

Energy-efficient wing design for flapping wing micro aerial vehicles†

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Abstract

Flapping wing micro aerial vehicles (FWMAVs) have attracted more attention during the development of the robotic systems field. The size of the flapping wing plays an important role in the lift force and torque generation based on quasi-steady aerodynamic model. Therefore, it is necessary to study energy-efficient design methods for wings to provide sufficient lift force and torque with minimal energy consumption for hovering flight. In this paper, the sensitive parameters for the lift force and power consumption were first selected based on design of experiment (DOE) and the parameter of the distributed wing stiffness was determined based on experimental data. Design optimization models for three different cases were then built by considering the lift force as one constraint and the energy consumption as the objective function. The combination of subset simulation and the gradient-based optimization was finally used for solving design optimization models, and the corresponding sensitivity analysis was provided.

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Keywords: Design optimization; Aerodynamic model; Energy-efficient; Hovering flight; Parameter determination; Sensitivity analysis

1. Introduction

Because of the advantages of small size, high agility, and the ability to hover in the air, FWMAVs have attracted more attention from academic and industrial fields. The flapping wing is an important part of the FWMAVs to produce lift force. Therefore, studying the morphology and kinematics of wings and then designing the corresponding energy-saving wings are new focuses.

There have been many studies on the properties of morphology, kinematics and aerodynamics for wings. Rayner proposed a theory on vorticity presented in the wake of hovering animal and estimated the rate of working [1]. Ellington et al*.* studied the aerodynamics of hovering insect flights, e.g.: Quasi-steady analysis, and morphological parameters [2-4]. Wang et al. studied the wing morphology, flapping kinematics and further proposed a quasi-steady aerodynamic model [5, 6]. Wang et al. also studied the design optimization of wings based on Beta probability density function (BPDF) [6-8]. For energy-effective hovering and roll control, Peters focused on a combined approach to finding an optimal wing design, including wing planform and pitching kinematics [9, 10]. Nan et al. conducted studies on the effect of the wing geometry on the performance including the lift force and energy consumption via experiments [11]. Sane provided a review on the high lift mechanism about insect flight [12]. Stewart et al. used a modified Zimmerman method to investigate the aerodynamic performance of flapping wings [13]. Ghommem et al. established the geometric model of the wing through B-spline representation and optimized the flapping wing shape [14]. For hummingbirds or hummingbird-like MAVs, there are some new developments. Warrick et al. tested and analyzed the differences in aerodynamic mechanisms of the hovering hummingbird between upstroke and downstroke using digital particle imaging velocimetry (DPIV) [15]. Keennon et al. provided state of the art of the development and design of the nano hummingbird [16]. Karásek et al. presented a newfangled flapping mechanism in a robotic hummingbird for the pitch moment generation [17].

A series of studies on the energy requirements of insects or FWMAVs have been carried out. Sun and Du studied energy requirements of eight species of insects including fruit fly, hawkmoth, etc. [18]. The study showed that the major part of energy consumption came from aerodynamic force or wing inertia associated with insect size and wingbeat frequency. Insects can use their muscles and elastic elements to recycle the energy for efficient flight, and change the flight attitude or flapping frequency to adapt to different flight states [19-21]. For the hovering insect flight, Berman et al. investigated efficient energy kinematics and conducted sensitivity analysis for the optimal solutions [22]. For the flight of MAVs, Woods et al. [23] provided energy requirements for the fixed wing mode, rotary wing mode and flapping wing mode, respectively. Ma-

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dangopal et al. [24] designed an energy storage mechanism and simplified the wing model by imitating insects, which could store part of the kinetic energy as potential energy similar to an insect thorax.

To further improve the energy efficiency of the hover flight under the satisfaction of constraints, we conducted design optimization of the wing of FWMAVs. The contribution of the paper can be summarized as: (1) The DOE is employed for sensitive parameter selection by considering lift force and energy consumption; (2) a method is provided to determine the distributed wing stiffness using experimental data; (3) design optimization models considering quasi-steady aerodynamic and power consumption model for three cases are built, respectively; (4) the combination of subset simulation and gradient-based optimization is used for solving the three design optimization models.

The paper is organized as follows. In Sec. 2, the quasisteady aerodynamic model is briefly introduced. Sec. 3 gives the DOE method for sensitive parameters selection by considering lift force and energy consumption. The distributed wing stiffness determination method based on experimental data is provided in Sec. 4. Sec. 5 elaborates on the details of design optimization and sensitivity analysis for the three cases. Conclusions are made in Sec. 6.

2. Quasi-steady aerodynamic and power consumption model

To understand clearly the DOE, parameter determination and design optimization in the next sections, the geometric and kinematic parameters of the wing, aerodynamic model, and energy consumption model are reviewed in this section.

2.1 Geometric parameters of the wing

The shape of the wing where the experiments are conducted is given in Fig. 1 [11]. For simplicity, we modify it as a right angle trapezoid and the diagram is shown in Fig. 2, where we can see the morphological parameters, wing span (*R*), wing root chord (C_R) and wing tip chord (C_T) . However the modification will impose little effect on the lift force and energy consumption. The aspect ratio (*AR*) is an important parameter, which is usually defined as the ratio of the wing span (*R*) to the mean chord length (\bar{c}) . The wing area (S) can be then expressed as Fo understand clearly the DOE, parameter determination
 I design optimization in the next sections, the geometric
 I kinematic parameters of the wing, aerodynamic model,
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Fig. 3. Definition of angles [6].

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trapezoid and the diagram is shown in Fig. 2, where we

be praised

$$
S = R\overline{c} = \frac{R^2}{AR} \,. \tag{1}
$$

The chord length can be expressed as a function of the spanwise radius *r*

$$
c = C_R - \frac{C_R - C_T}{R} \times r = C_R - \left(2C_R - \frac{2R}{AR}\right) \times \frac{r}{R} \,. \tag{2}
$$

To describe the spanwise area distribution, the normalized

Fig. 1. The geometric shape of the wing [11].

Fig. 2. The modified geometric shape of the wing.

Fig. 3. Definition of angles [6].

parameter \hat{d} expressing the chord-normalized distance from the pitching axis to the leading edge is introduced. Obviously, when the straight leading edge is considered as the pitching axis in our prototype, the parameter \hat{d} representing the position of the pitching axis will be set to 0.

2.2 Kinematic parameters of the wing

 $\left(\frac{2R}{x} \right)_{x} \frac{r}{(x)}$ in Fig. 3 [6]. With the defined framework in Fig. 3, the axis is *R* the wing where the experiments are conducted \overline{R} and the diagram is shown in Fig. 2, where we parameter \hat{d} expressing the chord-normalized distance and the diagram is shown in Fig. 2, where we parameter $\hat{$ The reciprocating motion of the wing can be generally divided into three progressive movements: sweeping motion, heaving motion, and pitching motion. Three Euler angles, including the sweeping angle ϕ , heaving angle θ and pitching angle η , are introduced to express these three mo-Fig. 3. Definition of angles [6].

parameter \hat{d} expressing the chord-normalized distance from

the pitching axis to the leading edge is introduced. Obviously,

when the straight leading edge is considered as the pitc Fig. 3. Definition of angles [6].

parameter \hat{d} expressing the chord-normalized distance from

the pitching axis to the leading edge is omsidered as the pitching

axis in our prototype, the parameter \hat{d} represen corresponding to the pitching axis and the axis is parallel to the plane with the wing in Fig. 2.

For hovering flight, the sweeping motion can be described

as

$$
\phi(t) = \frac{\phi_m}{\arcsin K} \arcsin \left[K \sin \left(2\pi f t \right) \right] + \phi_0 \tag{3}
$$

where ϕ_m , ϕ_0 and f denote the sweeping amplitude, horizontal offset and frequency, respectively. K is a coefficient to determine the motion pattern and $0 < K < 1$. If $K \rightarrow 1$, the motion will be a triangular pattern; if $K \to 0$, the motion will be sinusoidal. According to the motion requirements, we consider the sweeping motion to be approximately sinusoidal and set $K = 0.01$ in our paper.

The heaving motion is described by a sinusoidal function,

$$
\theta(t) = \theta_m \sin(2\pi Nft + \Phi_\theta) + \theta_0 \tag{4}
$$

where θ_{\ldots} , Φ_{θ} and θ_{0} are heaving amplitude, heaving phase offset and heaving offset, respectively. N is selected as 1 or 2. When $N = 1$, the shape of the wing motion will be vertical oscillation, and $N = 2$ corresponds to a figure "8" [22].

For the pitching motion, there is a relationship between the pitching angle and geometric angle of attack (AOA) on condition that the heaving motion is ignored [6], $\alpha_{\text{eq}} = 90^{\circ} - |\eta|$. According to Euler's second law, an implicit function of the pitching angle can be derived as [5, 6]

$$
I_{x,x_c}\ddot{\eta} = \frac{1}{2} I_{x_c x_c} \left[\dot{\phi}^2 \cos^2 \theta \sin(2\eta) - \dot{\theta}^2 \sin(2\eta) \right]
$$

+
$$
I_{x_c x_c} \left[2\dot{\phi}\dot{\theta} \cos \theta \cos^2 \eta + \ddot{\phi} \sin \theta \right]
$$

+
$$
I_{x_c z_c} \left[\dot{\theta} \sin \eta + \frac{1}{2} \dot{\phi}^2 \sin(2\theta) \sin \eta \right]
$$

+
$$
I_{x_c z_c} \left[2\dot{\phi}\dot{\theta} \sin \theta \cos \eta - \ddot{\phi} \cos \theta \cos \eta \right]
$$

+
$$
I_{x_c} \left[2\dot{\phi}\dot{\theta} \sin \theta \cos \eta - \ddot{\phi} \cos \theta \cos \eta \right]
$$

+
$$
I_{x_c} \left[\dot{\theta} \sin \theta \cos \eta - \ddot{\phi} \cos \theta \cos \eta \right]
$$

where $I_{x_c x_c}$ and $\tau_{x_c}^{\text{aero}}$ are $x_c x_c$ terms of the moment of inertia I and aerodynamic torque, respectively. In Nan's experiment, different elastic performance is exhibited due to different wing shape and the angle of stiffener but no quantitative relationship is provided [11]. While in Wang's wing model, a linear torsional spring linked to the wing root is used to represent the flexibility of the wing model $[6, 8]$. In this paper, we used a parameter k_n to express the distributed wing stiffness. Due to the lack of knowledge about the distribution of the wing mass, we also assumed that the distributed wing mass is uniformly as that in Ref. [6].

Using the "cans in series" approach for coordinate transformations [25], the angular velocity ω , and acceleration can be derived

$$
\mathbf{\omega}_{e} = \begin{bmatrix} \dot{\eta} - \dot{\phi}\sin\theta \\ \dot{\theta}\cos\eta + \dot{\phi}\cos\theta\sin\eta \\ \dot{\phi}\cos\eta\cos\theta - \dot{\theta}\sin\eta \end{bmatrix},
$$
(6)

$$
\boldsymbol{\alpha}_{\epsilon} = \left[\alpha_{x_{\epsilon}}, \alpha_{y_{\epsilon}}, \alpha_{z_{\epsilon}}\right]^{\mathsf{T}} = \left[\dot{\omega}_{x_{\epsilon}}, \dot{\omega}_{y_{\epsilon}}, \dot{\omega}_{z_{\epsilon}}\right]^{\mathsf{T}}.\tag{7}
$$

From the point of the wing with co-rotating coordinate $\mathbf{r} = (x_c \quad 0 \quad z_c)^T$, the translational velocity and acceleration can be calculated by

$$
v_{s} = \omega_{s} \times r \tag{8}
$$

$$
a_c = \alpha_c \times r + \omega_c \times v_c \,. \tag{9}
$$

2.3 Aerodynamic model

The aerodynamic model in Ref. [5] is employed here and the resultant aerodynamic load is decomposed into translationinduced load, rotation-induced load, coupling load between wing translation and rotation, and added-mass load. The wing plane is discretized into infinitesimal strips along the chordwise and spanwise directions with the use of the blade element method (BEM) [26]. Since the leading edge of the flapping wing is considered as the pitching axis, the parameter d representing the position of the pitching axis is set to 0.

The translation-induced load including force and torque can be expressed as

$$
F_{y_c}^{\text{trans}} = A^{\text{trans}} \int_0^R x_c^2 c dx_c ,
$$
\n(10a)

$$
\tau_{x_c}^{\text{trans}} = \begin{cases}\nA^{\text{trans}}(z_{cp}^{\text{max}} - d) \int_0^t x_c^c c^c dx_c, & \omega_{y_c} \le 0 \\
A^{\text{trans}} \left(1 - \hat{z}_{cp}^{\text{trans}} - \hat{d}\right) \int_0^R x_c^2 c^2 dx_c, & \omega_{y_c} > 0,\n\end{cases}
$$
\n(10b)

$$
\tau_{z_c}^{\text{trans}} = A^{\text{trans}} \int_0^R x_c^3 c dx_c \,, \tag{10c}
$$

$$
Atrans = -\frac{1}{2} \text{sgn}(\omega_{z_c}) \rho_f(\omega_{y_c}^2 + \omega_{z_c}^2) C_{F_{y_c}}^{trans}
$$
 (10d)

where sgn(•), ρ_f , $C_{F_{y_c}}^{\text{trans}}$ and $\hat{z}_{cp}^{\text{trans}}$ are the sign function, fluid density, translational force coefficient and normalized chordwise center of pressure, respectively. The translational force coefficient can be expressed as a function of the angle of attack $\hat{\alpha}$ and aspect ratio

$$
C_{F_{y_c}}^{\text{trans}} = \frac{2\pi AR \sin\left(\hat{\alpha}\right)}{2 + \sqrt{AR^2 + 4}}\,. \tag{11}
$$

 $\hat{z}_{c}^{\text{trans}}$ has the special relationship with the angle of attack based on experimental data as

$$
\hat{z}_{cp}^{\text{trans}} = 5.6 \times 10^{-3} \hat{\alpha} \tag{12}
$$

The rotation-induced load can be expressed as

$$
F_{y_c}^{\text{rot}} = \frac{1}{2} \rho_{\text{f}} \omega_{x_c} \left| \omega_{x_c} \right| C_D^{\text{rot}} \int_0^R \int_{\hat{d}c-c}^{\hat{d}c} z_c \left| z_c \right| dz_c dx_c, \qquad (13a)
$$

$$
\tau_{x_c}^{\rm rot} = -\frac{1}{2} \rho_{\rm r} \omega_{x_c} \left| \omega_{x_c} \right| C_D^{\rm rot} \int_0^R \int_{\hat{d}c-c}^{\hat{d}c} \left| z_c \right|^3 dz_c dx_c, \qquad (13b)
$$

2. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093~4104
\n
$$
\tau_{z_c}^{\text{rot}} = \frac{1}{2} \rho_t \omega_{x_c} \left| \omega_{x_c} \right| C_D^{\text{rot}} \int_0^R \int_{d\epsilon-c}^{d\epsilon} z_c \left| z_c \right| x_c dz_c dx_c
$$
\n(13c)\n
$$
P_{\text{are}} = -\tau_{x_c}^{\text{areo}} \omega_{x_c} - \tau_{z_c}^{\text{neto}} \omega_{z_c},
$$
\n(13c)\n
$$
P_{\text{are}} = -\tau_{x_c}^{\text{neto}} \omega_{x_c} - \tau_{z_c}^{\text{neto}} \omega_{y_c} - \tau_{z_c}^{\text{neto}} \omega_{z_c},
$$
\n(14c)

where C_D^{rot} indicates the rotational damping coefficient, which can be expressed as

2. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093-4104
\n
$$
\tau_{z_c}^{\text{net}} = \frac{1}{2} \rho_i \omega_{x_c} |\omega_{x_c}| C_D^{\text{net}} \int_0^R \int_{d_{c-c}}^{d_{c-c}} z_c |z_c| x_c dz_c dx_c
$$
\n(13c)\n
$$
\begin{aligned}\nP_{\text{area}} &= -\tau_{x_c}^{\text{line}} \omega_{x_c} - \tau_{y_c}^{\text{line}} \omega_{z_c}, & (18a) \\
P_{\text{area}} &= -\tau_{x_c}^{\text{line}} \omega_{x_c} - \tau_{y_c}^{\text{line}} \omega_{y_c} - \tau_{z_c}^{\text{line}} \omega_{z_c}, & (18b) \\
P_{\text{class}} &= k_{\eta} \eta \omega_{x_c}. & (18c)\n\end{aligned}
$$
\n(18c)\n
\n
$$
\begin{aligned}\nT_{\text{net}}^{\text{net}} &= -\tau_{x_c}^{\text{line}} \omega_{x_c} - \tau_{y_c}^{\text{line}} \omega_{z_c}, & (18b)\n\end{aligned}
$$
\n2. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093-4104\n
\n
$$
T_{\text{enc}}^{\text{net}} = -T_{x_c}^{\text{line}} \omega_{x_c}, & (18a)\n\end{aligned}
$$
\n2. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093-4104\n
\n
$$
T_{\text{enc}}^{\text{line}} = -T_{x_c}^{\text{line}} \omega_{x_c},
$$
\n(18a)\n
\n
$$
P_{\text{other}} = -T_{x_c}^{\text{line}} \omega_{x_c} - \tau_{y_c}^{\text{line}} \omega_{y_c}. & (18b)\n\end{aligned}
$$
\n2. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093-4104\n
\n
$$
T_{\text{line}}^{\text{line}} = -T_{x_c}^{\text{line}} \omega_{x_c},
$$
\n(18a)\n
\n
$$
T_{\text{line}}^{\text{line}} = -T_{x_c}^{\text{line}} \omega_{x_c} - \
$$

The coupling load between the wing translation and rotation can be calculated as

() coup coup ² coup 0 2 ˆ , 0, *c c ^c ^R c c y y R c c ^y A d c x dx F* - £ > ^ì ï í ï î ò ò

E
\n
$$
E_x^{mg} = \frac{1}{2} \rho_i \omega_{\kappa} \left| \omega_{\kappa} \right| \left| \frac{\cos \theta}{\theta_0} \right|_{\vec{\theta} \times \kappa}^{4\pi/2} \left| \frac{\sin \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\sin \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\cos \theta}{\omega_{\kappa}} \right| \left| \frac{\sin \theta}{\omega_{\kappa}}
$$

$$
P_{\text{max}} = \frac{1}{T} \int_{\tau} \mathbb{E} (P_{\text{ave}} + P_{\text{max}} + P_{\text{elas}})
$$
\n
$$
T_{\text{exp}}^{\text{sup}} = \begin{cases} A^{\text{sup}} \int_{0}^{s} \left(1 - \hat{d}\right) c^{2}(x_{c}) x_{c}^{2} dx_{c}, & \omega_{y_{c}} \leq 0 \\ A^{\text{sup}} = \pi \rho \omega_{x_{c}} \omega_{y_{c}}. & \omega_{y_{c}} > 0, \end{cases}
$$
\n(15c) The actual energy consumption
\n
$$
A^{\text{sup}} = \pi \rho \omega_{x_{c}} \omega_{y_{c}}. \qquad (15d)
$$
\n
$$
F_{\text{min}}^{\text{min}} = -\int_{0}^{s} m_{2} a_{y_{c}} dx_{c} - \alpha_{x_{c}} \int_{0}^{s} m_{2} dx_{c}, \qquad (16a)
$$
\n
$$
F_{\text{min}}^{\text{min}} = -\int_{0}^{s} m_{2} a_{y_{c}} dx_{c} - \alpha_{x_{c}} \int_{0}^{s} m_{2} dx_{c}, \qquad (16b)
$$
\n
$$
T_{\text{min}}^{\text{min}} = -\int_{0}^{s} m_{2} a_{y_{c}} dx_{c} - \alpha_{x_{c}} \int_{0}^{s} m_{2} dx_{c}, \qquad (16c)
$$
\n
$$
T_{\text{min}}^{\text{min}} = -\int_{0}^{s} m_{2} a_{y_{c}} x_{c} dx_{c} - \alpha_{x_{c}} \int_{0}^{s} m_{2} dx_{c}, \qquad (16e)
$$
\n
$$
T_{\text{min}}^{\text{min}} = -\int_{0}^{s} m_{2} a_{y_{c}} x_{c} dx_{c} - \alpha_{x_{c}} \int_{0}^{s} m_{2} dx_{c}, \qquad (16f)
$$
\nwhere Ξ is a sign, when $P_{\text{area}} \equiv P_{\text{max}} + P_{\text{class}} \equiv P_{\text{max}} \equiv$

$$
\begin{aligned}\n\left[A^{-1} \right]_0 & \text{ac} \left(\lambda_c \right) \lambda_c \text{a} \lambda_c, \qquad \omega_{y_c} > 0, \\
A^{\text{coup}} &= \pi \rho \omega_x \omega_y.\n\end{aligned}\n\tag{15d}
$$

The added-mass loads are given by

$$
F_{y_c}^{\rm am} = -\int_0^R m_{22} a_{y_c} dx_c - \alpha_{x_c} \int_0^R m_{24} dx_c , \qquad (16a)
$$

$$
\tau_{x_c}^{\text{am}} = -\int_0^R m_{a2} a_{x_c} dx_c - \alpha_{x_c} \int_0^R m_{44} dx_c , \qquad (16b)
$$

$$
\tau_{z_c}^{\rm am} = -\int_0^R m_{22} a_{y_c} x_c dx_c - \alpha_{x_c} \int_0^R m_{24} x_c dx_c, \qquad (16c)
$$

The added-mass loads are given by
\n
$$
F_{y_c}^{\text{nm}} = -\int_0^R m_{z2} a_{y_c} dx_c - \alpha_{x_c} \int_0^R m_{z4} dx_c, \qquad (16a)
$$
\n
$$
F_{z_c}^{\text{nm}} = -\int_0^R m_{z2} a_{y_c} dx_c - \alpha_{x_c} \int_0^R m_{z4} dx_c, \qquad (16b)
$$
\n
$$
\tau_{z_c}^{\text{nm}} = -\int_0^R m_{z2} a_{y_c} x_c dx_c - \alpha_{x_c} \int_0^R m_{z4} x_c dx_c, \qquad (16c)
$$
\nwhere Ξ is
\n
$$
\tau_{z_c}^{\text{nm}} = -\int_0^R m_{z2} a_{y_c} x_c dx_c - \alpha_{x_c} \int_0^R m_{z4} x_c dx_c, \qquad (16c)
$$
\n
$$
\Xi = P_{\text{aero}} + P_{\text{mer}}
$$
\n
$$
\Xi = P_{\text{arm}} + P_{\text{mer}}
$$
\n
$$
m_{z2} \ m_{z4} = \frac{\pi}{4} \rho \left[c^2 (1/2 - \hat{d}) - c^2/32 + c^2 (1/2 - \hat{d})^2 \right]. \qquad (16d)
$$

(15b) $F_{\text{max}} = \frac{1}{T} J_r F_{\text{max}} dt$,
 $A^{\text{max}} \int_{1}^{x} \left(\hat{d} - \frac{1}{4} \right) \left(\frac{1}{2} - \hat{d} \right) c^x x, \alpha, \omega_1 > 0$,
 $P_{\text{max}} = \frac{1}{T} J_r \mathbb{E} (P_{\text{max}} + P_{\text{max}} + P_{\text{max}}) dt$.
 $P_{\text{max}} = \frac{1}{T} \int_{2}^{x} \mathbb{E} (P_{\text{max}} + P_{\text{max}} + P_{\text{max}}) dt$. $A^{\text{new}} \int_{c}^{f} \left(\hat{d} - \frac{1}{4} \right) \left(\frac{1}{2} - \hat{d} \right) c^x x \, dx, \quad \omega_{\text{s}} > 0,$
 $P_{\text{max}} = \frac{1}{7}$
 $A^{\text{new}} \int_{0}^{R} (1 - \hat{d}) c^2 (x_{\text{s}}) x_{\text{s}}^2 dx, \quad \omega_{\text{s}} \le 0$
 $A^{\text{new}} \int_{0}^{R} \hat{d} c^2 (x_{\text{s}}) x_{\text{s}}^2 dx, \quad \omega_{\text{s}} > 0,$

readd Due to the assumption that the resultant force is perpendicular to the wing during the entire stroke, the force about x_c and z_c axis and the torque about y_c axis are zero. Then the $r_{z_c} = \int_0^{1} m_{22} w_{y_c}^2 \lambda_z \mu_z e^{-\alpha_z} \alpha_z \int_0^{1} m_{23} \lambda_z \mu_z \mu_z$, (160) $\frac{\Xi - P_{\text{mer}}}{\Xi}$
 $\left[\frac{m_{22}}{m_{42}} \frac{m_{34}}{m_{44}} \right] = \frac{\pi}{4} \rho \left[\frac{e^2}{e^2 (1/2 - \hat{d})} - \frac{e^2 (32 + e^2 (1/2 - \hat{d})}{(1/2 - \hat{d})} \right]$. (16d) tails, p

Due The added-mass loads are given by
 $r_{\infty}^{m} = -\int_{0}^{x} m_{\infty} a_{\infty} dx_{\infty} = \alpha_{\infty} \int_{0}^{x} m_{\infty} dx_{\infty}$,
 $r_{\infty}^{m} = -\int_{0}^{x} m_{\infty} a_{\infty} dx_{\infty} = \alpha_{\infty} \int_{0}^{x} m_{\infty} dx_{\infty}$,
 $r_{\infty}^{m} = -\int_{0}^{x} m_{\infty} a_{\infty} dx_{\infty} = \alpha_{\infty}$

$$
F_{y_c}^{\text{aero}} = F_{y_c}^{\text{trans}} + F_{y_c}^{\text{rot}} + F_{y_c}^{\text{coup}} + F_{y_c}^{\text{am}}, \qquad (17a)
$$

$$
\tau_{x_c}^{\text{aero}} = \tau_{x_c}^{\text{trans}} + \tau_{x_c}^{\text{rot}} + \tau_{x_c}^{\text{coup}} + \tau_{x_c}^{\text{am}}, \qquad (17b)
$$

$$
\tau_{z_c}^{\text{aero}} = \tau_{z_c}^{\text{trans}} + \tau_{z_c}^{\text{rot}} + \tau_{z_c}^{\text{coup}} + \tau_{z_c}^{\text{am}}.
$$
\n(17c)

2.4 Energy consumption model

Considering the flexible structure of the wing, there are three parts in the energy consumption model: the energy to tion is $\phi_n > AR > R > f > k_n > C_{\kappa}$, which is roughly similar to overcome the aerodynamic drag (P_{aero}) , the energy to acceler-
th ate the wing and the surrounding medium (P_{max}), and the en- ous for energy consumption, and $C_R = 25$ mm based on exergy stored in the elastic structure (P_{elas}) [27]. They are expressed as

$$
P_{\text{aero}} = -\tau_{x_c}^{\text{aero}} \omega_{x_c} - \tau_{z_c}^{\text{aero}} \omega_{z_c}, \qquad (18a)
$$

$$
P_{\text{iner}} = -\tau_{x_c}^{\text{iner}} \omega_{x_c} - \tau_{y_c}^{\text{iner}} \omega_{y_c} - \tau_{z_c}^{\text{iner}} \omega_{z_c}, \qquad (18b)
$$

$$
P_{\text{elas}} = k_{\eta} \eta \omega_{\mathbf{x}_c} \,. \tag{18c}
$$

chanology 33 (9) (2019) 4093~4104
 $P_{\text{aero}} = -\tau_{x_c}^{\text{aero}} \omega_{x_c} - \tau_{z_c}^{\text{aero}} \omega_{z_c}$, (18a)
 $P_{\text{iner}} = -\tau_{x_c}^{\text{iner}} \omega_{x_c} - \tau_{y_c}^{\text{iner}} \omega_{y_c} - \tau_{z_c}^{\text{iner}} \omega_{z_c}$, (18b)
 $P_{\text{elas}} = k_{\eta} \eta \omega_{x_c}$. (18c)

The inertial *c c c c c c ^P x x y y z z* = - - - ^t ^w ^t ^w ^t ^w (18b) elas . *cherology* 33 (9) (2019) 4093~4104
 $P_{\text{acc}} = -\tau_{x_c}^{\text{mer}} \omega_{x_c} - \tau_{z_c}^{\text{mer}} \omega_{z_c}$, (18a)
 $P_{\text{net}} = -\tau_{x_c}^{\text{iner}} \omega_{x_c} - \tau_{y_c}^{\text{iner}} \omega_{y_c} - \tau_{z_c}^{\text{iner}} \omega_{z_c}$, (18b)
 $P_{\text{elas}} = k_{\eta} \eta \omega_{x_c}$. (18c)

The inertial torque The inertial torque τ^{iner} can be calculated in the co-rotating frame as

$$
\tau^{\text{iner}} = -I\alpha_c - \omega_c \times (I\omega_c). \tag{19}
$$

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 $=\frac{1}{2}\rho_1\omega_x\left|C_{D}^{\text{ext}}\right|_0^2\left|\int_{\delta x-\bar{x}}^{\infty}z_c\right|z_c\left|x_cdz_cdx_c$ (13c) $P_{\text{area}} = -r_{xy}^{\text{exp}}\omega_x - r_{xy}^{\text{exp}}\omega_z$
 $P_{\text{max}} = -r_{xy}^{\text{exp}}$ $\omega_{y} > 0$, minimum energy consumption can be calculated as the drive mechanism, the actual flapping power consumption
(15a) can be estimated in two extreme cases. The maximum and The inertial torque τ^{inter} can be or

frame as
 $\tau^{\text{mer}} = -I\alpha_c - \omega_r \times (I\omega_c)$.
 \therefore
 $\tau^{\text{mer}} = -I\alpha_c - \omega_r \times (I\omega_c)$.
 \therefore
 $\tau^{\text{mer}} = -I\alpha_c - \omega_r \times (I\omega_c)$.

Since the kinetic energy and els

to compensate the power cons *c* $\int_{\pi-\pi}^{\pi} \int_{\pi}^{2\pi} \int_{\pi}$ $\frac{cm}{r} = \frac{2\pi A R}{4r^2 \int_{1}^{1} (x^2)^2 (x^2 dx, \omega_c \le 0)}$

The coupling load between the wing translation and rotation

the coloridated as
 $r_c^{\text{res}} = \int_{1}^{1} (x^2)^2 (x^2 dx, \omega_c \le 0)$
 $\int_{1}^{1} (x^2)^2 (x^2 dx, \omega_c \le 0)$
 $\int_{1}^{1} (x^2)^$ *z* $\frac{1}{\sqrt{2}}\left[\frac{1}{4-\frac{1}{2}}\left(\frac{1}{\sqrt{2}}\right)\right]z, \frac{1}{2}\phi$, (14)

the coupling band between the viring murstation and nutation

be calculated as
 $F_{\gamma}^{n} = -\left[4-\frac{1}{2}\left(\frac{1}{2}-\dot{a}\right)\right]z^2, \frac{1}{2}\phi, \frac{1}{2}\phi, \frac{1}{2}\phi$, ϕ *i f cehnology* 33 (9) (2019) 4093-4104
 $P_{\text{asco}} = -\tau_{x_c}^{\text{asco}} \omega_{x_c} - \tau_{x_c}^{\text{asco}} \omega_{x_c}$, (18a)
 $P_{\text{asco}} = -\tau_{x_c}^{\text{asco}} \omega_{x_c} - \tau_{y_c}^{\text{bas}} \omega_{y_c} - \tau_{z_c}^{\text{bas}} \omega_{z_c}$, (18b)
 $P_{\text{elas}} = k_{\eta} \eta \omega_{x_c}$. (18c)

The in Since the kinetic energy and elastic energy are either used to compensate the power consumed by the drag or recycled by $P_{\text{area}} = -\tau_{z_i}^{\text{zero}} \omega_{z_i} - \tau_{z_i}^{\text{zero}} \omega_{z_i}$, (18a)
 $P_{\text{max}} = -\tau_{z_i}^{\text{zero}} \omega_{z_i} - \tau_{z_i}^{\text{free}} \omega_{z_i} - \tau_{z_i}^{\text{free}} \omega_{z_i}$, (18b)
 $P_{\text{dus}} = k_{\eta} \eta \omega_{z_i}$. (18b)
 $P_{\text{dus}} = k_{\eta} \eta \omega_{z_i}$. (18c)

The inertial torque $\$ nce the kinetic energy and elastic energy are either used
mpensate the power consumed by the drag or recycled by
trive mechanism, the actual flapping power consumption
be estimated in two extreme cases. The maximum and
mu $P_{\text{max}} = -r_{x_c}^{\text{mean}} \omega_{x_c} - r_{y_c}^{\text{mean}} \omega_{x_c}$, (18b)
 $P_{\text{dus}} = k_{\eta} \eta \omega_{x_c}$. (18c)
 $P_{\text{dus}} = k_{\eta} \eta \omega_{x_c}$. (18c)

the inertial torque τ^{bare} can be calculated in the co-rotating

the metrial torque τ^{bare} c = $-\tau_{x_x}^{\text{ins}} \omega_{x_x} - \tau_{y_x}^{\text{ins}} \omega_{y_x} - \tau_{z_x}^{\text{ins}} \omega_{z_x}$, (18b)

= $k_{\eta} \eta \omega_{x_x}$. (18c)

nertial torque τ^{insr} can be calculated in the co-rotating

s

= $-I\alpha_z - \omega_z \times (I\omega_z)$. (19)

the kinetic energy and elastic Exercise energy and elastic energy are either used
ate the power consumed by the drag or recycled by
elechanism, the actual flapping power consumption
mated in two extreme cases. The maximum and
nergy consumption can be c ence energy and elastic energy are either used
the power consumed by the drag or recycled by
the mass and flapping power consumption
of in two extreme cases. The maximum and
y consumption can be calculated as
 t_n (20a)
nce the kinetic energy and elastic energy are either used

impensate the power consumed by the drag or recycled by

thrive mechanism, the actual flapping power consumption

be estimated in two extreme cases. The maximum a

$$
P_{\min} = \frac{1}{T} \int_{T} P_{\text{zero}} dt , \qquad (20a)
$$

$$
P_{\text{max}} = \frac{1}{T} \int_{T} \Xi \left(P_{\text{aero}} + P_{\text{iner}} + P_{\text{elas}} \right) dt \tag{20b}
$$

 $w > 0$. The actual energy consumption is between P_{min} and P_{max} . For the optimization, the maximum consumption is our focus; therefore, the total mass-normalized power consumption *P* * is accounted for $\frac{1}{T} \int_{T} \Xi (P_{\text{aero}} + P_{\text{iner}} + P_{\text{elas}}) dt$. (20b)

ctual energy consumption is between P_{min} and P_{max} .

pptimization, the maximum consumption is our focus;

e, the total mass-normalized power consumption P^*

$$
P^* = \frac{\frac{1}{T} \int_T \Xi \left(P_{\text{are}} + P_{\text{inter}} + P_{\text{elas}} \right) dt}{\text{total mass}}
$$
(21)

 $(1/2 - \hat{d})$] ping flight including upstroke and downstroke. For more de- $\int_{\alpha_{y}}^{a} (1 - d) e^{x} (x_{x}) x_{x}^{3} dx_{x}$, $\omega_{y_{x}} > 0$,
 $\int_{\alpha_{y}}^{a} \hat{d}e^{x} (x_{x}) x_{x}^{3} dx_{x}$, $\omega_{y_{x}} > 0$,
 $\int_{\alpha_{y}}^{a} \hat{d}e^{x} (x_{x}) x_{x}^{3} dx_{x}$, $\omega_{y_{x}} > 0$,
 $\int_{\alpha_{y}}^{a} \hat{d}x_{x} - \alpha_{x} \int_{0}^{b} m_{x} dx_{x}$, $\int_{\alpha_{y}}^{$ $P_{\text{max}} = \frac{1}{T} \int_{\tau} \Xi (P_{\text{max}} + P_{\text{max}} + P_{\text{obs}}) dt$
 $x_c^2 dx_c$, $\omega_z > 0$,
 $x_c^2 dx_c$, $\omega_z > 0$,

(15c) The actual energy consumption is

For the optimization, the maximum

therefore, the total mass-normalized

is accounted x_c) x_c^2 , x_c^2 , $\omega_{y_c} > 0$,
 dx_c , $\omega_{y_c} > 0$,

(15c) The actual energy consumption is

For the optimization, the maximum c

tierefore, the total mass-normalized

is accounted for

are given by
 $\int_0^s m_{x_x} dx_c$,

((16d) tails, please refer to Refs. [5, 6, 8, 27]. where $\sum_{m=0}^{\infty} \frac{1}{T} \int_{T} P_{\text{grav}} dt$, we lead in plying power consumption
 $P_{\text{min}} = \frac{1}{T} \int_{T} P_{\text{grav}} dt$, (20a)
 $P_{\text{max}} = \frac{1}{T} \int_{T} P_{\text{grav}} dt$, (20a)
 $P_{\text{max}} = \frac{1}{T} \int_{T} P_{\text{grav}} dt$, (20b)

The actual energy consump $P_{\text{max}} = \frac{1}{T} \int_{T} P_{\text{max}} dt$, (20a)
 $P_{\text{max}} = \frac{1}{T} \int_{T} \Xi(P_{\text{max}} + P_{\text{max}} + P_{\text{obs}}) dt$. (20b)
 $P_{\text{max}} = \frac{1}{T} \int_{T} \Xi(P_{\text{max}} + P_{\text{max}} + P_{\text{obs}}) dt$. (20b)

The actual energy consumption is between P_{max} and P_{max} .

3. Sensitive parameters selection for design optimization

c $f_{\text{res}} = -\int_{\text{S}}^{x_0} m_a a_x dx_x - a_x \int_{\text{S}}^{x} m_a dx_x$,
 $f_{\text{res}}^{x_0} = -\int_{\text{S}}^{x} m_a a_x dx_y - a_x \int_{\text{S}}^{x} m_a dx_x$,
 $f_{\text{res}}^{x_0} = -\int_{\text{S}}^{x} m_a a_x dx_y - a_x \int_{\text{S}}^{x} m_a dx_x$,
 $f_{\text{res}}^{x_0} = -\int_{\text{S}}^{x} m_a a_x dx_y - a_x \int_{\text{S}}^{x} m_a dx_x$,
 $f_{\$ It is necessary to select sensitive parameters for building design optimization models to reduce the computational expense. Therefore we use 3-level fractional factorial DOE approach to qualitatively selecting several sensitive parameters. The main effect analysis for the lift force to each variable is conducted and the main effects are shown in Fig. 4.

 $F_{\nu}^{w} = -\int_{0}^{x} m_{\nu} a_{\nu} dx, \ \ \alpha_{\nu} = -\int_{0}^{x} m_{\nu} a_{\nu} dx, \ \ \tau_{\nu}^{w} = -\int_{1}^{x} m_{\nu} a_{\nu} dx, \ \tau_{\nu}^{w} = -\int_{1}^{x} m_{\nu} a_{\nu} dx$ The order of the variables from the aspect of sensitivity to $P^* = \frac{1}{T} \int_{\gamma} \Xi (P_{\text{area}} + P_{\text{ens}} + P_{\text{ens}}) dt$ (21)
 total mass (21)
 the lift force in a sign, when $P_{\text{gen}} + P_{\text{gen}} + Q_{\text{ens}} \le 0$, $\Xi = 0$, else
 $\Xi = P_{\text{max}} + P_{\text{ear}} + P_{\text{ens}}$. T represents a stroke cycle of the flap the lift force is $AR > \phi_m > C_R > R > f > k_n$. The same procedure is used for energy consumption, the main effects are shown in Fig. 5. From Fig. 5, we know that the order of the variables from the aspect of sensitivity to the energy consumptails, please refer to Refs. [5, 6, 8, 27].
 3. Sensitive parameters selection for design optimization

It is necessary to select sensitive parameters for building de-

sign optimization models to reduce the computation that of the lift force. Since the main effect of C_R is not obviaalls, please refer to Refs. [5, 6, 8, 27].

3. Sensitive parameters selection for design optimiza-

tion

It is necessary to select sensitive parameters for building de-

sign optimization models to reduce the computatio perimental results. Therefore R, ϕ_m, f, k_n and AR are considered as design variables for design optimization.

			Z. Wang et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093~4104						
	Table 1. The lift at different frequency level.			quency level.				Table 2. The relationship between R, S and lift at different	
		Lift(mN)							
AR 2.8	$f = 10$ Hz 4.50	$f = 20$ Hz 62.82	$f = 22$ Hz 76.44	\boldsymbol{R}	S	$f = 14$ Hz	$f = 16$ Hz	Lift(mN) $f = 18$ Hz	
3.2	9.34	72.27	86.82	(mm)	mm^2)				
3.65	17.29	74.78	91.66	70	1059	15.80	21.32	28.09	43.77
4.13	23.17	80.91	98.92	75	1215	21.90	28.24	36.70	57.20
4.63	24.21	100.51	120.71	80	1383	29.52	40.24	52.05	78.52
5.14	24.90	106.34	128.32	85	1561	39.66	50.16	65.77	97.35
5.7	23.87	111.57	134.55	90 95	1750 1950	48.42 56.72	64.10 77.24	80.58 99.09	$f = 22$ 121.0 144.8

Table 1. The lift at different frequency level.

Fig. 4. Tree-level main effects for lift force.

Fig. 5. Tree-level main effects for normalized energy consumption.

4. Determination of the distributed wing stiffness based on experimental data

The distributed wing stiffness k_n is a morphological parameter that affects the lift force and also energy consumption. The parameter should be first determined for the design optimization for morphological parameters. However, it could not be measured via experiments. In this section, the parameters are determined based on the experimental data using the least square method, shown in Table 1 [11]. *s*. Iree-level mann effects for normalized energy consumption.

both morphological a

determination of the distributed wing stiffness

optimization and

the transference data

hovering status

lovering status

lovering s

The corresponding optimization model is built under the $\begin{bmatrix} C_R & f & \phi_m & S \end{bmatrix} = \begin{bmatrix} 25 & 10 & 90^\circ & 1750 \end{bmatrix}$, eral hundreds; (2) gra

\n $\begin{cases}\n \text{Find} & k_n \\ \min \quad L ^2 = \sum_{i=1}^7 \left \text{Lift}_{\text{theor}}(i) - \text{Lift}_{\text{exper}}(i) \right ^2 & \text{for } n = 1, \text{ and } n = 1, \text$
--

quency level.

	ing et al. / Journal of Mechanical Science and Technology 33 (9) (2019) 4093~4104						4097
level.							Table 2. The relationship between R, S and lift at different fre-
Lift(mN)		quency level.					
$f = 20$ Hz	$f = 22$ Hz					Lift(mN)	
62.82	76.44	\boldsymbol{R} (mm)	S mm^2	$f = 14$ Hz	$f = 16$ Hz	$f = 18$ Hz	$f = 22$ Hz
72.27	86.82	70	1059	15.80	21.32	28.09	43.77
74.78	91.66	75	1215	21.90	28.24	36.70	57.20
	98.92	80	1383	29.52	40.24	52.05	78.52
80.91			1561	39.66	50.16	65.77	97.35
100.51	120.71	85			64.10	80.58	121.08
106.34	128.32	90	1750	48.42			
111.57	134.55	95	1950	56.72	77.24	99.09	144.82
		100	2161	69.29	90.04	111.61	155.71
\varPhi_m	K_{η}						
							The optimization is carried out via a gradient-based optimi-
					zation method and the result is $k_n = 9.6 \times 10^{-4}$.		To verify whether the determined parameter fits the ex-
							perimental data well or not, graphical results for the model
							analysis are given in Fig. 6. From Fig. 6, we know that the
							theoretical values approximately fit the experimental values. The errors are from the theoretical model and also uncertainty

perimental data well or not, graphical results for the model analysis are given in Fig. 6. From Fig. 6, we know that the theoretical values approximately fit the experimental values. The errors are from the theoretical model and also uncertainty from experiments [28, 29]. To further verify the effectiveness of the determined parameter, other experiments data for different area are employed, shown in Table 2 [11]. The corresponding graphical results for the model analysis are given in Fig. 7. 90 1750 48.42 64.10 80.58 121.08

95 1950 56.72 77.24 99.09 144.82

100 2161 69.29 90.04 111.61 155.71

The optimization is carried out via a gradient-based optimi-

The optimization is carried out via a gradient-based op 48.42 64.10 80.58 121.08

56.72 77.24 99.09 144.82

69.29 90.04 111.61 155.71

and the result is a gradient-based optimi-

and the result is $k_q = 9.6 \times 10^{-4}$.

whether the determined parameter fits the ex-

ta well or no

From Figs. 6 and 7, we can conclude that the determined proximately suitable to describe the physical prototype. Therefore, it is proper to use the parameter and model to conduct sequential design optimization.

5. Design optimization and sensitivity analysis

Example 1.2 sequential design optimization.
 Example 1.4 Example 1.4 Examp 4 min is section, the easign optimization models
 ι_i o i -1 o Tre-level main effects for normalized energy consumption.

both morphological and kinematic parameters referemination of the distributed wing stiffness

optimization and sensitivity analysis are constructed wind data

obv *L* $\vec{a} \times \vec{b}$ in this section, three design optimization models is the time in the set of the distributed wing stiffness change of the imperiodal variant potential data be time section of the distributed wing stiffne 5. Iree-level mann effects for normalized energy consumption. both morphological and kinematic param
 Determination of the distributed wing stiffness the trapezoidal wings of the humming bit

reaction and gensitivity an In this section, three design optimization models will be elaborated considering the morphological parameters, and both morphological and kinematic parameters respectively for the trapezoidal wings of the hummingbird-like MAV. Design optimization and sensitivity analysis are conducted for the hovering status here. The combination of subset simulation and gradient-based optimization will be implemented for solving the three design optimization models. Subset simulation is generally robust to deal with high-dimensional constrained optimization problems [30-32]. There are two steps for the combined optimization algorithm: (1) Subset simulation optimization with *N* generated independent samples is employed for locating the initial point near the global optimal point for the gradient-based optimization and *N* is usually set to be several hundreds; (2) gradient-based optimization is conducted with the optimal point provided by subset simulation as the initial point for reducing the probability of sinking into local optimal points.

During the design optimization for morphological parameters, only the geometric parameters and flapping frequency are considered as design variables, e.g.: *R* , *AR* and *f* . While

Fig. 6. Comparison between theoretical and experimental results for the same area.

Fig. 7. Comparison between theoretical and experimental results for different area.

during the design optimization for both morphological and kinematic parameters, the morphological and kinematic parameters, i.e.: *R*, *AR*, *f*, ϕ_m and k_n , are considered as mation design variables. The objective is to minimize the energy consumption, while the constraints are the lift force and other performance and geometric parameters. The constraint boundaries for design variables are derived from experiments and theoretical analysis, shown in Table 3. For searching the

possible optimal results, we expand properly the design domain for the design optimization for morphological and kinematic parameters.

5.1 Design optimization for morphological parameters

Design optimization model for morphological parameters is given by

Parameters		DOMP		DOM&KP			
$\lceil 70, 100 \rceil$ $\lceil 70, 100 \rceil$ R (mm)							
AR		$\lceil 2.8, 5.7 \rceil$		$\lceil 2.5, 6 \rceil$			Wing chord $\begin{bmatrix} \text{mm} \\ \text{e} \end{bmatrix}$
f(Hz)		$\lceil 10, 24 \rceil$		$\lceil 5, 30 \rceil$			
ϕ_m (°)		90		$\lceil 0, 90 \rceil$			
k_n (Nm / rad)	9.6×10^{-4}		$\lceil 1, 15 \rceil \times 10^{-4}$				
% DOMP denotes design optimization for morphological parameters. % DOM&KP denotes design optimization for morphological and kine- matic parameters.		-40 Ω					
Table 4. Design optimization results for morphological parameters.						parameters.	Fig. 8. Optimal shape after
	R (mm)	AR	$f(\mathrm{Hz})$	$P^*(W/kg)$	$\mathcal E$		
Optimization results	96.9	3.32	18.7	23.83 42.9%			
Experimental results	90	4.63	18.0	41.73			
Find $X = \begin{bmatrix} R & AR & f \end{bmatrix}$ min $f(X) = P^*$ $70 \le R \le 100$; $2.8 \le AR \le 5.7$ s.t. $10 \le f \le 24$; $\frac{2 \text{Lift}_{mean}}{mg} \ge 1$.		Euler angles [rad] $^{-2}$ $_{0}^{\mathrm{L}}$ $\overline{0.5}$					
where the kinematic parameters are provided as $\begin{vmatrix} C_R & \phi_m & k_n \end{vmatrix}$							

Table 3. Design domain for the two design optimization models.

Table 4. Design optimization results for morphological parameters.

	R(mm)			$AR \mid f(Hz) \mid P^*(W/kg)$	
Optimization results	96.9	3.32	18.7	23.83	42.9%
Experimental results	90	4.63	18.0	41.73	---

$$
\begin{cases}\nFind & X = \lfloor R \ AR \ f \rfloor \\
min & f(X) = P^* \\
st. & 70 \le R \le 100; \ 2.8 \le AR \le 5.7 \\
10 \le f \le 24; & \frac{2Lift_{mean}}{mg} \ge 1.\n\end{cases}
$$
\n(23)

⁴ 25 90 9.6 10- ⁼ ^é ´ ^ù $\begin{bmatrix} 25 & 90^{\circ} & 9.6 \times 10^{-4} \end{bmatrix}$ according to the determination $\begin{bmatrix} 1 \\ 1 \end{bmatrix}$ method in Secs. 3 and 4; P^* and $Lift_{mean}$ denote the total mass-normalized energy consumption and average lift during a flapping cycle, respectively; *m* stands for the weight of the prototype, and takes 17.2 g.

 $N = 500$ is used for the subset simulation and gradientbased optimization is conducted with the initial point from the result of the subset simulation. The optimization results are shown in Table 4. From Table 4, we know that the energy consumption has decreased by 42.9 % compared with the experimental results.

The optimal geometric shape of the wing is provided in Fig. 8. Fig. 8 shows that the wing becomes greater far away from the flapping mechanism, which is contrary to the shape in the experiments. The time-varying Euler angles and produced lift force within two cycles are shown in Figs. 9 and 10, respectively. We see that the sweeping motion is approximately sinusoidal and the pitching angle also changes smoothly, which is greatly beneficial to the stable flapping flight. The lift force production is also a time-varying process during the flapping flight. From Fig. 10, we see that the produced lift force is less than 0 at some time duration. However, this does not affect the normal hovering of the MAV, as long as the average generated lift force can resist gravity.

The sensitivity of the lift force and energy consumption to each design parameters are provided in Fig. 11, which could verify why the results in Table 4 are globally optimal. Fig. 11

Fig. 8. Optimal shape after design optimization for morphological parameters.

Fig. 9. Time-varying process of the sweeping and pitching motion in two cycles.

Fig. 10. Time-varying process of the lift force production in two cycles.

shows that the sensitivity of the lift force and energy consumption to each parameter is obviously different, and the sensitivity of the lift force and energy consumption to a given parameter is also different during the cycle. Take Fig. 11(a) for example, the sensitivity of energy consumption to *R* increases approximately before $R \leq 80$, and then decreases when *^R* Îé ù 80,96.9 ë û , and increases again after *^R* ^³ 96.9; while the sensitivity of the lift force to *R* increases during the whole cycle. The optimal R is the minimal energy consumption while the lift force is satisfied. As shown in Fig. 11(c),

	R(mm)	$\mathcal{A}R$	f(Hz)	$\phi_{\scriptscriptstyle m}$	k_{η} (Nm / rad)	$P^*(W/\text{kg})$	$\boldsymbol{\varepsilon}$
Optimization results	$100\,$	4.14	18.7	88.9°	6.6×10^{-4}	22.70	45.6%
Experimental results	90 	4.65	$18.0\,$	90°	9.6×10^{-4}	41.73	---
90 ₁ 84.3 $\operatorname{Lift}\left[\text{mN}\right]$ 70 50 $\overline{80}$ $7\overline{0}$	$\overline{90}$ R [mm]	$\frac{-\text{Lift}}{\text{P*}}$ 96.9 100	$\frac{1}{27}$ P^* [W/kg] 25 23.8	110 $\begin{array}{c}\n\text{Liff} \\ \text{M3} \\ \text{M3}\n\end{array}$ 90 70 50 ⁵ $\overline{3}$ 3.32	$\overline{4}$ $\mathcal{A}R$	$\frac{-\text{Lift}}{\text{P*}}$ $\frac{1}{30}$ 40 30 23.8 ²⁰ 5	P^* [W/kg]
	$\left(\text{a}\right)$				(b)		
85 $\begin{array}{c}\n\overrightarrow{AB} \\ \overrightarrow{AB} \\ \overrightarrow{BA} \\ \end{array}$ 83 $\overline{20}$ $\frac{1}{22}$	$\frac{24}{C_R \text{ [mm]}}$ $\frac{1}{25}$	$\frac{-\text{Lift}}{\text{P*}}$ $\overline{26}$	-23.9 $\begin{array}{r} 23.83 \overline{\text{m}} \\ \underline{\text{m}} \\ 23.7 \overline{\text{m}} \end{array}$ 23.7 $^{123.5}$	84.3 80 Lift [mN] 70 ¹ 60 50 $\overline{10}$	18.720 $\overline{16}$ f [Hz]	$\overline{1}40$ $\frac{-\text{Lift}}{\text{P*}}$ 30 20 $\frac{1}{24}$ ¹⁰	P^* [W/kg]

Table 5. Design optimization results for both morphological and kinematic parameters.

Fig. 11. Sensitivity of the lift force and energy consumption to each parameter.

 (c)

numerical noise occurs when solving the ODE in Eq. (5), which is difficult to eliminate because of the existence of either truncation error or round-off error [8].

5.2 Design optimization for morphological and kinematic parameters

To further investigate the effects of the parameters of ϕ_m and k_n on the performance, e.g.: The lift force and power consumption, design optimization considering wing morphology and kinematics is carried out. The design optimization model is

$$
\begin{cases}\nFind & X = \begin{bmatrix} R & AR & f & \phi_m & k_n \end{bmatrix} \\
min & f(X) = P^* \\
s.t. & 70 \le R \le 100; \quad 2.5 \le AR \le 6; \\
5 \le f \le 30; \quad 1 \times 10^{-4} \le k_n \le 15 \times 10^{-4}; \\
0 \le \phi_m \le 90^{\circ}; \quad Lift_{mean} \ge 84.28.\n\end{cases}
$$
\n(24)

 (d)

Fig. 12. Optimal shape after design optimization for morphological and kinematic parameters.

The combination of the subset simulation and gradientbased optimization method is also employed to solve the design optimization model, and the optimal results are shown in Table 5. Compared with the results in experiments, the energy consumption decreases by 45.6 % when the design optimiza-

Fig. 13. Sensitivity of the lift force and energy consumption to each parameter.

tion is conducted by considering both morphological and kinematic parameters. Meanwhile, a conclusion is reached that the kinematic parameters still imposes effects on the lift force generation and energy consumption. Note that the optimization result of ϕ_m is very close to 90°, which is the optimal sweeping amplitude of many insects in the hovering flight, including fruitfly, bumblebee and hawkmoth [22].

The optimal shape of the wing by considering both morpho logical and kinematic parameters is shown in Fig. 12. From Fig. 12, we know that the shape is opposite by considering kinematic parameters additionally compared with only considering morphological parameters, which is consistent with the experimental shape. These two opposite results may be the embodiment of the low impact of C_R on lift force generation and energy consumption.

The influence of each parameter on the lift force generation and power consumption is shown in Fig. 13. The sensitivity curves are similar to those in Figs. $11(a)-(d)$ and $13(a)-(d)$. The sensitivity curves of the lift force and energy consumption to ϕ_m increase during the cycle. The sensitivity curves of the lift force and energy consumption to k_n rise first and then decrease during the cycle.

Fig. 14. Sensitivity of the lift force and energy consumption to each parameter.

5.3 Design optimization for the location of pitching axis

Although the wing shape, flapping kinematics and distributed wing stiffness have been investigated, the influence of the location of the wing pitching axis is not considered. However, the location of the pitching axis affects the lift force and power consumption. Therefore, it is selected as another design variable to study the effect in this part.

The normalized parameter \hat{d} changes along the span, and the linear model is employed [27],

$$
\hat{d}\left(x_{c}\right) = \frac{1}{R}\left(\hat{d}_{t} - \hat{d}_{r}\right)x_{c} + \hat{d}_{r}
$$
\n(25)

where \hat{d}_r and \hat{d}_t are the values of the normalized parameter \hat{d} at the wing root and wing tip, respectively, and

ness changes and therefore is considered as a design variable. Under the consideration of the effect of flapping frequency on the lift force and power consumption, the flapping frequency is also considered as the design variable. The design optimization model is constructed by d therefore is considered as a design variable.

leration of the effect of flapping frequency on

1 power consumption, the flapping frequency

d as the design variable. The design optimiza-

structed by
 $\hat{d}_r \quad \hat{d}_t \quad$

20.6
\n10 15 18.7 25 30
\n
$$
f
$$
 [Hz]
\n(d)
\n15 18.7 25 30
\n f [Hz]
\n16 1
\n17.

The optimal design results and the initial design variables are provided in Table 6. It can be seen that the optimal pitchto the initial condition $\hat{d} = 0$, more wing area is put in the α_c front of the pitching axis after optimization. Furthermore, ρ_f 13.7 % energy will be saved since extra kinetic energy can be saved.

The sensitivity curves about the lift force and power con- $\hat{\alpha}$ sumption to each design parameters are provided in Fig. 14. It C_n^{est} is clearly seen that \hat{d}_r , k_n and f are positively correlated is clearly seen that d_r , k_η and f are positively correlated with the lift force and energy consumption within the constraints. But for \hat{d}_t , the sensitivity curves of the lift force rises \hat{d}_t . first and then decreases during the cycle.

6. Conclusions

We first employed DOE to select sensitive parameters for optimally designing the wing of a hummingbird-like MAV to decrease the computational expense during the design optimization procedure. As an important kinematic parameter, the distributed wing stiffness, was approximately estimated based on experimental data with the least square method, which provides the deterministic parameter for the design optimization to obtain the morphological parameters. Three design optimization models were then built by considering the mor phological parameters, both morphological and kinematic parameters and the location of pitching axis, respectively. The combination of subset simulation and gradient-based optimization was finally presented to solve the three design optimization models and corresponding sensitivity analysis was con ducted. With the design optimization, the energy consumption decreased by 42.9 %, 45.6 % and 13.7 % under the satisfaction of the produced lift force.

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Nomenclature-

- *R* : Wing span
- C_R : Wing root chord
- C_T : Wing tip chord
- *AR* : Aspect ratio
- \overline{c} : Mean chord length
-
- *S* : Wing area \hat{d} : The chord-normalized distance from the pitching axis to the leading edge
- ϕ_m : Sweeping amplitude
- ϕ_0 : Horizontal offset
- *f* : Frequency
- θ_m : Heaving amplitude
- Φ_{θ} : Heaving phase offset
- θ_0 : Heaving offset
- k_n : The distributed wing stiffness
- **ω^c** : Angular velocity
- **α^c** : Angular acceleration
- $\rho_{\rm f}$: Fluid density
- $C_{F_{\bar{r}_e}}^{\text{trans}}$: Translational force coefficient
- $\begin{array}{c} C^{\rm trans}_{\mathit{F_{y_c}}} \\ 2^{\rm trans}_{\mathit{cp}} \end{array}$ $\hat{z}_{cp}^{\text{trans}}$ *z* : Normalized chordwise center of pressure
- : Angle of attack
- C_D^{rot} : Rotational damping coefficient
- **:** Inertial torque
- *P* * : Total mass-normalized power consumption
- \hat{d} at the wing root
- \hat{d}_t : \hat{d} at the wing tip

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