

## Experimental investigation of the preferred Strouhal number used in self-resonating pulsed waterjet<sup>†</sup>

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### Abstract

Self-resonating pulsed waterjet (SRPW) is superior to plain waterjet in many ways and is being employed in numerous applications. To further improve the performance of SRPW, the optimal value of the preferred Strouhal number ( $S_d$ ), which is used to determine the chamber length of a self-resonating nozzle, was experimentally studied at inlet pressures of 10 MPa and 20 MPa. The axial pressure oscillation peak and amplitude were used to evaluate the performance of SRPW, in order to find the optimum  $S_d$  value. Results show that  $S_d$  value determines the self-resonance behavior of an organ-pipe nozzle and greatly affects the intensity of the axial pressure oscillation. Under the experimental conditions, the optimum  $S_d$  values are 0.315 and 0.278 respectively, corresponding to inlet pressures of 10 MPa and 20 MPa. Compared with the default value of 0.3 obtained from air jet experiment, the optimum  $S_d$  value at inlet pressure of 10 MPa is a little larger and oppositely a bit smaller at inlet pressure of 20 MPa. Thus, if the inlet pressure is not considered,  $S_d$  value of 0.3 is reasonable for determining the chamber length of a self-resonating nozzle for generating effective SRPW.

**Keywords:** Strouhal number; Self-resonating waterjet; Pressure oscillation; Organ-pipe nozzle

### 1. Introduction

Due to the outstanding advantages of high pressure waterjet (HPW) such as no heat-affected zone, low cost, and environmentally friendly, it has been considered as a promising technology that can be employed in many areas [1-5]. For example, Huang et al. [2] used HPW to remove heat-formed coating from a titanium alloy and found this technology is capable for the removal of such hard and difficult-to-machine coatings. Through an exploratory experimental investigation, Careddu and Akkoyun [3] demonstrated that HPW can be used for efficient graffiti cleaning with specific operational conditions. Moreover, Jiang et al. [4] used HPW for dust suppression in coal mines and found the dust concentration could be reduced by over 90 % on average when waterjet was used during rock cutting. And in the medical field, Dunnen et al. [5] claimed that HPW drilling provides several advantages over classic drilling with rigid drilling bits in orthopedic surgery.

With the rapid development of HPW technology, new types of waterjet like cavitating waterjet, abrasive waterjet, and self-resonating pulsed waterjet have been put forward and studied in order to meet the demand for more powerful erosion ability

without increasing the pump pressure [6-8]. And among these novel waterjets, SRPW has been especially focused after being proposed by Johnson et al. [9].

SRPW is a newly proposed novel waterjet that can be yielded with the use of specially structured nozzles. To be more specific, by using a suitable self-resonating chamber, the turbulent shear layer of a HPW can be amplified and structured and the jet can be passively disrupted, leading to the generation of a train of discrete water slugs. Then these water slugs produce high-frequency and high pressure shock wave impacts able to enhance the erosion capability and working efficiency of the jet. On the other hand, SRPW also has very high amplitude, periodic, and oscillatory discharge without using moving parts in the supply system, which helps eliminate the cushion effect and improve the performance. In other words, SRPW is a totally acoustically generated jet that takes advantage of the natural tendency of an axisymmetric turbulent jet to organize into large-scale structures [10]. Therefore, SRPW efficiently avoids the drawbacks of mechanically generated pulsed waterjet like, high-levels of mechanical maintenance, high cost, limited durability in harsh environment, and short life-times [11].

Because of the foregoing unique advantages of SRPW, it has been regarded as one of the most promising novel waterjets and has been the subject of a relative intense research

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effort in the literature. In more specific terms, Yoshikawa et al. [12] carried out an experiment to examine the vortex-sound generation process happening in an organ-pipe and proposed a vortex-layer formation model to provide a united viewpoint for the edge-tone generation. To improve the exploration and production of petroleum, Wang et al. [13] applied self-resonance concept in the drilling method and came up with a self-excited pulsed jet drilling technology. Moreover, Arthurs and Ziada [14] experimentally studied the influence of thickness of nozzle exit on self-excited impinging jet and concluded that jet oscillation is controlled by a fluid-dynamic mechanism for small impinge distances. Indeed, our research team has also conducted a series of experimental and numerical investigations on SRPW, from feeding pipe diameter to surface roughness [15-25].

Even the generation mechanism and new applications of SRPW have been greatly studied, the value of the preferred Strouhal number,  $S_d$ , however does not appear to be adequately treated in the literature. As is known to all, Strouhal number,  $S_s$ , is a dimensionless number describing oscillating flow mechanism and is particularly important in studying self-excited or self-sustained oscillations. In the mechanism study of SRPW, Johnson et al. [9] established an equation for estimating the ratio of chamber length to jet diameter based on empirical acoustic analysis, in order to get an effective self-resonating nozzle. According to the research of self-sustained air jet carried out by Crow and Champagne [26],  $S_d = 0.3$  was used in the equation. And the same value was also used in the design of organ-pipe resonating nozzles by Li et al. [27], who employed SRPW in China oil field. However, as is well-known, air jet is much different from waterjet, and  $S_d = 0.3$  obtained in the air jet experiment may not satisfy the self-excited waterjet. So far, to the best of our knowledge, there is currently very little literature on this.

The present study provides an experimental investigation on the pressure oscillation characteristics of SRPW issuing from organ-pipe nozzles of different chamber lengths, in order to obtain the optimum  $S_d$  values for determining the nozzle structures under different working pressures. As a continuation of our previous research [15-25] on SRPW, this study also helps make a contribution to improving the working efficiency of SRPW.

## 2. Strouhal number in SRPW

In order to show the importance of  $S_t$  in SRPW, a brief description of the operating principles of the jet discharging from an organ-pipe nozzle is presented as follows.

As is depicted in Fig. 1, the typical self-resonating organ-pipe nozzle consists of two area contractions, defined by  $(D_i/D_c)^2$  and  $(D_c/D_e)^2$ , and a resonance chamber of diameter of  $D_c$  and length of  $L_c$ . Pressure waves will be generated at the downstream contraction if a flow is rapidly passing through the nozzle. The waves propagating upwards are then reflected by the upstream contraction and superimpose with the original ones.

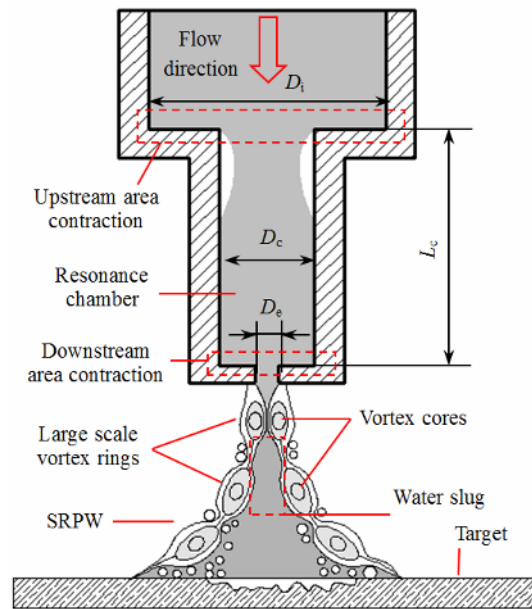


Fig. 1. Schematic of the working principles of SRPW.

And a standing wave will be formed in the resonance chamber if the fundamental frequency ( $f_0$ ) of the nozzle meets the main frequency ( $f_v$ ) of the strongest pressure wave. The standing wave is capable of structuring the shear layer of the jet into large scale vortex rings, resulting in large pressure oscillations of the jet and finally the emergence of SRPW. More detail information of the generation mechanism of SRPW can be found in Refs. [9, 10, 16, 17, 20-24].

Furthermore, it is well-known that  $f_0$  is determined by chamber length  $L_c$  and can be expressed as:

$$f_0 = K_n \frac{c}{L_c} \quad (1)$$

where  $c$  is local sound speed,  $K_n$  is model number.

And  $f_v$  can be expressed with the use of  $S_t$ :

$$f_v = \frac{S_t U_e}{D_e} \quad (2)$$

where  $U_e$  is the jet velocity, and here jet diameter has been equal to the exit diameter  $D_e$  for the sake of simplicity.

To get an effective SRPW, the two frequencies should be equal, which leads to the following equation:

$$f_0 = f_v \Rightarrow \frac{L_c}{D_e} = \frac{K_n c}{U_e S_d} \quad (3)$$

Therefore, once  $D_e$  is given,  $L_c$  can be obtained by the value of  $S_d$ . In other words, the value of  $S_d$  determines the effectiveness of the self-resonating organ-pipe nozzle and further the performance of SRPW.

Even  $S_d$  is dramatically important in determining the struc-

ture parameter ( $L_c$ ) of an organ-pipe nozzle, there is currently very little literature on verifying the accuracy of  $S_d = 0.3$  for SRPW. And this study is trying to provide an experimental attempt to figure out this problem.

### 3. Experimental setup and procedures

#### 3.1 Facilities and setup

Since the axial pressure oscillation peak and amplitude are among the most distinct characteristics of SRPW [9, 10, 16–24], these were used to evaluate the performance. Therefore, the axial pressure measurement experiment was performed.

Fig. 2 presents a schematic diagram of the experimental setup. The experiment was carried out on a multifunctional waterjet test bed that had been used in our previous study [15–25]. The test bed had a horizontal moving table with X-direction movement and a loading platform with Y-direction and Z-direction movements. High pressure water was provided by a plunger pump whose maximum working pressure and flow rate were 60 MPa and 120 L/min, respectively. The movements and the pump pressure could be regulated by the control table. The temperature of the tap water was 20 °C, consistent with that of the environment.

To minimize the flow rate and pressure disturbances, two bladder accumulators were employed in the system with one positioned near the pump and the other close to the testing nozzle, shown in Fig. 2.

A static pressure transducer (Model: BD Sensors DMK331P) was located as closely as possible to the entrance of the nozzle to get the inlet pressure ( $P_i$ ) for each test. During each test, SRPW impinged on a target plate with a 1 mm diameter hole located at its center. A dynamic pressure transducer (Model: XPM10) communicating with the pressure tap was mounted on the wall of the tank, so as to obtain the time-resolved axial pressure of the jet. The test results were then

acquired by the data logger (Model: MX840B) connected with a laptop. And the maximum and minimum pressures ( $P_{max}$  and  $P_{min}$ ) could be directly obtained from the laptop. The main parameters of the transducers are shown in the following tables.

Standoff distance,  $S$ , was defined as the distance between the exit of the nozzle and the surface of the target plate. Various standoff distances from 10 mm to 100 mm with an interval of 10 mm were applied during each test.

#### 3.2 Organ-pipe nozzles design

Based on Eq. (3), it can be found that  $U_c$  should be given to get the relation between  $L_c$  and  $S_d$ . In order to verify the rea-

Table 1. Main parameters of the static pressure transducer.

Model	Parameter	Value
BD DMK331P	Nominal pressure/MPa	40
	Burst pressure/MPa	100
	Accuracy	$\leq \pm 0.5\%$ FS
	Influence effects of supply/10 V	0.05 %FS
	Influence effects of load/k $\Omega$	0.05 %
	Thermal error/10 K	$\pm 0.5\%$ FS

Table 2. Main parameters of the dynamic pressure transducer.

Model	Parameter	Value
XPM 10	Nominal pressure/MPa	35
	Burst pressure/MPa	105
	Frequency response/KHz	288
	Accuracy	$\leq \pm 0.25\%$ FS
	Repeatability	$\pm 0.2\%$
	Nonlinearity	$\pm 0.25\%$

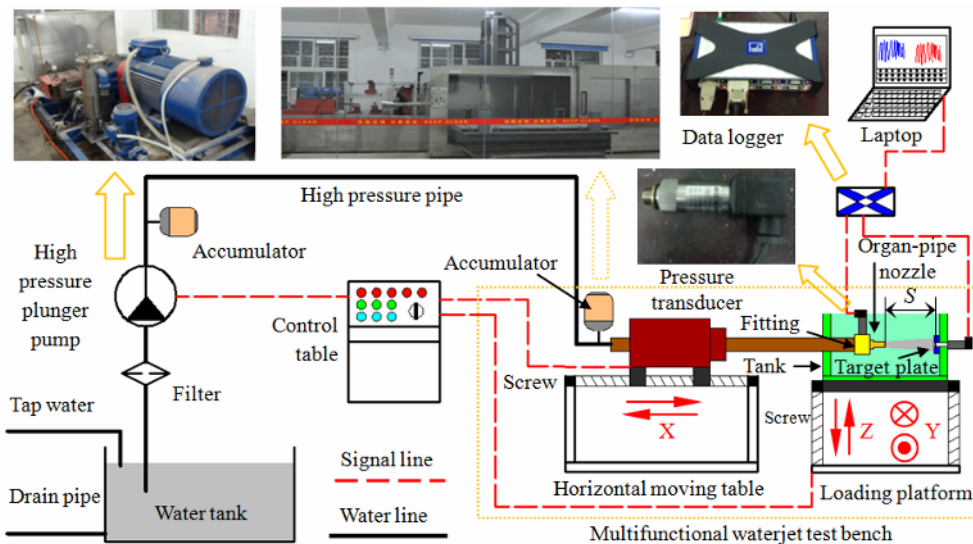


Fig. 2. Schematic diagram of the experimental setup for the pressure tests.

Table 3. Parameters of organ-pipe nozzles and corresponding values of  $S_d$ .

$P_i$ /MPa	$L_c$ /mm, $S_d$							$D_i$ /mm	$D_c$ /mm	$D_e$ /mm	$L_e$ /mm
	18	19	20	21	22	23	24				
10	0.35	0.33	0.315	0.3	0.286	0.27	0.26	13	5	2	5
	12	13	14	15	16	17	18				
20	0.37	0.34	0.318	0.3	0.278	0.262	0.25	13	5	2	5
	12	13	14	15	16	17	18				

sonability of  $S_d = 0.3$  and find the optimal value, different chamber lengths were used to get different  $S_d$  values. And it should be emphasized that organ-pipe nozzle has two abrupt area contractions, the energy loss under high pressure condition is relatively large and cannot be neglected. So, the process for calculating  $U_e$  is shown below:

The Bernouli equation gives the express of  $U_e$ :

$$\frac{P_i}{\rho g} + \frac{U_i^2}{2g} = \frac{P_a}{\rho g} + \frac{U_e^2}{2g} + h_f + h_j \tag{4}$$

where  $P_i$  is inlet pressure,  $\rho$  is the liquid density,  $g$  is the acceleration of gravity,  $h_f$  is the frictional head loss,  $h_j$  is the local head loss. And  $h_f$  and  $h_j$  can be written as:

$$h_f = \lambda \frac{l}{d} \frac{u^2}{2g} \tag{5}$$

$$h_j = k \frac{u^2}{2g} \tag{6}$$

where  $\lambda$  is the friction coefficient,  $l$  is the length of the flow channel (here it is the nozzle length),  $d$  is channel diameter,  $u$  is flow velocity, and  $k$  is the local resistance coefficient. Under the sudden contraction condition,  $k$  can be expressed as:

$$k = \frac{1}{2} \left( 1 - \left( \frac{A_2}{A_1} \right)^2 \right)^2 \tag{7}$$

where  $A_1$  and  $A_2$  are the cross-section areas at the contraction.

In Eq. (5),  $\lambda$ ,  $l$  and  $u$  are rather small. So, it can be neglected during the calculation. When compared with  $U_e$ ,  $U_i$  can also be neglected.

Then, based on the experimental results obtained by Li et al. [27], the structure parameters of the organ-pipe nozzles used in this experiment could be achieved by combining Eqs. (3), (4), (6) and (7). It should be noticed that during the calculation,  $S_d = 0.3$  was first assumed reasonable, as this value has already been used in many previous investigations [9, 10, 16-24]. After the resulted optimal  $L_c$  was obtained, six another lengths next to it were used to get the corresponding  $S_d$  values, shown in Table 3. And the pictures of the nozzles are shown in Fig. 3.

### 4. Results and discussion

The performance of SRPW, which could be characterized



Fig. 3. Pictures of the organ-pipe nozzles.

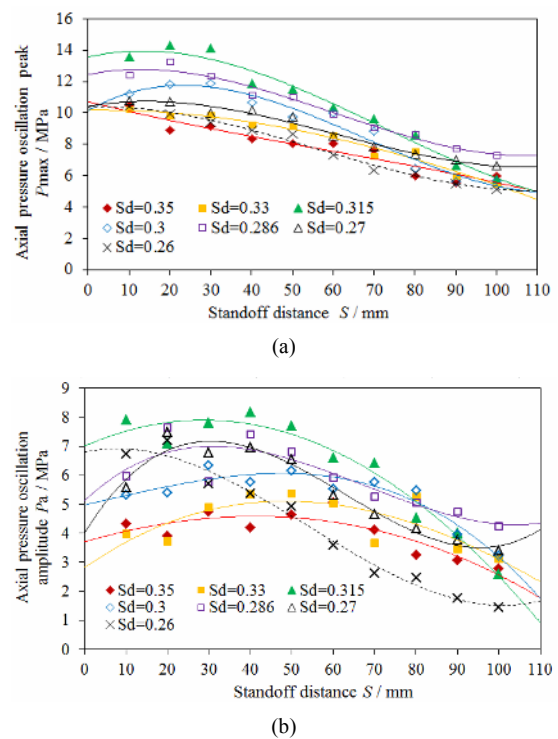


Fig. 4. Axial pressure oscillation peak (a); amplitude (b) against stand-off distance at  $P_i = 10$  MPa.

by the axial pressure oscillation peak ( $P_{max}$ ) and amplitude ( $P_a$ ), was used to evaluate the optimal value of  $S_d$ . And  $P_a$  was defined as the difference between  $P_{max}$  and  $P_{min}$ .

#### 4.1 Effects of the value of $S_d$ at $P_i = 10$ MPa

Fig. 4 is the axial pressure oscillation peak and amplitude of SRPW against standoff distance with respect to various  $S_d$  values under inlet pressure of 10 MPa. As can be observed in

the figure, the value of  $S_d$  greatly affects the pressure oscillations, especially the amplitude, because the curves are more scattered with each other compared with the situation of pressure oscillation peak.

Specifically, within the range of  $0.27 < S_d < 0.33$ , the pressure oscillation peak first gradually goes up and then drops gently with the increase of standoff distance, shown in Fig. 4(a). At standoff distance around 20 mm, which is defined as optimum standoff distance, the pressure oscillation peak reaches to its maximum. These are unique features of SRPW and are in good agreement with the previous study [9, 17, 21, 23]. On the other hand, when the value of  $S_d$  is out of the range, the pressure oscillation peak almost linearly decreases with standoff distance, indicating that the jet is losing self-resonance feature and seems to be transforming to a regular waterjet. And the pressure oscillation peak can hardly exceed the inlet pressure. Therefore, it can be expected that there is a specific range for the value of  $S_d$  in order to generate obvious pressure oscillation peaks and effective SRPWs.

For the pressure oscillation amplitude shown in Fig. 4(b), it first increases and then decreases with the increase of standoff distance for almost all  $S_d$  values. There also exists an optimum standoff distance for achieving the maximum pressure oscillation amplitude. And this optimum standoff distance is about 10 mm larger than that for the maximum pressure oscillation peak. Besides,  $S_d$  values of 0.35 and 0.33 both lead to relatively smaller pressure oscillation amplitudes within the testing standoff distance. Even 0.26 is the smallest value in this experiment, the pressure oscillation amplitude experiences a rapid decrease with increasing standoff distance. Contrary to the pressure oscillation peak under  $S_d = 0.26$  condition, the pressure oscillation amplitude is abnormally relatively larger at standoff distances less than 50 mm. And based on the previous study [17, 21, 23], this should not be the feature of SRPW. However, due to the very limited research on the value of  $S_d$  in SRPW, this phenomenon can hardly be clearly explained at the current stage.

In general, under the condition of  $S_d = 0.315$ , the axial pressure oscillation is more violent than any other case with respect to larger pressure oscillation peaks and amplitudes. This indicates that 0.315, which is a little larger than the commonly used value achieved from air jet, should be the optimal value at inlet pressure of 10 MPa.

#### 4.2 Effects of the value of $S_d$ at $P_i = 20$ MPa

Fig. 5 shows the axial pressure oscillation peak and amplitude against standoff distance at inlet pressure of 20 MPa under the condition of various  $S_d$  values. It is obvious that the value of  $S_d$  plays an important role in affecting the performance of SRPW, and a suitable value can greatly intensify the axial pressure oscillation, especially for the pressure oscillation amplitude at relatively larger standoff distances.

For the pressure oscillation peak illustrated in Fig. 5(a), it has the unique feature of SRPW only at  $S_d$  values of 0.318, 0.3

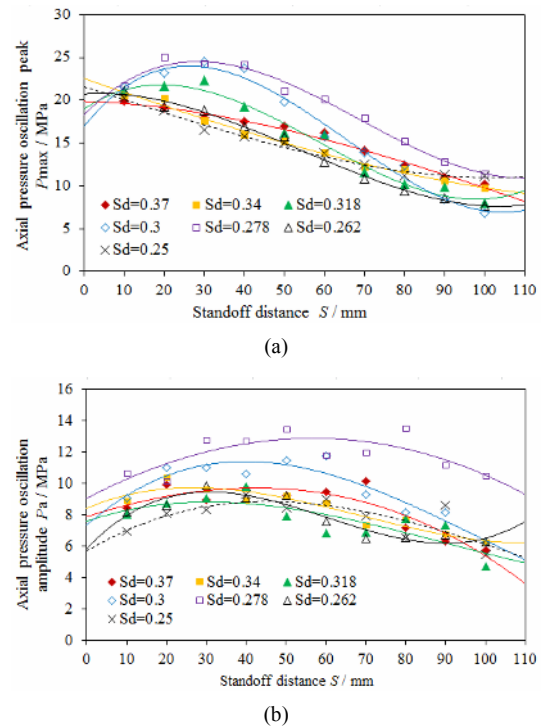


Fig. 5. Axial pressure oscillation peak (a); amplitude (b) against standoff distance at  $P_i = 20$  MPa.

and 0.278. While in the other cases, the pressure oscillation peak decreases with increasing standoff distance and is more like the feature of a traditional submerged turbulent jet at relatively larger standoff distances. It should be noticed when the value is larger than 0.318 or smaller than 0.278, the curves are almost overlapped in the testing standoff distance, meaning the jets have nearly the same features. This also indicates that the nozzles have little self-resonance behavior and thus, the value of  $S_d$  hardly affects the axial pressure characteristics. Furthermore, the optimum standoff distance here is about 10 mm larger than that under inlet pressure of 10 MPa, which is in consistent with the previous study [16-24] and can be evidence for the reasonability of the experimental results.

When it comes to the pressure oscillation amplitude shown in Fig. 5(b), it is found that  $S_d$  values of 0.3 and 0.278 result in relatively larger amplitudes compared with the other cases. In addition, the optimum standoff distances are obviously larger than the corresponding ones for the pressure oscillation peak. Except for two values of 0.3 and 0.278, the overlap of the curves of the other values indicates the pressure oscillation amplitude is more sensitive to the value of  $S_d$  at higher inlet pressures. Because  $S_d$  value of 0.318 affects the pressure oscillation peak more obviously.

Generally, as can be observed in Fig. 5,  $S_d$  value of 0.278 should be the optimum at inlet pressure of 20 MPa, and both the pressure oscillation peak and amplitude are superior to the others under the experimental conditions. This also means the commonly used value of 0.3 is a little larger than the optimum value in this experiment.

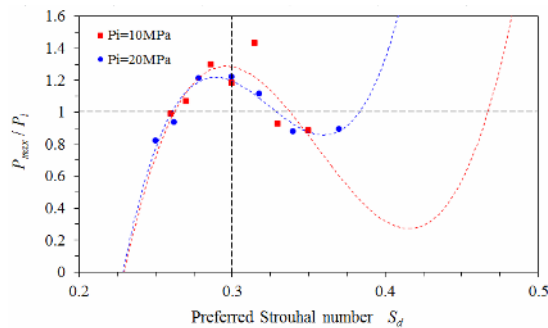


Fig. 6. Non-dimensional axial pressure oscillation peak as a function of the preferred Strouhal number at  $S = 20$  mm for  $P_i = 10$  MPa and  $S = 30$  mm for  $P_i = 20$  MPa.

### 4.3 Non-dimensional analysis

To further analyze the effects of  $S_d$  value on the performance of SRPW, the pressure oscillation peak has been non-dimensionalized by the inlet pressure. And the dimensionless pressure as a function of  $S_d$  under inlet pressures of 10 MPa and 20 MPa is plotted in Fig. 6. The pressure oscillation peaks at  $S = 20$  mm and  $S = 30$  mm for the two inlet pressures were used to analyze the effects. Moreover, cubic trend lines are added in the figure as well, in order to predict the effects of  $S_d$  values beyond the experimental ones.

As is shown in the figure, the axial pressure oscillation peak larger than the inlet pressure occurs in a small range of  $0.27 < S_d < 0.32$ . And the dots distribute on both sides of the line of  $S_d = 0.3$ , which indicates that this value could be the most suitable one for achieving higher pressure oscillation peaks without taking the inlet pressure into consideration. From this point of view, the use of  $S_d = 0.3$  in the previous study conducted by the pioneering researchers should be reasonable. And from the trendlines shown in the figure, it can be noticed that the pressure oscillation peak has a second increase when  $S_d$  value reaches to a certain range. Specifically, when  $S_d > 0.4$ , the pressure oscillation peaks starts to rise under  $P_i = 10$  MPa, while this value is 0.47 for the case of  $P_i = 20$  MPa. If the experimental results obtained by Johnson et al. [9, 10] are taken into account, this can be explained by the mode number shown in Eq. (2). And from their results, all integer times of 0.3 can create self-resonance, but the intensities are different. Therefore, this also provides evidence for the reasonability of the experimental results.

However, it should be addressed that due to the rather limited literature on this aspect, this is a preliminary study on the effects of  $S_d$  values on the performance of SRPW, and the discussion is somewhat prospective. Further investigations with the use of mathematic and quantitative methods should be performed.

### 5. Conclusions

(1) The value of  $S_d$  greatly determines the axial pressure oscillation peak and amplitude of SRPW, a suitable value can

obviously enhance the performance of the jet.

(2) At inlet pressure of 10 MPa, waterjet has the features of typical SRPW only when the value is within the range of  $0.27 < S_d < 0.33$ . And 0.315 is the optimum value for obtaining the maximum pressure oscillation peaks and amplitudes at the optimum standoff distances.

(3) When the inlet pressure is 20 MPa, the nozzles will become regular ones without self-resonance behavior under the experimental conditions if  $S_d$  value does not locate in the range of  $0.278 < S_d < 0.318$ . And 0.278 is superior to 0.3 for intensifying the oscillation.

(4) In general,  $S_d = 0.3$  is reasonable to be used for guiding the fabrication of self-resonating nozzles if the inlet pressure is not considered.

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