly, the incorporation of the concept of anomalous diffusion into mathematical models may provide a more convenient alternative to handling reservoir heterogeneity (natural fractures). It is our opinion that current mathematical formulations or models can be improved through the incorporation of more generalized constitutive equations due to the nature of the rock fabric i.e. equations relating the volumetric flux to pressure in oil reservoirs, and/or equations relating the conduction heat flux to the temperature.

## 2 Momentum Transport

There exist several momentum equations proposed to describe fluid flow in porous media, some of which were developed to match empirical observations and interestingly converge to corresponding free-fluid model (Navier–Stokes) when the porosity approaches 100% and permeability infinity. Accurate modeling of the momentum equation is of the utmost importance as improper velocity distribution introduces error to the temperature distribution through the convective energy flux no matter what sophisticated scheme or algorithm is used to handle the corresponding heat equation. The oldest and most used momentum equation is the Darcy flow model. This model is a form of linear momentum equation, which states that the volumetrically averaged velocity is directly related to the fluid potential gradient in that direction as presented in Eq. 1.

$$\vec{u} = \frac{K}{\mu} [G - \nabla P] = \frac{K}{\mu} \nabla \Phi \quad (1)$$

where  $G$  is the body force term due to gravity.

$$G = \rho g. \quad (2)$$

However, in the derivation of Darcy flow model, several simplifying assumptions were made concerning the porous medium and the nature or properties of the flowing fluid(s). It has been established [36], starting from the Navier–Stokes equation, that Darcy law is restricted to flows in which the viscous forces dominate over the inertia forces.

The Darcy flow model as described earlier makes the momentum equation linear, hence the resulting simplicity in solving the diffusivity equation. Amhalhel and Furmański [37] established that the Darcy equation is of one order less than the Navier–Stokes equation and that the no-slip hydrodynamic boundary condition cannot be applied. Furthermore, the maximum velocity is predicted at the impermeable surface. However, if any of the simplifying assumptions, for example, the porous medium, is heterogeneous, non-isothermal conditions prevail, or the fluid interacts either chemically with the rock surface. Darcy law in its simplest

form cannot be used to model fluid flow in such systems. There have been other fluid flow models proposed in the petroleum engineering literature; this includes Brinkman–Darcy model [38], Forchheimer–Darcy model [39], Darcy–Brinkman–Forchheimer model [40], and Hsu and Cheng generalized flow model [41], which were derived using some volume average technique from the Navier–Stokes equation see references [6,36,42–45] for more description. Choi et al. [46] studied the influence of inertia and viscous terms on velocity profile. Their results show that viscous forces contribute mostly to the deviation from Darcy flow model.

Fortunately, in many oil reservoirs and aquifers, the Reynolds number based on permeability ( $Re_K$ ) is  $\ll 1$ . In such flow conditions, Darcy equation has been established to be appropriate to describe the macroscopic fluid motion [47]. However, Darcy equation in its simplistic form does not account for evolution or variation in rock and fluid properties with time; therefore, it still has to be modified in some way. On the other hand, Darcy equation is not recommended when describing fluid flow in shale reservoirs, naturally fractured Karst reservoirs, artificially created porous media, worm-hole modeling in reservoir rocks, and nanomaterials. This is because the Reynolds number in such porous media is greater than unity.

A summary of some of the widely employed constitutive equations in porous media modeling applications is listed in Table 1. Please refer to nomenclature for the definition of terms.

## 3 Energy Transport

The importance of proper understanding the possible heat transfer modes within any porous media is of great significance for proper prediction of temperature distribution. Two different macroscopic descriptions are available in the literature namely the heterogeneous and homogeneous models.

### 3.1 Heterogeneous Formulation

This formulation considers the oil reservoir by two coexisting temperature fields, i.e., the solid and fluid phases. In the presence of two temperature fields, there is an additional heat exchange between both phases, i.e., no local thermal equilibrium (NLTE). This approach is key to accurate modeling of highly transient problems and few steady-state problems as pointed out by Nield [48]. The key to the accurate formulation of heterogeneous models lies in the determination of representative heat transfer coefficient ( $h_{sf}$ ) between both phases [49]. The literature is littered with experimental studies, see [50], and theoretical investigations, see [51–53], on estimating accurate and representative estimates of the heat transfer coefficient for different porous media. According

**Table 1** Constitutive equations describing fluid flow in a porous medium

Flow model and year	Equation	Key facts
Darcy law [45]	$\vec{u} = \frac{K}{\mu} [\langle G \rangle - \nabla \{P\}]$	<ol style="list-style-type: none"> <li>1. Appropriate when permeability-based Reynolds number is less than one</li> <li>2. Its main limitation is that the no-slip boundary condition cannot be imposed</li> </ol>
Forchheimer–Darcy model [39]	$\nabla P - \langle G \rangle = -\frac{\mu}{K} \vec{u} - \rho \frac{c_F \phi}{\sqrt{K}} \vec{u}  \vec{u} $	<ol style="list-style-type: none"> <li>1. Proposed a velocity square term addition to the Darcy term to account for the inertia effects in the pressure drop</li> <li>2. Appropriate for very high flow velocities in porous media</li> </ol>
Brinkman–Darcy model [38]	$\nabla P - \langle G \rangle = -\frac{\mu}{K} \vec{u} + \frac{\mu_e}{\phi} \nabla^2 \vec{u}$	<ol style="list-style-type: none"> <li>1. Derived for an assembly of spheres</li> <li>2. Proposed to account for transitional flow between boundaries, i.e., boundary layer flow</li> <li>3. Introduced the effective viscosity term and the Laplacian of velocity to account for the viscous effects which become significant as the porosity and permeability of the porous media becomes larger</li> <li>4. Due to the method of its derivation, it has been reported that it is only applicable to porous media with porosity values greater than 0.6</li> <li>5. Ambiguity in the effective viscosity term, some researchers concluded that the term depends on the porous media geometry</li> </ol>
Darcy–Brinkman–Forchheimer [40]	$\frac{\rho}{\phi} \left[ \frac{\partial \vec{u}}{\partial t} + \frac{(\vec{u} \cdot \nabla) \vec{u}}{\phi} \right] = \langle G \rangle - \nabla P + \mu_e \nabla^2 \vec{u} - \frac{\mu}{K} \vec{u} - \rho \frac{c_F}{\sqrt{K}} \vec{u}  \vec{u} $	<ol style="list-style-type: none"> <li>1. Difficult to solve numerically</li> <li>2. The convective term contributes to the inertia effects experienced in a porous media</li> <li>3. The presence of the convective term is important to high velocity and/or high porosity media. However, its role is not as clear as that of the Forchheimer inertia term and can be best understood as to that of the corresponding free-fluid flow</li> </ol>
Hsu and Cheng [41]	$\frac{\rho}{\phi} \left[ \frac{\partial \vec{u}}{\partial t} + \frac{(\vec{u} \cdot \nabla) \vec{u}}{\phi} \right] = \langle G \rangle - \nabla P + \mu \nabla^2 \vec{u} + B$ <p>Where</p> $B = - \left[ \frac{\mu}{K} \vec{u} + \rho \frac{c_F}{\sqrt{K}} \vec{u}  \vec{u}  \right]$ <p>B is the total drag force per unit volume (body force) due to the presence of the solid particles</p>	<ol style="list-style-type: none"> <li>1. Derived starting from starting the Navier–Stokes equations and utilizing the volume averaging</li> <li>2. As the porosity approaches unity and the permeability of the porous media approaches infinity, the equation reduces to the classical Navier–Stokes equation</li> <li>3. Difficulty to solve numerically</li> </ol>
Generalized model (2008)	$\rho \left[ \frac{\partial \vec{u}}{\partial t} + \nabla \left( \frac{\vec{u} \cdot \vec{u}}{\phi} \right) \right] = -\nabla \phi P + \mu_e \nabla^2 \vec{u} - \left[ \frac{\mu \phi}{K} \vec{u} - \rho \frac{c_F \phi}{\sqrt{K}} \vec{u}  \vec{u}  \right] + \langle G \rangle$	Similar to above (Hsu and Cheng model)

to Wakao et al. [54] for the heterogeneous description, the energy balance falls into three classes as presented below. *Schumann Model* This model neglects the heat conduction in both phases in the governing energy balance equations. *Continuous Solid Phase (C-S) Model* This model accounts for thermal conduction in both phases. In addition, the effec-

tive thermal conductivity is introduced, which includes the thermal dispersion effect.

*Dispersion–Concentric (D-C) Model* This model also uses one equation based on the average fluid temperature, which is coupled to the energy equation for the heat conduction in a single particle [42]. For more information, refer to the

**Table 2** Heterogeneous energy balance models

Model	Fluid phase	Solid phase
Schumann model	$\phi (\rho c_p)_f \frac{\partial T_f}{\partial t} + \phi (\rho c_p)_f \vec{u} \cdot \nabla T_f = h_{sf} a_{sf} [T_s - T_f]$	$(1 - \phi) (\rho c_p)_s \frac{\partial T_s}{\partial t} = h_{sf} a_{sf} [T_s - T_f]$
Continuous solid phase (C-S) model	$\phi (\rho c_p)_f \frac{\partial T_f}{\partial t} + \phi (\rho c_p)_f \vec{u} \cdot \nabla T_f = \nabla \cdot (k_e \cdot \nabla T_f) + h_{sf} a_{sf} [T_s - T_f]$	$\nabla \cdot (k_{es} \cdot \nabla T_s) + h_{sf} a_{sf} [T_s - T_f] = (1 - \phi) (\rho c_p)_s \frac{\partial T_s}{\partial t}$
Dispersion–concentric (D-C) model	$\frac{\partial T_f}{\partial t} + \vec{u} \cdot \nabla T_f = \alpha'_{ax} \nabla^2 T_f + \frac{h_{sf} a_{sf}}{\phi (\rho c_p)_f} [T_s - T_f]$	$\frac{\partial T_s}{\partial t} = \alpha'_s \left( \frac{\partial^2 T_s}{\partial r^2} + \frac{2}{r} \frac{\partial T_s}{\partial r} \right)$

manuscript of Wakao et al. [54]. For a more thorough review on the NLTE, readers should refer to the following references [55–58]. However, the heterogeneous energy balance models are unpopular in petroleum engineering literature and never really applied in practice. Recently, Hossain and Abu-Khamsin [59] employed the Schumann model to describe hot water injection process in an oil reservoir. Table 2 provides a summary of the above-described heterogeneous energy balance models.

### 3.2 Homogeneous Formulation

This formulation neglects the heat transfer between the fluid and solid phases. Hence, only a single temperature exists at any point in the porous media. A condition referred to in the literature as local thermal equilibrium (LTE), where only a single energy equation is required to describe the oil reservoir by two coexisting temperature fields, i.e., the solid and fluid phases [60].

The assumption of LTE was investigated by Wong and Dybbs [61]. They concluded that LTE holds for flow rates where the pore diameter-based Reynolds number,  $Re_p$  (refer to nomenclature for definition), is smaller than ten. It has also been reported in the literature that the Darcy number,  $Da$  (refer to nomenclature for definition), has the most influence in determining the validity of the LTE [62]. Therefore, it has been concluded that the LTE is only applicable for very small values of particle Reynolds number and the Darcy number. Vasdaz [63–65], pointed out that the LTE applies generally applied for boundary conditions which are a mixture of Dirichlet and insulation type. The complete form of the above-mentioned homogeneous energy equation is presented in Eq. 3.

$$\left[ \phi + (1 - \phi) \frac{(\rho c_p)_s}{(\rho c_p)_f} \right] \frac{\partial T}{\partial t} + \vec{u} \cdot \nabla T = \nabla \cdot (k_e \cdot \nabla T) \quad (3)$$

### 3.3 Heat Transfer Mechanisms

The major difference between the mechanism of heat transfer in an oil reservoir and a solid body stems from the inherent nature of porous materials. From the literature

[66–72], heat can be transferred in fluid-saturated porous media, by a combination of different mechanisms, namely heat convection, hydrodynamic/mechanical dispersion, radiation, thermal conduction, and forced convection.

Thermal conduction involves the transfer of heat from the porous media to the impermeable confining layers (overburden and under-burden) and also within the porous media. Its importance is dependent on the magnitude of thermal conductivities of the rock and fluid.

Convective heat transfer differs from the heat transfer due to forced convection. This accounts for the heat transferred between the injected fluid, the original reservoir fluids, and the solid material [73]. However, the low velocities encountered in most oil reservoirs justify the use of the LTE model. Generally, temperature equilibrium is attained under 1 s for 1 mm diameter grains, and in 1 min for 1 cm and in 2 h for 10 cm [74].

Hydrodynamic/mechanical dispersion results from velocity variations, which arise from the velocity profile in a single pore, the velocity differences between different pores in the porous media and the tortuosity. For instance, Fig. 2 shows two fluid parcels starting near each other at locations B and C dispersed to locations farther apart  $B'$  and  $C'$  during transport in the pore space.

The overall diffusion coefficients in the longitudinal and transverse directions are defined by:

$$\kappa_L = k + \alpha_L \rho c_p u_L \quad (4)$$

and

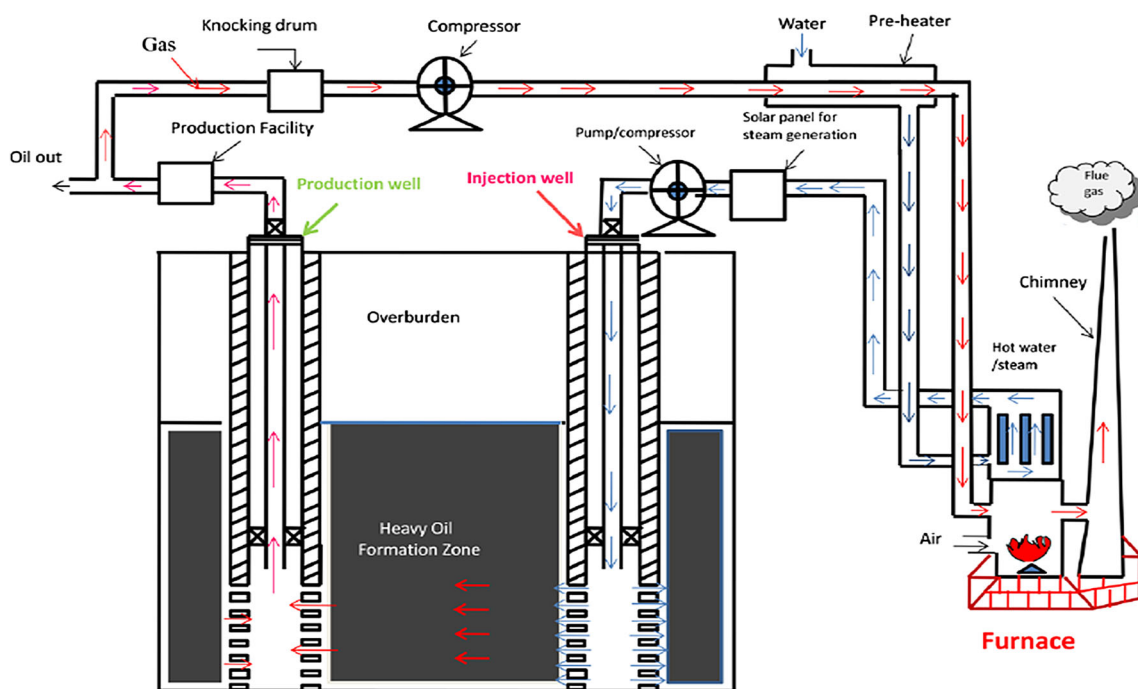
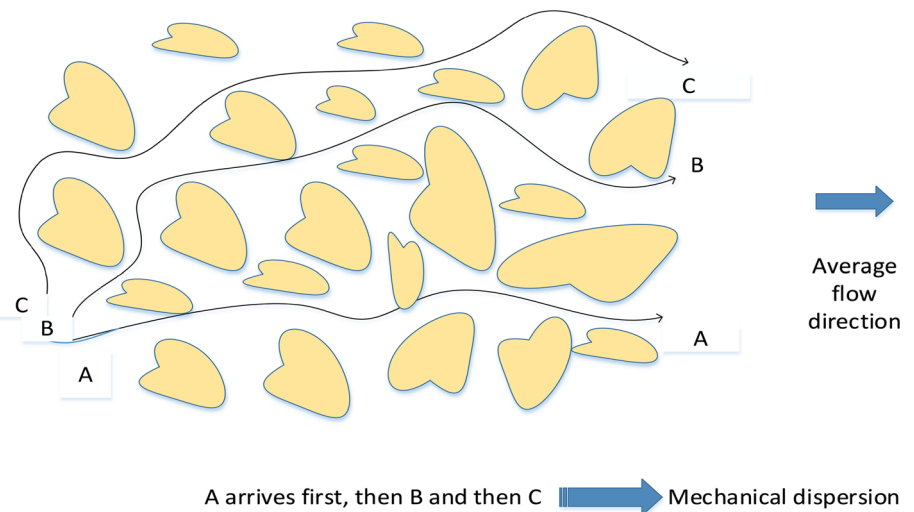
$$\kappa_t = k + \alpha_t \rho c_p u_t \quad (5)$$

Dispersion is usually several orders of magnitude lower than heat conduction leading to its effect neglected in many heat transfer models [75]. For a detailed overview on thermal dispersion, readers should refer to the following references [44, 76–82].

Radiation can be described through the electromagnetic wave theory, and it is independent of temperature and the thermodynamic properties of the medium. Its effect is usually neglected in many heat transfer models due to difficulty in its quantification at a given point in the medium [83].



**Fig. 2** Mechanical dispersion in ground water transport (Redrawn from Ref. [245])



**Fig. 3** Mechanism of oil recovery scheme using injection and production wells in an oil field reservoir. (Source: Ref. [200])

#### 4 Steam Flooding

The observed reduction in oil viscosity  $\mu_o$  during the injection of heat energy is key to effective recovery in heavy oil reservoirs. Similarly, there is an observed reduction in the viscosity of water  $\mu_w$  but to a lesser degree. However, it has been acknowledged that the benefit of increased temperature is improving the water-to-oil mobility ratio [84–87]. The most successful and widely used process for heating a reservoir is the steam injection.

Steam injection applications in heavy oil reservoirs date back to the 1960s. The most common application of steam injection is steam flooding, also referred to as steam drive

or steam displacement. The process in simple term describes the continuous injection of steam through injection well(s) with the aim of displacing original reservoir fluids toward the production wells as shown in Fig. 3. In a perfect scenario, a steam-saturated zone is formed around the injection well, with a temperature almost equal to that of the injected steam.

Predicting reservoir response during a steam flood is very important for reservoir engineering applications, and proper reservoir management and evaluation require tools or models, which accurately predict steam-flood parameters, for example the oil-to-steam ratios (OSR). A thorough review of some of the published steam-drive models indicates that the predictive capability of these models is inadequate and that

improvements are needed for improved evaluation of steam-drive projects [59,88]. The available prediction techniques can be classified into three groups, namely empirical correlations, analytical models, multi-component, and multiphase numerical simulations.

Four different zones have been observed between the steam injection well and the producer, each with its own pressure, temperature, and saturation [89].

#### 4.1 Empirical and Analytical Steam-Flood Models

In general, numerous analytic models have been presented to predict the temperature distribution in an oil reservoir. The earlier equations were derived based on pure convective type flow in linear and radial reservoirs, for example equations presented by Lauwerier [90], Marx and Langenheim [91], Ramey [92], Malofeev and Scheinman [93], Rubinshtein [94], Mandl and Volek [95], and Avdonin [96,97]. These authors presented analytical solutions to describe the temperature distribution, thermal invasion rate, heat injection ( $H_o$ ) rate required to raise the temperature to another temperature, heating efficiency ( $E_H$ ), cumulative heated area as a function of time ( $A$ ), theoretical economic limits for sustained hot fluid injection, to describe the injection of hot water into an oil-bearing layer. Each model is an improvement over the other by accounting for practical effects. Take for instance, radial heat conduction both within and outside the reservoir, vertical conduction within the reservoir, variable heat injection rate, no restriction on the direction of development of the heated area [92], and finite longitudinal and transverse conductivity [96,97]. Following the approach by Lauwerier [90], the temperature distribution in an oil layer can be described by Eq. 6. Refer to nomenclature for variables introduced.

$$T = T_i \operatorname{erfc} \left( \frac{\xi + |\eta| - 1}{2\sqrt{\theta}(\tau - \xi)} \right) \alpha(\tau - \xi) \tag{6}$$

where

$$\eta = \begin{cases} \frac{y}{b} & \text{for } |y| > b \\ 1 & \text{for } |y| < b \end{cases} \tag{7}$$

$$\alpha(\tau - \xi) = \begin{cases} 1 & \text{for } \tau \geq \xi \\ 0 & \text{for } \tau < \xi \end{cases} \tag{8}$$

The above quantities are defined as:

$$\xi = \frac{k_2}{b^2 c_p \rho_1 u} x, \quad \theta = \frac{\rho_1 c_{p1}}{\rho_2 c_{p2}}, \quad \tau = \frac{k_2}{b^2 \rho_1 c_{p1}} t \tag{9}$$

$x$  = distance in flow direction, m  
 $b$  = half the formation thickness, m  
 $\operatorname{erfc}$  = complementary error function

Marx and Langenheim [91] were able to predict the cumulative heated area within the oil layer when subjected to heat using Eq. 10. Refer to nomenclature for variables introduced.

$$A(t) = \left[ \frac{H_o M h \alpha'_2}{4k^2 (T_{inj} - T_0)} \right] \left[ e^{x^2} \operatorname{erfc} x + \frac{2x}{\sqrt{\pi}} - 1 \right] \tag{10}$$

where

$$x = \left( \frac{2k}{M h \alpha'_2} \right) t^{\frac{1}{2}}, \tag{11}$$

$$M = [(1 - \phi) \rho_r c_{pr} + S_w \phi \rho_w c_{pw} + S_o \phi \rho_o c_{po}]$$

Later, Ramey [92] extended the work of Marx and Langenheim [91] by considering the case of variable heat injection rate. He proposed that the cumulative heated area could be predicted with Eq. 12.

$$A(t) = \left( \frac{H_o(t)}{2\rho_1 c_1 b \Delta T} \right) * \left( e^{x^2} \operatorname{erfc} x \right) \tag{12}$$

The term (\*) refers to the convolution of two functions. Refer to nomenclature for variables presented.

Rubinshtein [94] derived an equation for predicting the heating efficiency ( $E_H$ ) in the oil layer in between the overburden and underburden formation using Eq. 13.

$$E_H = 1 - (1 - \beta) \times \left\{ \frac{\sqrt{\gamma\tau}}{\pi} \left[ 1 - (1 - \beta) \sum_{n=1}^m \beta^{n-1} \left( 1 + \frac{n^2}{\gamma\tau} \right) e^{-n^2/\gamma\tau} \right] + (1 - \beta) \sum_{n=1}^{\infty} n \beta^{n-1} \left( 1 + \frac{2n^2}{3\gamma\tau} \right) \operatorname{erfc} \frac{n}{\sqrt{\gamma\tau}} \right\} \tag{13}$$

where

$$\beta = \frac{\gamma a - 1}{\gamma a + 1}, \quad \gamma = \frac{k_1}{k_2}, \quad \text{and} \quad a^2 = \frac{k_2 \rho_1 c_1}{k_1 \rho_2 c_2} \tag{14}$$

Prats [98] investigated the thermal efficiency of thermal recovery processes in oil reservoirs using the same method as Marx and Langenheim [91]. However, he introduced far greater generality. The heat stored  $Q(t)$  in the formation was said to be divided into two parts: the heat in the pay zone near the injection well and the heat in the pay zone far from the injection well. Furthermore, he presented and solved an energy balance using the Laplace transform to obtain an estimate of the heat stored in the pay zone. The presented heat balance equation is of the form:

$$Q(t) = \frac{dH(t)}{dt} + 2 \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} U_{hz}(x, y, 0, t) dx dy \tag{15}$$

where  $z = 0$  is the interface plane between the pay zone and the adjacent zone. Refer to nomenclature for other terms. Furthermore, he solved Eq. 15 using the Laplace transform to get the heat stored in the pay zone as:

$$H(t) = \int_0^t Q(t') K(\theta_2 \sqrt{t-t'}) dt' - F \int_0^t H_o(t') dt K(\theta_2 \sqrt{t-t'}) dt' \quad (16)$$

The dimensionless parameters presented above can be expressed as follows:

$$K(z) = e^{z^2} \operatorname{erfc} z \quad (17)$$

$$\theta_2 = \frac{\lambda_{h2z}}{b \sqrt{\alpha_2} (\rho c)_1} \quad (18)$$

$$F = \frac{(\rho c)_1 - (\rho c)_2}{(\rho c)_1} \quad (19)$$

Davies and Silberberg [99] proposed a performance prediction model for five-spot steam floods based on the contributions of Marx and Langenheim [91] and Buckley and Leverett [100]. For each radial segment, they obtained the position of the steam front (through Buckley–Leverett solution to the material balance equation) and the temperature distribution ahead of the steam front by heat balance equations. They also estimated the amount of heat loss to the surrounding formations.

Ali [101] proposed a steam-flood model that incorporated the effect of relative permeability to predict the displaced oil from the moving steam chamber. In addition, he presented some post-breakthrough calculations; however, he neglected the effects of the heat lost through produced fluids.

Willman et al. [102] presented an analytical solution to the position of the steam front (steam zone advance) during a steam flood in a radial homogeneous reservoir. The solution can also be extended to include a variable rate by a superposition method. In addition, they suggested a calculative procedure using the temperature gradient to model the displacement in the hot liquid region moving ahead of the steam front.

Mandl and Volek [95] introduced for the first time the “critical time” to account for the heat transfer ahead of the steam front. They pointed out that after the critical time, it was paramount to account for convective heat transfer ahead of the steam front. The Myhill and Stegemeier [103] model combines some aspects of theories by Marx and Langenheim [91] and Ali (1982) to predict the volume of the steam zone and oil steam ratio. Using the principles of segregated flow, Van Lookeren [104] proposed a technique for predicting the geometry of the growing of steam envelope (zone) during steam injection applications into an oil reservoir (linear and radial steam-flood models). The author accounted for

the effect of the steam override. In addition, the model was in agreement with results obtained from scaled laboratory experiments, steam injection projects that were available, and calculations from a numerical simulation. He observed that the steam injection rate, pressure, and effective formation permeability to steam played an important role in determining the shape of the steam envelope (zone).

Jones [105] proposed model is an extension of the model proposed by Myhill and Stegemeier (1987) through the introduction of the capture efficiency. The model converts the oil displacement rate obtained from the results of Myhill and Stegemeier (1987) steam flood to the corresponding actual oil production rate using his correlation obtained from the results of 14 different steam-flood projects. He assumed that any steam flood consists of three production stages: The first stage is controlled by initial oil viscosity, the second stage is controlled by hot oil mobility and reservoir permeability, and the final stage is dominated by the remaining mobile fraction of original oil in place.

Jensen et al. [106] presented an improved steam-drive model over models proposed by Myhill and Stegemeier [103], van Lookeren [104]. The proposed model is based on reservoir information and operating conditions from various field-scale steam-drive projects. The new model employed the use of correlations to predict steam-flood process parameters both before and after steam breakthrough. Most importantly, the proposed model showed greater accuracy over existing models when compared with some 15 field-scale steam-drive projects.

Interestingly, Bødvarsson [107] was the first researcher to investigate the propagation of the thermal front in a porous media fully saturated with a single fluid (single-phase fluid flow). He assumed the heat transferred through thermal conduction was very small compared to that by thermal convection. This led to a simplified energy balance, from which he derived analytically an equation for the position of the thermal front. Furthermore, he showed that the temperature front always lagged the fluid front by a constant related to the ratio of the heat capacity of rock and fluid. Some years later, Bødvarsson and Tsang [68] proposed an analytical model that predicted the rate at which the thermal front advanced during the injection of cooler water into a fractured geothermal reservoir. They assumed the geothermal system was comprised of equal-spaced, horizontal fractures, each intersecting the injection well. Furthermore, it has been established that for many practical scenarios, the effect of thermal conductivity was negligible for heat transport in homogeneous geothermal porous media [108].

An analytical solution was derived by Ziagos and Blackwell [109] to predict the temperature in an underground thin aquifer. The authors in the proposed equation took into account the heat transferred through conduction into both overburden and under-burden layers following the injec-



tion of a hot fluid using Fourier transform technique. They assumed that the aquifer was of infinite size in the horizontal direction.

Chen and Sylvester [110] presented an evaluation of three existing analytical steam-flood models: Jones [105], Ali [101], and Miller and Leung [111]. For each model presented, they considered the oil recovery mechanism(s), the associated predictive capability and comparison with field data. In addition, each of the presented models was improved to help its ability to predict production rate and/or history match for typical field production data.

Chandra and Mamora [112] modified Jones' model due to its inadequacy in properly predicting the peak oil production. The predicted peak oil production was always smaller than that observed in the field. This modification was achieved through results obtained from the simulated performance of a 5-spot steam-flood pattern. Furthermore, the improved model was found to give satisfactory production performance up to 20 years for the simulated steam flood.

Shook [113] investigated the effect of thermal conductivity for flow in heterogeneous media and concluded that neglecting the effect of thermal conductivity was applicable for heterogeneous non-fractured media. Similarly, Stopa and Wojnarowski [114] derived an expression that can help predict the position of the velocity front, during cold water injection applications in geothermal reservoirs. They noticed the speed of the thermal front was not temperature-dependent. Furthermore, in the developed model, they accounted for temperature-dependent rock and water system.

Ziagos and Blackwell [115] proposed a method that predicts the temperature profile within an unconfined aquifer of semi-infinite thickness. Likewise, they deduced a technique that can predict the extent of the zone of influence as well as its magnitude for any combination of thermal and hydrological parameters. Further improvements to the temperature distribution models accounting for various scenarios have been proposed. Consider for example when we have finite confining layers separated by multiple fractures [116–119], or modeling thermal injection backflow tests [117, 120–125], or applications to naturally fractured geothermal reservoirs [126] and development of analogies to tracer transport [127–132].

Lawal and Vesovic [133] developed an analytical model to describe the possible buoyancy-induced natural convection in a one-dimensional, vertical, and semi-infinite reservoir column. The reservoir was assumed to be fully saturated with undersaturated heavy oil and was subjected to a constant temperature from the bottom. For analysis, they assumed that the density and viscosity were temperature-dependent using typical Athabasca bitumen correlation. They showed that the vertical distributions of in situ oil density, velocity, and Nusselt number were consistent with the induced temperature gradient. They concluded that at any time, the oil density

increases vertically away from the heat source, a gravitationally unstable condition, which can trigger the onset of convection. The temperature distribution was obtained by Eq. 20:

$$T(z, t) = T_0 + (T_1 - T_0) \operatorname{erfc}\left(\frac{z}{2\sqrt{\alpha t}}\right) \quad (20)$$

Barends [134] derived an analytical solution that predicts the temperature distribution in porous rocks while considering the effect of convection, conduction, dispersion, and thermal bleeding. He derived equations describing the temperature distribution considering both linear and radial flow situations using the Boltzmann and Laplace transformation methods. These analytical solutions were validated with COMSOL software giving an excellent match. In addition, a sensitivity study was performed with MAPLE for the assessment of specific effects. The solutions were all derived under the assumption of LTE between fluid and rock grains. Accordingly, the temperature profile in the oil layer can be described by Eq. 21.

$$T - T_0 = 2 \frac{(T_1 - T_0)}{\sqrt{\pi}} e^{\frac{xv}{2\alpha}} \int_{\frac{x}{2\sqrt{\alpha t}}}^{\infty} e^{-\sigma^2 - (\frac{xv}{4D\sigma})^2} \operatorname{erfc}\left[\left(\frac{x^2 h' \sqrt{\alpha'}}{8DH\sigma^2} + \frac{z}{2\sqrt{\alpha'}}\right)\left(t - \frac{x^2}{4\alpha\sigma^2}\right)\right] d\sigma \quad (21)$$

Miura and Wang [135, 136] proposed a modification to the Edmunds and Peterson model [137] used for to predict the cumulative steam-to-oil ratio (CSOR). The analytical model was derived by a combination of the material/energy balance and the gravity drainage theory. In the proposed model, the CSOR was allowed to be a function of the average reservoir properties as well as the time-dependent variables. The time-dependent variables include the injection temperature, temperature of the produced fluids, and the rising chamber height. They proved that the new model was able to predict the CSOR of a well more accurately than the Edmunds and Peterson model and was also verified with field data. In addition, the proposed model could be used to predict the instantaneous steam-to-oil ratio (iSOR).

Recently, Wei et al. [138] derived an analytical solution for the development of the steam chamber during steam-assisted gravity drainage (SAGD) applications in heavy oil reservoirs. According to the authors, the steam chamber shape is affected by the steam injection rate with a convex-like parabola for small injection rates and an inverted triangle shape with increasing steam injection.

All the above analytical models were developed based on flow in the transverse direction only. These one-dimensional models fall into two categories namely.

*Boundary conditions dominated models:* generally applicable to relatively thin productive layers. Examples of such models include but not limited to models proposed by [90,93,96,97]. This category of models is adequate for thin productive layers.

The second model assumes that the porous rock is so thick that the heat losses to the impermeable surrounding layers can be neglected. Conceptually, the reservoir is assumed to be made up of a highly permeable fracture network and a relatively small permeability matrix blocks superimposed together [73]. Most geothermal models fall into this category.

Kocabas [73] developed for the first time an analytical solution for predicting the transient temperature profile in two-dimensional laterally/vertically confined layer. He accounted for the heat loss to the surrounding strata and considered the effect of both the longitudinal and transverse heat dispersion. An advantage of this model is that it allows for the understanding of the role on the temperature profile the boundary conditions and fluid mechanics controls play at the same time. Due to much larger transition zones observed in reality, the concept of hydrodynamic heat dispersion was incorporated in the model. He assumed an incompressible fluid, with constant linear flow velocities in both directions. The dimensionless temperature profile ( $T_D$ ) was obtained by the application of Laplace transform; see Eq. 22. Interested readers are recommended to review original work by the author for definitions of dimensionless variables.

$$T_D = \frac{1}{2} \left\{ \operatorname{erfc} \left( \frac{x_D - \tau}{2\sqrt{\tau}} \right) + \exp(x_D) \operatorname{erfc} \left( \frac{x_D + \tau}{2\sqrt{\tau}} \right) \right\} - T_D \sin v \quad (22)$$

where

$$T_D \sin v = \sum_{n=0}^{\infty} \int_0^{t_D} \frac{\exp \left[ -\frac{\{(2n+1)z_{Db}-z_D\}^2}{4\tau} \right] + \exp \left[ -\frac{\{(2n+1)z_{Db}+z_D\}^2}{4\tau} \right]}{\pi \sqrt{\tau} (t_D - \tau)} F_{\sin v} d\tau \quad (23)$$

and

$$F_{\sin v} = \int_0^{\infty} \frac{1}{(\omega^2 + a)^{1/2}} \exp \left[ -\left\{ \frac{\omega^2 + a}{2} (t_D - \tau) \right\} \right] I_{n+\frac{1}{2}} \left[ \frac{\omega^2 + a}{2} (t_D - \tau) \right] \sin x \omega d\omega \quad (24)$$

In Eqs. (22) to (24),  $a$  is a constant equal to 0.25,  $I_{n+\frac{1}{2}}$  is the modified Bessel function of order  $n + \frac{1}{2}$ ,  $\omega$  is the Fourier sine transform variable,  $\sin v$  is the inverse Fourier sine, and  $\tau$  is the time convolution variable. Please refer to the original

manuscript for the definition of other dimensionless variables/terms.

Lastly, Li et al. [139] derived a mathematical model to predict the transient temperature distribution in a confined aquifer, bounded in both longitudinal directions by rocks each having different properties. They obtained the semi-analytic solution for the dimensionless temperature distribution using Laplace transform techniques. They considered in their model the effect of advection in the aquifer and the conduction in the porous media. Table 3 provides a summary of some of the various analytical solutions existing in the literature as regards the temperature distribution in a reservoir. In some cases, expressions were derived for the thermal efficiency or temperature profile in the reservoir. In addition, the assumptions made by the authors in terms of the thermal conductivity in different directions and overburden and under-burden rocks are presented.

In summary, the analytical solutions proposed in the literature eliminate a majority of the non-isothermal physical processes due to the simplifying assumptions usually invoked to arrive at them.

## 4.2 Experimental Studies

A plethora of experimental studies has been conducted over the years, each focusing on different issues some of which include improving oil recovery in heavy oil reservoirs through the addition of pure steam, steam mixed with surfactants, steam with hydrocarbons, and even hot water. Other studies include the effect of temperature on fines migration and subsequent pore throat plugging. In fact, experiments related to water re-injection into subsurface rocks, natural convection, and permeability impairment, etc., are rampant in the literature. Due to space restriction only, literature related to temperature distribution and EOR with hot water, and steam (or its variants) is discussed.

The first study devoted to hot fluid injection was presented by Cheppelear and Volek [140]. These authors considered experimentally and theoretically the heat transfer process by injecting a hotter fluid into an initially cool porous rock saturated with the same fluid and surrounded by heat conducting cap and base rocks. The viscosity dependence was accounted for in their mathematical model, but the specific heat and density of various materials were independent of temperature. The mathematical model was developed assuming a two-dimensional case using a finite difference numerical scheme. Both the theoretical and experimental results indicated that centerline temperatures were significantly higher than boundary temperatures. Likewise, comparisons of experimental and theoretical results with a cold-to-hot viscosity ratio of 19:1 were in reasonable agreement. Their theoretical calculations showed that the effect of the temperature dependence of viscosity was very significant at ratios of

**Table 3** Features of some available analytical models related to temperature distribution in an oil reservoir

Author	Flow geometry in reservoir layer	Rock thermal conductivity				Solutions derived for	
		In the reservoir layer		In the surrounding strata		Temp distribution	Thermal efficiency
		Horizontal direction	Vertical direction	Horizontal direction	Vertical direction		
Avdonin [96]	Linear, radial	Finite	Infinite	Zero	Finite	Yes	No
Avdonin [96]	Linear, radial	Zero	Finite	Zero	Finite	Yes	No
Lauwerier [90]	Linear	Zero	Infinite	Zero	Finite	Yes	Yes
Malofeev and Scheinman [93]	Radial	Zero	Infinite	Zero	Finite	Yes	Yes
Rubinshtein [94]	Radial	Finite	Finite	Finite	Finite	No	Yes
Marx and Langenheim [91]	Radial	Zero	Finite	Zero	Finite	No	Yes
Rubinshtein [94]	Radial	Finite	Infinite	Finite	Finite	Yes	No
Willman et al. [102]	Radial	Zero	Finite	Zero	Finite	No	Yes
Kocabas [73]	Linear	Finite	Infinite	Zero	Finite	Yes	No
Barends [134]	Linear, radial	Zero	Finite	Zero	Finite	Yes	No
Lawal and Vesovic [133]	Linear	Zero	Finite	Zero	Infinite	Yes	No
Li et al. [139]	Radial	Finite	Zero	Zero	Finite	Yes	No
Miura and Wang [136]	Linear	Zero	Infinite	Zero	Finite	No	No
Wei et al. [138]	Linear	Finite	Finite	Zero	Infinite	No	No

100:1 to 1000:1, which are typical of those that occur when injecting hot water to flood heavy oil reservoirs.

Similarly, Baker [141–143] conducted experiments to understand the heat transfer mechanisms in a reservoir using steam to displace a water-saturated porous rock assuming radial fluid flow model. In other words, he assumed the steam front to be a right circular cylinder. He was able to measure the temperature profile in the reservoir and the confining layers through some set of fixed thermocouples. This way he could quantify the fraction of the heat subsequently lost to the overburden and substratum. However, all flooding experiments were performed under low pressure (15 psig). Furthermore, the obtained radial temperature distributions matched perfectly the theoretical results of [90, 92, 144]. Additionally, he calculated the thermal efficiency both numerically and with the experimentally measured temperature values. He realized that higher thermal efficiencies were obtained at higher rates of heat injection. For the case of steam injection, the author found that thermal efficiency decreased with cumulative injected heat and that the heating process was more efficient at higher heat injection rates.

Ferguson [145] carried out further investigations based on the conclusions of Goite and his colleagues [146, 147]. He was interested in determining the optimum propane-to-steam mass ratio to achieve the best recovery. He observed a rapid increase in oil production with steam-to-propane experiments as opposed to using pure steam. He concluded that the

optimum mass ratio of propane to steam was approximately 5:100.

Tinss [148] conducted experiments on heavy oil samples from Kulin oil field in Indonesia using steam and propane combination. He observed rapid oil production in his experiments for a 5:100 propane-to-steam mass ratio. Furthermore, he noticed an increase in the API gravity of the produced oil, as well as a reduction in oil viscosity. In addition, better injectivity was achieved when propane was combined with steam leading to a reduction in the maximum injection pressure of 85–78 psig.

Rivero [149] conducted experiments using heavy oil samples from Hamaca field. His goal was to understand the benefits of steam additives in the recovery of Hamaca oil. He noticed a similar trend (as above studies) of accelerated oil production. He concluded that the optimum recovery was achieved with a propane-to-steam mass ratio of 2.5:100. Furthermore, Simangunsong [150] carried out both experimental investigation to understand how additives such as propane and petroleum distillate improve the recovery of heavy oil during steam injection. For his experiments, heavy oil samples from San Ardo field under the then current reservoir conditions were studied. He observed a rapid jump in oil production when the injected steam was mixed with the additives. In fact, the oil production increased by about 30% for 5:100 propane-to-steam injection and 38% for 5:100 petroleum distillate-to-steam injection. In addition, he

**Table 4** Typical data required for thermal reservoir simulation

Group	Property	Requirements
Reservoir	Principal values of the anisotropic absolute permeability and thermal conductivity, assigned to the directions $x$ , $y$ , and $z$	Three values of permeability and conductivity, respectively, for each block
	Porosity and heat capacity of reservoir rocks	Two values, respectively, for each block
	Relative permeability for each phase	One relation for each phase at each grid block; each relation is a function of saturations and temperature
	Capillary pressure	Two relations as functions of saturations; several pairs allowed
	Reservoir geometry	Specify coordinate system to be used and locations of wells and boundaries
Overburden and under-burden formations	Rock matrix compressibility	One value for each block
	Thermal conductivity and heat capacity	At least one of each for both Caprock and base rock
Initialization values	Rock density	
Fluid property	Saturations, pressure, temperature, and composition	One value for each variable at each grid block
	Density and viscosity of each phase; compressibility of the fluids	One relation for each phase; each relation should depend on temperature, pressure, and possibly composition
Well and boundary conditions	Component properties and $K$ values (for compositional simulation)	Should be a function of pressures and temperature
	Latent heat of vaporization and saturation pressure	Latent heat of vaporization and pressure/temperature relation at saturation for each component that undergoes a phase change
	Enthalpy and internal energy of each phase	A relation for each quantity for each phase as a function of temperature, pressure, and possibly composition
	Specify well type, and inner boundary conditions, rates, pressures, and temperatures	Maximum and minimum values, constraints and penalties

derived a simplified analytical model capable of predicting the steam front position and the cumulative oil recovery for a one-dimensional steam flood. He concluded that the rapid increase in oil production observed was due to the reduction in oil viscosity as a result of mixing steam with propane and petroleum distillate. More recent literature related to experimental studies on thermal EOR applications can be found in references listed here [151–177].

### 4.3 Numerical Simulation

The use of numerical reservoir simulation for steam flood or hot water injection performance prediction has been reported in the literature with applications dating to over 20 years. With rapid increasing numerical, simulation, and computational capabilities, almost all important reservoir phenomena can be modeled adequately. Non-isothermal numerical models are similar to the conventional black-oil simulation with the additional modeling of the energy balance. That is, thermal effects are considered.

Numerical models have the advantage of encompassing all important physics in terms of accurate modeling of the temperature transients in a reservoir. However, the implementation of numerical models requires proper understanding of the issues that are relevant and important. Table 4 describes the amount of data required for the development of any numerical thermal model.

The earliest numerical models were developed for varying applications encompassing a large spectrum some of which are one-dimensional/two-phase flow and heat transfer neglecting the effect of heat losses [178], estimation of the recharge rate and the time of evolution for a fault charged hydrothermal system [179], economic analysis for comparing costs associated with different thermal recovery schemes [180], compositional steam flooding numerical models [181, 182], equation of state thermal simulation [183], investigating multidimensional heat transfer problems associated with hot water or steam injection into an oil reservoir [144, 184–186], heat flow in fractured carbonate reservoir [187–191], natural convection [192, 193], understanding the effect of

temperature-dependent rock properties on three-phase fluid flow during a steam flood [186, 194–196], oil recovery correlation applicable to typical heavy oil reservoirs [197, 198], studies devoted to investigating the effect of steam distillation and solution gas during steam flooding [199] and application of steam injection for removal of non-aqueous phase liquids from subsurface [70, 71]. However, a major drawback of the above studies was the simplistic assumptions incorporated into their numerical models. Take for instance the injected fluid was considered to be non-condensable, temperature-independent rock properties, convection only in one direction, etc.

Hossain et al. [16, 200] developed a one-dimensional numerical model to investigate the effects of the reservoir fluid and injection steam velocities on the temperature distribution in a one-dimensional reservoir. For the first case, they assumed that the reservoir rock and fluids had different temperatures. They observed little or no difference between the fluid and rock temperatures. Secondly, they considered when the reservoir rock and fluid temperature were equal. The authors solved the governing energy balance equations using an explicit finite difference scheme. The convective term and diffusive term were discretized using central differencing. Results showed that fluid velocity, initial steam injection rate, and time have strong effects on the temperature profile. However, in both cases, the fluid velocity was assumed to be a linear function of time and was a function of rock and fluid properties.

Cicek [201] considered the steam displacement of oil in a naturally fractured reservoir by developing a three-dimensional, three-phase, compositional, dual-porosity/dual-permeability model. The effects of capillary pressure and gravity were all incorporated into the simulation model. Cicek [202] again presented a detailed study on the effects of the reservoir and operational properties on the performance of steam displacement considering an inverted nine-spot pattern in a naturally fractured reservoir. In both studies, a fully implicit numerical scheme was developed. Subsequently, the Newton–Raphson method was employed to linearize the resulting sets of equations.

Wu et al. [203] developed a model for predicting the breakthrough time for steam during steam injection into heavy oil reservoirs based on the production performance data. However, the authors concluded that due to some features of the model, the model is best applied during the early time period of steam-drive applications and numerical simulations during the latter stages. Recent investigations have shown that temperature variations can lead to continuous alterations in rock and fluid properties [204–206]. This continuous alteration of fluid and pore space can be captured or modeled by fluid memory models especially in geothermal areas [19].

Again, Hossain et al. [15] developed a finite difference numerical model to investigate the permeability, porosity, and pore volume changes that occur during steam flooding process in a reservoir. The following assumptions were during their analysis; instantaneous thermal equilibrium between rock and fluid, the Boussinesq approximation was applicable. Their results showed the reduction in permeability, increase in porosity, and increase in pore volume during the steam injection process. They concluded that higher cumulative oil recovery would be predicted when the alterations of rock properties are included in recovery calculations. However, the authors assumed a constant fluid velocity for the energy balance.

Recently, new mathematical models have been proposed to describe the temperature evolution in a reservoir during steam injection process [59, 207]. They included the effects of fluid memory through a modified Darcy law. The model was derived assuming a one-dimensional linear reservoir for both the case of instantaneous thermal equilibrium and unequal fluid and rock temperatures. Their study produced new dimensionless numbers that are specific to and influence the performance of a thermal process in an oil reservoir.

Civan [8] proposed an empirical model to describe the permeability impairment in porous rocks incorporating the contributions from fines deposition and non-isothermal conditions such as steam flooding or hot water injection. He developed a one-dimensional, finite difference, numerical scheme to predict the temperature distribution in a reservoir during non-isothermal conditions assuming thermal equilibrium between the flowing fluid system and the porous matrix. From the numerical results, it became evident that temperature variation had a significant effect on permeability impairment, with a higher degree of permeability impairment observed during non-isothermal conditions than isothermal conditions. The proposed model could easily be extended to two- or three-dimensional cases to account for the dispersion in various directions.

Yoshida et al. [208, 209] developed a mathematical model capable of predicting the flow and temperature profile for a system comprising of horizontal wells intersected by transverse fractures. They assumed only single-phase gas flow conditions. Sensitivity studies were conducted to understand the influence of fracture conductivity and the fracture half-length on the temperature behavior of the system. They observed that the wellbore temperature was strongly affected by the fracture half-length and the fracture conductivity. The proposed model is very useful for evaluating created fracture parameters with real-time post-fracture temperature measurements.

Mozaffari et al. [69] developed a three-dimensional, three-phase simulation model to investigate the steam injection process in heavy oil reservoir using a finite difference



scheme. Although the proposed numerical approach was rigorous, in that it accounted for three-phase relative permeability, capillary pressure, pressure- and temperature-dependent fluid properties, and interphase mass transfer between water and steam. Yet, the effect of temperature on rock properties (porosity, absolute permeability, and relative permeability) was not included, the oil was assumed to be nonvolatile, and the hydrocarbon gas was considered insoluble in the liquid phases. The authors pointed out that steam injection could result in an overall recovery improvement of almost 60% from nothing for a fixed period of time.

Very recently, Irawan and Bathaee [210] developed a three-phase mathematical model for the prediction of flow and temperature distribution for water-alternating-gas (WAG) process in a heterogeneous porous media. They included the effects of gravity, turbulence, relative permeability, and capillary pressure. The flow model was developed in the cylindrical coordinate, with the flow in the tangential direction neglected. The governing equations were solved using implicit finite difference scheme. However, the model did not consider temperature-dependent relative permeability and changing rock properties.

Tables 5, 6, and 7 present a summary of the comparison of the treatment of some rock and fluid properties, distribution of components in fluid phases, and features, respectively, in randomly selected non-isothermal simulation models in the literature.

Under certain scenarios, steam injection is not the best possible option for the production of heavy oil reservoirs, for example some shallow reservoirs, or a very deep reservoir due to heat losses either within the reservoir or across the wellbore [199]. Recently, Lasgari [200,201] developed an electrical joule heating simulation model applicable for heavy oil reservoir production applications for the prediction of temperature distribution. They were able to study the effect of water vaporization near the wellbore as well as the effect of water-saturated fractures during the heating process. They concluded that the vaporization of water reduces the generation of heat within the reservoir and that the water saturation and the electrical conductivity of the water within the fractures are critical to the success of the heating process. Other recent technologies include electromagnetic heating [211–220] and downhole heaters [221–229]. It is worth reiterating that although numerical methods incorporate most of the physical processes, they usually need to be validated against benchmark solution to ascertain their accuracy, suffer from reliability problems, large memory requirements, and sometimes excessive computation time.

## 5 Summary

The literature review reveals a collection of mathematical formulations that vary in their assumptions and approaches

to model non-isothermal flow in porous media and, consequently, their accuracies. Unfortunately, most of all the above-formulated lack in a fundamental aspect that is pertinent to thermal EOR operations. That is, accounting for continuous thermal alteration of the characteristics of reservoir rock and fluid.

Additionally, the disordered structure of naturally fractured reservoir rocks (see Fig. 4) has been pointed to be more in line with the anomalous diffusion models, characterized by the mean displacement of particles proportional to the fractional power of time [230,231]. In fact, the transport pathways created by the natural and induced fractures have been shown to be fractals [230]. The petroleum engineering literature is littered with models based on fractal derivatives; for example, refer to references [232–234]. This approach has successfully been applied to capture the stochastic nature of heterogeneity, i.e., natural fractures in reservoirs.

Furthermore, classical diffusion model(s) assume that the random motion of diffusing particles follows a Gaussian probability density characterized with a variance proportional to the first power of time, i.e., the mean square displacement of a particle is a linear function of time. Thus, one might ask what happens when the mean square displacement (variance) grows faster or perhaps slower than the Gaussian diffusion process? A general relationship between the mean square variance and time was presented by [230] as follows:

$$\sigma_r^2 \sim Dt^\gamma \text{ where } \begin{cases} \gamma = 1 & \text{Normal diffusion} \\ \gamma \neq 1 & \text{Anomalous diffusion} \\ \gamma > 1 & \text{Super diffusion} \\ \gamma < 1 & \text{Sub diffusion} \end{cases} \quad (25)$$

A comprehensive mathematical model that can incorporate all the factors discussed thus far will be a huge step up to more realistic, robust, and accurate simulation for non-isothermal fluid flow. One of such approaches is the memory-based models (fractional diffusion models), in that they capture the hereditary nature of the porous media. Applications of such models in petroleum engineering are few and can be found in references [19,21,59,235,236].

## 6 Suggested Future Trends

We propose a generalized constitutive equation of the form presented in Eq. 26 to relate the volumetric flux to the fluid potential in an oil reservoir.

$$\vec{u} = -\eta D_t^{1-\gamma} \nabla \Phi \quad (26)$$

Equation 26 implies a fluid velocity proportional to the time fractional derivative of the gradient of fluid potential in the

**Table 5** Comparison of treatment of rock and fluid properties in some available steam-flood simulation models in the literature

Author	Rel perm	Oil/water viscosity	Gas viscosity	Oil/water density	K values ( $K_v$ )	Porosity	Permeability
Spillate [144]	$k_r(S)$	$\mu_o(T)$	$\mu_g(P, T)$	$\rho_o(T)$	Not applicable	Const.	Const.
Shutler [195]	$k_r(S)$	$\mu_o(P, T)$	$\mu_g(P, T)$	$\rho_o(P, T)$	Not applicable	Const.	Const.
Shutler (1970)	$k_r(S)$	$\mu_o(P, T)$	$\mu_g(P, T)$	$\rho_o(P, T)$	Not applicable	Const.	Const.
Abdalla and Coats [198]	$k_r(S)$	$\mu_o(T)$	$\mu_g(T)$	$\rho_o(T)$	Not applicable	$\phi(P)$	Const.
Vinsome [185]	$k_r(S)$	$\mu_o(T)$	$\mu_g(T)$	$\rho_o(P, T, C)$	Not applicable	Const.	Const.
Coats et al. [247]	$k_r(S, T)$	$\mu_o(T)$	$\mu_g(T)$	$\rho_o(P, T, C)$	Not applicable	$\phi(P)$	Const.
Weinstein et al. [246]	$k_r(S)$	$\mu_o(T, C)$	$\mu_g(T)$	$\rho_o(P, T, C)$	Not applicable	Const.	Const.
Coats [248]	$k_r(S, T)$	$\mu_o(T, C)$	Not clear	$\rho_o(P, T, C)$	$K_v(P, T)$	$\phi(P)$	Const.
Abou-Kassem [182]	$k_r(S, T)$	$\mu_o(T, C)$	$\mu_g(T, P)$	$\rho_o(T)$	$K_v(P, T)$	Const.	Const.
Rubin and Buchanan [249]	$k_r(S, T)$	$\mu_o(T)$	$\mu_g(T)$	$\rho_o(T)$	$K_v(P, T, C)$	$\phi(C_c, \rho_c)$	Const.
Ishimoto et al. [183]	$k_r(S, T)$	$\mu_o(T, C, P)$	$\mu_g(T, P, C)$	$\rho_o(T, P, C)$	$K_v(P, T, C)$	Const.	Const.
Sarathi [194]	$k_r(S, T)$	$\mu_o(T, C)$	$\mu_g(T, P, C)$	$\rho_o(T, P, C)$	$K_v(P, T)$	$\phi(P)$	Const.
Jensen et al. [187]	$k_r(S, T)$	$\mu_o(T)$	$\mu_g(T, C)$	$\rho_o(T, C)$	$K_v(P, T, C)$	$\phi(P)$	Const.
Class et al. [186]	$k_r(S)$	$\mu_o(T)$	$\mu_g(T, C)$	$\rho_o(T)$	Not applicable	Const.	Const.
Cicek [201]	$k_r(S)$	$\mu_o(T)$	$\mu_g(T)$	$\rho(T)$	Yes	Const.	Const.
Hossain et al. [204]	Not applicable	Const.	Not applicable	Const.	Not applicable	$\phi(T)$	$K(T)$
Hossain et al. [235]	Not applicable	Const.	Not applicable	Const.	Not applicable	Const.	Const.
Agarwal et al. [240]	$k_r(S)$	$\mu_o(T, P)$	$\mu_g(P, T)$	$\rho(P, T)$	$K_v(P, T)$	Const.	Const.
Rousset [250]	Not applicable	$\mu_o(T)$	Not applicable	$\rho(P, T)$	Not applicable	Const.	Const.
App [251]	Not applicable	$\mu_o(T, P)$	Not applicable	$\rho(P, T)$	Not applicable	$\phi(P, T)$	Const.
Hossain et al. [59]	Not applicable	$\mu_o(T)$	Not applicable	Const.	Not applicable	$\phi(T)$	$K(T)$
Civan [8]	Not applicable	$\mu_w(T)$	Not applicable	Const.	Not applicable	$\phi(T)$	$K(T)$
Mozaffari et al. [69]	$k_r(S)$	$\mu_o(T)$	$\mu_g(T)$	$\rho(P, T)$	Not applicable	$\phi(P)$	Const.
Yoshida et al. [209]	$k_r(S)$	$\mu_o(T)$	Not applicable	$\rho(P, T)$	Not applicable	Const.	Const.
Irawan and Bathae [210]	$k_r(S)$	Not clear	Not clear	$\rho(P, T)$	Not applicable	$\phi(P)$	Const.

**Table 6** Distribution of components in fluid phases typical of non-isothermal simulation

No	Component	Phases		
		Aqueous	Oleic	Vapor
1	Water	X	–	X
2	Light oil	–	X	X
3	Intermediate oil	–	X	X
4	Heavy oil	–	X	X

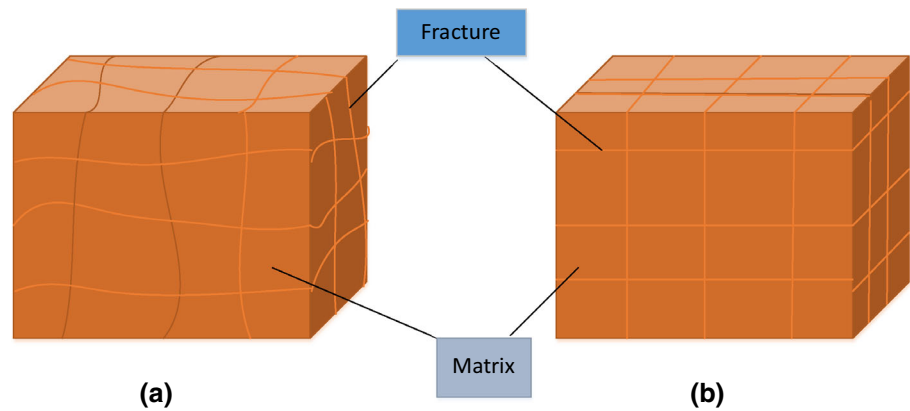
reservoir. Subsequently, the fractional derivative operator  $D_t^{1-\gamma}$  must be interpreted using a suitable definition; Caputo [237, 238], or Reimann–Liouville [239, 240]. Such a generalized constitutive equation has the inherent ability to capture both the classic physics and hereditary nature (long memory) of subsurface reservoir rocks. In fact, Eq. 26 simplifies to the classic Darcy equation for certain value of the fractional-order derivative  $\gamma$ .

Incorporating the proposed constitutive equation into the fluid mass balance results in a nonlinear fractional diffusion

**Table 7** Major features of some numerical simulation models devoted to modeling steam-flood process in the literature

Researcher	Steam distillation effect	Dimension of reservoir geometry	No of phases	Gravity override effect	No of components in phases		Memory effect	Capillary pressure effect
					Oil	Gas		
Spillete [144]	No	2	2	Yes	1	0	No	Yes
Shutler [195]	No	1	3	Yes	1	2	No	Yes
Shutler (1970)	No	2	3	Yes	1	2	No	Yes
Abdalla and Coats [198]	No	2	3	No	1	1	No	Yes
Vinsome [185]	No	3	3	Yes	1	2	No	Yes
Coats et al. [247]	Yes	3	3	Yes	3	3	No	Yes
Coats [199]	Yes	3	3	Yes	2	2	No	Yes
Weinstein et al. [246]	No	1	3	No	2	2	No	No
Coats [248]	Yes	3	3	Yes	2	2	No	Yes
Abou-Kassem [182]	Yes	2	3	Yes	3	4	No	Yes
Rubin and Buchanan [249]	Yes	2	4	Yes	2	4	No	Yes
Ishimoto et al. [183]	Yes	1	3	Yes	3	3	No	Yes
Sarathi [194]	Yes	2	3	Yes	3	4	No	Yes
Jensen et al. [187]	No	2	3	Yes	1	1	No	Yes
Cicek [201]	No	3	3	Yes	Not applicable	Not applicable	No	Yes
Hossain et al. [204]	No	1	1	No	1	Not applicable	No	No
Hossain et al. [235]	No	1	3	No	1	No	No	No
Civan [8]	Not applicable	1	3	No	Not applicable	No	No	No
App [251]	No	1	2	No	1	No	No	No
Mozaffari et al. [69]	No	3	3	Yes	1	1	No	Yes
Hossain et al. [72]	No	1	3	No	1	1	Yes	No
Lashgari et al. (2015)	No	3	3	Yes	Not applicable		No	Yes
Irawan and Bathaee [210]	Not applicable	2	3	Yes	Not applicable		No	Yes

**Fig. 4** **a** Actual, and **b** idealized naturally fractured dual-porosity reservoir model (Redrawn from Ref. [252])



model. Numerical schemes for handling fractional diffusion equations are well established in the literature [241–244]. Lastly, calibrating the accuracy of such fractional diffusion models (memory-based models) with thermal flooding experimental data is recommended.

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