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Numerical simulation of fow on circular crested stepped spillway

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Abstract

The spillways are one of the most important hydraulic structures used in river engineering, dam construction, irrigation, and drainage engineering projects. Recently, a new type of such spillways with a circular crest has been proposed. In this paper, the hydraulic properties of the circular crested stepped spillway (CCSS) including fow pattern, distribution of velocity on the crest and pressure, turbulence intensity, discharge coefficient (C_d) and energy dissipation ratio (EDR) were investigated numerically. To model the free surface of flow the volume of fluid technique, and for modeling the turbulence of flow, $k - \varepsilon$ (RNG) was utilized. Results declared that there is a good agreement between the laboratory observations and numerical simulation. The C_d of the CCSS changes between 0.9 and 1.4 considering the range of relative upstream head (h_{μ} /*R*) between 0.33 m and 2.67. The observation of the fow streamlines showed that they are tangential to the curvature of the crest and there is no separation of the fow from the crest. Examination of the pressure distribution on the CCSS model shows that just downstream part of the crest, the pressure is partially negative. Of course, the same partial negative pressure is observed on the edge of the steps. The steps increase the maximum intensity turbulence by 50%. The CCSS can dissipate the energy of fow between 90 and 30%, and in the skimming fow regime, the portion of each step in the energy dissipation regardless of their position is almost identical.

Keywords CFD simulation · Discharge capacity · Energy dissipation · Skimming fow · Spillway

List of symbols

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Introduction

Modeling the important hydraulic structures such as spillways always is one of the fundamental steps in hydraulic engineering projects such as dam construction and river diversion works. The design of spillways in each project according to their specifc conditions can be a unique case. Although there are technical manuals for designing spillways, the fnal plan should be tested using a physical model (Hager and Pfster [2010](#page-8-0); Bagheri and Kabiri-Samani [2020a](#page-8-1); Haghiabi et al. [2018\)](#page-8-2). The modeling aims to investigate the hydraulic properties including the fow pattern, discharge capacity (determine the rating curve), and distribution of velocity and pressure of fow through the spillway. Over the past decades, the hydraulic properties of various components of the spillway, including the guide walls (Wang and Chen [2010](#page-9-0)), approach channel (Zhang et al. [2015;](#page-9-1) Parsaie et al. [2018](#page-9-2)), crest (Mohammadzadeh-Habili et al. [2016;](#page-9-3) Shamsi et al. [2022](#page-9-4); Parsaie and Haghiabi [2021](#page-9-5)), chute (Parsaie et al. [2016](#page-9-6)) and energy dissipator located at the toe of the chute (Valero et al. [2018](#page-9-7)), have been investigated using physical and numerical simulation. Today, the use of numerical simulation as a powerful tool in the design of spillways and evaluation of their alternative plans is common. In this regard, numerous studies have been published, most of which confrm the accuracy of numerical modeling results (Ghaderi et al. [2020a,](#page-8-3) [2020b;](#page-8-4) Rahimzadeh et al. [2012](#page-9-8); Tadayon and Ramamurthy [2009;](#page-9-9) Bai et al. [2017;](#page-8-5) Zhan et al. [2016](#page-9-10); Mohammad Rezapour Tabari and Tavakoli [2016;](#page-9-11) Bagheri and Kabiri-Samani [2020b\)](#page-8-6). The velocity of the fow on the spillway's chute is very high, which increases the risk of cavitation as well as scouring downstream of the spillway. Spreading downstream scour to the toe of the spillway may damage the dam body (Nou et al. [2021;](#page-9-12) Ghaderi et al. [2020b](#page-8-4); Haghiabi [2017;](#page-8-7) Samadi et al. [2015](#page-9-13)). One of the efective ways to dissipate the energy of the fow passing through the

spillway's chute is to step its surface (Parsaie and Haghiabi [2019a](#page-9-14)), which both cause energy dissipation and increase the aeration of the fow (Parsaie and Haghiabi [2019c](#page-9-15)).

Recently, a circular crest stepped spillway (CCSS) due to having high C_d , suitable ability in energy dissipation and flow aeration has been proposed (Parsaie and Haghiabi [2019b](#page-9-16)). Hence, in this study, the hydraulic properties of the CCSS including flow pattern according to the flow streamlines, the distribution of fow velocity on the crest and pressure on the stepped chute, and energy dissipation are investigated using computational fuid dynamic (CFD) method.

Materials and methods

A CCSS consists of a circular crest that its curvature is an arc of a circle as well as a downstream stepped chute. The CCSS as shown in Fig. [1](#page-1-0) is categorized as a short crested weir. In this figure, h_s and L_s are the height and length of step, respectively. *R* is the radius of crest. *P* is the height of spillway. h_{up} and y_{up} are the head and depth of flow over the crest upstream. Equation [1](#page-1-1) can demonstrate the discharge capacity of the short crested weirs (Bos [1976](#page-8-8)). In this equation, *q* is the discharge per width, *g* is acceleration due to gravity, and C_d is the discharge coefficient.

$$
q = \frac{2}{3} C_d h_{up} \sqrt{\frac{2}{3} g h_{up}}
$$
 (1)

To estimate the performance of CCSS regarding energy dissipation, the total head of flow downstream (Eq. $2:H_{dyn}$ $2:H_{dyn}$) mins the total head fow upstream (Eq. [2:](#page-2-0)*Hup*). The ratio of their difference to the H_{up} (Eq. [4](#page-2-1)) is defined as the performance of CCSS regarding energy dissipation (Haghiabi et al. [2022](#page-8-9)).

Fig. 1 The sketch of the stepped spillway with circular crest

$$
H_{up} = P + y_{up} + \frac{V_{up}^2}{2g} = P + y_{up} + \frac{q^2}{2g(P + y_{up})^2}
$$
 (2)

$$
H_{dwn} = y_1 + \frac{V_1^2}{2g} = y_1 + \frac{q^2}{2gy_1^2}
$$
 (3)

$$
\frac{\Delta H}{H_{up}} = \frac{H_{up} - H_{dwn}}{H_{up}} = \left(1 - \frac{H_{dwn}}{H_{up}}\right) \times 100\tag{4}
$$

where the V_{up} and V_1 are the flow velocity at the upstream and downstream of the CCSS.

Experimental setups

The laboratory investigation on hydraulic models of the CCSS was carried out in a fume of 12 m long, 0.5 m wide and 0.45 m deep. The sidewalls of the fume were made of Plexiglas. The water surface profle in the fume was measured using a point gauge. The fow discharge was measured by a calibrated triangular weir located at the end of the fume. The height of the models was 0.3 m. The radius of the crest of the models was considered to be 0.06 m. The slope of the downstream step chute equal to (V: H) was 1:1.25. The models were made of concrete. The details of the smooth and CCSS models and the range of discharge of flow during the test are presented in Table [1.](#page-2-2)

CFD modeling

In this study, the numerical modeling of hydraulic properties of flow over the CCSS was performed using Flow 3D. The Flow 3D uses the fnite volume method for the discretization of the governing equations of fow (Navier–Stokes equations). The strategy of this software to discretize the computational domain is using the structured mesh. The volume of fuid (VOF) method is utilized for modeling the free surface of the flow. The standard flow equations such as continuity equations (Eq. [5\)](#page-2-3) and Navier–Stokes equations

Table 1 The details of the CCSS models and flow rate

Model	$R(m)$ $P(m)$ N			$S (V:H)$ $Q (m^3/s)$
Smooth spillway 0.06 0.3			1:1.25	0.003–0.045
			1:1.25	
Stepped spillway 0.06 0.3		3,5 and $7 \quad 1:1$		
			1:0.75	

(Eq. [6\)](#page-2-4) are solved numerically for all computational domains (Moukalled et al. [2015;](#page-9-17) Parsaie et al. [2016\)](#page-9-6).

$$
v_f \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} \left(u A_x \right) + \frac{\partial}{\partial x} \left(v A_y \right) = \frac{PSOR}{\rho}
$$
 (5)

where u, v, z are the velocity components in the x, y and w directions. A_x , A_y are the cross-sectional area of the flow, ρ is fluid density, *PSOR* is the source term, v_f is the volume fraction of the fuid and three-dimensional momentum equations given in Eq. (6) (6) .

$$
\frac{\partial u}{\partial t} + \frac{1}{v_f} \left(u A_x \frac{\partial u}{\partial x} + v A_y \frac{\partial u}{\partial y} + w A_z \frac{\partial u}{\partial z} \right) = -\frac{1}{\rho} \frac{\partial P}{\partial x} + G_x + f_x
$$

$$
\frac{\partial v}{\partial t} + \frac{1}{v_f} \left(u A_x \frac{\partial v}{\partial x} + v A_y \frac{\partial v}{\partial y} + w A_z \frac{\partial v}{\partial z} \right) = -\frac{1}{\rho} \frac{\partial P}{\partial y} + G_y + f_y
$$
(6)

where *P* is the fluid pressure, G_x , G_y the acceleration created by body fluids, f_x, f_y viscosity acceleration in three dimensions and v_f is related to the volume of fluid, defined by Eq. ([7\)](#page-2-5). For modeling of free surface profle, the VOF technique based on the volume fraction of the computational cells has been used. Since the volume fraction F represents the amount of fuid in each cell, it takes a value between 0 and 1.

$$
\frac{\partial F}{\partial t} + \frac{1}{v_f} \left[\frac{\partial}{\partial x} \left(F A_x u \right) + \frac{\partial}{\partial y} \left(F A_y v \right) \right] = 0 \tag{7}
$$

To model turbulent flow, numbers of turbulence models including one-equation turbulent model (Prandtl mixing length), two types of two-equation k-ɛ models, renormalized group model $(RNG - k - \varepsilon)$ and large-eddy simulation (LES) have been proposed. The RNG- k-ɛ model is a powerful turbulence model that has suitable performance for modeling the fne vortex; therefore, this model is very useful for modeling fow pattern problems. Most equations used in the RNG model are given in Eqs. ([8](#page-2-6) and [9](#page-2-7)).

$$
\frac{\partial}{\partial t}(\rho k) + \frac{\partial}{\partial x}(\rho u_i k) = \frac{\partial}{\partial x_i}(\alpha_k \mu_{\text{eff}} \frac{\partial k}{\partial x_i}) + G_k + G_b - \rho \varepsilon
$$
 (8)

$$
\frac{\partial}{\partial t}(\rho \varepsilon) + \frac{\partial}{\partial x}(\rho u_i \varepsilon) = \frac{\partial}{\partial x_i} \left(\alpha_k \mu_{\text{eff}} \frac{\partial \varepsilon}{\partial x_i} \right) \n+ C_{1\varepsilon} \frac{\varepsilon}{k} (G_k + C_{3\varepsilon} G_b) \n- C_{2\varepsilon} \rho \frac{\varepsilon^2}{k} - R
$$
\n(9)

in which G_k is the rate of kinetic energy creation, and R is the density of turbulence defned as below.

$$
R = \frac{C_{\mu}\rho\eta^3\left(1 - \eta/\eta_0\right)}{1 + \beta\eta^3} \frac{\varepsilon^2}{k}, \quad \mu_t = \rho C_{\mu} \frac{k^2}{\varepsilon}
$$
(10)

In these equations $\beta = 0.012$, $\eta_0 = 1.38$.

CFD model Setup

To simulate the flow over the CCSS, the number of steps including the definition of the two-dimensional models, definition and meshing of the computational domain, setting the boundary and initial conditions, and selecting the turbulence model should be followed. In this case, the upstream fow head and output were selected as the upstream and downstream boundary conditions, respectively. The symmetry was chosen as the boundary condition of the upper faces of the computational domain. For the lower face of the computational domain, the wall boundary condition was chosen. In this study, the $k - \varepsilon$ (RNG) model is used as the turbulence model. The size of the meshes was equal to 1.0 cm (horizon- $\text{tal})\times 0.5$ cm(vertical). These mesh dimensions are chosen to meet the criteria of mesh independence.

Results and discussion

In this section, the results of numerical simulation are presented. For Flow 3D validation, the water surface pro-file, flow head and flow discharge were used. Figure [2](#page-3-0) shows the measured water surface profle and the results of numerical simulation in the validation stage.

numerical simulation

In this stage, the measured piezometric pressure and results of numerical simulation are plotted in Fig. [3.](#page-4-0) In these fgures, the calibration results for the smooth and stepped model are shown. The examination of the calibration results shows that there is good agreement between the numerical results and the laboratory measurements.

In Fig. [4](#page-4-1)a and b, the measured discharge capacity, discharge coefficient (C_d) and the results of numerical simulations are shown, as well. In this fgure, only the size of the steps is changed and it has been attempted to consider the slope of the stepped chute as constant. This figure shows that the geometric properties of the steps including size and slope of steps do not affect the discharge coefficient and the stage–discharge relation. These results support the theory of supercritical fow which defnes that in the supercritical fow, the downstream cannot afect the upstream flow properties.

One of the most important reasons for the high value of C_d of CCSS is the curvature of its crest. This curvature prevents the separation of the streamlines from the surface of the crest and results in a signifcant reduction in the energy loss of the fow passing over the crest part. Figure [5](#page-5-0) shows the streamlines over the CCSS. As shown in this figure, all the streamlines are tangential to the curvature of the crest surface. Figure [5](#page-5-0)b shows the streamlines on the stepped chute. As can be seen from this figure, a vortex flow is formed between the stairs, which also reduces the fow energy. The examination of Fig. [5a](#page-5-0) shows that the flow regime at the upstream part of the crest is subcritical and the other part (downstream part) is supercritical. In other words, the critical point is formed on the crest. According to the subcritical fow theory, the crest geometry (crest radius) can afect the C_d . It should be noted that the critical depth increases with increasing fow discharge and scrolls upstream. This means Fig. 3 The results of the measured piezometric pressure and the that by increasing the discharge of flow the effect of the

Fig. 4 The results of numerical modeling of stage–discharge (**a**) and discharge coefficient (**b**) of the CCSS

Fig. 6 The distribution of pressure over the smooth (**a**) and stepped chute (**b**)

Fig. 7 The energy dissipation of flow on the CCSS

upstream part of the crest on the C_d is decreased. For this reason, as shown in Fig. [4,](#page-4-1) the C_d remains relatively constant after the y_{up} more than 0.05 m ($y_{up}/R \approx 1$).

Figure [6](#page-5-1) illustrates the pressure distribution over the circular crested spillway with a smooth (Fig. [6](#page-5-1)a) and stepped chute (Fig. [6](#page-5-1)b). Although the flow velocity at the downstream part of the crest increases, the main cause of the negative pressure zone is the sudden change in the direction of flow velocity and the formation of the wake region. Of course, this decrease in pressure even down to negative pressure values can be justifed. It should be noted that the negative pressure in this section does not lead to cavitation and structural degradation. However, it leads to an increase in the discharge capacity due to a rise in the suction zone of flow. This is reminiscent of the criteria of the head design of the Ogee spillway, where the design head can be considered up to 1.5 times the head of the design flood discharge without any problems in the crest part. As the flow passes on the smooth chute, the fow velocity increases, and subsequently, the fow pressure decreases according to the energy equation. As the fow reaches the stilling basin or energy dissipator structure due to the collision of the fow with the surface of the concrete slab, the fow energy dissipation increases. It is noteworthy that the energy dissipation is refected in the decrease in fow velocity and the amount of pressure at the toe of the chute increases. In the following, the distribution of pressure over the CCSS is investigated. As shown in Fig. [6](#page-5-1), the value of pressure on the surface of

steps is positive and at their edges is negative. Of course, it should be checked whether this amount of negative pressure will damage the structure and how much of it (negative) is allowed. To do this, the cavitation index should be calculated. In other words, this will investigate the potential of the CCSS for cavitation to occur. According to Fig. [6,](#page-5-1) the inlet fow pressure to the stilling basin downstream of the stepped spillway is lower than that of the smooth chute spillway, which is due to the greater dissipation of the flow energy through the stepped chute. The results show that as the size of the steps increases, the energy dissipation of fow increases. This is because the intensity of the turbulence of the flow increases with the increase of the step size.

The performance of CCSS in terms of energy dissipation is shown in Fig. [7.](#page-6-0) As presented in this fgure, the CCSS can dissipate the fow energy between 90 and 30%. The stepped chute can dissipate energy about 50% more than the smooth chute even in the skimming fow regime. As presented in this fgure, by increasing the discharge, the performance of smooth and stepped chute decreases signifcantly. Of course, the intensity of the reduction of smooth chute performance is more than the stepped chute. In a smooth chute with an increasing fow rate, its performance decreases by about

91%, while in the same range of increasing flow, the performance of a stepped chute decreases by about 37%.

The effect of each part of CCSS including the crest and steps on the energy dissipation is shown in Fig. [8](#page-6-1). As shown in this fgure the portion of the crest in the energy dissipation is negligible. That reason is the least disturbance in the flow path (streamlines) due to its proper curvature of the crest. When the flow enters the first step, a considerable amount of its energy is dissipated compared to the crest part. At the low value of discharges, the energy loss is due to the collision of the fow with the surface of the steps. By increasing the flow discharge as shown in Fig. [5a](#page-5-0), the vortex flow is generated between the steps. The structure of the vertex flows from a larger view is shown in Fig. [5](#page-5-0)b. After the formation of the vortex fow in the steps, a pseudo-boundary is formed between the steps and the main fow jet that reduces the efect of the steps' size on energy dissipation. This mode of flow is called the skimming flow regime. The h_s/L_s of laboratory models was 0.8, 1.0 and 1.33; hence, when the ratio of y_c/h_s is more than the 0.5, the skimming flow is formed. Theis fnding confrms with criteria presented by Rajaratnam ([1990](#page-9-18)), Chamani and Rajaratnam ([1999\)](#page-8-10), and James et al. (2001) (2001) (2001) . After the full development of flow on the stepped chute and formation of the skimming fow on the steps, the

Fig. 9 The distribution of turbulent intensity percentage over the smooth (**a**) and CCSS (**b**)

portion of each of the steps regardless of their position in the energy dissipation is almost identical. In other words, with the increase of the discharge, the contribution of each step to the energy dissipating decreases. Increasing the step size increases the intensity of fow turbulence, which further increases the rate of fow energy dissipation. It should be noted that the total head of flow must decrease during the spillway, but as seen, on each of the steps, the head of fow frstly decreases and then increases. This is because the direction of velocity in the vortex flow that forms on the steps is the opposite of the fow direction. Moving from the vertex flow to the tip of the steps, the direction of flow is synchronized with the overall direction of the flow passing through the stepped chute, As a result, the fow head is increased relative to the areas affected by the vortex flow.

The effect of the step size as the roughness of the chute on the turbulence intensity is shown in Fig. [9](#page-7-0)c and d. As shown in these fgures, the maximum turbulence intensity of fow on the smooth chute is about 40, whereas this value is increased by up to 48 for the smallest steps, and as the steps become larger, this value is increased to about 78. As the size of the steps increases, the intensity of the fow turbulence increases, which in turn increases the amount of energy loss caused by them.

Conclusion

In this study, the hydraulic properties of circular crested stepped spillways including the pattern of streamlines, distribution of pressure, rating curve, discharge coefficient and mechanism of energy dissipation were investigated. Results declared that the length and curvature of the crest make it possible for the streamlines to adapt to the surface of the crest. The results also showed that crest curvature does not lead to dissipation of the flow energy. The downstream part of the crest performs the smooth transfer of fow from the crest to the stepped chute. By the formation of the skimming flow regime, the contribution of the steps to the energy dissipation decreases, and their portion in the energy dissipation is almost identical regardless of their position. As the step size increases, the intensity of the flow turbulence increases, resulting in a greater amount of energy dissipation.

Author contribution Dr. Haghiabi conceived of the presented idea; Dr. Parsaie developed the theory and performed the computations; Dr. Suleiman Shareef, Dr. Irzooki and Dr. Khalaf performed experimental tests.

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Data availability Some or all data, models or code that supports the fndings of this study is available from the corresponding author upon reasonable request.

Declarations

Competing interests The authors declare that they have no competing interests.

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