Closed-form solution to thin-walled box girders considering effects of shear deformation and shear lag

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Abstract: Considering three longitudinal displacement functions and uniform axial displacement functions for shear lag effect and uniform axial deformation of thin-walled box girder with varying depths, a simple and efficient method with high precision to analyze the shear lag effect of thin-walled box girders was proposed. The governing differential equations and boundary conditions of the box girder under lateral loading were derived based on the energy-variational method, and closed-form solutions to stress and deflection corresponding to lateral loading were obtained. Analysis and calculations were carried out with respect to a trapezoidal box girder under concentrated loading or uniform loading and a rectangular box girder under concentrated loading. The analytical results were compared with numerical solutions derived according to the high order finite strip element method and the experimental results. The investigation shows that the closed-form solution is in good agreement with the numerical solutions derived according to the high order finite strip method and the experimental results, and has good stability. Because of the shear lag effect, the stress in cross-section centroid is no longer zero, thus it is not reasonable enough to assume that the strain in cross-section centroid is zero without considering uniform axial deformation.

Key words: shear lag effect; thin-walled box girder; energy-variational method; shear deformation; closed-form solution

1 Introduction

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Box girders have good structural properties of light weight and large flexural and tortional rigidity, and desirable lateral-distributed load could be obtained by installing a small amount of diaphragms. The prestressed reinforcements in the top and bottom plates could effectively resist the positive and negative moments, which are suitable for continuous beam bridges, rigid frame bridges and especially continuous horizontally curved box girder bridges. Cantilever erection, cantilever casting and incremental launching method are all available in dealing with construction of box girder, which satisfies the demand of modern construction. Therefore, box girders have been widely used and promoted in bridge construction [1−3]. However, as for the wide box girder with web plates of large spacing under a symmetrical loading, the distribution of bending stresses in thin-walled box girders at any transverse section is non-uniform, and in general, the stresses at the web-flange conjunction reach their maximum, decreasing towards the middle point of the top and bottom slabs and cantilever flanges. Its deformation does not obey the elementary beam theory. This phenomenon is called the shear lag effect [4−5]. The stress concentration induced by shear lag effect is likely to cause failure of the box girder when it is severe [1, 6−8].

The shear lag of the box girder has been studied without considering the effect of shear deformation [9−13]; NI [14] proposed warping displacement mode to consider the self-balancing of axial force, yet without taking the shear deformation of cross-section into account; the researches (e.g., Refs. [11−14]) were all based on the assumption that the functions of longitudinal warping amplitude of the upper and lower flange and the cantilever plate were the same. While LUO [15−18] proved that the assumption was valid only for rectangular box girders but invalid for trapezoidal box girders, so this assumption made in Refs. [11−14] was abandoned. Three longitudinal displacement functions and uniform axial displacement functions for thin-walled box girder with varying depths for uniform axial deformation were taken into account, which were

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proved to be more reasonable. However, LUO [15−18] didn't consider the uniform axial deformation of the whole cross-section due to shear lag effect. The uniform axial deformation made the centroidal axis no longer the neutral axis, which meant that axial stress existed in the centroid of the cross section.

In this work, a simple and efficient analytical model with high precision was proposed, taking shear lag, shear deformation and uniform axial deformation into account. The governing differential equations and boundary conditions of the box girder were derived based on energy-variational method [2, 19−20], and the closed-form solution was found. Finally, results were compared with numerical solutions derived according to the high order finite strip method and the experimental results [17]. The calculation method was based on box girder under bending load.

2 Basic assumptions

1) Let $g_i(x, y)(i = 1, 2, 3)$ represent longitudinal warping displacement function of the cantilever plate, the center of top plate and bottom plate, respectively (Fig. 1). According to the fact that the lateral shear strains of the edge of cantilever plate, the center of the roof plate and base plate are all zero, and considering the compatibility of the displacement of the flange and the web and the compatibility coordination of the displacement of the cantilever plate and the top plate, we can obtain

$$
g_1(x, b_1) = g_2(x, b_1) = g_3(x, b_3) = 0
$$

$$
\frac{\partial g_1(x, 0)}{\partial y} = \frac{\partial g_2(x, b_1 + b_2)}{\partial y} = \frac{\partial g_3(x, 0)}{\partial y} = 0
$$

Therefore, it could be assumed that the longitudinal warping displacement of the top plate, cantilever plate and bottom plate are expressed as

Fig. 1 Box girder of trapezoidal cross section: (a) Cross section; (b) Coordinate and load

$$
g_1(x, y) = h_1 \left(1 - \frac{y^2}{b_1^2} \right) U_1(x), \quad -b_1 \le y \le b_1 \tag{1a}
$$

$$
g_2(x, y) = h_2 \left(1 - \frac{\overline{y}^2}{b_2^2} \right) U_2(x), \quad 0 \le \overline{y} \le b_2
$$
 (1b)

$$
g_3(x, y) = h_3 \left(1 - \frac{y^2}{b_3^2} \right) U_3(x), \quad -b_3 \le y \le b_3 \tag{1c}
$$

where $\overline{y} = b_1 + b_2 - y$; 2*b*₁, *b*₂, 2*b*₃ are the depths of top plate, cantilever plate, and bottom plate, respectively; h_i $(i=1, 2, 3)$ are coordinate values of the middle section of top plate, cantilever plate and bottom plate, respectively; *H* is the total height of the girder.

2) The longitudinal displacement of each point on the cross-section of a box girder can be assumed to be the sum of the plane surface deflection, warping displacement, and uniform axial displacement. Using $u_0(x)$, the self-balanced axial force can be obtained:

$$
u_i(x, y) = h_i \left[-\theta(x) + \left(1 - \frac{y^2}{b_i^2} \right) U_i(x) \right] + u_0(x),
$$

$$
-b_i \le y \le b_i, \quad i = 1, 3
$$
 (2a)

$$
u_2(x, y) = h_2 \left[-\theta(x) + \left(1 - \bar{y}^2 / b_2^2 \right) U_2(x) \right] + u_0(x),
$$

0 \le \bar{y} \le b_2, \quad \bar{y} = b_1 + b_2 - y \tag{2b}

$$
u_4(x, z) = -z\theta(x) + u_0(x), -h_1 \le z \le h_3
$$
 (2c)

where $\overline{y} = b_1 + b_2 - y$; $\theta = w'(x) - \gamma(x)$; $\gamma = \alpha_s Q/GA_w$) is the average shear strain; α_s is the shear coefficient; A_w is the cross-sectional area of the two web plates; *G* is the shear modulus; $Q(x)$ is the shear force; $W(x)$ is the vertical deflection of the box girder; $U(x)$ ($i=1, 2, 3$) are the functions of longitudinal warping amplitude; U_0 is the uniform longitudinal displacement; $2b_1$, b_2 , $2b_3$ and t_1 , t_2 , t_3 are the depths and thicknesses of top plate, cantilever plate, and bottom plate, respectively; β and t_4 are the slope angle and thickness of the web.

3) Vertical compression of the flange, lateral strain and bending, and shear deformation out of the plate are small enough to be neglected.

3 Governing differential equation and its solution

3.1 Expression of total potential energy

Based on the aforementioned expression of the vertical displacement on the cross-section, the strain of flange at any point is given by

$$
\varepsilon_i = \frac{\partial u_i}{\partial x} = h_i \left[-\theta' + \left(1 - \frac{y^2}{b_i^2} \right) U'_i \right] + u'_0, \ i = 1, 2, 3 \tag{3a}
$$

$$
\gamma_4 = \gamma = \alpha_s Q / (G A_w) \tag{3c}
$$

$$
\varepsilon_4 = -z\theta' + u'_0 \tag{3d}
$$

where *y* is replaced by \bar{y} when *i*=2; ε_i (*i* = 1, 2, 3, 4) are the longitudinal positive strains of the top plate, cantilever plate, bottom plate and web plate, respectively; γ_i (*i* = 1, 2, 3, 4) are the shear strains of the top plate, cantilever plate, bottom plate and web plate, respectively.

According to law of conservation of energy, the external potential energy caused by bending can be expressed in terms of internal force as

$$
\overline{V} = \int_{0}^{L} M \theta' dx - \int_{0}^{L} Q \gamma dx
$$
 (4)

The strain energies of top plate, cantilever plate, bottom plate and web plate can be derived based on energy-variational method in conjunction with above assumption as follows:

$$
\overline{U}_t = \frac{1}{2} \int_0^L \int_0^{b_1} 2t_1 \left(E \varepsilon_1^2 + G \gamma_1^2 \right) \mathrm{d}y \mathrm{d}x \tag{5}
$$

$$
\overline{U}_c = \frac{1}{2} \int_0^L \int_0^{b_2} 2t_2 \left(E \varepsilon_2^2 + G \gamma_2^2 \right) \mathrm{d}y \mathrm{d}x \tag{6}
$$

$$
\overline{U}_b = \frac{1}{2} \int_0^L \int_0^{b_3} 2t_3 \left(E \varepsilon_3^2 + G \gamma_3^2 \right) \mathrm{d}y \mathrm{d}x \tag{7}
$$

$$
\overline{U}_{\rm w} = \int_0^L \int_{h_1}^{h_3} E t_4 \varepsilon_4^2 / \cos \beta dz dx + \frac{G A_{\rm w}}{2} \int_0^L \gamma^2 dx \tag{8}
$$

Substituting Eq. (3) into Eqs. (5)−(8) results in

$$
\overline{U}_{t} = \frac{EI_{1}}{2} \int_{0}^{L} \left[\frac{u_{0}'^{2}}{h_{1}^{2}} + \frac{8}{15} U_{1}'^{2} + \theta'^{2} - \frac{4}{3} \theta' U_{1}' + 2 \frac{u_{0}'}{h_{1}} \left(\frac{2}{3} U_{1}' - \theta' \right) + \frac{4}{3} \frac{G}{E h_{1}^{2}} U_{1}^{2} \right] dx
$$
\n(9)

$$
\overline{U}_{c} = \frac{EI_{2}}{2} \int_{0}^{L} \left[\frac{u_{0}'^{2}}{h_{2}^{2}} + \frac{8}{15} U_{2}'^{2} + \theta'^{2} - \frac{4}{3} \theta' U_{2}' + 2 \frac{u_{0}'}{h_{2}} \left(\frac{2}{3} U_{2}' - \theta' \right) + \frac{4}{3} \frac{G}{E b_{2}^{2}} U_{2}^{2} \right] dx
$$
\n(10)

$$
\overline{U}_{\text{b}} = \frac{EI_3}{2} \int_0^L \left[\frac{u_0^2}{h_3^2} + \frac{8}{15} U_3^2 + \theta'^2 - \frac{4}{3} \theta' U_3' + 2 \frac{u_0'}{h_3} \left(\frac{2}{3} U_3' - \theta' \right) + \frac{4}{3} \frac{G}{E b_3^2} U_3^2 \right] dx \tag{11}
$$

$$
\bar{U}_{\rm w} = \frac{EI_{\rm w}}{2} \int_0^L \left[\theta'^2 - \frac{3\left(h_3^2 - h_1^2\right)}{\left(h_3^3 - h_1^3\right)} \theta' u_0' \right] +
$$

$$
\frac{3(h_3 - h_1)}{(h_3^3 - h_1^3)} u_0^2 \left[dx + 0.5GA_w \int_0^L \gamma^2 dx \right]
$$
 (12)

where E is the elastic modulus; G is the shear modulus; $I_i = 2b_i t_i h_i^2 (i = 1, 2, 3)$; $I_w = 2t_4 (h_3^3 - h_1^3)/$ (3cos*β*).

If the origin of coordinate system is set at the mid-height location of the cross-section, we get

$$
h_3 = -h_1 = -h_2 = H/2 \tag{13}
$$

Consequently, the potential energy of the total system is

$$
\Pi = \overline{V} + \overline{U}_t + \overline{U}_c + \overline{U}_b + \overline{U}_w \tag{14}
$$

3.2 Governing differential equations

Substituting Eqs. (4), (9) − (12) into the above equation, and letting δ*Π*=0, we get the governing differential equations through integration of each term:

$$
\theta''' - \varsigma u_0''' - \frac{2}{3} \sum_{i=1}^3 \alpha_i U_i''' - \frac{q}{EI} = 0 \tag{15}
$$

$$
\varsigma \theta'' - \xi u_0'' - \frac{2}{3} \sum_{i=1}^3 \frac{\alpha_i}{h_i} U_i'' = 0
$$
 (16)

$$
\theta'' - \frac{4}{5}U_i'' + \frac{2G}{Eb_i^2}U_i - \frac{u_0''}{h_i} = 0, \ i = 1, 2, 3
$$
 (17)

$$
\left(\theta'' - \zeta u_0'' - \frac{2}{3} \sum_{i=1}^3 \alpha_i U_i'' + \frac{Q}{EI}\right) \delta w \Big|_0^L = 0
$$
\n(18)

$$
\left(\theta'-\varsigma u_0'-\frac{2}{3}\sum_{i=1}^3\alpha_i U_i'+\frac{M}{EI}\right)\delta\theta\Big|_0^L=0\tag{19}
$$

$$
\left(-\zeta \theta' + \xi u_0' + \frac{2}{3} \sum_{i=1}^3 \frac{\alpha_i}{h_i} U_i'\right) \delta u_0 \Big|_0^L = 0
$$
 (20)

$$
\left(\frac{4}{5}U_i' - \theta' + \frac{u_0'}{h_i}\right) \delta u_i \Big|_0^L = 0, \ i = 1, 2, 3 \tag{21}
$$

where
$$
I = \sum_{i=1}^{3} I_i + I_w
$$
; $\alpha_i = I_i / I(i = 1, 2, 3)$; $\varsigma = \sum_{i=1}^{3} \frac{\alpha_i}{h_i}$;
\n $\xi = \frac{12I_w}{IH^2} + \sum_{i=1}^{3} \frac{4\alpha_i}{H^2}$.

Equations (15) − (17) are the governing differential equations of the deformation of the box girder, and Eqs. (18)−(21) are the boundary conditions of the deformation of the box girder.

3.3 Closed-form solution of governing differential equations

The first integration of Eq. (16) gives

J. Cent. South Univ. (2012) 19: 2650−2655 2653

$$
\theta'' - \varsigma u_0'' = \frac{2}{3} \sum_{i=1}^3 \alpha_i U_i'' + \frac{qx}{EI} + c_1
$$
 (22)

Substituting Eq. (22) into Eq. (16), we obtain

$$
\theta'' = \sum_{i=1}^{3} \kappa_i U_i'' + \xi_0 \frac{qx}{EI} + \xi_0 c_1 \tag{23}
$$

$$
u_0'' = \sum_{i=1}^3 \mathcal{G}_i U_i'' + \mathcal{G}_0 \frac{qx}{EI} + \mathcal{G}_0 c_1 \tag{24}
$$

where $\kappa_i = \frac{2\alpha_i(\xi - \zeta/h_i)}{2\alpha_i}$ $(\xi-\varsigma^2)$ 2 3 $\dot{a} = \frac{2a_i (s - s/n_i)}{2(n-2)}$ $\kappa_i = \frac{2\alpha_i(\xi - \zeta/h_i)}{3(\xi - \zeta^2)}$, $\theta_i = \frac{2\alpha_i(\zeta - 1/h_i)}{3(\xi - \zeta^2)}$ $2\alpha_i(\varsigma -1)$ 3 $R_i = \frac{2a_i (s - 1/n_i)}{2(n-2)}$ $heta_i = \frac{2\alpha_i(\varsigma - 1/h)}{h}$ $=\frac{2\alpha_i(\zeta-1/h_i)}{3(\zeta-\zeta^2)}$, $\xi_0=\zeta/$ $(\xi-\zeta^2)$, and $\zeta_0 = \zeta/(\xi-\zeta^2)$.

Substituting Eqs. (23)−(24) into Eq. (17) gives

$$
\begin{bmatrix} U_1^{\dagger} \\ U_2^{\dagger} \\ U_3^{\dagger} \end{bmatrix} - A^{-1} B \begin{bmatrix} U_1 \\ U_2 \\ U_3 \end{bmatrix} = A^{-1} \begin{bmatrix} \psi_1 \\ \psi_2 \\ \psi_3 \end{bmatrix}
$$
(25)

$$
A = \begin{bmatrix} \kappa_1 - \mathcal{A}/h_1 - 0.8 & \kappa_2 - \mathcal{A}/h_1 & \kappa_3 - \mathcal{A}/h_1 \\ \kappa_1 - \mathcal{A}/h_2 & \kappa_2 - \mathcal{A}/h_2 - 0.8 & \kappa_3 - \mathcal{A}/h_2 \\ \kappa_1 - \mathcal{A}/h_3 & \kappa_2 - \mathcal{A}/h_3 & \kappa_3 - \mathcal{A}/h_3 - 0.8 \end{bmatrix}
$$

$$
B = \frac{-2G}{E} \begin{bmatrix} 1/b_1^2 & 0 & 0 \\ 0 & 1/b_2^2 & 0 \\ 0 & 0 & 1/b_3^2 \end{bmatrix}
$$

where $\psi_i = -1.5 \kappa_i \left[q x / (EI) + c_1 \right] / \alpha_i$. The homogeneous solution of Eq. (26) is

$$
U_{i0} = \sum_{j=1}^{3} c_{ij} \left(d_j \cosh \sqrt{\lambda_j} x + e_j \sinh \sqrt{\lambda_j} x \right) \tag{26}
$$

When *q* is constant, the particular solution of Eq. (25) is

$$
U_{i1} = 0.5 E b_i^2 \psi_i / G \tag{27}
$$

The general solution of Eq. (26) is

$$
U_i = \sum_{j=1}^{3} c_{ij} \left(d_j \cosh \sqrt{\lambda_j} x + e_j \sinh \sqrt{\lambda_j} x \right) +
$$

$$
Eb_i^2 \psi_i / (2G)
$$
 (28)

$$
U'_{i} = \sum_{j=1}^{3} c_{ij} \sqrt{\lambda_{j}} \left(d_{j} \sinh \sqrt{\lambda_{j}} x + e_{j} \cosh \sqrt{\lambda_{j}} x \right) +
$$

$$
Eb_{i}^{2} \psi_{i}' / (2G)
$$
 (29)

where c_{ij} is the element of eigenvector of $A^{-1}B$, and c_3 ^{$=$ −1; λ ^{*i*} (*i*=1, 2, 3) are three eigen values of $A^{-1}B$; d_1}

and e_i ($i=1, 2, 3$) are the constants of integration. Substituting Eq. (28) into Eq. (23)−(24) yields

$$
\theta = \sum_{j=1}^{3} \Omega_j \left(d_j \cosh \sqrt{\lambda_j} x + e_j \sinh \sqrt{\lambda_j} x \right) +
$$

$$
\xi_0 q x^3 / (6EI) + 0.5 \xi_0 c_1 x^2 + c_2 x + c_3
$$
 (30)

$$
\theta' = \sum_{j=1}^{3} \Omega_j \sqrt{\lambda_j} \left(d_j \sinh \sqrt{\lambda_j} x + e_j \cosh \sqrt{\lambda_j} x \right) +
$$

$$
\xi_0 q x^2 / (2EI) + \xi_0 c_1 x + c_2
$$
 (31)

$$
U_0 = \sum_{j=1}^{3} \Phi_j \left(d_j \cosh \sqrt{\lambda_j} x + e_j \sinh \sqrt{\lambda_j} x \right) +
$$

$$
\frac{c_0 q x^3}{6EI} + 0.5 \frac{c_0 c_1 x^2}{6} + c_5 x + c_6
$$
 (32)

$$
U_0' = \sum_{j=1}^3 \Phi_j \sqrt{\lambda_j} \left(d_j \sinh \sqrt{\lambda_j} x + e_j \cosh \sqrt{\lambda_j} x \right) +
$$

$$
\zeta_0 q x^2 / (2EI) + \zeta_0 c_1 x + c_5
$$
 (33)

The first derivative of *w* is $w' = \theta + \frac{a_s}{\epsilon}$ w $w' = \theta + \frac{\alpha_s Q}{G A_w},$ $\alpha' = \theta + \frac{\alpha}{\alpha}$ therefore,

$$
w = \sum_{j=1}^{3} \frac{\Omega_{j}}{\sqrt{\lambda_{j}}}\left(d_{j} \sinh \sqrt{\lambda_{j}} x + e_{j} \cosh \sqrt{\lambda_{j}} x\right) + \frac{\xi_{0} q x^{4}}{24EI} + \frac{\xi_{0} c_{1} x^{3}}{6} + \frac{c_{2} x^{2}}{2} + c_{3} x + c_{4} + \int \frac{\alpha_{s} Q}{G A_{w}} dx
$$
(34)

where $Q_i = \sum_{i=1}^{3}$ 1 $j = \sum c_{ij} \kappa_i$ *i* $Q_i = \sum c_{ii} \kappa_i$ $=\sum_{i=1}^{3}c_{ij}\kappa_i$, and $\Phi_j=\sum_{i=1}^{3}$ 1 $\mathcal{V}_j = \sum c_{ij} \mathcal{F}_i$ *i* $\Phi_i = \sum c_{ii} \theta_i$ $=\sum_{i=1}c_{ij}\theta_i$ are the constants which are related to the section dimension; c_i ($i=1, \dots, 6$)

are the constants of integration. According to the boundary conditions Eqs. (18)−

(21), we can obtain the unknown coefficients. Then, by substituting Eqs. (29), (31) and (33) into the following stress calculating formula, we can obtain the close-form solution of longitudinal stress at each point of the web and flange:

$$
\sigma_{x4} = -zE\theta' + Eu_0' \tag{35}
$$

$$
\sigma_{xi} = Eh_i \left[-\theta' + \left(1 - y^2 \middle/ b_i^2 \right) U_i' \right] + Eu_0', \ i = 1, 2, 3 \quad (36)
$$

where *y* is replaced by \overline{y} when *i*=2.

From Eqs. (28) − (34) , we can learn that only Eq. (34) has the term of shear deformation, which means that shear deformation has influence on deflection, while it has no influence on normal stress and shear lag effect. From Eq. (35), we can learn that the centroid of cross-section is no longer zero at the location of the neutral axis, which means that longitudinal stress exists at the centroid of cross-section due to shear lag effect.

4 Verification of closed-form solution

4.1 Comparison of closed-form solution, finite strip method and experimental results of a rectangular box girder

To verify the proposed formula, a rectangular beam model made by organic glass is investigated [17]. The section size and the measurement point arrangement of this beam model with a span of 0.8 m and diaphragms installed at beam ends are shown in Fig. 2. Two concentrated loads equal in magnitude to 0.272 2 kN are symmetrically applied on the top of the web plate at the mid-span section. The elastic modulus of PMMA is measured to be 3 000 MPa, the Poisson ratio is taken as 0.385, and the strain of mid-plane of the plate is taken as the average value of the results of upper and lower measurement points.

Fig. 2 Rectangular box girder section (Unit: 10^{-2} m)

The comparison of results from closed-form solution (*P*1), the experimental results (*P*) and the numerical solutions (*P*2) derived according to the high order finite strip method are shown in Fig. 3. It could be calculated from Fig. 3 the average values of *P*1/*P* and *P*2/*P* are 1.000 2 and 1.012 2, respectively, and the

Fig. 3 Section strain at mid-span of simply-supported rectangular section box girder under concentrated load

variances of the ratios are 0.002 4 and 0.001 6. The stress at the centroid point of the cross-section of box girder is no longer zero. It can be concluded that results from the close-form solution correlated are in good agreement with the experimental results, and have good stability.

4.2 Comparison of closed-form solutions and numerical solutions derived according to high order finite strip method in a trapezoidal box girder

A simply supported trapezoidal box girder with a span of 50 m and the section size shown in Fig. 4 is investigated. The elastic modulus of concrete $E=3.1\times10^4$ MPa, and the Poisson ratio *μ*=0.17. 1) Two concentrated loads equal in magnitude to 20 kN are symmetrically applied on the top of the web plate at the mid-span section. 2) A uniform load of $q=2$ kN/m is applied on the top of the web along the span.

Table 1 and Fig. 5 show the analytical results obtained from different methods. It could be concluded from Table 1 and Fig. 5 that the average values of the ratios of the closed-form solutions to the numerical

Fig. 4 Trapezoidal box girder section (Unit: 10^{-2} m)

Table 1 Comparison of section strain at mid-span of simply-supported trapezoidal section box girder under uniform load (10^{-6})

Calculation point designation		Present method	Finite strip method $[17]$	Ratio
Cantilever slab	1	-97.7	-96.9	1.008
	2	-98.5	-97.6	1.009
	3	-99.8	-98.9	1.009
	4	-102.6	-100.7	1.019
Roof plate	4	-102.6	-100.7	1.019
	5	-100.5	-99.6	1.009
	6	-99.1	-99.4	0.997
	7	-98.3	-99.3	0.990
Cross section centroid	8	-7.7	-7.3	1.055
Baseboard	9	199.6	196.8	1.014
	10	196.5	195.7	1.004
	11	194.2	195.4	0.994
Average value				1.011
Variance				0.0003

Fig. 5 Section strain at mid-span of simply-supported trapezoidal section box girder under concentrated load

solutions for two different kinds of loadings are 1.02 2 and 1.011, respectively, and the variances of the ratios are 0.000 7 and 0.000 3, respectively. For trapezoidal box girder, we can draw the same conclusion as that for rectangular box girder. At the same time, the stress at the centroid point of the cross-section of box girder is no longer zero. It is not reasonable enough to neglect the uniform axial deformation. The proposed method in this work is more reasonable.

5 Conclusions

1) A method is proposed for direct calculation of stress in box girder. Verification of the method is performed by comparing the results with other analytical results using finite strip method and experimental results.

2) The shear effect influences only the deflection, while has no impact on longitudinal positive strain and shear lag. Therefore, both shear lag and shear effect should be considered when calculating the deflection while only shear lag should be considered when calculating the section stress.

3) The stress at cross-section centroid is no longer zero due to shear lag effect. Therefore, it is not scientific enough to assume the strain in cross-section centroid to be zero without considering the uniform axial deformation. The proposed method is more reasonable.

4) Theoretical analysis method for the shear lag effect in box girder is explored, which provides important basis for the shear lag calculation in thin-walled box girder bridges.

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