# An Improved Result on Stability Analysis of Delayed Load Frequency Control Power Systems

Shiyu Jiao, Jianwei Xia, Zhen Wang\* 6, Xiangyong Chen\*, Jing Wang, and Hao Shen

**Abstract:** This paper investigates the stability of power systems with load frequency control considering time delays (constant and time-varying delays). A new criterion for ensuring the stability of the system is proposed on the basis of Lyapunov stability theory and a further strengthened inequality. Finally, taking a single-area load frequency control scheme with the proportional-integral controller as an example, according to the stability criterion obtained, the relationship between the maximum allowable delay and the gain of proportional-integral controller is discussed. Besides, in case studies, the effectiveness of our method is also demonstrated.

Keywords: Load frequency control, power systems, proportional-integral controller, time delays.

# 1. INTRODUCTION

Load frequency control (LFC), because of its excellent ability to maintain frequency and power exchange with neighboring areas at predetermined values, has been used in power systems for many years and has also laid the foundation for the development of smart grids [1]. Subsequently, with the continuous development of smart grids, a lot of research emerged [2-4]. The LFC scheme model with communication channels is usually regarded as a typical delayed system [5]. In the LFC system, due to the introduction of the communication channel, there will be a constant delay in the signal transmission process. However, in most LFC designs and operations, because the value of the delay is relatively small, this delay is often ignored. In addition, it is also worth noting that the data packed dropout and disordering may occur in communication, which may cause time-varying delays, and the unavailability of data due to communication failure can also be equivalent to time-varying delays.

At present, the time delay phenomenon has become one of the most important unreliable factors, it will weaken the dynamic performance of the system, and even cause the instability of the LFC scheme. This means that the control area does not meet the control standards, resulting in deviation, which will have a negative impact on the stable operation of power systems. Therefore, finding the maximum allowable delay is of great significance for power systems with LFC schemes to maintain the stable operation [6].

There are two common methods for determining the maximum allowable delay, each of which has advantages and limitations. First, the critical eigenvalues and eigenroots can be obtained according to the characteristic equation of the system to calculate the accurate maximum allowable delay directly [7]. However, when the scale of the system increases, this method is very time-consuming and inefficient. And because it can only deal with constant delays, an indirect method on the basis of Lyapunov stability theory gradually replaced it as the main method for calculating the maximum allowable delay [8-10]. Because of the fact that the second method can handle both constant and time-varying delays, it has become a mainstream method, although this method is somewhat conservative. In recent years, many excellent methods have been proposed for the stability analysis of systems with time delays [11–16]. However, how to apply these methods to the stability analysis of power systems with LFC scheme still

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arouses the attention of many scholars. In order to reduce the limitation and conservatism of the results, a further strengthened inequality technique that is less conservative than the Jensen and Wirtinger-based integral inequality is adopted in this paper [17].

Also a fascinating topic is what control methods should be adopted when studying LFC schemes, many excellent control strategies have always attracted much attention [18–21]. Among them, it is worth mentioning that proportional-integral (PI) control, because of its simplicity and high economic benefits [22–25]. In this paper, time delays are taken into account during the design phase of the PI-type controller, which is an issue that has been ignored in many previous literatures. Moreover, the relationship between the maximum allowable delay and the controller gain is analyzed.

In summary, this paper discusses the stability of the delayed LFC scheme equipped with a PI-type controller, the main contributions are divided into the following three aspects. 1) The single-area LFC scheme is more comprehensive by considering the influence of time delays when designing the PI-type controller. 2) The stability criterion is improved by constructing a suitable Lyapunov-Krasovskii functional (LKF) and an enhancement inequality technique. 3) The relation between the controller gain and the maximum allowable delay is discussed from two aspects: constant delay and time-varying delay, which provides auxiliary conditions for the design and adjustment of the LFC scheme. And by comparison, it not only proves the effectiveness of our method, but also validates that our results are less conservative than [9] and [26].

## 2. PROBLEM FORMULATION AND PRELIMINARIES

Consider time delays in the LFC scheme, as shown in Fig. 1 [1]. The LFC system model of a single-area can be expressed as follows:

$$\begin{cases} \tilde{x}(t) = \tilde{A}\tilde{x}(t) + \tilde{B}u(t) + \tilde{D}_w w(t), \\ \tilde{y}(t) = \tilde{C}\tilde{x}(t), \end{cases}$$
(1)

where

$$\tilde{x}^{T}(t) = \begin{bmatrix} \Delta f & \Delta P_{m} & \Delta P_{v} \end{bmatrix}$$



Fig. 1. The LFC scheme of the single-area.

Terminology	Meaning		
$\Delta P_d$	deviation of load		
$\Delta P_m$	deviation of generator mechanical output		
$\Delta P_{\nu}$	deviation of valve position		
$\Delta f$	deviation of frequency		
D	damping coefficient of generator		
М	moment of inertia of generator		
R	speed droop		
$T_{g}$	time constant of governor		
$T_{ch}$	time constant of turbine		
β	frequency bias factor		
ACE	area control error		

Table 1. The explanation of terminologies.

$$\tilde{y}(t) = ACE, \quad \tilde{A} = \begin{bmatrix} -\frac{D}{M} & \frac{1}{M} & 0\\ 0 & -\frac{1}{T_{ch}} & \frac{1}{T_{ch}} \\ -\frac{1}{RT_g} & 0 & -\frac{1}{T_g} \end{bmatrix},$$
$$\tilde{B} = \begin{bmatrix} 0\\ 0\\ \frac{1}{T_g} \end{bmatrix}, \quad \tilde{D}_w = \begin{bmatrix} -\frac{1}{M} \\ 0\\ 0 \end{bmatrix}, \quad \tilde{C} = \begin{bmatrix} \beta\\ 0\\ 0 \end{bmatrix}^T.$$

And explanations of some terms are shown in Table 1.

Since the power exchange phenomenon of the network tie-line does not occur in the LFC scheme of a single-area, the *ACE* can be expressed as

$$\tilde{y}(t) = ACE = \beta \Delta f, \tag{2}$$

in which,  $\beta > 0$ .

Considering *ACE* as the input if the controller, the form of PI-type LFC is as follows:

$$u(t) = -\mathcal{K}_P A C E - \mathcal{K}_I \int A C E, \qquad (3)$$

where  $\mathcal{K}_P$  and  $\mathcal{K}_I$  represent the proportional gain and the integral gain of the controller respectively. Defining  $y(t) \stackrel{\Delta}{=} \begin{bmatrix} \tilde{y}(t) & \int \tilde{y}(t) \end{bmatrix}^T$ ,  $\mathcal{K} \stackrel{\Delta}{=} \begin{bmatrix} \mathcal{K}_P & \mathcal{K}_I \end{bmatrix}$ , so expression (3) can be converted into

$$u(t) = -\mathcal{K}y(t).$$

However, it is not difficult to find from Fig. 1 that there are time delays in the transmission of the control signal, so time delays need to be taken into account in the controller, naturally

$$u(t) = -\mathcal{K}y(t - h(t)), \tag{4}$$

where h(t) represents time delays, and  $0 < h(t) < \overline{h}, \dot{h}(t) \le \mu \le 1$ .

Define the new state variable as  $x(t) \stackrel{\Delta}{=} [\Delta f \ \Delta P_m \ \Delta P_v \ \int ACE ]^T$ . For ease of understanding, the PI-type control problem can be translated into output feedback control problem. By introducing the controller (4) into system (1),

the closed-loop system in the following form can be obtained

$$\begin{cases} \dot{x}(t) = \mathcal{A}x(t) + \mathcal{A}_d x(t - h(t)) + \mathcal{D}_w w(t), \\ y(t) = \mathcal{C}x(t), \\ x(t) = \phi(t), \ t \in [-\bar{h}, 0], \end{cases}$$
(5)

where

$$\mathcal{A} = \begin{bmatrix} -\frac{D}{M} & \frac{1}{M} & 0 & 0\\ 0 & -\frac{1}{T_{ch}} & \frac{1}{T_{ch}} & 0\\ -\frac{1}{RT_g} & 0 & -\frac{1}{T_g} & 0\\ \beta & 0 & 0 & 0 \end{bmatrix}, \quad \mathcal{D}_w = \begin{bmatrix} -\frac{1}{M} \\ 0\\ 0\\ 0\\ 0 \end{bmatrix},$$
$$\mathcal{A}_d = \begin{bmatrix} 0 & 0 & 0 & 0\\ 0 & 0 & 0 & 0\\ -\frac{K_p\beta}{T_g} & 0 & 0 & -\frac{K_I}{T_g}\\ 0 & 0 & 0 & 0 \end{bmatrix}, \quad \mathcal{C} = \begin{bmatrix} \beta & 0\\ 0 & 0\\ 0 & 0\\ 0 & 1 \end{bmatrix}^T.$$

**Remark 1:** When dealing with unknown external load disturbances in power systems, it can be modeled as a nonlinear perturbation in the current and delayed state vectors [26].

$$\mathcal{D}_w w(t) = g(x(t), x(t-h(t))), \tag{6}$$

meeting the following condition

$$||g(\cdot)|| \le \varepsilon ||x(t)|| + \theta ||x(t-h(t))||, \tag{7}$$

in which  $\varepsilon$  and  $\theta$  are known non-negative scalars. A more generalized form of the condition is adopted, as follows:

$$g(\cdot)^{T}g(\cdot) \leq \varepsilon^{2} x^{T}(t) E^{T} E x(t) + \theta^{2} x^{T}(t-h(t)) F^{T} F x(t-h(t)), \qquad (8)$$

where *E* along with *F* are both known constant matrices with appropriate dimensions. The non-negative scalars  $\varepsilon$ ,  $\theta$  and the matrices *E*, *F* can be used to quantify the impact of load disturbances on power systems.

**Lemma 1** [17]: Given a matrix  $Z(\in \mathbb{R}^{n \times n}) > 0$ , if there exists any matrix  $H_i \in \mathbb{R}^{n \times m}$   $(i = 0, 1, 2, \dots, N)$  and a vector  $\delta \in \mathbb{R}^m$ , for any continuous and differentiable function  $x(\cdot)$  from [a,b] to  $\mathbb{R}^n$ , and integer  $N \in \mathbb{N}$ , the following inequality holds

$$-\int_{a}^{b} \dot{x}^{T}(s) Z \dot{x}(s) ds$$
  
$$\leq \sum_{k=0}^{N} [2\sigma_{N}^{T} \eta_{N}^{T}(k) H_{k} \delta + \frac{b-a}{(2k+1)} \delta^{T} H_{k}^{T} Z^{-1} H_{k} \delta], \quad (9)$$

where

$$\boldsymbol{\sigma}_{N} \triangleq \begin{cases} \begin{bmatrix} x^{T}(b) \ x^{T}(a) \end{bmatrix}^{T}, & N = 0, \\ \begin{bmatrix} x^{T}(b) \ x^{T}(a) \ \frac{1}{b-a} \Delta_{0}^{T} \ \cdots \ \frac{1}{b-a} \Delta_{N-1}^{T} \end{bmatrix}^{T}, \\ N > 0, \end{cases}$$

$$\begin{split} \eta_N(k) &\triangleq \begin{cases} [I \ -I], & N = 0, \\ [I \ (-1)^{k+1}I \ \tau_{Nk}^0 I \ \cdots \ \tau_{Nk}^{N-1}I], & N > 0, \end{cases} \\ \tau_{Nk}^i &\triangleq \begin{cases} -(2i+1)(1-(-1)^{k+i}), & i \le k, \\ 0, & i \ge k+1, \end{cases} \\ \Delta_k &\triangleq \int_a^b Y_k(s)x(s)ds, \end{cases} \\ Y_k(s) &\triangleq (-1)^k \sum_{e=0}^k \left[ (-1)^e \binom{k}{e} \binom{k+e}{e} \right] \left( \frac{s-a}{b-a} \right)^e. \end{split}$$

**Remark 2:** The stability of the LFC scheme is extremely important for the normal operation of power systems. When analyzing the stability of the LFC scheme with time delays, advanced inequality technology plays an important role. Compared with common inequalities, Jensen inequality and Wirtinger-based integral inequality, in order to effectively reduce the conservatism, a low-conservative inequality is applied this paper. Interestingly, as  $\delta$  introduces more state variables, the less conservative it is.

#### 3. MAIN RESULTS

To simplify the following calculation process, some relevant explanations are given here.

$$\begin{split} \Omega &= \begin{bmatrix} I_n & -I_n & 0 & 0\\ I_n & I_n & -2I_n & 0\\ I_n & -I_n & 0 & -6I_n \end{bmatrix},\\ \overline{\boldsymbol{\omega}}(t) &= \begin{bmatrix} \boldsymbol{\omega}_1^T(t) & \boldsymbol{\omega}_2^T(t) & \boldsymbol{\sigma}_3^T(t) \end{bmatrix}^T,\\ \overline{\boldsymbol{\omega}}_1(t) &= \begin{bmatrix} x(t)\\ x(t-h(t))\\ x(t-\bar{h})\\ g(\cdot) \end{bmatrix}, \quad P &= \begin{bmatrix} P_{11} & P_{12} & P_{13}\\ * & P_{22} & P_{23}\\ * & * & P_{33} \end{bmatrix},\\ \overline{\boldsymbol{\omega}}_2(t) &= \begin{bmatrix} \frac{1}{h(t)} \int_{t-h(t)}^t \int_{t-h(t)}^t x(s) ds\\ \frac{1}{h(t)} \int_{t-h(t)}^t (2\frac{s-t+h(t)}{h(t)} - 1)x(s) ds \end{bmatrix},\\ \overline{\boldsymbol{\omega}}_3(t) &= \begin{bmatrix} \frac{1}{h-h(t)} \int_{t-\bar{h}}^{t-h(t)} (2\frac{s-t+\bar{h}}{d-h(t)} - 1)x(s) ds\\ \frac{1}{h-h(t)} \int_{t-\bar{h}}^{t-h(t)} (2\frac{s-t+\bar{h}}{d-h(t)} - 1)x(s) ds \end{bmatrix},\\ Z_n &= \operatorname{diag}\{Z, 3Z, 5Z\},\\ c_i &= \begin{bmatrix} 0_{n\times(i-1)n} & I_n & 0_{n\times(9-i)n} \end{bmatrix}, \quad i = 1, 2, \dots, 9,\\ l_i &= \begin{bmatrix} 0_{n\times(i-1)n} & I_n & 0_{n\times(8-i)n} \end{bmatrix}, \quad i = 1, 2, \dots, 8. \end{split}$$

**Theorem 1:** For given scalars  $\varepsilon \ge 0$ ,  $\theta \ge 0$ ,  $\lambda \ge 0$ , assuming  $Q_2 > 0$ ,  $Q_1 > 0$ , P > 0, Z > 0, and any matrices  $\tilde{M}_1^{3\times9}$ ,  $\tilde{M}_2^{3\times9}$ , the system (5) is asymptotically stable if the following conditions hold

$$\Theta_1 = \begin{bmatrix} \tilde{\Gamma}(0) & \tilde{M}_2^T \\ * & -Z_n \end{bmatrix}_{12 \times 12} < 0, \tag{10}$$

$$\Theta_2 = \begin{bmatrix} \tilde{\Gamma}(\bar{h}) & \tilde{M}_1^T \\ * & -Z_n \end{bmatrix}_{12 \times 12} < 0, \tag{11}$$

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where

$$\begin{split} \tilde{\Gamma} &= \sum_{i=1}^{8} \tilde{\Gamma}_{i}, \\ \tilde{\Gamma}_{1} &= sym\{c_{1}^{T}P_{11}\mathcal{A}c_{1} + c_{1}^{T}P_{11}\mathcal{A}_{d}c_{2} + c_{1}^{T}P_{11}c_{4}, \\ &+ c_{1}^{T}P_{12}c_{1} - c_{1}^{T}P_{12}c_{3} + c_{1}^{T}P_{13}c_{1}\} - \varepsilon c_{4}^{T}Ic_{4}, \\ \tilde{\Gamma}_{2} &= sym\{(\bar{h} - h(t))c_{7}^{T}P_{12}^{T}(\mathcal{A}c_{1} + \mathcal{A}_{d}c_{2} + c_{4}) \\ &+ h(t)c_{5}^{T}P_{12}^{T}(\mathcal{A}c_{1} + \mathcal{A}_{d}c_{2} + c_{4})\} + \varepsilon \varepsilon^{2}c_{1}^{T}E^{T}Ec_{1}, \\ \tilde{\Gamma}_{3} &= sym\{(\bar{h} - h(t))c_{8}^{T}P_{13}^{T}(\mathcal{A}c_{1} + \mathcal{A}_{d}c_{2} + c_{4}), \\ &+ h(t)c_{6}^{T}P_{13}^{T}(\mathcal{A}c_{1} + \mathcal{A}_{d}c_{2} + c_{4})\} + \varepsilon \theta^{2}c_{2}^{T}F^{T}Fc_{2}, \\ \tilde{\Gamma}_{4} &= sym\{[h(t)c_{5}^{T} + (\bar{h} - h(t))c_{7}^{T}]P_{22}(c_{1} - c_{3})\}, \\ \tilde{\Gamma}_{5} &= sym\{[h(t)c_{6}^{T} + (\bar{h} - h(t))c_{8}^{T}]P_{23}^{T}(c_{1} - c_{3})\}, \\ \tilde{\Gamma}_{6} &= sym\{[h(t)c_{6}^{T} + (\bar{h} - h(t))c_{7}^{T}P_{23}c_{1} + h(t)c_{5}^{T}P_{23}c_{1} \\ &+ (\bar{h} - h(t))c_{8}^{T}P_{33}c_{1} + h(t)c_{5}^{T}P_{23}c_{1} \\ &+ (\bar{h} - h(t))c_{8}^{T}P_{33}c_{1} + h(t)c_{5}^{T}P_{23}c_{1} \\ &+ (\bar{h} - h(t))c_{8}^{T}P_{33}c_{1} + h(t)c_{6}^{T}P_{33}c_{1}\}, \\ \tilde{\Gamma}_{7} &= c_{1}^{T}(Q_{1} + Q_{2})c_{1} - (1 - \mu)c_{2}^{T}Q_{1}c_{2} \\ &- c_{3}^{T}Q_{2}c_{3} + sym\{\bar{h}\tilde{\Pi}_{1}^{T}\Omega^{T}M_{1} + \bar{h}\tilde{\Pi}_{2}^{T}\Omega^{T}M_{2}\}, \\ \tilde{\Gamma}_{8} &= \bar{h}c_{1}^{T}\mathcal{A}^{T}Zc_{9} + \bar{h}c_{2}^{T}\mathcal{A}_{d}^{T}Zc_{9} + \bar{h}c_{4}^{T}Zc_{9} + c_{9}^{T}Zc_{9}. \end{split}$$

Proof: Choose the appropriate LKF

$$\mathcal{V}(t) = \mathcal{V}_1(t) + \mathcal{V}_2(t) + \mathcal{V}_3(t), \qquad (12)$$

$$\mathcal{V}_1(t) = \boldsymbol{\rho}^T(t) \boldsymbol{P} \boldsymbol{\rho}(t), \tag{13}$$

$$\mathcal{V}_2(t) = \int_{t-h(t)}^t x^T(s) Q_1 x(s) ds$$

$$+ \int_{t-h(t)}^t x^T(s) Q_2 x(s) ds \qquad (14)$$

$$+\int_{t-\bar{h}} x^T(s)Q_2x(s)ds,\tag{14}$$

$$\mathcal{V}_3(t) = \bar{h} \int_{-\bar{h}}^0 \int_{t+\gamma}^t \dot{x}^T(s) Z \dot{x}(s) ds d\gamma, \tag{15}$$

where

$$\rho(t) = \left[ x^T(t) \quad \int_{t-\bar{h}}^t x^T(s) ds \quad \int_{t-\bar{h}}^t (2\frac{s-t+\bar{h}}{\bar{h}} - 1) x^T(s) ds \right]^T.$$

Take the derivative of the above functional, and the following expressions hold

$$\dot{\mathcal{V}}_1(t) = sym\{\boldsymbol{\rho}^T(t)P\dot{\boldsymbol{\rho}}(t)\},\tag{16}$$
$$\dot{\mathcal{V}}_2(t) < x^T(t)O_1x(t) + x^T(t)O_2x(t)$$

$$-x^{T}(t-\bar{h})Q_{2}x(t-\bar{h}) -(1-\mu)x^{T}(t-h(t))O_{1}x(t-h(t)),$$
(17)

$$\dot{\mathcal{V}}_{3}(t) = \bar{h}^{2} \dot{x}^{T}(t) Z \dot{x}(t) - \bar{h} \int_{t-\bar{h}}^{t} \dot{x}^{T}(s) Z \dot{x}(s) ds.$$
(18)

Setting N = 2, and according to Lemma 1, the following expressions hold

$$-\bar{h}\int_{t-\bar{h}}^{t}\dot{x}^{T}(s)Z\dot{x}(s)ds$$

$$= -\bar{h} \int_{t-\bar{h}}^{t-h(t)} \dot{x}^{T}(s) Z \dot{x}(s) ds - \bar{h} \int_{t-h(t)}^{t} \dot{x}^{T}(s) Z \dot{x}(s) ds,$$
(19)

$$-\bar{h} \int_{t-h(t)} \dot{x}^{T}(s) Z \dot{x}(s) ds$$

$$\leq \boldsymbol{\varpi}^{T}(t) [\bar{h}(sym\{\Pi_{1}^{T} \boldsymbol{\Omega}^{T} M_{1}\} + h(t)M_{1}^{T} Z_{n}^{-1} M_{1})] \boldsymbol{\varpi}(t), \qquad (20)$$

$$-\bar{h} \int_{t-\bar{h}}^{t-h(t)} \dot{x}^{T}(s) Z \dot{x}(s) ds$$

$$\leq \boldsymbol{\varpi}^{T}(t) [\bar{h}(sym\{\Pi_{2}^{T} \boldsymbol{\Omega}^{T} M_{2}\} + (\bar{h} - h(t))M_{1}^{T} Z_{n}^{-1} M_{2})] \boldsymbol{\varpi}(t), \qquad (21)$$

where  $M_1$  and  $M_2$  are matrices with suitable dimensions, and

$$\begin{split} \tilde{M}_{1} &= \begin{bmatrix} M_{1} & 0_{3\times 1} \end{bmatrix}, \quad \tilde{M}_{2} = \begin{bmatrix} M_{2} & 0_{3\times 1} \end{bmatrix}, \\ \Pi_{1} &= \begin{bmatrix} l_{1}^{T} & l_{2}^{T} & l_{5}^{T} \end{bmatrix}^{T}, \quad \Pi_{2} = \begin{bmatrix} l_{2}^{T} & l_{3}^{T} & l_{7}^{T} & l_{8}^{T} \end{bmatrix}^{T}, \\ \tilde{\Pi}_{1} &= \begin{bmatrix} c_{1}^{T} & c_{2}^{T} & c_{5}^{T} & c_{6}^{T} \end{bmatrix}^{T}, \quad \tilde{\Pi}_{2} = \begin{bmatrix} c_{2}^{T} & c_{3}^{T} & c_{7}^{T} & c_{8}^{T} \end{bmatrix}^{T}. \end{split}$$

For any  $\varepsilon > 0$ , the inequality hold

$$-\varepsilon g^{T}(\cdot)g(\cdot) + \varepsilon \varepsilon^{2} x^{T}(t) E^{T} E x(t) + \varepsilon \theta^{2} x^{T}(t-h(t)) F^{T} F x(t-h(t)) \ge 0.$$
(22)

By summarizing the above calculation, it is not difficult to get the following inequality

$$\dot{\mathcal{V}}(t) \leq \boldsymbol{\varpi}^{T}(t) \{ \sum_{i=1}^{7} \Gamma_{i}(h(t)) + \bar{h}[h(t)M_{1}^{T}Z_{n}^{-1}M_{1} + (\bar{h} - h(t))M_{2}^{T}Z_{n}^{-1}M_{2}] \} \boldsymbol{\varpi}(t) + \bar{h}^{2}\dot{x}^{T}(t)Z\dot{x}(t),$$
(23)

where

$$\begin{split} &\Gamma_{1} = sym\{l_{1}^{T}P_{11}\mathcal{A}l_{1} + l_{1}^{T}P_{11}\mathcal{A}_{d}l_{2} + l_{1}^{T}P_{11}l_{4} \\ &+ l_{1}^{T}P_{12}l_{1} - l_{1}^{T}P_{12}l_{3} + l_{1}^{T}P_{13}l_{1}\} - \varepsilon l_{4}^{T}Il_{4}, \\ &\Gamma_{2} = sym\{(\bar{h} - h(t))l_{7}^{T}P_{12}^{T}(\mathcal{A}l_{1} + \mathcal{A}_{d}l_{2} + l_{4}) \\ &+ h(t)l_{5}^{T}P_{12}^{T}(\mathcal{A}l_{1} + \mathcal{A}_{d}l_{2} + l_{4})\} + \varepsilon \varepsilon^{2}l_{1}^{T}E^{T}El_{1}, \\ &\Gamma_{3} = sym\{(\bar{h} - h(t))l_{8}^{T}P_{13}^{T}(\mathcal{A}l_{1} + \mathcal{A}_{d}l_{2} + l_{4}) \\ &+ h(t)l_{6}^{T}P_{13}^{T}(\mathcal{A}l_{1} + \mathcal{A}_{d}l_{2} + l_{4})\} + \varepsilon \theta^{2}l_{2}^{T}F^{T}Fl_{2}, \\ &\Gamma_{4} = sym\{[h(t)l_{5}^{T} + (\bar{h} - h(t))l_{7}^{T}]P_{22}(l_{1} - l_{3})\}, \\ &\Gamma_{5} = sym\{[h(t)l_{6}^{T} + (\bar{h} - h(t))l_{8}^{T}]P_{23}^{T}(l_{1} - l_{3})\}, \\ &\Gamma_{6} = sym\{(\bar{h} - h(t))l_{7}^{T}P_{23}l_{1} + h(t)l_{5}^{T}P_{23}l_{1} \\ &+ (\bar{h} - h(t))l_{8}^{T}P_{3}l_{1} + h(t)l_{6}^{T}P_{3}l_{1}\}, \\ &\Gamma_{7} = l_{1}^{T}(Q_{1} + Q_{2})l_{1} - (1 - \mu)l_{2}^{T}Q_{1}l_{2} - l_{3}^{T}Q_{2}l_{3} \\ &+ sym\{\bar{h}\tilde{\Pi}_{1}^{T}\Omega^{T}M_{1} + \bar{h}\tilde{\Pi}_{2}^{T}\Omega^{T}M_{2}\}. \end{split}$$

According to Schur complement, the expressions  $\Theta_1$  and  $\Theta_2$  hold, since the condition  $\tilde{\Gamma}(0) + \bar{h}^2 M_2^T Z_n^{-1} M_2 < 0$ and the condition  $\tilde{\Gamma}(\bar{h}) + \bar{h}^2 M_1^T Z_n^{-1} M_1 < 0$  can guarantee the establishment of the condition  $\tilde{\Gamma}(h(t)) +$ 

Table 2. Parameters of single-area LFC system.

Parameter	β	R	D	M(s)	$T_{ch}(s)$	$T_g(s)$
Area1	21	0.05	1.0	1.0	0.3	0.1

Table 3. The maximum allowable delay  $\bar{h}$  for LFC system when  $\mu = 0$ .

Methods		$\mu = 0$			
$\mathcal{K}_P$	$\mathcal{K}_I$	Theorem1	[ <mark>9</mark> ]	[26]	
0	0.2	11.70	7.33	6.69	
0	0.4	6.15	3.38	3.12	
0	0.6	4.17	2.04	1.91	
0.1	0.2	10.96	7.79	6.94	
0.1	0.4	5.83	3.61	3.29	
0.1	0.6	4.05	2.19	2.02	

Table 4. The maximum allowable delay  $\bar{h}$  for LFC system when  $\mu = 0.9$ .

Methods		$\mu = 0.9$			
$\mathcal{K}_P$	$\mathcal{K}_I$	Theorem1	[ <mark>9</mark> ]	[26]	
0	0.2	9.98	6.43	6.25	
0	0.4	4.44	2.91	2.85	
0	0.6	2.80	1.71	1.68	
0.1	0.2	9.17	6.59	5.93	
0.1	0.4	4.31	3.11	2.87	
0.1	0.6	2.83	1.84	1.75	

 $\bar{h}[h(t)M_1^T Z_n^{-1} M_1 + (\bar{h} - h(t))M_2^T Z_n^{-1} M_2] < 0 \text{ for } \forall h(t) \in [0, \bar{h}].$  Based on Schur complement, inequalities (10) and (11) can be derived. This completes the proof of Theorem 1.

### 4. CASE STUDIES

In this section, taking a single-area LFC system as an example, the superiority of our results is proved by comparison with the previous literature [9] and [26]. The parameters of the system are shown in Table 2.

Suppose time delays h(t) as two cases : constant or time-varying delays ( $\mu = 0/0.9$ ). In these two cases, analyze the maximum allowable delay for the normal operation of the system. Consider the load disturbance  $\mathcal{D}_w w(t)$ as formula (8) and set the parameters  $\varepsilon = 0$ ,  $\theta = 0$ , E = $F = 0.1I_4$ . With the help of Matlab, the maximum allowable delay under diverse values of controller gains ( $\mathcal{K}_P$ ,  $\mathcal{K}_I$ ) can be solved. The specific data are shown in Table 3 and Table 4.

It can be found from Table 3 that when considering h(t) as constant delay ( $\mu = 0$ ), under the same controller gain, the maximum allowable delay calculated by our method is greater than than the values in [9] and [26]. And Fig.



Fig. 2. The state response trajectory of LFC system.

2 shows the state response of the single-area LFC system at  $\mathcal{K}_P = 0$ ,  $\mathcal{K}_I = 0.2$  when  $\mu = 0$ . It is proved that the system finally achieves asymptotic stability. Table 4 shows the maximum allowable delay obtained under different controller gains when h(t) is time-varying delay ( $\mu = 0.9$ ). By comparing Table 3 and Table 4 longitudinally, it is not difficult to see that under the premise of ensuring the same controller gain, the maximum allowable upper bound of constant delay is generally higher than that of time-varying delay.

In addition, according to Table 3 and Table 4, it is obvious that compared with literature [9] and [26], whether h(t) is constant delay ( $\mu = 0$ ) or time-varying delay ( $\mu = 0.9$ ), the maximum allowable delay obtained by our method is larger. This proves that our results are less conservative than those in [9] and [26], and the effectiveness and superiority of our method are also confirmed, directly.

#### 5. CONCLUSIONS

In this paper, the stability of frequency load control scheme with time delays has been studied. By Lyapunov stability theory and an improved inequality technology, the stability criterion has been further optimized. In addition, the maximum allowable delay under different controller gain values has been calculated, and the relationship between the two has been discussed. By comparing the examples, the validity and excellence of our method have been fully confirmed whether under the constant delay or time-varying delay.

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