

Temperature distribution and stress analysis of end pumped lasers under Gaussian pump profle

Mohammed Jalal AbdulRazzaq[1](http://orcid.org/0000-0002-0804-0205) · Khalid S. Shibib1 · Sudad Issam Younis1

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Abstract

Modifed modeling of thermal efects within a dual end-pumped cylindrical shape laser medium was presented. The temperature distribution and the maximum hoop stress were determined by solving the heat conductance equation in a cylindrical medium. Two different incident beams of 0.6 mm waist radii were suggested, namely, efective and the Gaussian beam radii, where the effective beam radius is smaller by a factor of $\sqrt{2}$ than the Gaussian beam radius. Under these considerations, the radial temperature diference due to an efective beam radius was found to be 23% higher than that for a Gaussian beam radius. The calculations show that the pump power does not afect the location of the maximum hoop stress as the spot does. A discrepancy within 1% was obtained by comparing the obtained results with the experimental work in the literature.

Keywords Dual-end pumped · Efective Gaussian beam · Laser damage · Thermal efects

1 Introduction

In end-pumped lasers, the input face is under a high thermal loading and the upper limit on the critical input power density is determined by the fracture strength of the medium. Therefore, the temperature distribution inside the medium due to power and a spot diameter of the pump light usually dominate the system design considerations for the high average power laser. Both theoretical models and experimental settings were carried out by many researchers to reduce thermal effects generated due to high pumping power (Abdul-Razzaq et al. [2019;](#page-7-0) Ahmed et al. [2018;](#page-7-1) Cini and Mackenzie [2017](#page-8-0); El-Agmy and Al-Hosiny [2017;](#page-8-1) Ghadban et al. [2020;](#page-8-2) Kim et al. [2019;](#page-8-3) Liu et al. [2016](#page-8-4); Mojahedi and Shekoohinejad [2018;](#page-8-5) Shen et al. [2015;](#page-8-6) Shibib et al. [2017\)](#page-8-7). In this work, a modifed thermal model was presented by considering two different beams, namely, effective and the Gaussian beam radii. Accordingly, the maximum temperature diference on each face was obtained taking into account the dependence of thermal conductivity on temperature, and the maximum hoop stress was obtained on each incident face of the medium by considering the

 \boxtimes Mohammed Jalal AbdulRazzaq 140041@uotechnology.edu.iq

¹ Laser and Optoelectronics Engineering Department, University of Technology –Iraq, Baghdad, Iraq

dependence of thermal expansion coefficient on the temperature. A small discrepancy was obtained between our analysis and the experimental work in the literature.

2 Temperature distribution analysis

The confguration illustrated in Fig. [1](#page-1-0), concerns a cylindrical Nd:YAG laser rod with unequally absorbed pump power density from two end faces. The nonlinear heat equation was solved to estimate the temperature distribution and the maximum hoop stress with the help of Kirchhof's transformation.

By considering the dependency on the temperature of the active medium thermal conductivity, the nonlinear heat equation is written as:

$$
\frac{1}{r}\frac{d}{dr}\left[rk(T)\frac{dT(r,z)}{dr}\right] + Q(r,z) \tag{1}
$$

where $Q(r, z)$, is the dissipation thermal power-per-unit rod volume and $k(T)$ is the depend-ence of thermal conductivity on temperature, given by (Cini and Mackenzie [2017\)](#page-8-0):

$$
k(T) = k_o \left(\frac{T}{T_o}\right)^m
$$
 (2)

where k_0 , is the laser material thermal conductivity at temperature To. In this equation, m is unknown value and selected to be an agreement with the experimental work presented by Shen et al. ([2015\)](#page-8-6). In this study (i.e. double–Gaussian-distribution), $Q(r, z)$ can be written as:

$$
Q(r, z) = Q_{0,1} e^{-2\frac{r^2}{w_1^2}} e^{-\alpha z} + Q_{0,2} e^{-2\frac{r^2}{w_2^2}} e^{-\alpha(L-z)}
$$
(3)

 w_1 , are the effective and the waist radius of the left and right pump beams intensity profile, α and L are the absorption coefficient and the length of the laser medium respectively. In the following, two unequal spot radii are assumed i.e. $(w_1 \neq w_2)$. The two normalization constants $(Q_{0,1,2})$ can separately be calculated according to:

$$
Q_{0,1,2} = \frac{\eta_h P_{1,2}}{V_{1,2}}\tag{4}
$$

where η_h P_{1,2} are the two absorbed incident powers, and (η_h is the rate of heat-loading). V_{1,2} are the volumetric heating of the distributed-pump in the laser rod, i.e.

$$
V_1 = \frac{\pi w_1^2 \eta_{\text{abs}}}{2\sqrt{2\alpha}}\tag{5a}
$$

$$
V_2 = \frac{\pi w_1^2 \eta_{abs}}{2\alpha} \tag{5b}
$$

Given that the absorption efficiency is $(1 - e^{-\alpha z})$. Defining P_{in,1} and P_{in,2} the incident pump powers at the faces of the laser medium, and substituting the expression of the two normalization constants into Eq. (3) (3) , With the help of Eq. (2) , Eq. (1) has the form of:

$$
\nabla^2 \mathbf{U}(\mathbf{r}, \mathbf{z}) = -\mathbf{Q}(\mathbf{r}, \mathbf{z}) = \frac{\partial^2 \mathbf{U}(\mathbf{r}, \mathbf{z})}{\partial \mathbf{r}^2} + \frac{1}{r} \frac{\partial \mathbf{U}(\mathbf{r}, \mathbf{z})}{\partial \mathbf{r}} \tag{6}
$$

where

$$
U = \frac{K_0}{(m+1)T_0^m}T^{m+1} + C
$$
 (7)

C, is the integration constant (an arbitrary).

According to Fig. [1](#page-1-0), the boundary conditions are:

$$
-\frac{dU_{pump}}{dr}\Big|_{r=0} = 0
$$
 (8)

and

$$
-\frac{dU_{\text{unpump}}}{dr}\bigg|_{r=b} = h[T_2(r=b) - T_c]
$$
 (9)

h is the coefficient of heat transfer, and Tc is the temperature of the coolant. Finally, by imposing the temperature continuity and its derivative, such that,

$$
\left. \frac{d T_{pump}}{dr} \right|_{r=2w} = \left. \frac{d T_{unpump}}{dr} \right|_{r=2w} \tag{10}
$$

One obtained two solutions of Eq. [\(6](#page-2-0)), (i.e. $T_{pump}(r,z)$ in the region ($0 \le r \le w$)

$$
T_{pump1}(\mathbf{r}, \mathbf{z}) = \left\{ \frac{\eta_{h} \alpha (m+1) T_{0}^{m}}{4\pi K_{0}} \left(P_{in,1} e^{-\alpha z} \right) \left[\ln \left(\frac{b^{2}}{r^{2}} \right) + E_{1} \left(\frac{2b^{2}}{w_{1}^{2}} \right) - E_{1} \left(\frac{2r^{2}}{w_{1}^{2}} \right) \right] + \left[T_{c} + \frac{\eta_{h} \alpha}{4\pi b h} \left(P_{in,1} e^{-\alpha z} \right) \right]^{m+1} \right\}^{\frac{1}{m+1}}
$$
\n(11a)

and

$$
T_{pump2}(\mathbf{r}, \mathbf{z}) = \left\{ \frac{\eta_{\mathbf{h}} \alpha (m+1) T_0^{\mathbf{m}}}{4\pi K_0} \left(P_{\mathbf{in}, 2} e^{-\alpha (L-z)} \right) \left[\ln \left(\frac{b^2}{r^2} \right) + E_1 \left(\frac{2b^2}{w_2^2} \right) - E_1 \left(\frac{2r^2}{w_2^2} \right) \right] + \left[T_c + \frac{\eta_{\mathbf{h}} \alpha}{4\pi b \mathbf{h}} \left(P_{\mathbf{in}, 2} e^{-\alpha (L-z)} \right) \right]^{m+1} \right\}^{\frac{1}{m+1}}
$$
\n(11b)

and outside the pump region, $T_{\text{unpump}}(r,z)$ in (w $\leq r \leq b$)) as follows:

$$
T_{unpump1}(r, z) = \left\{ \frac{\eta_h \alpha (m+1) T_0^m}{4\pi K_0} \left(P_{in,1} e^{-\alpha z} \right) \ln \left(\frac{b^2}{r^2} \right) + \left[T_c + \frac{\eta_h \alpha}{4\pi b h} \left(P_{in,1} e^{-\alpha z} \right) \right]^{m+1} \right\}^{\frac{1}{m+1}} (12a)
$$

and

$$
T_{unpump2}(r,z) = \left\{ \frac{\eta_{h}\alpha(m+1)T_{0}^{m}}{4\pi K_{0}}\left(P_{in,2}e^{-\alpha(L-z)}\right)\ln\left(\frac{b^{2}}{r^{2}}\right) + \left[T_{c} + \frac{\eta_{h}\alpha}{4\pi bh}\left(P_{in,2}e^{-\alpha(L-z)}\right)\right]^{m+1}\right\}^{\frac{1}{m+1}}(12b)
$$

where E1, is the exponential integral function.

Therefore, the temperature diference between the rod center and its edge, is given as

$$
\Delta T(r, z) = T_{pump}(r, z) - T_{unpump}(b, z).
$$

3 Thermal stress analysis

The generation of heat in solid-state lasers is correlated with the optical pumping process. As a consequence of the nonuniform absorbed pumping, the heat generation is nonuniform which leads to the temperature gradient along the cross-section of the laser medium where the center is hotter than the outer surface of the medium and consequently internal thermal stress was induced. As the pumping power increased the induced thermal stress may exceed the maximum tensile hoop stress of the laser material and leading to fracture. The maximum hoop stress is defned and related as (Xie et al. [2003](#page-8-8)):

$$
\sigma_{\theta}(r) = \frac{\alpha_T E}{(1-v)} \left[\frac{1}{b^2} \int_0^b \Delta T r dr + \frac{1}{w^2} \int_0^w \Delta T r dr - \Delta T \right]
$$
(13)

Where E, α_T , and v are the medium parameters i.e. modulus of elasticity, thermal coefficient of expansion, and the Poisson's ratio, respectively. In this model, the dependence of thermal expansion on the temperature was considered and given as (Cini and Mackenzie [2017\)](#page-8-0):

$$
\alpha_T(T) = 4.2 \times 10^{-6} + 0.62 \times 10^{-8} (T_c)
$$
\n(14)

Therefore, Eq. ([13](#page-3-0)) was solved at the two incident medium faces in the location where z equal to zero to estimate the maximum hoop stress.

4 Results and discussions

To evaluate our proposed model, we made a comparison with the earlier experimental work by Shen et al. [\(2015](#page-8-6)). The heat loading and the radial temperature distribution were simulated for the proposed model considering Nd:YAG as a gain medium. The parameters used in this analysis are listed in Table [1](#page-4-0).

Figure [2](#page-4-1) shows the heat loading generated on each face of the rod (i.e. $z=0$, L). According to an unequal spot radius imposed on each medium-end face, the dissipated heat per unit volume was diferent. Furthermore, the heat loading increased as the pump radius become smaller.

The combination of the volumetric absorbed power and the surface cooling leads to a non-uniform temperature diference in the medium as shown in Fig. [3.](#page-5-0)

Fig. 2 Heat load distribution on each rod's faces

Fig. 3 The temperature diference on each rod faces

Fig. 4 Temperature distribution along the medium

Due to an unequal spot radius i.e. $w_1 = \frac{w_2}{\sqrt{2}}$ $\frac{2}{2}$ on the right face where $w_2 = 0.6$ mm on left face), the radial temperature difference (i.e. $z=0$, *L*) was different on each rod's faces. In other words, at a pump power of 46.4 W imposed on each rod faces, the maximum critical temperature diference according to an efective Gaussian spot of 0.4 mm radius was found to be 276.36 K as compared to 274 K listed in Shen et al. (2015) (2015) . While the maximum radial critical temperature diference of 213.23 K was estimated according to a spot of 0.6 mm radius. Figure [4](#page-5-1) shows how the Gaussian beam spot radius variation afects the axial medium steady-state temperature distribution. It can be seen that on each face of the local absorption region, the temperature has a perfectly Gaussian due to the Gaussian profle of the input power and gradually declined outside the pump region. Along the z-axis, the temperature decays exponentially as a consequence of the exponential absorption of the incident power.

The characteristics of the medium fracture stress were analyzed by calculating the temperature distribution at a maximum critical pump power as shown in Fig. [5](#page-6-0).

According to Fig. [5](#page-6-0), as the pump power increased, the thermal stress increased too and reaching its ultimate value (i.e. fracture point) of 235.2 MPa corresponding to 0.4 mm

Fig. 5 Fracture stress on each face of the medium

spot radius and 186.4 MPa according to 0.6 mm pump radius at a maximum critical power of 46.4 W. It's clear that, due to an efective spot of 0.4 mm radius, the medium is more compressed as compared to 0.6 mm spot radius imposed on the other end face. Also, the calculations show that the incident power has no efect on the location of the maximum hoop stress and the two end faces of the medium are subject to fracture at a spot of $\sqrt{2}w_o$ imposed on the right face and at $2w_o$ spot incident on the left face. Table [2](#page-7-2) summarized how the obtained results are in agreement with previous works presented by Shen et al. ([2015\)](#page-8-6).

The maximum hoop stress was also calculated were the Nd:YAG laser medium under lasing and non-lasing situations as illustrated in Fig. [6](#page-7-3). At the same input power of 46.4 W, the maximum hoop stress can be varied from a theoretical lower limit where the rate of heating equal to 24% to the practical value of 32% under lasing and 43% under non-lasing situations (Koechner [2006\)](#page-8-9). Generally, the theoretical value of the rate of the heat of 24% for Nd:YAG laser pumped by 810 nm laser diode and lasing at 1064 nm wavelength was admitted by Koechner (2006) (2006) which corresponds to a quantum efficiency of unity, while 32% corresponds to 90% quantum efficiency under lasing action. The value of 43% rating heat was recorded due to the absence of stimulated emission. Therefore, these variations have their signature on the maximum hoop stress.

Figure [6a](#page-7-3), b shows the maximum hoop stress variations as a function of rod radius on both sides of the medium-end faces at an input power of 46.4 W. it can be observed that at a rate of 50% heating, the maximum hoop stress of 235.2 MPa was located on the position of 0.4 mm spot radius while a 186.4 MPa was obtained at 0.6 mm radius.

Active medium	Length (mm)	Diam- eter (mm)	Pump profile	Pump power (W)	Maximum temperature difference (K)	hoop stress (MPa)
Nd:YAG (analytical model)	10	3	Gaussian	46.4	276.36	235.2
Nd:YAG (Shen et al. 10 2015		3	Gaussian	46.4	274	235

Table 2 Comparison between theoretical results and experimental work by Shen et al. [\(2015](#page-8-6))

Fig. 6 Maximum hoop stress on both sides faces at diferent heating rates. Redline for 50% rating heat, blue line for 24%, dot blue line for 43%, and dot red line for 32%

5 Conclusions

In conclusion, a modifed thermal model was presented to predict the temperature distribution and the maximum hoop stress within the end-pumped cylindrical laser medium. To work laser safely, the efective spot radius has been suggested as a spot pump imposed on one side face in comparison to the Gaussian spot radius was incident on the other medium side face. The radial temperature diference due to an efective beam radius was found to be 23% higher than that for a Gaussian beam radius. The results also show that the location of the maximum hoop stress is independent of the pump power and each face of the medium reaches its ultimate fracture according to the value of the spot radius. A minimum discrepancy (1%) was obtained by making a comparison with earlier works of literature. The model presented in this work can open the way for the laser designers to evaluate a diferent laser crystal under diferent environmental conditions and one of the main challenges for implementing the model will be to replace the Gaussian pumping profle by another profle like super-Gaussian or Donut shape.

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