Variable Impedance Control for a Single Leg of a Quadruped Robot Based on Contact Force Estimation

Yanan Fan* (, Zhongcai Pei, and Zhiyong Tang

Abstract: A quadruped robot interacts with the ground during the stance phase. This interaction will have a great impact on the feet, torso and joints of the robot, thus affecting the stability of its movement and reducing its adaptability in complex environments with features such as uneven terrain. The contact between each foot of the quadruped robot and the ground should not only control the movement trajectory of the leg but also control the force between the leg and the ground to comply with the environmental constraints. In general, the environment is constantly changing, whereas the traditional impedance control parameters are fixed and thus impose fixed-point constraints. To improve the compliance of the feet of a robot and achieve flexible interactions with the ground in various complex environments, such as pipelines, ruins and forests, variable impedance control is proposed. Based on variable inertia, damping and stiffness parameters, a new Lyapunov function is selected to analyse the stability of the closed-loop system. Furthermore, a force estimator is applied to estimate the contact forces, thereby reducing the burden of structural design and the cost of the robot. The effectiveness of the proposed variable impedance control scheme and contact force estimator is verified through numerical simulations in MATLAB.

Keywords: Dynamics, force estimation, quadruped robot, variable impedance control.

1. INTRODUCTION

Legged robots have attracted extensive attention in recent years due to their strong environmental adaptability, high flexibility and fast movement speed. Legged robots can be divided into two legs, four legs, six legs and eight legs. In particular, quadruped bionic robots have become key research objects because of their superior field walking ability and load capacity [1]. At present, quadruped robots have been applied in material transportation [2,3], emergency rescue [4], home services [5,6], military surveys [7], and engineering exploration [8], among other fields. Most quadruped robots have rigid structures composed of a series of rigid links, prismatic pairs and revolute pairs. Consequently, large rigid collisions will occur between the robot and the ground. This can cause such a robot to be prone to vibrations, which may even result in loss of stability. Therefore, a simple position control strategy cannot satisfy the design needs of a corresponding controller. As an alternative, a flexible force control strategy requires an accurate dynamics model, which increases the difficulty of robot control [9]. Therefore, hybrid force/position control and impedance control are two commonly used approaches in the field of robotics. Compliance control can achieve the desired motion while also

ensuring that a robot can interact with its environment in a friendly manner. In hybrid force/position control, the task space is decoupled into a position subspace and force subspace; accordingly, position control and force control are carried out separately in these two subspaces, ignoring the dynamic coupling between the robot and the environment [10]. Hogan [11] proposed the impedance control approach by establishing the dynamic relationship between the motion of the robot and the external force acting on it, thereby providing a unified framework for the position control of the robot in free space and the compliance control of environment interactions.

In view of the advantages of impedance control, reference [12] applied impedance control in a quadruped robot to compensate for the influence of the weight of the legs on the centre of gravity of the robot to achieve walking stability. Considering the compliance of the end effector of a surgical robot, Xiao *et al.* [13] investigated the impedance control of a minimally invasive surgical robot and determined the impedance control parameters in accordance with collected data so as to avoid excessive insertion force between the robot end effector and human tissue. The impedance control strategy has been widely employed in exoskeleton robots for rehabilitation [14] because impedance control can not only control the posi-

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tion of a robot but also maintain the human-robot interaction force in a safe range, allowing the patient to perform rehabilitation training in accordance with predetermined movements [15]. Ochoa et al. [16] realized a mould polishing operation for a 7 degree of freedom (DOF) robot within the impedance control framework. Notably, the previous research work on impedance control has mainly focused on constant impedance control; that is, the inertia parameters, damping parameters and stiffness parameters have been expressed as constant matrices. In practical applications, however, the typical working environment of a robot is relatively complex. For example, when a quadruped robot walks on a rough road with inconsistent stiffness, it will be difficult to achieve good walking performance if the impedance control parameters remain unchanged. Therefore, constant impedance control cannot guarantee the motion accuracy of a robot or the dynamic performance of its control system. Accordingly, in recent years, some scholars have begun to explore the potential of variable impedance control for improving the flexibility and safety of robot operations. Lee et al. [17] defined the stiffness parameters as a time-varying matrix to track the contact forces of a 7-DOF manipulator platform. Reference [18] proposed an impedance control scheme with time-varying damping and stiffness parameters based on the pin insertion task on the KUKA iiwa collaborative robot test platform. In [19], variable impedance control was applied for apple picking. To reduce the damage caused to the apples during the picking process, three selftuning control parameters were defined to enhance the performance of the force-based impedance control system.

Although variable impedance control improves the dexterity and dynamic performance of the controlled system [20], its control parameters make the traditional system stability analysis method no longer applicable and increase the difficulty of performance analysis for the controlled system. In [21], a 7-DOF manipulator was used to simulate a surgical suturing process. Time-varying stiffness parameters can allow the impedance characteristics to be adjusted in accordance with the unique properties of different people and different tissue layers. However, such time-varying stiffness parameters will also inject additional energy into the system and make the system unstable. To allow a manipulator to stably interact with its external environment, the concept of a virtual energy tank has been introduced to store the energy dissipated by the controlled system in order to adjust the time-varying stiffness so as to ensure the stability of the system [22]. Unfortunately, this method depends on the initial state of the system. Park et al. [23] proposed an input-state-stable variable impedance controller for a manipulator by defining a Lyapunov function that includes both the kinetic energy and potential energy and designing a variable stiffness update law to ensure the asymptotic stability of the system. Kronander et al. [24] proposed state-independent stability conditions for time-varying stiffness and damping matrices by referring to the method of adaptive selection of the energy function, namely, a weighted quadratic energy function that includes both the velocity error and position error. Variable impedance control can simultaneously adjust the dynamic relationship between the motion and contact force of a robot and flexibly adjust the dynamic characteristics of the system in accordance with environmental changes that occur during the execution of interactive tasks [25]. Recently, variable impedance control has attracted the attention of many scholars; however, there has been little related academic literature in the field of quadruped robots to date. Reference [26] studied constant impedance control for a quadruped robot. This method does not allow for dynamic changes in the operating stiffness of the foot, although variable stiffness is quite significant for enabling dynamic interaction between a quadruped robot and its environment, which is beneficial for helping the robot to walk smoothly. The ability to perceive force is essential for adapting to an unknown environment and achieving compliance control of a robot [27]. The traditional method is to install force sensors on the robot. However, the installation of force sensors will impose a burden on the overall structure of the robot, raise the complexity of system operation and increase the cost of the system. As an alternative, designing a force estimator for a robot rather than directly equipping it with sensors can reduce the design difficulty, and enable the robot to sense collisions, external interference, environmental changes, etc. [28].

In this paper, a variable impedance control scheme for a single leg of a quadruped robot is primarily studied. The main contributions are as follows:

- To endow the quadruped robot with a force sensing ability while avoiding complex structural components such as force sensors, a contact force estimator is designed based on a nonlinear disturbance observer for the single-leg workspace. Moreover, the convergence of the contact force tracking error is proven.
- 2) To ensure the contact flexibility of the quadruped robot and enable the robot to independently adjust its control parameters when moving on complex and uneven terrain, a variable impedance controller based on the operation space is proposed for a single-leg model of the quadruped robot, in which the time-varying impedance control parameters include a time-varying inertia matrix, a time-varying damping matrix and a time-varying stiffness matrix.
- 3) Because the control parameters are varying in time, the traditional Lyapunov function selection method will continuously inject potential energy into the system. In this paper, a new Lyapunov structure including time-varying impedance control parameters is adopted to analyse the stability of the system, and

state-independent constraint conditions are proposed to ensure system stability.

4) The proposed variable impedance controller is implemented in the operation space, thus allowing the quadruped robot to perform the desired task by means of coupled force-position constraints and avoiding the occurrence of an acceleration term.

The structure of this paper is as follows: In Section 2, the dynamics modelling for a single leg of a quadruped robot is briefly introduced. In Section 3, a force estimator is designed. Section 4 introduces the implementation of the proposed variable impedance controller in the operation space. In Section 5, MATLAB numerical simulations are presented to verify the feasibility of the proposed variable impedance control method and force estimator. Finally, in Section 6, the conclusion is given.

2. DYNAMICS MODELLING FOR A SINGLE LEG OF A QUADRUPED ROBOT

In this paper, a single leg of a quadruped robot is taken as the research object. The single-leg structure model of the quadruped robot is shown in Fig. 1. Each leg has 3 degrees of freedom and is composed of three connecting rods connected through revolute pairs.

To model a single leg of the robot, the single-leg structure model is simplified as shown in Fig. 2, where $x_{\text{H}}-y_{\text{H}}-z_{\text{H}}$ is the base frame of the leg; $x_i-y_i-z_i$ for i = 1, 2, 3 and 4 are the coordinate frames of the rolling hip joint, pitching hip joint, knee joint and foot, respectively; and l_1 , l_2 and l_3 are the rol lengths.

In this paper, the dynamics model of a single leg mechanism is established based on the Lagrange function [29], which is an effective and widely used method. By applying the Euler-Lagrange equation [30], the dynamics of a single leg mechanism can be described in the following normal form



Fig. 1. The single-leg structure of the quadruped robot.



Fig. 2. Schematic diagram of the single-leg structure.

$$\boldsymbol{M}\ddot{\boldsymbol{\theta}} + \Gamma \begin{bmatrix} \dot{\theta}_{1} \dot{\theta}_{1} \\ \dot{\theta}_{2} \dot{\theta}_{2} \\ \dot{\theta}_{3} \dot{\theta}_{3} \\ \dot{\theta}_{1} \dot{\theta}_{2} \\ \dot{\theta}_{1} \dot{\theta}_{3} \\ \dot{\theta}_{2} \dot{\theta}_{3} \end{bmatrix} + \boldsymbol{G} = \boldsymbol{\tau}.$$
(1)

In (1), $\boldsymbol{M} \in \Re^{3 \times 3}$ is the inertia matrix, $\Gamma \in \Re^{3 \times 6}$, $\boldsymbol{G} \in \Re^{3 \times 1}$ is the gravity term, $\boldsymbol{\tau} \in \Re^{3 \times 1}$ is the actuation torque, and

$$\ddot{\boldsymbol{\theta}} = \begin{bmatrix} \ddot{\boldsymbol{\theta}}_1 & \ddot{\boldsymbol{\theta}}_2 & \ddot{\boldsymbol{\theta}}_3 \end{bmatrix}^{\mathrm{T}} \text{ is the angular acceleration. } \boldsymbol{\Gamma} \begin{vmatrix} \dot{\theta}_1 \dot{\theta}_1 \\ \dot{\theta}_2 \dot{\theta}_2 \\ \dot{\theta}_3 \dot{\theta}_3 \\ \dot{\theta}_1 \dot{\theta}_2 \\ \dot{\theta}_1 \dot{\theta}_3 \\ \dot{\theta}_2 \dot{\theta}_3 \end{vmatrix}$$

is the Coriolis force and centrifugal force term, in accordance with the properties of the robot dynamics model [29]

$$\mathbf{\Gamma} = \frac{\partial M_{ij}}{\partial \theta_k} - 0.5 \frac{\partial M_{jk}}{\partial \theta_i}, \ (i, \ j, \ k = 1, \ 2, \ 3).$$
(2)

The Coriolis force and centrifugal force term can be further simplified as

$$\boldsymbol{C}(\boldsymbol{\theta}, \dot{\boldsymbol{\theta}}) = \boldsymbol{\Gamma} \dot{\boldsymbol{\theta}}. \tag{3}$$

Then, the dynamics model based on the configuration space can be rewritten as

$$\boldsymbol{M}\boldsymbol{\ddot{\theta}} + \boldsymbol{C}(\boldsymbol{\theta}, \boldsymbol{\dot{\theta}})\boldsymbol{\dot{\theta}} + \boldsymbol{G} = \boldsymbol{\tau}.$$
(4)

Usually, the desired trajectory of the robot is planned in the operation space. Thus, to facilitate the compliance control of the robot in the standing phase, the dynamics model in the configuration space should be further trans-

lated into the operation space. By defining
$$\boldsymbol{q}_e = \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$
 as the

trajectory of the foot, the differential of q_e with respect to time can be derived as follows:

$$\dot{\boldsymbol{q}}_e = \boldsymbol{J}\boldsymbol{\dot{\boldsymbol{\theta}}}.$$

 $J \in \Re^{3 \times 3}$ represents the Jacobian matrix for the transformation from the configuration space to the operation space. Taking the derivative of (5) results in

$$\ddot{\boldsymbol{q}}_{e} = \boldsymbol{J}\ddot{\boldsymbol{\theta}} + \dot{\boldsymbol{J}}\dot{\boldsymbol{\theta}}.$$
(6)

Now, by multiplying both sides of (4) by J^{-T} and substituting (5) and (6) into (4), the dynamics equation of the leg can be rewritten in the following compact form

$$\boldsymbol{M}_{e}\boldsymbol{\ddot{q}}_{e} + \boldsymbol{C}_{e}\boldsymbol{\dot{q}}_{e} + \boldsymbol{G}_{e} = \boldsymbol{\tau}_{e}.$$
(7)

In (7), $M_e = J^{-T}MJ^{-1}$ denotes the inertia matrix in the operation space, $C_e = J^{-T} (C - MJ^{-1}JJ^{-1})$ represents the Coriolis force and centrifugal force matrix in the operation space, $\boldsymbol{G}_{e} = \boldsymbol{J}^{-\mathrm{T}}\boldsymbol{G}$ is the gravitational force vector in the operation space, and $\boldsymbol{\tau}_e = \boldsymbol{J}^{-\mathrm{T}} \boldsymbol{\tau}$ is the actuation torque in the operation space.

In the stance phase, considering the contact force, the dynamics model of (7) becomes [31]

$$\boldsymbol{M}_{e} \boldsymbol{\ddot{q}}_{e} + \boldsymbol{C}_{e} \boldsymbol{\dot{q}}_{e} + \boldsymbol{G}_{e} = \boldsymbol{\tau}_{e} + \boldsymbol{\tau}_{out}, \qquad (8)$$

where $\boldsymbol{\tau}_{out}$ is the contact force. A quadruped robot is a relatively complex system. However, the installation of a force sensor to sense the external contact force has several disadvantages, such as increasing the costs of implementation and maintenance of the robot. Instead of relying on a force sensor, designing a force estimator to indirectly sense the external contact force is an economical and effective alternative.

3. FORCE ESTIMATOR DESIGN

3.1. Force estimator design

Sensorless technology provides an alternative means of determining contact forces [32]. Previous research on external force estimators has been based on Newton's law [33] and disturbance estimation [34], among other approaches. These force estimation models usually include an acceleration parameter, although a recently proposed method based on generalized momentum overcomes the problems presented by the acceleration term and the impact of system noise [35]. However, the accuracy with which the contact force can be estimated based on the generalized momentum is dependent on the gain parameters, and the stability of such estimators has rarely been proven in the literature. In this paper, the foot contact force in the closed-loop system is estimated based on a nonlinear disturbance observer [28].

For estimating the contact force τ_{out} , according to the dynamic system given in (8), the auxiliary variable z is defined as

$$\boldsymbol{z} = \hat{\boldsymbol{\tau}}_{out} - \boldsymbol{p}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e), \tag{9}$$

where the vector $p(q_e, \dot{q}_e)$ can be obtained from the nonlinear estimator gain matrix $L(q_e, \dot{q}_e)$

$$\frac{d}{dt}\boldsymbol{p}(\boldsymbol{q}_{e}, \dot{\boldsymbol{q}}_{e}) = \boldsymbol{L}(\boldsymbol{q}_{e}, \dot{\boldsymbol{q}}_{e})\boldsymbol{M}_{e} \boldsymbol{\ddot{q}}_{e}.$$
(10)

In (10), the gain matrix $L(q_e, \dot{q}_e)$ can be expressed as

$$\boldsymbol{L}(\boldsymbol{q}_{e}, \dot{\boldsymbol{q}}_{e}) = X^{-1} \boldsymbol{M}_{e}^{-1}, \qquad (11)$$

where X is a constant matrix; then, **p** can be derived as

$$\boldsymbol{p} = \boldsymbol{X}^{-1} \dot{\boldsymbol{q}}_e. \tag{12}$$

Taking the derivative of (9), we obtain

$$\begin{aligned} \dot{\boldsymbol{z}} &= \hat{\boldsymbol{\tau}}_{out} - \dot{\boldsymbol{p}}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e) \\ &= \dot{\boldsymbol{\tau}}_{out} - \boldsymbol{L}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e) \boldsymbol{M}_e \ddot{\boldsymbol{q}}_e, \end{aligned} \tag{13}$$

where $\dot{\hat{\tau}}_{out}$ is defined as

$$\dot{\hat{\boldsymbol{\tau}}}_{out} = -\boldsymbol{L}\hat{\boldsymbol{\tau}}_{out} + \boldsymbol{L}(\boldsymbol{M}_{e}\boldsymbol{\ddot{\boldsymbol{q}}}_{e} + \boldsymbol{C}_{e}\boldsymbol{\dot{\boldsymbol{q}}}_{e} + \boldsymbol{G}_{e} - \boldsymbol{\tau}_{e}).$$
(14)

Substituting (8), (9) and (14) into (13) yields

$$\dot{\boldsymbol{z}} = -\boldsymbol{L}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e)\boldsymbol{z} + \boldsymbol{L}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e)(\boldsymbol{C}_e \dot{\boldsymbol{q}}_e + \boldsymbol{G}_e - \boldsymbol{\tau}_e - \boldsymbol{p}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e)).$$
(15)

Then, according to (12) and (15), the contact force τ_{out} can be represented as

$$\hat{\boldsymbol{\tau}}_{out} = \boldsymbol{z} + \boldsymbol{p}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e). \tag{16}$$

The process of the design and analysis of the contact force estimator has been presented above. To ensure that the appropriate X is obtained, the convergence of the tracking error of the contact force estimator should be further analysed.

3.2. Stability of the contact force estimator

As shown in the previous subsection, the contact force between the foot of the quadruped robot and the ground can be estimated by means of (9). Suppose that the quadruped robot is moving on homogeneous ground and that the external contact force between the foot end and the ground remains basically unchanged, that is, $\dot{\boldsymbol{\tau}}_{out} \approx 0$; then, the following theorem can guarantee the asymptotic stability of the foot-end force estimation error for the quadruped robot.

Theorem 1: For a quadruped robot, the contact force can be estimated by means of (9). If *X* satisfies the following conditions, then the contact force tracking error $\tilde{\boldsymbol{\tau}}_{out} = \boldsymbol{\tau}_{out} - \boldsymbol{\hat{\tau}}_{out}$ will converge to zero

- 1) The matrix *X* is invertible.
- 2) There exists a positive definite and symmetric matrix χ satisfying $X + X^T X^T \dot{\mathbf{M}}_e X \ge \chi$.

Proof: The following energy function is considered

$$W = (X \widetilde{\boldsymbol{\tau}}_{out})^{\mathrm{T}} \boldsymbol{M}_{e} (X \widetilde{\boldsymbol{\tau}}_{out})$$
$$= \widetilde{\boldsymbol{\tau}}_{out}^{\mathrm{T}} X^{\mathrm{T}} \boldsymbol{M}_{e} X \widetilde{\boldsymbol{\tau}}_{out}.$$
(17)

Since M_e is a symmetric and positive matrix, X is invertible, and $X^T M_e X$ is positive definite, the energy function W is also positive definite and $W \rightarrow \infty$.

When $\dot{\boldsymbol{\tau}}_{out} \approx 0$, the derivative of (17) can be calculated to obtain

$$\dot{W} = \dot{\tilde{\boldsymbol{\tau}}}_{out}^{\mathrm{T}} X^{\mathrm{T}} \boldsymbol{M}_{e} X \tilde{\boldsymbol{\tau}}_{out} + \tilde{\boldsymbol{\tau}}_{out}^{\mathrm{T}} X^{\mathrm{T}} \boldsymbol{M}_{e} X \dot{\tilde{\boldsymbol{\tau}}}_{out} + \tilde{\boldsymbol{\tau}}_{out}^{\mathrm{T}} X^{\mathrm{T}} \dot{\boldsymbol{M}}_{e} X \tilde{\boldsymbol{\tau}}_{out}.$$
(18)

Because $\widetilde{\boldsymbol{\tau}}_{out} = \boldsymbol{\tau}_{out} - \hat{\boldsymbol{\tau}}_{out}$, it holds that

$$\hat{\boldsymbol{\tau}}_{out} = \dot{\boldsymbol{\tau}}_{out} - \dot{\boldsymbol{\dot{\tau}}}_{out}.$$
(19)

Substituting (10) into (19) results in

$$\widetilde{\boldsymbol{\tau}}_{out} = \dot{\boldsymbol{\tau}}_{out} - \boldsymbol{L}(\boldsymbol{q}_e, \dot{\boldsymbol{q}}_e) \widetilde{\boldsymbol{\tau}}_{out}.$$
(20)

Substituting (12) and (20) into (18) yields

$$\dot{W} = -\widetilde{\boldsymbol{\tau}}_{out}^{\mathrm{T}} (X + X^{\mathrm{T}} - X^{\mathrm{T}} \dot{\boldsymbol{M}}_{e} X) \widetilde{\boldsymbol{\tau}}_{out}.$$
⁽²¹⁾

According to the energy functions W > 0, $X + X^T - X^T \dot{M}_e X \ge \chi$, and $\dot{W} < 0$, it can be concluded that the contact force tracking error asymptotically converges to zero. This completes the proof.

4. DESIGN OF THE VARIABLE IMPEDANCE CONTROLLER

The essence of impedance control is based on the virtual concept of "mechanical impedance" and the principle of feedback equivalence. One of the most important advantages of impedance control is that the force control target and displacement control target are unified in the target system to dynamically adjust the motion and contact force of the end effector. In invariant impedance control, the force and position constraints in the control task target are constant constraints, so the control problem is a problem of fixed-point adjustment. However, in variable impedance control, the control target is a "time-varying impedance contour or trajectory". For complex systems such as assembly robots, grinding robots, and exoskeleton robots, the inertia, damping and stiffness parameters are varying in time during the execution of the operation tasks, so constant impedance control cannot guarantee the performance of such a system.

To improve the dynamic performance of the foot of a quadruped robot in contact with the ground and adjust the interaction state between the quadruped robot and its environment in real time, the dynamics model of the closed-loop system for variable impedance control is established as follows [37]:

$$\boldsymbol{H}(t)\ddot{\boldsymbol{q}}_{e} + \boldsymbol{D}(t)\dot{\boldsymbol{q}}_{e} + \boldsymbol{K}(t)\boldsymbol{\bar{q}}_{e} = \boldsymbol{\tau}_{out}, \qquad (22)$$

where $\boldsymbol{H}(t)$, $\boldsymbol{D}(t)$, and $\boldsymbol{K}(t)$ are symmetric positive timevarying matrices. $\boldsymbol{H}(t)$ represents the desired inertia matrix, $\boldsymbol{D}(t)$ denotes the desired damping matrix, and $\boldsymbol{K}(t)$ is the desired stiffness matrix. It is supposed that $\boldsymbol{\ddot{q}}_{e} =$ $\boldsymbol{\ddot{q}}_{e} - \boldsymbol{\ddot{q}}_{ed}$, $\boldsymbol{\ddot{q}}_{e} = \boldsymbol{\dot{q}}_{e} - \boldsymbol{\dot{q}}_{ed}$, and $\boldsymbol{\bar{q}}_{e} = \boldsymbol{q}_{e} - \boldsymbol{q}_{ed}$ are the errors in acceleration, velocity and position, respectively, where $\boldsymbol{\ddot{q}}_{ed} = \begin{bmatrix} \ddot{x}_{d} & \ddot{y}_{d} & \ddot{z}_{d} \end{bmatrix}^{\mathrm{T}}$, $\boldsymbol{\dot{q}}_{ed} = \begin{bmatrix} \dot{x}_{d} & \dot{y}_{d} & \dot{z}_{d} \end{bmatrix}^{\mathrm{T}}$, and $\boldsymbol{q}_{ed} = \begin{bmatrix} x_{d} & y_{d} & z_{d} \end{bmatrix}^{\mathrm{T}}$ are the desired acceleration, desired speed and desired position of the foot, respectively, associated with the desired gait.

By combining the single-leg dynamics model (8) of the quadruped robot with the variable impedance control dynamics model (22), the control law for variable impedance control can be derived as follows:

$$\begin{cases} \boldsymbol{\tau}_{e} = \boldsymbol{M}_{e}\boldsymbol{v} + \boldsymbol{C}_{e}\dot{\boldsymbol{q}}_{e} + \boldsymbol{G}_{e} - \boldsymbol{\tau}_{out}, \\ \boldsymbol{v} = \ddot{\boldsymbol{q}}_{ed} + (\boldsymbol{H}(t))^{-1}(\boldsymbol{\tau}_{e} - \boldsymbol{K}(t)\overline{\boldsymbol{q}}_{e} - \boldsymbol{D}(t)\dot{\overline{q}}_{e}). \end{cases}$$
(23)

To ensure the stability of the closed-loop system, in accordance with Lyapunov stability theory, the following energy function is selected [38]

$$\mathbf{V} = \frac{1}{2} (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e)^{\mathrm{T}} \boldsymbol{H}(t) (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e) + \frac{1}{2} \bar{\boldsymbol{q}}_e^{\mathrm{T}} \boldsymbol{\beta}(t) \bar{\boldsymbol{q}}_e, \qquad (24)$$

where α is a constant satisfying $\alpha > 0$ and $\beta(t)$ is an undetermined time-varying matrix satisfying $\beta(t) > 0$. The above conditions $\alpha > 0$ and $\beta(t) > 0$ ensure the positive definiteness of the Lyapunov function *V*.

Assumption 1: The inertia $\boldsymbol{H}(t)$, damping $\boldsymbol{D}(t)$, and stiffness $\boldsymbol{K}(t)$ are set to be symmetric, uniformly continuous functions, and $\boldsymbol{H}(t) > 0$, $\boldsymbol{D}(t) > 0$, and $\boldsymbol{K}(t) > 0$. In other words, the time derivatives $\boldsymbol{H}(t)$, $\boldsymbol{D}(t)$, and $\boldsymbol{K}(t)$ are bounded for all t > 0.

Assumption 2: There always exists a matrix $D_0(t)$ such that $\dot{H}(t) - 2D_0(t)$ is skew symmetric.

The inertia H(t) determines the stability of the system, and too large or too small of a value will affect the system performance.

Assumption 3: Conservative constraints on H(t) are given by M_e , and H(t) is chosen such that

$$\lambda_{\min}(\boldsymbol{M}_e)\mathbf{I} \leq \boldsymbol{H}(t) \leq \lambda_{\max}(\boldsymbol{M}_e)\mathbf{I}.$$

Theorem 2: Suppose that Assumptions 1-3 hold. Considering the single-leg dynamics model (8) of the quadruped robot and the target variable impedance control dynamics model (22), suppose that the following set of inequalities holds

$$\begin{cases} \boldsymbol{\alpha} > 0, \\ \boldsymbol{K}(t) + \boldsymbol{\alpha} \boldsymbol{D}(t) - \boldsymbol{\alpha}^{2} \boldsymbol{H}(t) - 2\boldsymbol{\alpha} \boldsymbol{D}_{0}(t) > 0, \\ \boldsymbol{D}(t) - \boldsymbol{\alpha} \boldsymbol{H}(t) - \boldsymbol{D}_{0}(t) > 0, \\ \boldsymbol{\alpha} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \boldsymbol{\alpha} \dot{\boldsymbol{D}}(t) + \boldsymbol{\alpha} \dot{\boldsymbol{D}}_{0}(t) > 0. \end{cases}$$
(25)

When $\tau_{out} = 0$, the origin $\bar{q}_e = 0$ of the variable impedance control dynamics model (22) is globally asymptotically stable. When $\tau_{out} \neq 0$, the origin $\bar{q}_e = 0$ of the system described by (22) is globally stable under the same conditions.

Proof: The time derivative of the Lyapunov energy function can be calculated as follows:

$$\dot{\mathbf{V}} = (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e)^{\mathrm{T}} \boldsymbol{H}(t) (\ddot{\boldsymbol{q}}_e + \alpha \dot{\bar{\boldsymbol{q}}}_e) + \frac{1}{2} (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e)^{\mathrm{T}} \dot{\boldsymbol{H}}(t) (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e) + \dot{\bar{\boldsymbol{q}}}_e^{\mathrm{T}} \beta(t) \bar{\boldsymbol{q}}_e + \frac{1}{2} \bar{\boldsymbol{q}}_e^{\mathrm{T}} \dot{\beta}(t) \bar{\boldsymbol{q}}_e.$$
(26)

Upon substituting the resulting dynamics function $\boldsymbol{H}(t)\ddot{\boldsymbol{q}}_e = \boldsymbol{\tau}_{out} - \boldsymbol{K}(t)\boldsymbol{\bar{q}}_e - \boldsymbol{D}(t)\dot{\boldsymbol{\bar{q}}}_e$ into (26), it follows that

$$\dot{\mathbf{V}} = (\dot{\overline{\boldsymbol{q}}}_{e} + \alpha \overline{\boldsymbol{q}}_{e})^{\mathrm{T}} \boldsymbol{\tau}_{out} - \dot{\overline{\boldsymbol{q}}}_{e}^{\mathrm{T}} (\boldsymbol{D}(t) - \alpha \boldsymbol{H}(t) - \frac{1}{2} \dot{\boldsymbol{H}}(t)) \dot{\overline{\boldsymbol{q}}}_{e} - \dot{\overline{\boldsymbol{q}}}_{e}^{\mathrm{T}} (\boldsymbol{K}(t) + \alpha \boldsymbol{D}(t) - \alpha^{2} \boldsymbol{H}(t) - \alpha \dot{\boldsymbol{H}}(t) - \beta(t)) \overline{\boldsymbol{q}}_{e} - \overline{\boldsymbol{q}}_{e}^{\mathrm{T}} (\alpha \boldsymbol{K}(t) - \frac{1}{2} \alpha^{2} \dot{\boldsymbol{H}}(t) - \frac{1}{2} \dot{\beta}(t)) \overline{\boldsymbol{q}}_{e}.$$
(27)

By eliminating the nonquadratic term in (26), the positive definiteness of the Lyapunov function is automatically guaranteed. By setting $\beta(t) = \mathbf{K}(t) + \alpha \mathbf{D}(t) - \alpha^2 \mathbf{H}(t) - \alpha \mathbf{\dot{H}}(t) > 0$ and taking the derivative of $\beta(t)$, one obtains

$$\dot{\boldsymbol{\beta}}(t) = \boldsymbol{\dot{K}}(t) + \alpha \boldsymbol{\dot{D}}(t) - \alpha^2 \boldsymbol{\dot{H}}(t) - \alpha \boldsymbol{\ddot{H}}(t) > 0.$$
(28)

By substituting (28) into (27), (27) can be written in a more concise form

$$\dot{\mathbf{V}} = (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e)^{\mathrm{T}} \boldsymbol{\tau}_{out} - \dot{\bar{\boldsymbol{q}}}_e^{\mathrm{T}} (\boldsymbol{D}(t) - \alpha \boldsymbol{H}(t) - \frac{1}{2} \dot{\boldsymbol{H}}(t)) \dot{\bar{\boldsymbol{q}}}_e - \bar{\boldsymbol{q}}_e^{\mathrm{T}} (\alpha \dot{\boldsymbol{K}}(t) - \frac{1}{2} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \alpha \dot{\boldsymbol{D}}(t) + \frac{1}{2} \alpha \ddot{\boldsymbol{H}}(t)) \bar{\boldsymbol{q}}_e.$$
(29)

For a given $\boldsymbol{H}(t)$, there always exists a $\boldsymbol{D}_0(t)$ such that $\dot{\boldsymbol{H}}(t) - 2\boldsymbol{D}_0(t)$ is skew symmetric, i.e., $(\dot{\boldsymbol{q}}_e + \alpha \boldsymbol{\bar{q}}_e)^{\mathrm{T}}(\dot{\boldsymbol{H}}(t) - \boldsymbol{\bar{H}}(t))$

 $2\boldsymbol{D}_0(t))(\ddot{\boldsymbol{q}}_e + \alpha \dot{\boldsymbol{q}}_e) = 0$. Substituting $\boldsymbol{D}_0(t)$ for $\dot{\boldsymbol{H}}(t)$ in (29) leads to

$$\dot{\mathbf{V}} = (\dot{\boldsymbol{q}}_{e} + \alpha \boldsymbol{\bar{q}}_{e})^{\mathrm{T}} \boldsymbol{\tau}_{out} - \dot{\boldsymbol{q}}_{e}^{\mathrm{T}} (\boldsymbol{\mathcal{D}}(t)) - \alpha \boldsymbol{\mathcal{H}}(t) - \boldsymbol{\mathcal{D}}_{0}(t)) \dot{\boldsymbol{\bar{q}}}_{e} - \boldsymbol{\bar{q}}_{e}^{\mathrm{T}} (\alpha \dot{\boldsymbol{\mathcal{K}}}(t)) - \frac{1}{2} \dot{\boldsymbol{\mathcal{K}}}(t) - \frac{1}{2} \alpha \dot{\boldsymbol{\mathcal{D}}}(t) + \alpha \dot{\boldsymbol{\mathcal{D}}}_{0}(t)) \boldsymbol{\bar{q}}_{e}.$$
(30)

Next, the stability of the closed-loop system in the noncontact phase and in the contact phase, i.e., when $\tau_{out} = 0$ and $\tau_{out} \neq 0$, will be discussed.

1) When the contact forces satisfy $\boldsymbol{\tau}_{out} = 0$, then (30) will result in

$$\dot{\mathbf{V}} = -\dot{\bar{\boldsymbol{q}}}_{e}^{1}(\boldsymbol{D}(t) - \boldsymbol{\alpha}\boldsymbol{H}(t)) - \boldsymbol{D}_{0}(t))\dot{\bar{\boldsymbol{q}}}_{e} - \bar{\boldsymbol{q}}_{e}^{\mathrm{T}}(\boldsymbol{\alpha}\dot{\boldsymbol{K}}(t)) - \frac{1}{2}\dot{\boldsymbol{K}}(t) - \frac{1}{2}\boldsymbol{\alpha}\dot{\boldsymbol{D}}(t) + \boldsymbol{\alpha}\dot{\boldsymbol{D}}_{0}(t))\bar{\boldsymbol{q}}_{e}.$$
(31)

To ensure that $\dot{V} < 0$, the following set of inequalities must be satisfied

$$\begin{cases} \boldsymbol{D}(t) - \alpha \boldsymbol{H}(t) - \frac{1}{2} \dot{\boldsymbol{H}}(t) > 0, \\ \alpha \dot{\boldsymbol{K}}(t) - \frac{1}{2} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \alpha \dot{\boldsymbol{D}}(t) + \frac{1}{2} \alpha \dot{\boldsymbol{D}}_{0}(t) > 0. \end{cases}$$
(32)

The energy function V is positive definite and \dot{V} is negative definite, then when $\bar{q}_e \to \infty$, $\bar{q}_e \to \infty$, $V \to \infty$ is unbounded. When $t \to \infty$, $\bar{q}_e \to 0$, $\bar{q}_e \to 0$, the closed-loop system is globally asymptotically stable. Thus, the closed-loop system (23) is stable in the noncontact phase.

2) When the contact forces satisfy $\tau_{out} \neq 0$, if the inequalities in (32) still hold, then

$$\dot{\mathbf{V}} \leq \left(\dot{\boldsymbol{q}}_e + \alpha \boldsymbol{\bar{q}}_e\right)^{\mathrm{T}} \boldsymbol{\tau}_{out}.$$
(33)

Then,

$$\mathbf{V} \leq \mathbf{V}(0) + \int_0^t (\dot{\bar{\boldsymbol{q}}}_e + \alpha \bar{\boldsymbol{q}}_e)^{\mathrm{T}} \boldsymbol{\tau}_{out} d\rho.$$
(34)

The above inequality (34) is called the dissipation function [31]. Dissipation ensures stable behaviour of the system in the contact phase.

Based on the above analysis, the following must hold

$$\begin{cases} \boldsymbol{\alpha} > 0, \\ \boldsymbol{K}(t) + \boldsymbol{\alpha} \boldsymbol{D}(t) - \boldsymbol{\alpha}^{2} \boldsymbol{H}(t) - 2\boldsymbol{\alpha} \boldsymbol{D}_{0}(t) > 0, \\ \boldsymbol{D}(t) - \boldsymbol{\alpha} \boldsymbol{H}(t) - \boldsymbol{D}_{0}(t) > 0, \\ \boldsymbol{\alpha} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \dot{\boldsymbol{K}}(t) - \frac{1}{2} \boldsymbol{\alpha} \dot{\boldsymbol{D}}(t) + \boldsymbol{\alpha} \dot{\boldsymbol{D}}_{0}(t) > 0. \end{cases}$$
(35)

The closed-loop system (23) is stable. This completes the proof. $\hfill \Box$

Remark 1: The inequalities $\alpha > 0$ and $\beta(t) = \mathbf{K}(t) + \alpha \mathbf{D}(t) - \alpha^2 \mathbf{H}(t) - \alpha \mathbf{\dot{H}}(t) > 0$ guarantee the positive definiteness of the Lyapunov energy function. To better adapt to changes in the environment, the update law for the stiffness matrix $\mathbf{K}(t)$ should be selected in accordance with the specific terrain.

Remark 2: In this paper, the variable impedance control performance in the stance phase and the swing phase is analysed; however, a more suitable controller for the swing phase still needs to be designed.

5. NUMERICAL SIMULATION

To verify the effectiveness of the proposed variable impedance control scheme for a single leg of a quadruped robot in the stance phase, a simulation model was built in MATLAB. The simulation model is shown in Fig. 3.

As shown in Fig. 3, the simulation model is mainly composed of five parts: the desired gait, the control law, the controlled plant, the force estimator, and the output module. In Fig. 3, the desired gait is described by (36) below. With the desired gait, the kinematics and dynamics models and the proposed variable impedance control scheme, the control law for the closed-loop system can be expressed as shown in (23). Through the control law module, the control torque is obtained and transmitted to the controlled plant (7); then, the actual position and attitude of the system can be acquired. The foot follows a simple cycloid trajectory, as follows:

$$\begin{cases} x = (x_f - x_s)\frac{\sigma - \sin\sigma}{2\pi} + x_s, \\ z = h\frac{1 - \cos\sigma}{2} + z_s, \end{cases} \quad \sigma = \frac{2\pi t}{T_s}. \tag{36}$$

In this paper, only x and z are planned; the trajectory parameters, the structural parameters of the single leg and the variable impedance control parameters are given in Table 1.

The closed-loop system (23) was simulated in MAT-LAB. Under the assumption that the contact forces are $\boldsymbol{\tau}_{out} = \begin{bmatrix} 0, 0, 0 \end{bmatrix}^{\mathrm{T}}$, the position response curve, position error curve, actuation torque response curve and contact force estimation curve in a single cycle are illustrated in Figs. 4-7.

It can be seen from Figs. 4 and 5 that when the contact forces are $\boldsymbol{\tau}_{out} = \begin{bmatrix} 0, 0, 0 \end{bmatrix}^{T}$, the quadruped robot starts from -0.1 m to 0.1 m in x and lifts its leg up 0.15 m in z at -0.482 m. When the variable impedance control scheme is used, the quadruped robot can track the desired trajectory well. In this paper, since only x and z of the foot are planned, y is always maintained at -0.15 m. Moreover, 0



Fig. 3. Simulation model of variable impedance control.

Parameter	Symbol	Value
Period	Т	1 s
Swing period	T_s	0.25 s
Starting point	x _s	-0.1 m
Terminal point	x_f	0.1 m
Initial height	Z_s	-0.482 m
Leg lift height	h	0.15 m
Hip length	l_1	0.15 m
Thigh length	l_2	0.35 m
Calf length	l_3	0.382 m
Hip mass	m_1	0.1 kg
Thigh mass	m_2	0.3 kg
Calf mass	m_3	0.4 kg
Variable inertia matrix	$\boldsymbol{H}(t)$	$\frac{diag(0.00025 + 0.0001*}{\sin(\frac{\pi}{2}*t))}$
Variable damping matrix	$\boldsymbol{D}(t)$	$diag(0.05+0.01*\sin(\frac{\pi}{2}*t))$
Variable stiffness matrix	$\boldsymbol{K}(t)$	$diag(4+0.01*\sin(\frac{\pi}{2}*t))$
Constant	α	2

 Table 1. Trajectory parameters, structural parameters, and variable impedance control parameters.



Fig. 4. Position response curve under variable impedance control.



Fig. 5. Position error response curve under variable impedance control.



Fig. 6. Actuation torque response curve under variable impedance control.



Fig. 7. Force estimation curve under variable impedance control.

0.25 s is the swing period, and 0.25-1 s is the stance phase. The controller presented in this paper is mainly designed for the stance phase. In the future, it will be necessary to further adjust the variable impedance control parameters or improve the design of the controller to overcome the problems of the large swing period error and driving torque fluctuations observed in Figs. 5 and 6. Fig. 7 shows the force estimation curve, from which it is obvious that the force estimator designed in this paper can successfully estimate the foot forces.

For contact forces of $\boldsymbol{\tau}_{out} = \begin{bmatrix} 0, 0, 0.3 \end{bmatrix}^{T}$ in the stance phase, the closed-loop system (23) was further simulated to obtain the corresponding position response curve, position error curve, driving torque response curve and force estimation curve in a single period, which are shown in Figs. 8-11.

The simulation results in Figs. 4-7 and Figs. 8-11 show that the foot end of the quadruped robot can track the desired trajectory and that the position tracking error converges to 0. The variable impedance control of the closedloop system is stable. In contrast to Fig. 4, the contact



Fig. 8. Position response curve under variable impedance control.



Fig. 9. Position error response curve under variable impedance control.



Fig. 10. Actuation torque response curve under variable impedance control.

force in Fig. 8 is not zero in the stance phase, and the variable impedance controller has to adjust the foot position to ensure that the foot contact force is stable at the desired value. As seen by comparing Fig. 6 and Fig. 10, due to the different external forces, the value of the actuation torque



Fig. 11. Force estimation curve under variable impedance control.

will change accordingly. The simulation results in Figs. 7 and 11 further illustrate that the proposed force estimator based on a nonlinear disturbance observer is feasible. Taken together, Figs. 4-7 and Figs. 8-11 show that the theoretical analysis is consistent with the simulation results.

6. CONCLUSION

Currently, quadruped robots are attracting increasing attention in patrol, rescue and home service applications due to various advantages, such as their adaptability to rough terrain. One important issue in the field of quadruped robots is the stability of their motion, especially when such a robot interacts with its environment, resulting in impact. Impedance control can stabilize the desired motion and the interaction forces during a given operation task. In this paper, variable impedance control based on the operation space is studied to ensure the stability of the system. However, the time-varying impedance control parameters continuously inject potential energy into the system, making traditional stability analysis methods no longer applicable. Instead, the stability of variable impedance control in the operation space is analysed by applying a new Lyapunov function. Moreover, by designing a force estimator acting in the operation space, the installation of force sensors is avoided. The design process and the convergence of the contact force tracking error are analysed in detail. The feasibility of the proposed variable impedance controller and the nonlinear contact force estimator acting in the operation space is demonstrated through numerical simulations.

CONFLICT OF INTEREST

The authors declare that there is no competing financial interest or personal relationship that could have appeared to influence the work reported in this paper.

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