#### **REVIEW**



# **Rigid Precision Reducers for Machining Industrial Robots**

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#### **Abstract**

Machining robots are expected to signifcantly change existing production systems in the near future. The quality of the machining process with robots is mainly governed by the accuracy and stifness of the robots. Therefore, a precision reducer for the robot joint is an important component that governs the accuracy of machining robots. This paper presents a review of rigid precision reducers for machining robots. Initially, an overview of the machining robots and their features is introduced. The importance of a precision reducer as a component of a robot for machining is explored. A cycloid reducer is the best candidate among precision reducers, considering both the structural compliance and kinematic accuracy of the machining robots. This is followed by reviews of various cycloid reducers and their operating principles. The design issues of the cycloid reducer for performance improvement are then presented. Additionally, the methodology and analysis to assess the performance of the cycloid reducers are discussed. The machining and fault detection of a cycloid reducer are briefy addressed. Finally, other applications of cycloid reducers are introduced.

**Keywords** Rigid precision reducer · Cycloid reducer · High stifness reducer · Machining robot · Industrial robot

## **1 Introductions**

Smart manufacturing is a strong industrial driver for reshaping the current competitive landscape and establishing new market leaders. Manufacturing has evolved to become more automated, computerized, and complex. Smart manufacturing can be defned as a set of technologies that employ a computer-integrated process, high levels of adaptability, rapid design changes, digital information technology, and more fexible technical workforce training [[1\]](#page-13-0). Smart manufacturing allows building of new value-added processes and networks to improve and optimize the fexibility, adaptability, and efficiency of business processes.

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Cyber-physical systems (CPS)—a new trend in smart manufacturing-related research—integrates the cyber world and the dynamic physical world by combining computing, communication, and control [\[2\]](#page-13-1). The main feature is the integration of cyber-physical systems to enable intersystem communication and self-controlled system operation [\[3](#page-13-2)]. Moreover, smart factories integrated with CPS can be easily applied to a big data analytics platform to collect data on industrial processes [[4](#page-13-3), [5\]](#page-13-4). This would enable physical entities in smart manufacturing to be controlled and supervised in a safe, efficient, and reliable manner  $[6]$  $[6]$ .

Additive manufacturing (AM) or three-dimensional (3D) printing is considered as a CPS owing to the interlacing of virtual production with physical production [\[7\]](#page-14-0). AM has introduced new production methods to design, manufacture, and distribute to end users. Moreover, AM plays an important role in smart manufacturing owing to its various benefts such as time and material savings, rapid prototyping, high efficiency, and decentralized production  $[8, 9]$  $[8, 9]$  $[8, 9]$  $[8, 9]$ .

Although AM technologies have experienced substantial growth in recent decades, machining manufacturing (MM) has played a dominant role in manufacturing. AM is highly suitable for green manufacturing owing to its advantages such as material saving and waste minimization [[9\]](#page-14-2). However, it is not relevant to the manufacturing of large-scale

products with metals and high-standard surface fnishing [\[10\]](#page-14-3). In addition, AM has limitations in terms of material use, whereas MM is easy to apply because of the variety of materials available on the market. Finally, metal parts manufactured by AM require additional trimming with computer numerical control (CNC) machines in the case of a high precision finish [[11\]](#page-14-4).

Even though manufacturing automation has become extremely popular in the past 30 years, advances in AI and robots will ensure signifcant development in this feld [[12,](#page-14-5) [13\]](#page-14-6). AI is comparable to the brain in the automation process, with industrial machining robots being the body cells. In future factory automation, robots will play an important role in customizing the production process, including picking and placing, welding, painting, packaging and labeling, palletizing, and product inspection.

Machining robots are expected to signifcantly change existing production systems for a variety of reasons [[14](#page-14-7)]. Many industrial robots are being utilized in the machining process to increase productivity and reduce labor costs. In general, a typical machining robot system includes a serial robotic arm with a machining tool attached to the endeffector of the robot  $[15]$  $[15]$  $[15]$ . Compared to traditional machining systems, these machining robots can easily expand the workspace having additional mobile platforms attached [\[16](#page-14-9)]. The rise of machining robots in modern factories is undeniable. In particular, 78% of industrial robot productivity in the market is used in welding and handling operations, of which more than 40% is used in the automotive and metal processing industries [[17\]](#page-14-10). However, these machining robots have diverse applications because the tools mounted on the end-efector easily can be changed for diferent tasks, such as grinding and welding [\[18](#page-14-11)].

The quality of the machining process with robots is mainly governed by the accuracy and stifness of the robots [\[19\]](#page-14-12). There are three types of errors affecting the accuracy of a robot: errors due to the working environment, machining process errors, and robot self-dependent errors [[20\]](#page-14-13). The errors in a robot due to structural deformations of load-transmitting components, links, energy-transforming devices, or wear and nonlinear effects are hardly controllable [[13,](#page-14-6) [20](#page-14-13)]. However, some of the self-dependent errors in a robot can be compensated for by suitable control or dedicated calibration [[21\]](#page-14-14).

The precision reducer is an important component governing the accuracy of the machining robot. High-quality mechanical and electrical parts can be used to improve the accuracy of the robot to obtain the best precision. Among these, precision reducers used in joint actuators contribute signifcantly to the kinematic and nongeometrical position errors of robots [\[22](#page-14-15)].

This paper presents a review of rigid precision reducers of industrial robots for machining. Initially, an overview of the machining robots and their features are introduced. The importance of a precision reducer as a key component of a robot for machining is explored. A cycloid reducer is the best candidate among precision reducers, considering both the structural compliance and kinematic accuracy of the machining robots. Various cycloid reducers and their operating principles are reviewed. The design issues of the cycloid reducer for performance improvement are presented. Additionally, the methodology and analysis for assessing the performance of the cycloid reducers are discussed. Afterward, the machining and fault detection of a cycloid reducer are briefy presented. Finally, other applications of cycloid reducers are introduced.

### **2 Machining Robot and its Characteristics**

Prototypical machining robots—6-or 7-degree-of-freedom (DOF) structures of industrial robotic arms—have been used for various tasks such as milling, grilling, polishing, and cutting. An example of a 6-DOF robotic arm for the milling process is shown in Fig. [1](#page-1-0) [[23\]](#page-14-16). In this task, a machining tool is attached to the end-efector of an industrial robot. The tool can be conveniently changed on the spindle attached to the end-efector of the robot to execute other machining tasks [[16\]](#page-14-9). Various studies on machining robots are presented in Table [1](#page-2-0) [\[24](#page-14-17)].

The machining robot system offers many advantages over conventional CNC machines, as shown in Table [2](#page-2-1) [[25](#page-14-18), [26](#page-14-19)]. Primarily, the workspace of the machining robot is very large and can be shared owing to the high fexibility of their arms [\[27](#page-14-20)]. As a result, complex bulky 3D shapes, such as an aircraft, can be machined directly with the robot. Moreover, a tool path for its smoothness can be further optimized using

<span id="page-1-0"></span>

**Fig. 1** KUKA milling robot [[23](#page-14-16)] (open accessed)

Fields	<b>Process</b>	Product		
Foundries	Deburring, milling, routing, drilling, finishing	Mold and dies, casting		
Automotive	Milling, de-flashing, drilling, grinding, cutting	Engines, vehicle frame, body panels, bumpers, door- knobs, stamping dies, sand cores		
Aerospace	Grinding, drilling polishing, cutting	Turbine blades, wing segments, bulkheads, insulation		
Medical equipment	Grinding, polishing	Prosthesis		

<span id="page-2-0"></span>**Table 1** Applications of machining robot in diferent manufacturing felds [[24](#page-14-17)]

<span id="page-2-1"></span>**Table 2** Comparison between machining robots and CNC machines

Properties	Machining robots	CNC machines	
Overall			
Kinematic architecture	Serial	Cartesian	
Number of axes	$6+$	3 or 5	
Kinematic redundancy	At least 1-DOF	None	
Dynamic properties	Heterogeneous within the workspace	Homogeneous within the workspace	
Control algorithm	Point-to-point control Continues path control	Continues path control	
Error compensation	Mechanical: gravity compensators Control algorithms: Off-line and/or on-line	Not required	
<b>Actuator</b> feedback	Single or double encoder	Single encoder	
	Advantage	Disadvantage	
Workspace	Large	Limited	
Extendable capacity	Possible with extra actuators or mobile platform	Impossible	
Flexibility	High	Low	
Working objects	Highly flexible (easily machining oversize elements)	Limited (just for the limited size of elements)	
Complexity of trajectory	Any complex trajectory	Just suitable for 3/5 axes machining	
Machine operator workload	Any type of operation Several parts at one	Single or several similar operations One part at the time	
Maintenance	Simple	Complex	
User-friendly	Without programming knowledge	Need programming knowledge	
Competitive for 6-DOF robot Price		Competitive for 3-axes machine Expensive for a 5-axes machine	
Accuracy	$\pm 0.1$ to $\pm 1$ mm	$+0.005$ mm	
Repeatability	$\pm$ 0.03 to $\pm$ 0.3 mm	$\pm 0.002$ mm	
Mechanical compliance	Relatively low	Relatively high	
The relation between actuated and operation space	Non-linear	Linear	

redundant joint actuators, although trajectory planning is signifcantly complex compared to conventional CNC. Time and substantial effort are required to train operators who have little or no software knowledge can be saved by using the robot system.

Operational accuracy and structural stifness are crucial in machining robots. During processes with negligible machining forces, such as welding or routing, only kinematic position errors need to be considered [[28\]](#page-14-21). Compliance is very important in resisting the structural deformation caused by machining forces during the cutting or milling process [[29\]](#page-14-22). Some methods have been proposed to improve the operational accuracy and structural stifness of machining robots [[21](#page-14-14), [29\]](#page-14-22). The spindle of the robot is directly attached to its ffth joint to reduce the efect of the machining forces, which limits the fexibility and workspace of the robot [\[16](#page-14-9)]. Compliance compensation also helps to improve the performance of a machining robot with kinematic position and nongeometrical errors. Nevertheless, the control system of machining robots is extremely expensive and complicated because several high-resolution sensors and tracker-checking systems are required [\[30\]](#page-14-23).

Choice of proper components in the machining robot is crucial for reducing compliance issues. The ability and performance of robots are infuenced by the working environment, accuracy of the robot control system, and robot dependent errors (both kinetics and dynamics) [[20\]](#page-14-13). While factors such as the working environment and control system accuracy can be adjusted during robot operation [\[21,](#page-14-14) [22,](#page-14-15) [28](#page-14-21)], the robot-dependence errors need to be considered in the robot manufacturing process. Robot-dependence errors include geometrical and nongeometrical errors [\[20\]](#page-14-13). Most of the geometrical errors result from the quality of actuators located in the joints of the robot, which is a combination of motors and high-precision reducers [\[31](#page-14-24), [32\]](#page-14-25). Furthermore, nongeometrical errors are caused by stick–slip motion, hysteresis, and nonlinear deformation due to impact and cutting forces during robot operations [[20](#page-14-13), [33\]](#page-14-26). The robot joints need to be integrated with a reducer with a high torsional stifness and the ability for long-term operation in environments requiring high-speed performance [[34,](#page-14-27) [35\]](#page-14-28) to minimize nongeometrical impacts. Therefore, it is extremely important to choose a reducer with good structure, high torsional rigidity, high-speed operatability with a large payload ratio, and antivibration ability during machining [\[14](#page-14-7)]. Examples of reducers in the machining robot are compared in Table [3](#page-3-0) [\[34](#page-14-27), [35](#page-14-28)].

## **3 Overview of Rigid Precision Reducers Used in the Actuator of the Machining Robot**

Although several types of precision reducers are used in industrial robots [[36\]](#page-14-29), reducers with cycloidal disk and eccentric shaft are ideal for machining robots. Compared with a planetary reducer [\[37\]](#page-14-30), the cycloid reducer can easily achieve a high reduction ratio without occupying a large working space. Owing to its circular tooth profle and multiple contact points, the cycloid reducer can work more efficiently and accurately than the planetary reducer. As shown in Table [3](#page-3-0), the cycloid reducer has a much higher torque-toweight ratio than the planetary reducer in a similar reduction ratio range, while still working with maximum efficiency under high-speed conditions. There is the other possible option for a joint reducer with a harmonic drive [\[35](#page-14-28)]. However, the harmonic drive has a continuously deformed and moving thin-wall structure causing it to have low torsional rigidity. This makes it suitable for applications with low load capacity and narrow space. [[38\]](#page-14-31). The cycloid reducer is desirable for the machining robot under conditions requiring high accuracy and payload capacity, such as machining. Furthermore, the cycloid reducer is also known to exhibit high robustness and torsional stifness with low required maintenance [[34](#page-14-27)].

<span id="page-3-0"></span>**Table 3** Effects of the reducer on the performance of the machining robot [\[34,](#page-14-27) [35\]](#page-14-28)

Effects on perfor- mance of machin- ing robot	Performance index Cycloid reducer				Harmonic drive	Planetary reducer
		Nabtesco RV-25N	Spinea-Twinspin TS110	Sumitomo Fine Cyclo F2C- T <sub>155</sub>	CSG-25-160-2UJ- LW	Wittenstein-Alpha $SP+075MF$ (2 stage)
Payload capacity, speed	Transmission ratio Acceleraton/nomi- nal torques	1:108 612/245 Nm	1:119 244/122 Nm	1:118 417/167 Nm	1:100 204/87 Nm	1:100 105/84 Nm
	Torque-to-weight ratios	161/64 Nm/kg	64/32 Nm/kg	87/29 Nm/kg	208/79 Nm/kg	35/28 Nm/kg
	Efficiency and subjective dependency on operatiing condi- tions	87%, high (speed and torque)	74%, high (speed and torque)	87%, high (speed and torque)	84%, high (speed) and torque)	94%, low (speed and torque)
Accuracy; repeat- ability; fast accelaracy; smooth trajectory	Backlash	$<$ 1 Arcmin	$<$ 1 Arcmin	$< 0.75$ Arcmin	$\overline{0}$	4–6 Arcmin
	Lost motion	$<$ 1 Arcmin	$<$ 1 Arcmin	$< 0.75$ Arcmin	$<$ 1 Arcmin	4–6 Arcmin
	Maximum input speed		$4500$ rpm	8500 rpm	7500 rpm	8500 rpm
	Torsional rigidity	61 Nm/Arcmin	$>22$ Nm/Arcmin	25–41 Nm/Arc- min	$11-16$ Nm/Arc- min	10 Nm/Arcm

#### **3.1 Operating Principle**

Depending on the specifc requirements of the applications, the cycloid reducer may have various structures, as shown in Fig. [2](#page-4-0). The kinematic diagram of the cycloid reducer can be divided into fve types: 2K-H, K-H-V, rotate vector (RV), China Bearing Reducer (CBR), and outputpin-wheel. The 2K-H type cycloid reducer consists of a planet carrier (H) and two central gears (K), as illustrated in Fig. [2](#page-4-0)a [\[39\]](#page-14-32). However, the 2K-H type cycloid reducer may have a high reduction ratio but low efficiency because it always need at least two gear pairs to transfer movement between input and output  $[40]$  $[40]$ . Thus, the K-H-V type cycloid reducer was invented to overcome the drawback of the 2K-H type by introducing an equal angular mechanism (V). The K-H-V type cycloid reducer is compact, light-weight, and highly efficient (Fig. [2b](#page-4-0))  $[41]$  $[41]$ . The RV-type cycloid reducer using a combination of K-H and K-H-V cycloid reducers is proposed, as shown in Fig. [2c](#page-4-0), to further increase the reduction ratio and enhance the fexibility of the output input  $[36]$  $[36]$ . Recently, a CBR-type cycloid reducer (Fig. [2d](#page-4-0)) was suggested for minimizing the volume; the outer diameter of the cycloid reducer would be similar in size to the harmonic drive [[42\]](#page-14-35). The CBR-type cycloid reducer has disc connectors instead of the pin roller in the K-H-V-type cycloid reducer. Furthermore, an output-pin-wheel mechanism that works by switching functions between the output and housing using four cycloid disks, as shown in Fig. [2](#page-4-0)e, is proposed [[43](#page-14-36)]. The kinematic diagram of the output-pin-wheel reducer is similar to that of the K-H type. Furthermore, the installed size can be increased. In general, the RV type cycloid reducer is the most widely used option in industrial robots nowadays.

#### **3.2 Design Optimizations of the Cycloid Reducer**

Design optimization is key to improving the performance of cycloid reducers. The performance of the cycloid reducer can be improved by using materials, lubrication



<span id="page-4-0"></span>**Fig. 2** Operating principle of various cycloid reducers

methods, and design optimization. Various materials, from low-carbon steels to high-quality alloy steel, are used for cycloid reducers [[44](#page-14-37)]. Alloy steels with Mn and Cr are widely used because of their hardenability, high bending strength, and fatigue strength [[45\]](#page-15-0). Although Al is used in some parts of the harmonic drive owing to its lightweight, material choices for the cycloid reducer are limited because they have to guarantee structural stifness of the reducer. Oil, grease, and mixed lubrication methods for cycloid reducers have advanced over the years [[46,](#page-15-1) [47](#page-15-2)]. However, research related to the optimization of cycloid designs, since the frst appearance of the cycloid reducer (over 60 years ago) (structure, tooth profle, etc.), is still ongoing from [[48](#page-15-3)].

Design optimizations for cycloid reducers can be classifed into three main groups: generating method of tooth profle, modifcation or correlation of the profle, and structural design. Several methods have been proposed to generate a cycloid profle for manufacturability. Tooth profle modifcations or corrections are proposed to minimize the gaps between engaging elements. In particular, minimizing the initial gap (ideally 0) between engaged teeth pairs can signifcantly decrease transmission errors between output and input, which increases the accuracy of reducer. Lastly, most research related to reducer designs concentrates on the structural optimization of the cycloid reducer.

Methods for generating the cycloid tooth profile are divided into three main categories: circle enveloping [\[41,](#page-14-34) [49](#page-15-4)], transmitted coordinate systems [[50–](#page-15-5)[52](#page-15-6)], and instantvelocity center method [[53](#page-15-7), [54](#page-15-8)], as shown in Fig. [3.](#page-5-0) The circle-enveloping method generates a trace of a point of a circle rolling on the other fxed circle, which is called an epitrochoid curve, in coordinate system  $F_x$  ( $O_x$ ,  $X$ ,  $Y$ ) [\[41](#page-14-34)]. The transmitted coordinate system method generates the



(a) Circle-enveloping method



(b) Transmitted coordinate system method



(c) Instant-velocity center method.

<span id="page-5-0"></span>**Fig. 3** Generating methods of cycloid tooth profle



#### <span id="page-6-0"></span>**Table 4** Equations for cycloid tooth profle with diferent methods

Circle-enveloping method [\[41\]](#page-14-34) Transmitted coordinate system method [[56](#page-15-11)] Instant velocity center method [[53](#page-15-7)]

Point on epicycloid curve:

$$
\begin{cases}\nx_c = R\left(\sin\theta - \frac{R_b}{RZ_p}\sin Z_p\theta\right) \\
y_c = R\left(\cos\theta - \frac{R_b}{RZ_p}\cos Z_p\theta\right)\n\end{cases}
$$

Curvature radius of epicycloid curve:

$$
\rho_c = \frac{R/K (1 + K^2 - 2K \cos \theta_b)^{2/3}}{(1 + Z_p) \cos \theta_b - (1 + Z_p K^2)}
$$

 $(K=R_b/R)$ with:

*R:* Radius of the pin-roller distributed circle

 $R_b$ : Pitch radius of the pin-roller distributed circle

*Zp*: Number of pin-roller of the cycloid reducer

*θ*: Rotation angle of pin-roller around center  $O$ 

 $\theta_b$ : Rotation angle of pin-roller around center *Ox*

*R<sub>p</sub>*: radius of the pin-roller

Point on epicycloid curve:  
\n
$$
\begin{cases}\nx_c(\alpha_i, \phi_p) = R \sin(\theta_i - \phi_p + \phi_g) \\
+ R_p \sin(\alpha_i + \phi_p - \phi_g - \theta_i) \\
- e \sin \phi_g \\
y_c(\alpha_i, \phi_p) = R \cos(\theta_i - \phi_p + \phi_g) \\
- R_p \cos(\alpha_i + \phi_p - \phi_g - \theta_i) \\
- e \cos \phi_g\n\end{cases}
$$

Curvature radius of epicycloid curve:

$$
\rho_c(\alpha_i, \phi_p) = \begin{bmatrix} x_c(\alpha_i, \phi_p) \\ y_c(\alpha_i, \phi_p) \\ 1 \end{bmatrix}
$$

with:

*ϕp, ϕg:* angular movement of coordinate systems  $F_p$  and  $F_g$  in comparison with  $F_x$ 

 $\alpha_i$ : the angle between the vertical axis of  $F_p$  and horizontal axis of  $F<sub>x</sub>$ 

 $\theta_i$ : the position of ith pin-roller in coordinate system  $\overline{F}$ ,

*R:* Radius of the pin-roller distributed circle

*R<sub>p</sub>*: radius of the pin-roller

*e*: eccentricity

Point on epicycloid curve:  
\n
$$
\begin{cases}\nx_c = R\cos(\phi_g - \phi_p) \\
-R_p\cos(\phi_g - \phi_p - \psi_i) \\
-e \cos \phi_g \\
y_c = -R\sin(\phi_g - \phi_p) \\
+R_p\sin(\phi_g - \phi_p - \psi_i) \\
+e \sin \phi_g\n\end{cases}
$$

Curvature radius of epicycloid curve:

$$
\rho_c = \begin{bmatrix} x_c \\ y_c \\ 1 \end{bmatrix}
$$

with:

- $\phi_p$ ,  $\phi_g$ : angular movement of coordinate systems  $F_p$  and  $F_q$  in comparison with  $F_x$
- *ψi* : Pressure angle between ith pin-roller and the cycloid gear
- *R:* Radius of the pin-roller distributed circle
- *R<sub>p</sub>*: radius of the pin-roller
- *e*: eccentricity



(a) 3D shape of the cycloid tooth profile (b) Four modified cases of a cycloid tooth profile

<span id="page-6-1"></span>**Fig. 4** Cases of profle modifcations of the cycloid reducer [[64](#page-15-12)] (open accessed). **a** 3D shape of the cycloid tooth profle, **b** Four modifed cases of a cycloid tooth profle. \*PM: positive roller radius mod-

epitrochoid curve by transforming points in the coordinate system with diferent frames of reference. This method normally uses three frames of reference  $F_g$  ( $O_g$ ,  $X_g$ ,  $Y_g$ ),  $F_p$  ( $O_p$ ,  $(X_p, Y_p)$ ,  $F_f$  ( $O_p$ ,  $X_p$ ,  $Y_f$ ) that are attached to the cycloid disk, cycloid pin (or cycloid wheel), and fxed-frame or housing [\[55](#page-15-9)[–57](#page-15-10)]. The  $F_p$  of a cycloid pin coincides with  $F_f$ , whereas  $F_p$  revolves around the origin of  $F_p$  as illustrated in Fig. [3b](#page-5-0). The instant-velocity center method utilizes three frames

ifcation; NM: negative roller position modifcation. NNM: negative roller position and negative roller radius modifcation. PPM: positive roller position and positive roller radius modifcation

of reference similar to the transmitted coordinate system method and one more frame for the pitch point of the cycloid reducer (instant-velocity center). Thus, the generated epicycloid curve is determined according to the positions of the pitch point and contact point of the conjugating surfaces between the cycloid reducer and cycloid pins [[53](#page-15-7)]. A comparison among the generating equations used in the three methods is presented in Table [4.](#page-6-0) The shape of the designed

cycloid disk or the epicycloid curve depends on the radius of the pin roller, radius of the circle passing through the centers of the pin-rollers, eccentricity, number of pin-rollers, and number of teeth (or lobes of the profle) of the cycloid reducer.

Modifcations or corrections of the tooth profle should be performed to tolerate errors during both assembly and manufacturing processes [[58](#page-15-13), [59\]](#page-15-14). A small change in the theoretical tooth profle is made at the conjugated section of the tooth contact. Modifcations or corrections of the tooth profle guarantee smooth engagement, increase the contact number, and provide good lubrication conditions [\[60,](#page-15-15) [61](#page-15-16)]. There are two types of tooth modifcation methods: single modifed parameter and multi-modifed parameter methods. The isometric, offset, and angular rotation modifications correspond to the single modifed parameter method [[54,](#page-15-8) [62](#page-15-17)]. Isometric and ofset modifcations are used to adjust the radius of the pin roller and the radius of the circle passing through the center of the pin rollers, respectively [[62–](#page-15-17)[64\]](#page-15-12). In contrast, the angular rotation modifcation changes the center position of the pin rollers along the circumferential direction [\[54,](#page-15-8) [62\]](#page-15-17). Isometric and offset modifications can be applied independently [\[63\]](#page-15-18) or together [[58\]](#page-15-13), whereas the angular rotation modifcation can only be applied along with other methods [[54\]](#page-15-8). Examples of single modifed parameter and multimodifed parameter methods are shown in Fig. [4](#page-6-1) [\[64](#page-15-12)]. A comparison between various cases of a modifed cycloid tooth profle with a theoretical profle is introduced by sole isometric modifcation (both positive and negative), and a combination of isometric and offset modifications. Furthermore, the multimodifed parameter method of changing the tooth shape considering the pressure angle [[59](#page-15-14)], increasing contact points [\[55\]](#page-15-9), or under-cutting phenomenon [\[57\]](#page-15-10) are presented. Finally, a tooth modifcation method is studied as a unifed process of combining parameters such as machining tolerances and axial play in bearings [\[65](#page-15-19)].

Various design problems of the cycloid reducer have been studied because the structure of a cycloid reducer has a significant effect on its performance [\[39](#page-14-32), [52](#page-15-6), [59\]](#page-15-14). As mentioned in the previous section, fve types of cycloid reducers are introduced to improve their performance, including size, payload, strength, and stifness [[52](#page-15-6), [66](#page-15-20)]. Furthermore, the output mechanism of a cycloid reducer has a signifcant impact on the torsional stifness and vibration [\[67–](#page-15-21)[69\]](#page-15-22). A cycloid reducer structure without pin rollers was proposed to decrease the stress fuctuation associated with conventional designs [\[70](#page-15-23)]. This research also recognized that the presence of more than two tooth diferences reduces stress fuctuations and velocity ripple, in addition to improving the stress distributions of the cycloid reducer. In contrast, reducing the diference in tooth number between two pin-rollers of the 2K-H type cycloid reducer can improve the self-locking

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performance of the mechanism but degrade the backlash angle of the reducer [\[39](#page-14-32)].

All aspects of the tooth profle and the cycloid reducer designs are summarized in Fig. [5](#page-8-0).

## **4 Performance Analysis of the Cycloid Reducer**

Internal and external factors afecting the performance of the cycloid reducer, as shown in Fig. [6](#page-9-0). The tooth geometry and bearing characteristics are the most signifcant internal factors afecting the load capacity, rigidity, bending stress, and torque ripple of the transmission device [[71](#page-15-24)]. On the other hand, machining errors or tolerances of the components of a cycloid reducer increase backlash, lost motion, and transmission errors, in addition to causing vibration and noise in the system [\[58](#page-15-13), [72](#page-15-25)]. Nevertheless, both the payload and speed are external factors that signifcantly infuence the efficiency of the reducer, whereas the assembling misalignment between the reducer and input or output parts afect the vibration/noise and shock resistance of the system [\[73](#page-15-26), [74](#page-15-27)].

A mathematical model of the contact point and force distribution of a cycloid reducer is required, considering that the contact force distribution is a fundamental in investigating tooth stress and dynamic transmission performance analysis. Dynamic equations for the contact force distribution in the meshing area of a 2K-H cycloid reducer are derived in [[39\]](#page-14-32). A detailed analysis of the dynamic model of the eccentric shaft bearing is performed to evaluate the engagement of a cycloid reducer [[75](#page-15-28), [76](#page-15-29)]. Recently, stress analysis of a cycloid reducer was performed by combining the kinematics and dynamics of rigid bodies and nonlinear stifness based on contact dynamics [[77](#page-15-30)].

As mentioned in the previous section, the structures of the reducer as well as the tooth profle and its modifcation strongly afect the force distribution, hysteresis curve, transmission errors, and efficiency  $[67]$  $[67]$  $[67]$ . The contact number and force distribution of a cycloid reducer are numerically studied considering two tooth modifcation methods, i.e., isometric and offset methods, and their efects on the torque performance of the cycloid reducer are analyzed [[64](#page-15-12)]. Tooth profle modifcations based on the contact angle (straight line, catenary, and cycloid functions) reduce the backlash and lost motion of the cycloid reducer signifcantly compared to conventional tooth modifcations, as shown in Fig. [7](#page-9-1) [[59](#page-15-14)]. In addition, high stress concentrated on the contact points of a cycloid reducer such as pin-rollers could be relieved by performing FE analysis and design modifcations [[78,](#page-15-31) [79](#page-15-32)]. Longitudinal tooth profle modifcations are also efective in smoothing the contact distribution  $[64, 80]$  $[64, 80]$  $[64, 80]$  $[64, 80]$  as well as performance efficiency  $[66]$  $[66]$  $[66]$ , transmission error  $[60, 76]$  $[60, 76]$  $[60, 76]$  $[60, 76]$ , and torsional



<span id="page-8-0"></span>**Fig. 5** Design optimizations of the cycloid reducer

rigidity [[81\]](#page-15-34). As an example, the transmission error of an RV-type reducer can be reduced by 24%, with proper tooth modifcation, as shown in Fig. [8](#page-10-0) [[52](#page-15-6)]. The nonpin A cycloid reducer without pinwheels [[51\]](#page-15-35) might exhibit higher efficiency under certain conditions than those with pinwheels [[82\]](#page-15-36).

Bearings used in a cycloid reducer signifcantly infuence the performance of the cycloid reducer, which can be studied using numerical or FEM analysis [\[83](#page-16-0)]. In particular, the



<span id="page-9-0"></span>**Fig. 6** Factors affecting the performance of a cycloid reducer

RV-type reducer has many bearings, making it suitable for investigating the efects of bearings on the dynamic perfor-mance of reducer [[76,](#page-15-29) [84](#page-16-1), [85\]](#page-16-2). The stiffness and geometric defects of bearings may afect the contact force distribution and fnally lead to an improvement in the vibration sensitivity and reliability of the cycloid reducer [[75,](#page-15-28) [85\]](#page-16-2). Moreover, the clearance of the support bearings might affect the transmission errors of a cycloid reducer [[84](#page-16-1)]. Lastly, a cycloid reducer with needle roller bearings has better efficiency than that with sleeve bearings [[86\]](#page-16-3).

The machining quality of the tooth profle is also a signifcant contributor to the performance of cycloid reducers [[87,](#page-16-4) [88\]](#page-16-5). Specifcally, the torsional stifness of a cycloid gear is strongly related to the roughness of the tooth surface [\[87](#page-16-4)]. In addition, the clearances of the cycloid reducer due to machining errors have an impact on the hysteresis curve of the reducer, such as backlash and lost motion [\[88](#page-16-5)]. Finally, the manufacturing tolerance of eccentricity might also afect the force distribution and power losses of a cycloid reducer [\[74\]](#page-15-27).

The efficiency and life cycle of the cycloid reducer are governed by friction and lubricants. The friction forces of both sliding and rolling motions cause losses in the cycloid reducer [[69](#page-15-22)]. Experiments were conducted to investigate the correlation between the friction/lubricant and power losses/efficiency of the cycloid reducer  $[86]$  $[86]$  $[86]$ . As the speed increases, the friction coefficient  $[89]$  $[89]$  $[89]$  and the contact ratio decrease [\[61\]](#page-15-16). The contact force distribution of a cycloid reducer also influences the average film thickness and efficiency of the reducer [[61](#page-15-16)]. Oil lubrication enables a large compound cycloid reducer to dissipate heat efectively from friction loss [\[90\]](#page-16-7). Furthermore, oil flling can be used in diagnosing a cycloid reducer, and abrasive friction particles should be carefully checked to extend the life cycle of the cycloid reducer [\[91\]](#page-16-8).

The performances of the cycloid reducers are investigated experimentally, as shown in Figs. [9](#page-11-0) and [10](#page-12-0). The experimental setup is divided into four principal subsystems: assembly, instrumentation, supervision, and interface, as shown in Fig. [9](#page-11-0). All components connected in the transmission system are denoted by "assembly." "Instrumentation" includes high precision sensors, such as an angle encoder, torque meter, and speed meter. Usually, the control system drives the input angle of the motor through a motor driver and applies a torque load via a brake or servomotor. Finally, the "interface" helps to collect and show the data. All data from "instrumentation" are collected and displayed on a computer.

The impacts of operating payload, speed, and misalignment errors on the performance of the cycloid reducer can be experimentally studied. Integrated with internal effects, an experimental system with a precision reducer might operate with slightly lower performance, including having higher transmission error [\[71](#page-15-24), [73\]](#page-15-26), lower torsional stiffness  $[67, 92]$  $[67, 92]$  $[67, 92]$ , and more vibrations  $[92, 93]$  $[92, 93]$  $[92, 93]$  $[92, 93]$ . Even though the efficiency could be improved under certain conditions, such as high input torque and low speed, many losses would still be



<span id="page-9-1"></span>**Fig. 7** Infuence of modifed tooth profle methods on the backlash and lost motion of a cycloid reducer [[59](#page-15-14)] (open access)

<span id="page-10-0"></span>**Fig. 8** Initial clearances, normal contact deformations, and transmission errors of a proposed tooth profle [[52](#page-15-6)] (open access)



(a) Normal contact deformation under initial clearance



present in the entire experimental system [\[86,](#page-16-3) [89\]](#page-16-6). Thus, vibration suppression and suitable lubrication approaches would have to be carefully considered to improve the experimental conditions of the cycloid reducer [[73,](#page-15-26) [91\]](#page-16-8).

## **5 Machining Process and Fault Detections of the Cycloid Reducer**

Grinding is the principal method for producing an accurate epicycloid profle of a cycloid reducer [\[94](#page-16-11)]. Generally, the milling process with a 5-axis CNC machine can create the tooth profle of a cycloid gear [\[95](#page-16-12)], but the grinding process can provide better roughness for the tooth surface of a cycloid gear [\[87\]](#page-16-4). With the recent improvement in precision measurement technology and gear grinding devices [[96](#page-16-13)], all technical aspects of forming a complex surface, such as an epicycloid, can be handled using the grinding process. A grinding system for cycloid gear quantitatively corrects the error of the tooth profle of a cycloid reducer, with an eccentric shaft and pin rollers. The only diference is that one of the pin rollers is replaced with the grinding wheel, as shown in Fig. [11](#page-12-1) [\[94](#page-16-11)].

In addition, selective parts of cycloid reducers are assembled using a genetic algorithm for better performance [\[97](#page-16-14)]. A fault detection method is applied to discard poor quality products of cycloid reducers [\[98](#page-16-15)], which are usually based on vibration signals [[99\]](#page-16-16) or acoustic emission [[100,](#page-16-17) [101\]](#page-16-18).



Experiment system



<span id="page-11-0"></span>**Fig. 9** Principal subsystems of an experimental setup of a cycloid reducer

## **6 Other Applications of Rigid Precision Reducer**

Owing to their high-performance characteristics such as accuracy, reliability, and rigidity, cycloid reducers are widely being used in various applications, as illustrated in Fig. [12.](#page-13-6) First, a cycloid reducer is designed with an extremely compact size and unified with a servomotor in a light-weight actuator, which is suitable for mobile robots [[102–](#page-16-19)[104\]](#page-16-20). Moreover, medical equipment and surgical robots use a cycloid reducer as a component in an automatic tool changer or actuator [[105](#page-16-21)]. In the automotive industry, a cycloid reducer is used in E-CVVT  $[106, 107]$  $[106, 107]$  $[106, 107]$  $[106, 107]$  or the new upgraded system E-CVVD  $[108]$  $[108]$  $[108]$ as an actuator. In addition, a cycloid reducer is used as an actuator in vehilce radar systems [[109](#page-16-25)].

<span id="page-12-0"></span>

<span id="page-12-1"></span>**Fig. 11** Grinding system and process for machining the cycloid gear [[93](#page-16-10)]. (open access)



(a) CanguRo  $[103]$ 



(b) Surgical robot  $[104]$ :



(c) E-CVVD system in automotive engine [107]

<span id="page-13-6"></span>

## **7 Conclusion**

Although smart manufacturing is a strong industrial driver for reshaping the current competitive landscape and establishing new market leaders, machining manufacturing plays an irreplaceable and dominant role in manufacturing. Machining robots are leading the change in the existing manufacturing system, and the precision reducer is an important component governing the accuracy of the machining robot. The cycloid reducer is the best candidate among precision reducers when considering both the structural compliance and kinematic accuracy of the machining robots. Many types of cycloid reducers have been developed, and numerous design approaches to improve the performance of cycloid reducers have been proposed. Performance evaluations for the cycloid reducer are necessary for validating the proposed designs. In addition, fault detection and diagnosis issues are gaining interest from researchers. Currently, cycloid reducers are fnding industrial applications such as in mobility, automotive, and precision industrial robots. Thus, the application of cycloid reducers will continue to expand.

 $(d)$  Radar system [108]

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