## THEORY AND PRACTICAL USE OF MODULAR-POURING ELECTRIC FURNACES FOR FIRING VERMICULITE

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Results are presented from theoretical and experimental studies of modular-pouring electric furnaces. The process of the absorption of heat energy by vermiculite was examined and the optical-geometric parameters of the firing modules were determined. Efficient values were found for the temperature and duration of the firing of vermiculite concentrates. It is shown that the existing modular-pouring electric furnaces can be improved and new, energy-saving furnaces of this type can be developed.

**Keywords:** vermiculite, modular-pouring electric furnace, heat assimilation, thermal radiation, firing module, furnace efficiency, unit energy content of the firing operation.

Products based on expanded vermiculite occupy an important place among the wide range of refractory materials that are made. They have good heat-insulating properties thanks to their porous layered structure, which allows them to be used to line different types of high-temperature equipment. Vermiculite is also an effective heat-insulating material and serves as a porous filler for lightweight concretes. It can be used to prepare dry mixes, including mixes with fireprotection properties. Vermiculite is employed in the acoustic insulation of buildings, the casting of steel, and other applications [1, 2].

The idea of creating a modular-pouring electric furnace for firing vermiculite was conceived in 2003 [2-4], and refinements were made to the original concept during the ensuing decade of practical use of such furnaces. Improvements were made to the furnaces on the basis of results obtained not only from their operation but also studies of analytical models of their working processes.

The goals of the studies were to analyze the process by which heat energy is absorbed by vermiculite, determine the optical-geometric parameters of the firing modules, find efficient temperature-time regimes for the firing operation, and determine the efficiency and unit energy content of the firing process.

Modular-pouring electric furnaces can be used as a basis for the construction of small production units designed to obtain different fractions of high-quality expanded products from vermiculite concentrates (including concentrates that have already undergone preliminary beneficiation) and conglomerates with a high content of inert material [2].

Figure 1 shows a block diagram of the process of obtaining an expanded product from high-purity concentrates with a vermiculite content of 94 - 96%. Charging the raw material into the furnace in different fractions makes it possible to obtain vermiculite of any desired granulometric composition. If



Fig. 1. Block diagram of the process of obtaining expanded vermiculite.

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Fig. 2. Modular electric furnace.

that is not necessary, then the raw material is delivered to the furnace by elevator for immediate batch charging. Rapid cooling of expanded vermiculite increases the strength of its granules to some extent [5], although this operation can be by-passed and the vermiculite can be sent directly to the storage hopper after firing.

Figure 2 presents a diagram of a three-module furnace. The furnace includes a drum-type charge-material batcher 1 provided with a small hopper 2, fluted drum 3, and discharge chute 4. The charge material is heated to an initial temperature of  $95 - 100^{\circ}$ C before it is dropped into the furnace. This maximizes the rate of heating of the granules inside the furnace and ensures that the furnace operates in the thermal-shock regime (at least  $200^{\circ}$ C/sec [5]). Frames 5 located inside the furnace housing support electric firing modules 6 with heat-retaining covers 7. Electric heaters attached to current leads 8 are positioned in a row which extends in the longitudinal direction underneath the covers. All of the furnace elements are secured to framework 9, which is composed of

two thermally insulated steel walls and doors (not shown in the figure).

When the vermiculite concentrates are fired the expanded material enters storage hopper 10, which extends below bottom firing module 6. If concentrate in another size group is being fired in the furnace, the optical-geometric parameters of the modules' working chambers are changed by replacing the removable blocks of heating elements [6]. As the vermiculite travels through the firing modules it absorbs heat energy and is mechanically transformed and expands as a result of the expansion of superheated steam and adsorbed gases [2]; a new material with an exfoliated structure is formed in the process.

The unit amount of assimilated (absorbed) heat  $\theta_{\Sigma}$ , J/kg, is expended on dehydration and the release of chemically bound water  $\theta_c$ , heating of the dry part of the mineral  $\theta_d$ , phase transformation of the physical (interlaminar) and hydrate water  $\theta_w$ , and superheating of the water vapor  $\theta_v$  and adsorbed gases  $\theta_{a.g.}$ 

The thermal-energy balance is determined by the sum

$$Q_{\Sigma} = Q_{c} + Q_{d} + Q_{w} + Q_{v} + Q_{a.g}.$$

The heat  $\theta_d$  does not take part in the mechanical transformation of the material and accumulates in the expanded granules, gradually being dissipated into the surrounding medium.

The energy of mechanical transformation  $E_{m,t}$  is equal to the sum:

$$E_{\rm m.t} = Q_{\rm c} + Q_{\rm w} + Q_{\rm v} + Q_{\rm a.g}$$

and its ratio to the absorbed energy determines the efficiency coefficient (EFC) of the expansion process:

$$\eta_{\mathrm{m.t}} = \frac{E_{\mathrm{m.t}}}{\theta_{\mathrm{y}}}.$$
 (1)

The EFC in the structural transformation of Kovdor vermiculite is approximately 0.544, while the corresponding quantity for vermiculite from the Tatar deposit is 0.577 [2].

The temperature-time relation that characterizes the regime state of the three-module furnace has the form:

$$T_{\rm h} = \left\{ \frac{\theta_{\Sigma} \cdot \frac{\pi DB[n_{\rm m}l_{\rm w} + (n_{\rm m} - 1] + l]}{18\kappa_{\rm e}(1 - \kappa_{\rm p})}}{t_{\rm fr} \left[ \varphi_{\rm in} + \frac{\rho_{\rm f}}{3} \alpha_{34}^{\rm y}(\varphi_{\rm ba}k_{\rm o} + \varphi_{\rm hs}k_{\rm k}) \right] \varepsilon_{\rm n} \sigma 2\chi n_{\rm m}n_{\rm h} (2l + 4l_0 + r_{\rm min})} \right\}^{\frac{1}{4}},$$
(2)

where  $T_{\rm h}$  is the temperature of the surface of the heaters, K; *D* is the size group of the initial concentrate, m; *B* is the width of the module, m;  $n_{\rm m}$  is the number of modules in the furnace;  $l_{\rm w}$  is the working length of the modules, m;  $l_0$  is the Theory and Practical Use of Modular-Pouring Electric Furnaces for Firing Vermiculite

length of the cantilevered sections of the heaters, m; l is the total length of the modules, m;  $\lambda$  is the size of the pour zone, m;  $\kappa_{e}$  is the expansion ratio, m<sup>3</sup>/kg'  $\kappa_p$  is the porosity coefficient of the expanded vermiculite in the mix;  $t_{\rm fr}$  is the duration of the firing operation, sec;  $\phi_{in}$ is the angle of incidence of the radiation reaching the vermiculite from the surface of the heater;  $\phi_{hs}$  and  $\phi_{ba}$  are the angles of incidence of the effective radiation reaching the vermiculite from the heat-retaining cover and the base of the module;  $\rho_f$  is the reflectivity of the surface of fireclay brick;  $\alpha_{34}^{y}$  is the absorptivity of the vermiculite medium in the direction from the base of the module to the cover;  $k_0$  and  $k_k$  are coefficients accounting for the optical-geometric parameters of the radiating, absorbing, and reflecting surfaces of the modules' working chambers and the bulk medium [7, 8];  $\varepsilon_n$  is the emissivity of nichrome;  $\sigma$ is the Stefan-Boltzmann constant,  $\sigma = 5.67 \times 10^{-8} \text{ W/(m^2 \cdot K^4)}; 2\chi \text{ is the per$ imeter of the heaters' nichrome bands about their cross section, m;  $r_{\min}$  is the distance between the heaters, m.

Equation (2) was used to plot a graph of temperature-time relation 1 (Fig. 3) for a three-module furnace in which the firing modules had the following design parameters: B = 0.92 m,  $n_{\rm m} = 3$ ,  $l_{\rm w} = 0.53$  m,  $l_0 = 0.575$  m,  $\lambda = 0.05$  m.

The operating point *E* of an experimental furnace having modules of the same dimensions as above corresponds to a heater temperature  $T_{\rm h} = 758$  °C, a furnace time constant which is equal to the average firing time  $T_{\rm fr} = 2.72$  sec, and a heating rate  $(T_{\rm h} - 100$  °C)/ $T_{\rm fr} = 241.9$  °C/sec. Point *c* on curve *l* corresponds to the analytical model of the three-module furnace with a time constant of 2.72 sec, although the heating temperature in the model is significantly higher than it was in the experiment and is equal to 830 °C,.

We introduce a correction factor to correct the modules by converting the temperature values to the Kelvin scale:

$$k_{\rm T} = T_{\rm h}^{\rm E} / T_{\rm h(c)}^{\rm w} = (758 + 273) / (830 + 273) = 0.935.$$

It follows from Eq. (3) that the difference between the analytical and empirical models is only 6.5%.

We make curve 2 undergo a parallel shift to position 2 so that the ratio of the temperatures at points *E* and *c* are equal to the coefficient  $k_{\rm T}$  (0.935).

Point B will not lie on curve 2, since the temperature at this point



i.e. curve 2 should be rotated clockwise around point E, as indicated in Fig. 3 (curve 3).

Segment *BD* on curve 3 in Fig. 3 is a nonworking section, since the rate of temperature increase here is below the allowable minimum (200°C/sec). Thus, the process of dehydrating the vermiculite raw material will be incomplete. Line x - x reflects the set of points corresponding to the regime with the lowest allowable rate of increase in temperature. As a result, the hatched region near sections *AD* and *DB'* is the region occupied by the furnace's operating points based on the empirical model.

Thus, the analytical model of the process is very accurate in relation to the experimental results (6.5%) even though curves 1 and 3 are shifted relative to one another by  $72^{\circ}$ C. The overall result obtained here can therefore be considered satisfactory.

With allowance for the efficiency of the process of vermiculite's mechanical transformation  $\eta_m$ , the efficiency of the furnace  $\eta_f$  is calculated from the formula

$$\eta_{\rm f} = \frac{\theta_{\Sigma} \Pi_G \eta_{\rm m.t}}{N_{\star}},\tag{5}$$



**Fig. 3.** Graphs of the temperature-time characteristic of a three-module furnace according to an analytical model (1) and an empirical model (2) ( $\eta_f$  — efficiency coefficient of furnace).

 $e_{\rm u}$ , mJ/m<sup>3</sup>

230

220

210

process.



Kovdor

concentrate KVK-2

0.355

. 710

 $e_{\rm u} = 210 \text{ mJ/m}^3$   $710 \quad 720 \quad 730 \quad 740 \quad 750 \quad T, \text{ °C}$ Fig. 4. Temperature dependence of the unit energy content of the

A

while unit energy content in the firing operation is determined by the ratio  $e_u$ , mJ/m<sup>3</sup>:

$$e_v = 3600 N_e / \Pi_{V_e}$$
 (6)

where  $\Pi_G$  is the mass-based productivity of the furnace, kg/sec;  $\Pi_V$  is the volume-based productivity of the furnace, m<sup>3</sup>/sec;  $N_e$  is electric-power consumption, W.

We use Eqs. (5) and (6) to find the values of  $\eta_f$  and  $e_u$  at characteristic points on the temperature dependence (Table 1).

Figures 4 and 5 show relations constructed with the use of the above data. The furnace operating regime at point *A* is advantageous from an energy standpoint; the extrema of the main indices  $\eta_f$  and  $e_u$  are reached at this point. The regime at point *B'* is best for maximizing productivity, since the time constant  $T_{t,c}$  is highest here. However, the unit heat content of the process is 11.6% higher and furnace efficiency is 10% lower than at point A.

Regime point D is a compromise variant: here, the decrease in productivity relative to the maximum is about 6%, while efficiency and unit energy content respectively decrease by 2.5 and 1.7% relative to point A.

**TABLE 1.** Values of Furnace Efficiency and Unit Energy Content

 Determined from the Empirical Model

Point	Furnace efficiency	Unit energy content of the process, mJ/m <sup>3</sup>
A	0.40	210.5
D	0.39	214.1
B'	0.36	243.2
C	0.38	222.0



Fig. 5. Temperature dependence of the efficiency of the furnace.

730

740

750

T. °C

720

Thus, The efficient regime values for the temperature and duration of the firing of vermiculite concentrates lie within the hatched region near points B', D, and A in Fig. 3.

The study results and the knowledge gained over a decade of experience in the use of modular electric furnaces has made it possible to design and build optimum variants of these units — furnaces that have the best possible characteristics. Table 2 shows the most important indices of the five main furnaces (in all, eight furnaces have been built and placed in service).

In addition to these units, two other furnaces have been in use since 2010 in Tashkent (in the Republic of Uzbekistan) to process concentrates prepared from the Karakalpak deposit.

TABLE 2. Power and Unit Energy Content of the Modular Furnaces

Furnace	Rated power, kW	Unit energy content of the firing operation, mJ/m <sup>3</sup>
Commercial prototype*:		
second	112.2	290.6
third	108.7	249.9
fourth**	105.6	250.1
Experimental:		
fifth***	75.2	235.4
sixth****	63.4	196.8

\* The data obtained for the first commercial prototype was not saved.

\*\* The fourth commercial prototype (Fig. 6) was used to fire vermiculite concentrates from the Tatar deposit (in the Krasnoyarsk Krai) and the other furnaces were used to fire concentrates from the Kovdor deposit (in the Murmansk Region).

\*\*\* The results presented above from theoretical and experimental studies were obtained for the fifth experimental furnace.

\*\*\*\* The sixth experimental furnace was of a different design; the firing modules were series-parallel-connected [2].



Fig. 6. Commercial prototype

It was shown in [9] that approximately 42 - 46% of the heat energy  $\theta_d$  absorbed by the vermiculite remains in the expanded material, which is heated to 730 - 750°C. If the material is produced in the slightly under-expanded state, then part of this "latent" energy can be directed into deeper layers of the vermiculite granules. This can be done simply by lengthening the process to 0.6 - 0.8 sec in order to complete the material's dehydration. One prerequisite to the success of this approach is that none of the latent energy can be lost to the surrounding medium. An additional non-electric module needs to be built into the furnace to receive the flow of recuperated energy.

Such a structural transformation of the three-module furnace will make it possible to reduce energy consumption and the unit energy content of the firing operation and increase the efficiency  $\eta_f$  of the furnace, bringing it close to the efficiency of the mechanical transformation process  $\eta_{m,t}$ .

The experience gained in the use of modular furnaces has made it possible to come up with the following practical recommendations:

 an efficient design for the firing modules includes lengthwise chambers formed by heaters that are positioned on edge and are made of metal with a high electrical resistivity (such as nichrome X20H80);

the width of the working chambers should satisfy the condition

$$r_{\min} \approx (9 - 15)D,\tag{7}$$

where D is the center of the group of nominal diameters of the initial particles of vermiculite concentrate; this parameter determines the size group of the concentrate [2] (the value of D is lower for coarse fractions and higher for fine fractions). Condition (7) prevents the formation of lumps of expanded vermiculite in the working chambers of the modules;

– to make the temperature field in the firing zones of the modules as uniform as possible, heat-insulating inserts need to be installed along their side walls to reduce heat flow through the walls; it is also necessary to use heating elements spaced different distances apart, with fewer elements being installed near the walls while still meeting condition (7);

— it is best to use furnaces whose housings have thermally insulated walls and door leaves; in production operations in which hot air needs to be used as a heat carrier, the housing should be designed with hollow walls to allow the air to be pumped into the walls.

Research on the topic discussed in this article will be continued.

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