#### **ORIGINAL PAPER**



# Space-time computational analysis of tire aerodynamics with actual geometry, road contact, tire deformation, road roughness and fluid film

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#### **Abstract**

The space-time (ST) computational method "ST-SI-TC-IGA" has recently enabled computational analysis of tire aerodynamics with actual tire geometry, road contact and tire deformation. The core component of the ST-SI-TC-IGA is the ST Variational Multiscale (ST-VMS) method, and the other key components are the ST Slip Interface (ST-SI) and ST Topology Change (ST-TC) methods and the ST Isogeometric Analysis (ST-IGA). These ST methods played their parts in overcoming the computational challenges, including (i) the complexity of an actual tire geometry with longitudinal and transverse grooves, (ii) the spin of the tire, (iii) maintaining accurate representation of the boundary layers near the tire while being able to deal with the flow-domain topology change created by the road contact, and (iv) the turbulent nature of the flow. The combination of the ST-VMS, ST-SI and the ST-IGA has also recently enabled solution of fluid film problems with a computational cost comparable to that of the Reynolds-equation model for the comparable solution quality. This was accomplished with the computational flexibility to go beyond the limitations of the Reynolds-equation model. Here we include and address the computational challenges associated with the road roughness and the fluid film between the tire and the road. The new methods we add to accomplish that include a remedy for the trapped fluid, a method for reducing the number of control points as a space occupied by the fluid shrinks down to a narrow gap, and a method for representing the road roughness. We present computations for a 2D test problem with a straight channel, a simple 2D model of the tire, and a 3D model with actual tire geometry and road roughness. The computations show the effectiveness of our integrated set of ST methods targeting tire aerodynamics.

**Keywords** Tire aerodynamics  $\cdot$  Actual tire geometry  $\cdot$  Road contact  $\cdot$  Road roughness  $\cdot$  Fluid film  $\cdot$  ST Variational Multiscale (ST-VMS) method  $\cdot$  ST Slip Interface (ST-SI) method  $\cdot$  ST Topology Change (ST-TC) method  $\cdot$  ST Isogeometric Analysis (ST-IGA)

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#### 1 Introduction

Computational analysis of tire aerodynamics with actual tire geometry, road contact and tire deformation poses tough challenges [1]. The challenges include (i) the complexity of an actual tire geometry with longitudinal and transverse grooves, (ii) the spin of the tire, (iii) maintaining accurate representation of the boundary layers near the tire while being able to deal with the flow-domain topology change created by the road contact, and (iv) the turbulent nature of the flow. These computational challenges were overcome in [1] with the space—time (ST) computational method "ST-SI-TC-IGA" [1,2]. The core component of the ST-SI-TC-IGA is the ST Variational Multiscale (ST-VMS) method [3–5], which subsumes its precursor "ST-SUPS" (see Sect. 2). The other key



components are the ST Slip Interface (ST-SI) [6,7] and ST Topology Change (ST-TC) [8,9] methods and the ST Isogeometric Analysis (ST-IGA) [3,10,11].

The VMS feature of the ST-VMS addresses the challenge created by the turbulent nature of the flow, the moving-mesh feature of the ST framework enables high-resolution flow computation near the moving fluid-solid interfaces, and the higher-order accuracy of the ST framework strengthens both features. The ST-SI enables moving-mesh computation with the tire spinning. The mesh covering the tire spins with it, and the SI between the spinning mesh and the rest of the mesh accurately connects the two sides of the solution. The ST-TC enables moving-mesh computation even with the TC created by the contact between the tire and the road. It deals with the contact while maintaining high-resolution flow representation near the tire. Integration of the ST-SI and ST-TC enables high-resolution representation even though parts of the SI are coinciding with the tire and road surfaces. It also enables dealing with the tire-road contact location change and contact sliding. By integrating the ST-IGA with the ST-SI and ST-TC, in addition to having a more accurate representation of the tire geometry and increased accuracy in the flow solution, the element density in the tire grooves and in the narrow spaces near the contact areas is kept at a reasonable level.

The work report in [1] included computations with the ST-SI-TC-IGA and two models of flow around a rotating tire with road contact and prescribed deformation. One was a simple 2D model for verification purposes, and one was a 3D model with an actual tire geometry and a deformation pattern provided by the tire company. The computations showed the effectiveness of the ST-SI-TC-IGA in tire aerodynamics with actual tire geometry, road contact and tire deformation.

The combination of the ST-VMS, ST-SI and the ST-IGA also created an ST computational method [12] with built-in Reynolds-equation limit, enabling solution of fluid film problems with a computational cost comparable to that of the Reynolds-equation model for the comparable solution quality. This was accomplished with the computational flexibility to go beyond the limitations of the Reynolds-equation model. With the ST-IGA, even with just one quadratic NURBS element across the gap of the fluid film, we reach a solution quality comparable to that of the Reynolds-equation model.

The work reported in [12] included detailed 2D test computations. The computations showed how the ST computational method performs compared to the Reynolds-equation model, and also compared to finite element discretization. The test cases included different circumferential and normal mesh refinement levels. Also included were test cases with an SI in the mesh and cases where the no-slip boundary conditions are enforced weakly.

In this article, we include and address the computational challenges associated with the road roughness and the fluid film between the tire and the road. To accomplish that we add some new methods to the ST computational methods we described in this section so far. The added methods include a remedy for the trapped fluid and a method for reducing the number of control points as a space occupied by the fluid shrinks down to a narrow gap. They also include a method for representing the road roughness. We present computations for a 2D test problem with a straight channel, a simple 2D model of the tire, and a 3D model with actual tire geometry and road roughness.

In Sect. 2, we provide an overview of the ST-VMS and ST-SUPS. The overviews of the ST-SI, ST-TC, ST-SI-TC, ST-IGA and ST-SI-TC-IGA are provided in Sects. 3–7. The remedy for the trapped fluid is described in Sect. 8. The 2D test computations with a straight channel are presented in Sect. 9. In Sect. 10, we present, in the context of a simple 2D tire model, test computations with different contact representations. The computation with the 3D tire model with actual tire geometry and road roughness is presented in Sect. 11, and the concluding remarks are given in Sect. 12.

#### 2 ST-VMS and ST-SUPS

The Deforming-Spatial-Domain/Stabilized ST (DSD/SST) method [13-15] was introduced for computation of flows with moving boundaries and interfaces (MBI), including fluid-structure interaction (FSI). In MBI computations the DSD/SST functions as a moving-mesh method. Moving the fluid mechanics mesh to follow an interface enables mesh-resolution control near the interface and, consequently, high-resolution boundary-layer representation near fluidsolid interfaces. The stabilization components of the original DSD/SST are the Streamline-Upwind/Petrov-Galerkin (SUPG) [16] and Pressure-Stabilizing/Petrov-Galerkin (PSPG) [13] stabilizations, which are used widely. Because of the SUPG and PSPG components, the original DSD/SST is now called "ST-SUPS." The ST-VMS is the VMS version of the DSD/SST. The VMS components of the ST-VMS are from the residual-based VMS (RBVMS) method [17– 20]. The ST-VMS has two more stabilization terms beyond those in the ST-SUPS, and the additional terms give the method better turbulence modeling features. The ST-SUPS and ST-VMS, because of the higher-order accuracy of the ST framework (see [3,4]), are desirable also in computations without MBI.

The DSD/SST is an alternative to the Arbitrary Lagrangian–Eulerian (ALE) method, which is an older and more commonly used moving-mesh method. The ALE-VMS method [21–27] is the VMS version of the ALE. It succeeded the ST-SUPS [13] and ALE-SUPS [28] and preceded the ST-VMS. To increase their scope and accuracy, the ALE-VMS and RBVMS are often supplemented with special methods, such as those for weakly-enforced no-slip



boundary conditions [29–31], "sliding interfaces" [32,33] and backflow stabilization [34]. The ALE-SUPS, RBVMS and ALE-VMS have been applied to many classes of FSI, MBI and fluid mechanics problems. The classes of problems include ram-air parachute FSI [28], wind-turbine aerodynamics and FSI [35-45], more specifically, vertical-axis wind turbines [44–47], floating wind turbines [48], wind turbines in atmospheric boundary layers [43–45,49], and fatigue damage in wind-turbine blades [50], patient-specific cardiovascular fluid mechanics and FSI [21,51-56], biomedicaldevice FSI [57-62], ship hydrodynamics with free-surface flow and fluid-object interaction [63,64], hydrodynamics and FSI of a hydraulic arresting gear [65,66], hydrodynamics of tidal-stream turbines with free-surface flow [67], passive-morphing FSI in turbomachinery [68], bioinspired FSI for marine propulsion [69,70], bridge aerodynamics and fluid-object interaction [71-73], and mixed ALE-VMS/Immersogeometric computations [60–62,74,75] in the framework of the Fluid-Solid Interface-Tracking/Interface-Capturing Technique [76]. Recent advances in stabilized and multiscale methods may be found for stratified incompressible flows in [77], for divergence-conforming discretizations of incompressible flows in [78], and for compressible flows with emphasis on gas-turbine modeling in [79].

The ST-SUPS and ST-VMS have also been applied to many classes of FSI, MBI and fluid mechanics problems (see [80] for a comprehensive summary). The classes of problems include spacecraft parachute analysis for the landing-stage parachutes [24,81–84], cover-separation parachutes [85] and the drogue parachutes [86–88], wind-turbine aerodynamics for horizontal-axis wind-turbine rotors [24,35,89,90], full horizontal-axis wind-turbines [41,91-93] and vertical-axis wind-turbines [6,44,45], flapping-wing aerodynamics for an actual locust [10,24,94,95], bioinspired MAVs [92,93,96,97] and wing-clapping [8,98], blood flow analysis of cerebral aneurysms [92,99], stent-blocked aneurysms [99–101], aortas [102–105] and heart valves [2,8,9,93,104,106,107], spacecraft aerodynamics [85,108], thermo-fluid analysis of ground vehicles and their tires [5,106], thermo-fluid analysis of disk brakes [7], flow-driven string dynamics in turbomachinery [109–111], flow analysis of turbocharger turbines [11,112–114], flow around tires with road contact and deformation [1,106,115,116], fluid films [12], ram-air parachutes [117], and compressible-flow spacecraft parachute aerodynamics [118,119].

The ST-SUPS, ALE-SUPS, RBVMS, ALE-VMS and ST-VMS all have some embedded stabilization parameters that play a significant role (see [24]). There are many ways of defining these stabilization parameters (for examples, see [1, 5,6,14,15,91,120–136]). We will specify which ones we use here when we describe the computations in Sects. 9–11.

For more on the ST-VMS and ST-SUPS, see [24]. In the flow analyses presented here, the ST framework provides higher-order accuracy in a general context. The VMS and SUPS features of the ST-VMS and ST-SUPS address the computational challenges associated with the multiscale nature of the unsteady flow. The moving-mesh feature of the ST framework enables high-resolution computation near the tire surfaces.

#### 3 ST-SI

The ST-SI was introduced in [6], in the context of incompressible-flow equations, to retain the desirable movingmesh features of the ST-VMS and ST-SUPS when we have spinning solid surfaces, such as a turbine rotor. The mesh covering the spinning surface spins with it, retaining the high-resolution representation of the boundary layers. The starting point in the development of the ST-SI was the version of the ALE-VMS for computations with sliding interfaces [32,33]. Interface terms similar to those in the ALE-VMS version are added to the ST-VMS to account for the compatibility conditions for the velocity and stress at the SI. That accurately connects the two sides of the solution. An ST-SI version where the SI is between fluid and solid domains was also presented in [6]. The SI in this case is a "fluid-solid SI" rather than a standard "fluid-fluid SI" and enables weak enforcement of the Dirichlet boundary conditions for the fluid. The ST-SI introduced in [7] for the coupled incompressible-flow and thermal-transport equations retains the high-resolution representation of the thermo-fluid boundary layers near spinning solid surfaces. These ST-SI methods have been applied to aerodynamic analysis of vertical-axis wind turbines [6,44,45], thermo-fluid analysis of disk brakes [7], flow-driven string dynamics in turbomachinery [109–111], flow analysis of turbocharger turbines [11,112–114], flow around tires with road contact and deformation [1,106,115,116], fluid films [12], aerodynamic analysis of ram-air parachutes [117], and flow analysis of heart valves [2,104,107].

In the ST-SI version presented in [6] the SI is between a thin porous structure and the fluid on its two sides. This enables dealing with the porosity in a fashion consistent with how the standard fluid–fluid SIs are dealt with and how the Dirichlet conditions are enforced weakly with fluid–solid SIs. This version also enables handling thin structures that have T-junctions. This method has been applied to incompressible-flow aerodynamic analysis of ram-air parachutes with fabric porosity [117]. The compressible-flow ST-SI methods were introduced in [118], including the version where the SI is between a thin porous structure and the fluid on its two sides. Compressible-flow porosity models were also introduced in [118]. These, together with the compressible-flow ST SUPG method [137], extended the ST computational analysis range to compressible-flow aerodynamics of parachutes with fab-



ric and geometric porosities. That enabled ST computational flow analysis of the Orion spacecraft drogue parachute in the compressible-flow regime [118,119].

For more on the ST-SI, see [6,7]. In the tire-aerodynamics computational analysis here, the mesh covering the tire spins with it, and the SI between the spinning mesh and the rest of the mesh accurately connects the two sides of the solution. This enables high-resolution representation of the boundary layers near the tire. In the computations here, the ST-SI is used also in combination with the ST-TC, and we will describe the ST-SI-TC in Sect. 5.

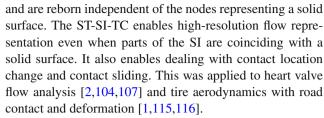
#### 4 ST-TC

The ST-TC [8,9] was introduced for moving-mesh computation of flow problems with TC, such as contact between solid surfaces. Even before the ST-TC, the ST-SUPS and ST-VMS, when used with robust mesh update methods, have proven effective in flow computations where the solid surfaces are in near contact or create other near TC, if the nearness is sufficiently near for the purpose of solving the problem. Many classes of problems can be solved that way with sufficient accuracy. For examples of such computations, see the references mentioned in [8]. The ST-TC made moving-mesh computations possible even when there is an actual contact between solid surfaces or other TC. By collapsing elements as needed, without changing the connectivity of the "parent" mesh, the ST-TC can handle an actual TC while maintaining high-resolution boundary layer representation near solid surfaces. This enabled successful moving-mesh computation of heart valve flows [2,8,9,93,104,106,107], wing clapping [98], and flow around a rotating tire with road contact and prescribed deformation [1,106,115,116].

For more on the ST-TC, see [8,9]. In the computational analyses here, the ST-TC enables moving-mesh computation even with the TC created by the actual contact between the tire and the road. It deals with the contact while maintaining high-resolution flow representation near the tire.

#### 5 ST-SI-TC

The ST-SI-TC is the integration of the ST-SI and ST-TC. A fluid–fluid SI requires elements on both sides of the SI. When part of an SI needs to coincide with a solid surface, which happens for example when the solid surfaces on two sides of an SI come into contact or when an SI reaches a solid surface, the elements between the coinciding SI part and the solid surface need to collapse with the ST-TC mechanism. The collapse switches the SI from fluid–fluid SI to fluid–solid SI. With that, an SI can be a mixture of fluid–fluid and fluid–solid SIs. With the ST-SI-TC, the elements collapse



For more on the ST-SI-TC, see [2,115]. In the computational analyses presented here, the ST-SI-TC enables contact location change and contact sliding between the tire and road surfaces.

#### 6 ST-IGA

The ST-IGA is the integration of the ST framework with isogeometric discretization. It was introduced in [3]. Computations with the ST-VMS and ST-IGA were first reported in [3] in a 2D context, with IGA basis functions in space for flow past an airfoil, and in both space and time for the advection equation. Using higher-order basis functions in time enables getting full benefit out of using higher-order basis functions in space. This was demonstrated with the stability and accuracy analysis given in [3] for the advection equation.

The ST-IGA with IGA basis functions in time enables a more accurate representation of the motion of the solid surfaces and a mesh motion consistent with that. This was pointed out in [3,4] and demonstrated in [10,94,96]. It also enables more efficient temporal representation of the motion and deformation of the volume meshes, and more efficient remeshing. These motivated the development of the ST/NURBS Mesh Update Method (STNMUM) [10,94,96]; the name was given in [91]. The STNMUM has a wide scope that includes spinning solid surfaces. With the spinning motion represented by quadratic NURBS in time, and with sufficient number of temporal patches for a full rotation, the circular paths are represented exactly. A "secondary mapping" [3,4,10,24] enables also specifying a constant angular velocity for invariant speeds along the circular paths. The ST framework and NURBS in time also enable, with the "ST-C" method, extracting a continuous representation from the computed data and, in large-scale computations, efficient data compression [5,7,106,109-111,138]. The STNMUM and the ST-IGA with IGA basis functions in time have been used in many 3D computations. The classes of problems solved are flapping-wing aerodynamics for an actual locust [10,24,94,95], bioinspired MAVs [92,93,96,97] and wingclapping [8,98], separation aerodynamics of spacecraft [85], aerodynamics of horizontal-axis [41,91–93] and verticalaxis [6,44,45] wind-turbines, thermo-fluid analysis of ground vehicles and their tires [5,106], thermo-fluid analysis of disk brakes [7], flow-driven string dynamics in turbomachin-



ery [109–111], and flow analysis of turbocharger turbines [11,112–114].

The ST-IGA with IGA basis functions in space enables more accurate representation of the geometry and increased accuracy in the flow solution. It accomplishes that with fewer control points, and consequently with larger effective element sizes. That in turn enables using larger time-step sizes while keeping the Courant number at a desirable level for good accuracy. It has been used in ST computational flow analysis of turbocharger turbines [11,112–114], flow-driven string dynamics in turbomachinery [110,111], ram-air parachutes [117], spacecraft parachutes [119], aortas [104,105], heart valves [2,104,107], tires with road contact and deformation [1,116], and fluid films [12]. Using IGA basis functions in space is now a key part of some of the newest arterial zero-stress-state (ZSS) estimation methods [139–142] and related shell analysis [143].

For more on the ST-IGA, see [11,24,94,117]. In the computational flow analyses presented here, the ST-IGA enables more accurate representation of the tire geometry and increased accuracy in the flow solution.

#### 7 ST-SI-TC-IGA

The turbocharger turbine analysis [11,112–114] and flow-driven string dynamics in turbomachinery [110,111] were based on the integration of the ST-SI and ST-IGA. The IGA basis functions were used in the spatial discretization of the fluid mechanics equations and also in the temporal representation of the rotor and spinning-mesh motion. That enabled accurate representation of the turbine geometry and rotor motion and increased accuracy in the flow solution. The IGA basis functions were used also in the spatial discretization of the string structural dynamics equations. That enabled increased accuracy in the structural dynamics solution, as well as smoothness in the string shape and fluid dynamics forces computed on the string.

The ram-air parachute analysis [117] and spacecraft parachute compressible-flow analysis [119] were based on the integration of the ST-IGA, the ST-SI version that weakly enforces the Dirichlet conditions, and the ST-SI version that accounts for the porosity of a thin structure. The ST-IGA with IGA basis functions in space enabled, with relatively few number of unknowns, accurate representation of the parafoil and parachute geometries and increased accuracy in the flow solution. The volume mesh needed to be generated both inside and outside the parafoil. Mesh generation inside was challenging near the trailing edge because of the narrowing space. The spacecraft parachute has a very complex geometry, including gores and gaps. Using IGA basis functions addressed those challenges and still kept the ele-

ment density near the trailing edge of the parafoil and around the spacecraft parachute at a reasonable level.

The heart valve analysis [2,104,107] was based on the integration of the ST-SI, ST-TC and ST-IGA. The ST-SI-TC-IGA, beyond enabling a more accurate representation of the geometry and increased accuracy in the flow solution, kept the element density in the narrow spaces near the contact areas at a reasonable level. When solid surfaces come into contact, the elements between the surface and the SI collapse. Before the elements collapse, the boundaries could be curved and rather complex, and the narrow spaces might have high-aspect-ratio elements. With NURBS elements, it was possible to deal with such adverse conditions rather effectively.

In computational analysis of flow around tires with road contact and deformation [1], the ST-SI-TC-IGA enables a more accurate representation of the geometry and motion of the tire surfaces, a mesh motion consistent with that, and increased accuracy in the flow solution. It also keeps the element density in the tire grooves and in the narrow spaces near the contact areas at a reasonable level. In addition, we benefit from the mesh generation flexibility provided by using SIs.

An SI provides mesh generation flexibility in a general context by accurately connecting the two sides of the solution computed over nonmatching meshes. This type of mesh generation flexibility is especially valuable in complex-geometry flow computations with isogeometric discretization, removing the matching requirement between the NURBS patches without loss of accuracy. This feature was used in the flow analysis of heart valves [2,104,107], turbocharger turbines [11,112–114], and spacecraft parachute compressible-flow analysis [119].

For more on the ST-SI-TC-IGA, see [1,2]. In the computations presented here, the ST-SI-TC-IGA is used for the reasons given and as described in the earlier paragraphs of this section.

### 8 Remedy for the trapped fluid

With the ST-SI-TC, we can deal with a TC in the domain, which may create isolated subdomains with trapped fluid. When a subdomain is isolated it may be difficult to satisfy the incompressibility constraint as the volume changes its shape, and that would impact to the solution process. As one way of remedying that, we create a path that enables mass flow between the isolated subdomain and the rest of the domain. To reduce the unintended effects of the path, we make it a narrow gap such that the flow going through it is laminar.

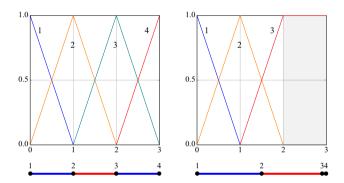
As stated in [12], with a consistent formulation, the Reynolds-equation limit can be represented by a quadratic NURBS element. This means that when the height of the gap is small enough, just one quadratic NURBS element would



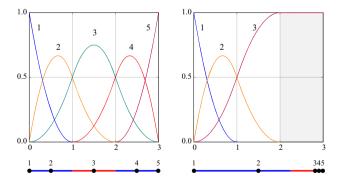
be enough. This motivates us to use a single quadratic element across the gap. As also stated in [12], the quadratic NURBS element would need to have a constant parametrization across the gap. Here we explore how to create that constant-parametrization quadratic NURBS element.

With the ST-SI-TC, we can collapse or re-form space by using a master–slave relationship. With a similar process, we can reduce the number of control points as a space occupied by the fluid shrinks down to a narrow gap. Before describing the process for quadratic NURBS elements, we explain the concept for linear elements in Fig. 1.

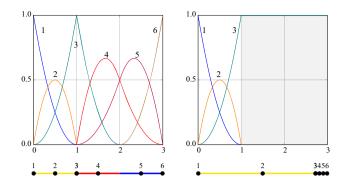
While reducing the number of control points is a simple process when we are using linear elements, that is not the case for higher-order NURBS elements. Figure 2 is an example of starting with three quadratic elements and reducing the number of control points. To reduce the number of control points to three, we make two of the control points slaves of the last control point. This results in two elements that are not desirable.



**Fig. 1** Meshes and shape functions for the linear elements. The left mesh, which we see as the "parent" mesh, is for a wider stage of the gap. The right mesh is for when the gap is narrow and we reduce the number of control points to three by TC collapse. This results in two elements. The *gray* zone indicates the unused elements in the parametric space



**Fig. 2** Meshes and shape functions for the quadratic NURBS elements. The left mesh, which we see as the "parent" mesh, is for a wider stage of the gap. The right mesh is for when the gap is narrow and we reduce the number of control points to three by TC collapse. This results in two elements that are not desirable. The *gray* zone indicates the unused elements in the parametric space



**Fig. 3** Meshes and shape functions for the quadratic NURBS elements. The left mesh, which we see as the "parent" mesh, is for a wider stage of the gap. The right mesh is for when the gap is narrow and we reduce the number of control points to three by TC collapse. This results in one element with constant parametrization, which is what we wanted. The gray zone indicates the unused elements in the parametric space. We note that the parent mesh has a special element, which introduces a  $C^0$ -continuity

**Remark 1** The quadratic-NURBS framework is not suitable for creating slaves to reduce the number of control points. A better framework would involve knot removal, which would basically require knot changes in time. However, that is beyond the scope of NURBS and may involve ST elements that are not tensor-products.

In the meantime, we propose a simpler solution, explained in Fig. 3. This solution includes accepting a  $C^0$ -continuity in the mesh.

## 9 2D test computation with a straight channel

We test the process of reducing the number of control points with the ST-SI-TC. We use an SI at one of the horizontal boundaries. The SI has an important role in a typical use of the ST-SI-TC. It enables contact location change and contact sliding. It also enables changing the location, with respect to the solid surface, where we reduce the number of control points.

### 9.1 Problem setup

Figure 4 shows the domain size and the boundary conditions. To test the SI part of the ST-SI-TC, we have an SI at the bottom boundary. The domain represents a narrow gap between two solid surfaces, after the SI, with TC collapse, coincides with the bottom solid surface. The flow is from left to right, and at the top and bottom boundaries we have no-slip condition. Because the bottom solid surface coincides with the SI, the no-slip condition is enforced weakly there. The Reynolds number is 1.0.



#### 9.2 Meshes

Having a special quadratic NURBS element as explained in Fig. 3 can be accomplished in two ways. In the first way, shown in Fig. 5, we have one layer of special quadratic NURBS elements near the top solid surface. In the second way, shown in Fig. 6, the layer of special quadratic NURBS elements are just above the SI. In both cases, the parent mesh has elements below the SI, but the physical mesh will not. How the elements in these two cases collapse with the TC mechanism is explained in Figs. 7 and 8.

**Remark 2** In theory, collapsing elements to the SI as shown in Fig. 7 is acceptable. However, then in the implementation of the SI formulation, such as numerical integration, we have to use elements that are not neighboring the SI in the parent mesh. That is not convenient for parallel implementation. Therefore we use the way described in Figs. 6 and 8.



**Fig. 4** Domain size and boundary conditions. *Red, blue, black*, and *orange* indicate the inflow, outflow, wall, and the SI. The domain represents a narrow gap between two solid surfaces, after the SI, with TC collapse, coincides with the bottom solid surface. (Color figure online)

**Fig. 5** Mesh-1. The *yellow-gray* elements, which are between the SI and the bottom solid surface, will collapse. The *red-gray* elements are the special quadratic NURBS elements we want to be left with, and they are placed at the top. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)

**Fig. 6** Mesh-2. The *yellow–gray* elements, which are between the SI and the bottom solid surface, will collapse. The *red–gray* elements are the special quadratic NURBS elements we want to be left with, and they are placed just above the SI. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)

**Fig. 7** Mesh-1-reduced. At the center part, the *blue-gray* elements collapse to the SI, and only the special quadratic NURBS elements remain, expanding to the SI. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)

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**Fig. 8** Mesh-2-reduced. At the center part, the *blue-gray* elements collapse to the top solid surface, and only the special quadratic NURBS elements remain, expanding to the top solid surface. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)

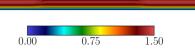


Fig. 9 Mesh-2-reduced. Velocity magnitude

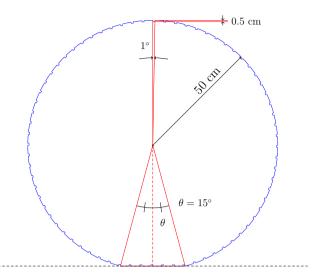


Fig. 10 A 2D tire model with grooves. There is a groove for every  $5^{\circ}$ . The deformation region is the circular sector with central angle  $30^{\circ}$ 

#### 9.3 Computation

We use Mesh-2-reduced. The method is the ST-VMS, and the stabilization parameters are those given by Eqs. (4)–(9) in [1]. The number of nonlinear iterations per time step is 3, and the number of GMRES [144] iterations per nonlinear iteration is 100. Figure 9 shows the velocity magnitude. The mesh transitions are handled well, resulting in a good solution.

## 10 2D test computations with different contact representations

We test our method on a multiscale problem in 2D, with different ways of handling the region near the contact.

#### 10.1 Problem setup

A 2D tire model with a groove for every  $5^{\circ}$  is in contact with a planar solid surface and undergoes deformation (see Fig. 10). The contact angle  $2\theta=30^{\circ}$ . The rotation speed corresponds to a linear speed of U=100 km/h at the undeformed tire periphery. There is no slip between the spinning and planar surfaces. The speed of the planar surface becomes

$$U_0 = \frac{\sin \theta}{\theta} U,\tag{1}$$



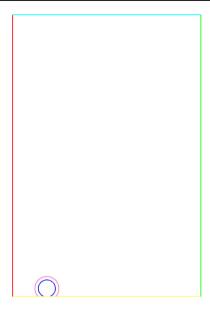


Fig. 11 A 2D tire model with grooves. Computational domain and boundary conditions. The left (red) and bottom (yellow) boundaries are the inflow and the moving planar surface, where the velocity is  $U_0$ . The innermost (blue) circle is the spinning surface, where the velocity is U. The larger (pink) circle is the SI. The bottom of the SI is coinciding with the planar surface and the interface of the spinning and planar surfaces. The conditions at the right (green) and upper (cyan) boundaries are stress-free and slip, respectively. (Color figure online)

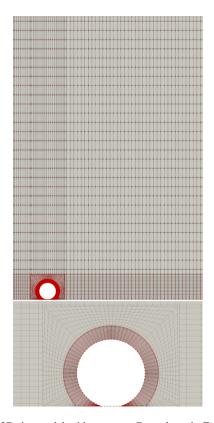
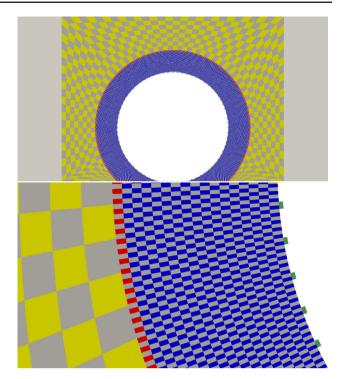


Fig. 12 A 2D tire model with grooves. Control mesh. Full view and focus view around the tire



**Fig. 13** A 2D tire model with grooves. The domain and elements near the tire. The *gray* and *yellow-gray* areas are the stationary domains, and the area with the other colors is the rotating domain. The *red-gray* elements are the special quadratic NURBS elements. *Green-gray* indicate the groove elements. *Blue-gray* indicate the remaining elements. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)



**Fig. 14** A 2D tire model with grooves. Contact representations: "closing deisolation gaps (CDG)" (*top*), "untrapping the fluid with deisolation gaps (UFDG)" (*middle*), and "all deisolation gaps (ADG)" (*bottom*)

giving  $U_0 = 98.86 \text{ km/h}$ . The density and kinematic viscosity of the air are  $1.205 \text{ kg/m}^3$  and  $1.822 \times 10^{-5} \text{ m}^2/\text{s}$ .

## 10.2 Computational domain, boundary conditions and mesh

Figure 11 shows the computational domain and boundary conditions. The domain size is  $11.00 \times 16.48$  m<sup>2</sup>. The left boundary is 2.0 m from the tire center. We have no-slip condition on the planar and tire surfaces. The no-slip condition is enforced weakly on the part of the planar surface coinciding with the SI. We use quadratic NURBS. Figure 12 shows the



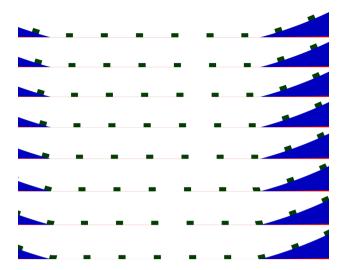


Fig. 15 A 2D tire model with grooves. Mesh motion for UFDG. We see that the full-contact zone switches (see the 5th and 6th frames)

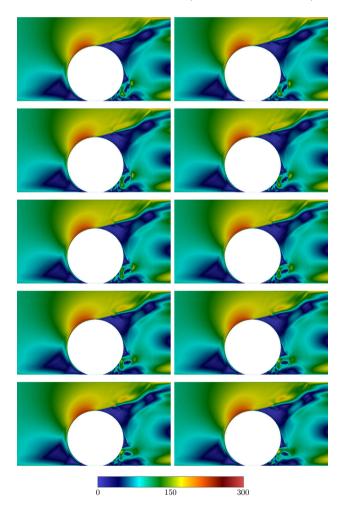


Fig. 16  $\,$  A 2D tire model with grooves. Velocity magnitude (km/h) for CDG near the contact region

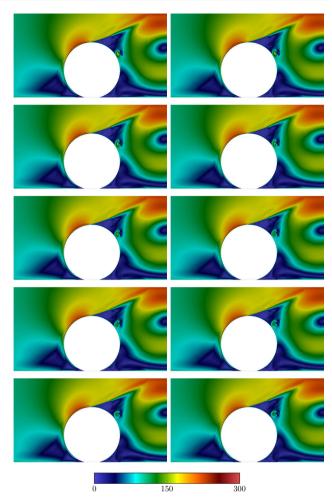


Fig. 17  $\,$  A 2D tire model with grooves. Velocity magnitude (km/h) for UFDG near the contact region

control mesh. The number of control points is 26,902, and the number of elements is 18,348.

Figure 13 shows the domain and elements near the tire. The number of elements in the circumferential direction is 720. In the rotating domain, we have three regions: the region with special quadratic NURBS elements, the grooves, which have 2 elements in the circumferential direction and 18 elements in the radial direction, and the rest of the domain, which has 16 elements in the radial direction.

We consider three types of the contact representations: "closed deisolation gaps (CDG)," "untrapping the fluid with deisolation gaps (UFDG)," and "all deisolation gaps (ADG)." Figure 14 shows each representation.

The CDG case has no gaps, and the fluid is trapped in the grooves. The UFDG case has only one full-contact zone. The rest of the gap has a height of 0.1 mm. The ADG case does not have a full-contact zone at all and the gap height is the same as in the UFDG case. Figure 15 shows the mesh motion for UFDG.



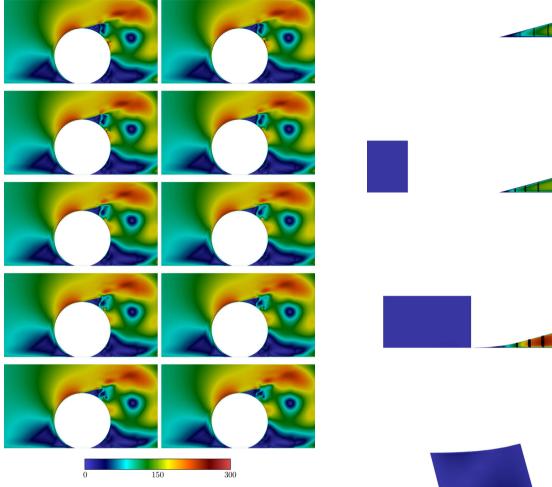


Fig. 18  $\,$  A 2D tire model with grooves. Velocity magnitude (km/h) for ADG near the contact region

#### 10.3 Computational conditions

In all three computations, the method is the ST-SUPS, and the stabilization parameters are those given by Eqs. (4)–(9) in [1]. There are 1000 time steps per rotation, which is equivalent to a time-step size of  $1.131 \times 10^{-4}$  s. The number of nonlinear iterations per time step is 3, and the number of GMRES iterations per nonlinear iteration is 300.

#### 10.4 Results and discussion

Figures 16, 17, and 18 show the ninth rotation. The common point for all three cases is that the flow is affected by the fluid in the groove. In front of the tire the flow fields are comparable in all three cases. At the back of the tire the flow fields are close between UFDG and ADG, but CDG is quite different. This indicates that the last contact zone at the back of the tire makes a big difference.

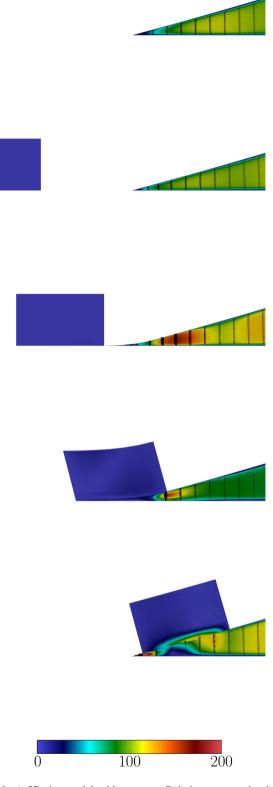


Fig. 19 A 2D tire model with grooves. Relative-to-ground velocity magnitude (km/h) for CDG near the contact region. The *black* vectors indicate the velocity



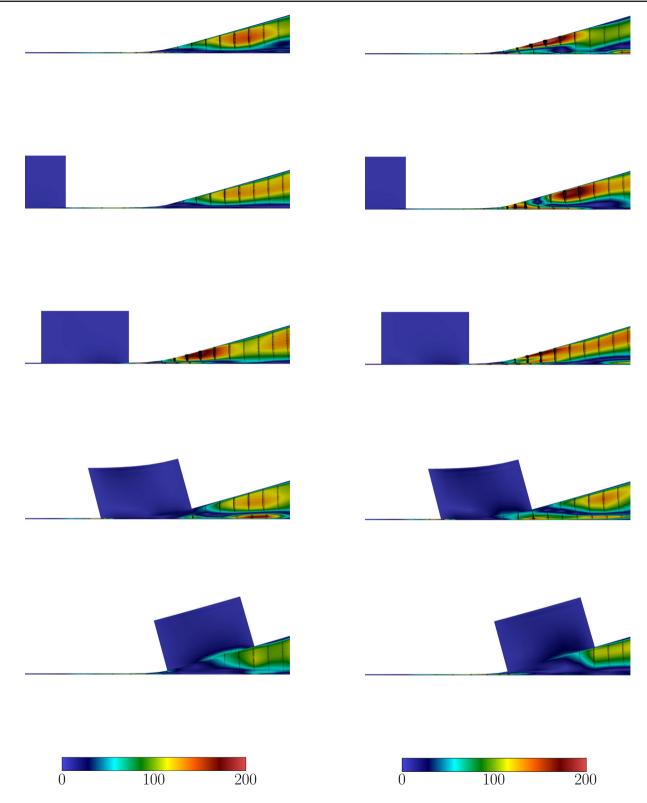
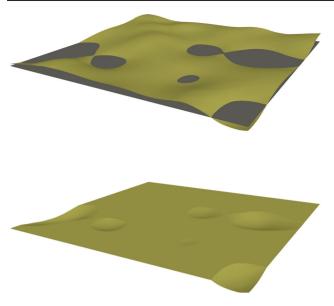


Fig. 20 A 2D tire model with grooves. Relative-to-ground velocity magnitude (km/h) for UFDG near the contact region. The *black* vectors indicate the velocity

Fig. 21 A 2D tire model with grooves. Relative-to-ground velocity magnitude (km/h) for ADG near the contact region. The black vectors indicate the velocity





**Fig. 22** Road surface with dents and bumps (*top*). The *gray* plane represents the tire surface. Road surface with dents only (*bottom*)

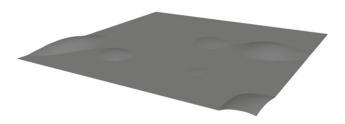


Fig. 23 Modified tire surface representing the road dents

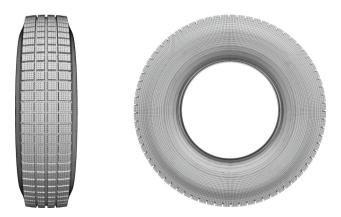


Fig. 24 Tire aerodynamics with actual tire geometry and road roughness. Tire model

We zoom in on the back of the contact region. Figures 19, 20, and 21 show the velocity relative to the ground. In the CDG case, most of the flow is from right to left, as expected (tire is moving to the left). In the other two cases, however, some of the flow is from left to right. This is because the lower pressure at the back of the tire causes flow from the deisolation gap.





**Fig. 25** Tire aerodynamics with actual tire geometry and road roughness. Deformed shape

Because this is a 2D model, the effect of the deisolation might be emphasized. We conclude that the ST-SI-TC can handle the contact and near contact representations very well.

## 11 Tire aerodynamics with actual tire geometry and road roughness

In this section, we present a tire-aerodynamics computation with actual tire geometry, road contact, tire deformation, road roughness and fluid film. Here we see the effect of the road roughness, with the dents decreasing the full-contact area, and that gives us a more realistic contact model.

#### 11.1 Representation of the road roughness

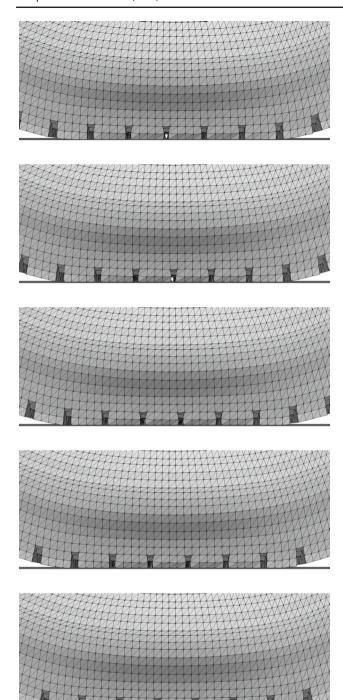
We describe how we deal with the road roughness. Figure 22 shows the road surface and the plane representing the tire surface. Surface regions below the plane are dents and they contain air. In our mesh, the road representation is relatively coarse, not suitable for roughness modeling. Therefore, we flip the situation and the tire surface represents the dents, as shown in Fig. 23. In addition, we create paths on the tire surface to deisolate the dents.

**Remark 3** We note that we do not model the dents coinciding with the grooves. That is because we assume that the groove depth is significantly larger than the dent depth.

#### 11.2 Tire model

The tire model is shown in Fig. 24. The model is the same as the one in [1]. The diameter and width are 1.03 m and 260 mm. There are three longitudinal grooves, and a transverse groove for every 5°. The depth and width of the grooves are 11.071 mm and 11.692 mm for the center groove, 10.974 mm and 7.177 mm for the side grooves, and





**Fig. 26** Tire aerodynamics with actual tire geometry and road roughness. Tire deformation near the contact region at five instants of a 5° rotation, provided by the tire company

11.085 mm and 8.489 mm for the transverse grooves. Tire with the prescribed deformation is shown in Fig. 25.

The tire deformation is represented in time based on the deformation at five instants of a 5° rotation, which was provided by the tire company. Figure 26 shows the tire deformation at those five instants. The deformation representation

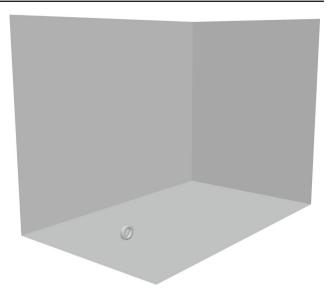
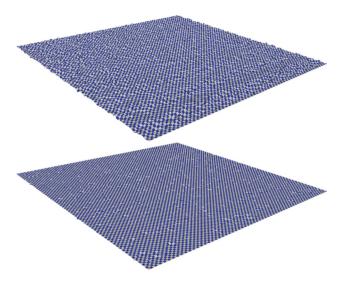


Fig. 27 Tire aerodynamics with actual tire geometry and road roughness. Computational domain



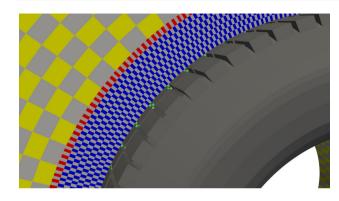
**Fig. 28** Road surface (*top*) and road surface with dents only (*bottom*). For visualization purposes the roughness is amplified in the height direction

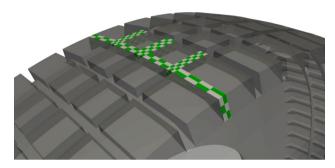
in time is with cubic NURBS basis functions and obtained by projection from the five-instant data. The projection is done with the ST-C [138]. The rotation speed corresponds to a linear speed of 100 km/h at the undeformed tire periphery, and  $U_0 = 98.86$  km/h. The density and kinematic viscosity of the air are 1.205 kg/m<sup>3</sup> and  $1.822 \times 10^{-5}$  m<sup>2</sup>/s.

## 11.3 Computational domain and boundary conditions

The computational domain is shown in Fig. 27. The domain size is 12.000 m and 14.489 m in width and height, and 19.000 m in the flow direction. The tire is placed at 4.000 m







**Fig. 29** Tire aerodynamics with actual tire geometry and road roughness. The mesh near the tire (*top*) and in the grooves (*bottom*). The *red–gray* elements are the special quadratic NURBS elements. *Green–gray* indicate the groove elements. *Blue–gray* indicate the remaining elements. The checkerboard coloring is for differentiating between the NURBS elements. (Color figure online)

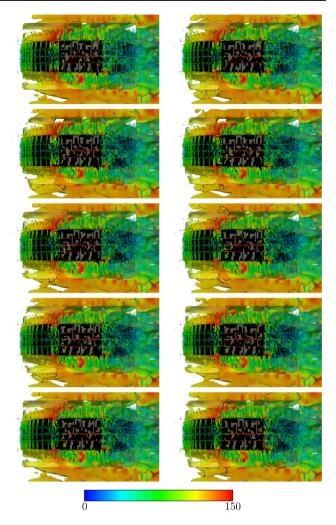
from the inflow boundary. The boundary conditions are 3D extensions of the conditions in the 2D model, with slip conditions on the boundary planes perpendicular to the tire axis. While we have no-slip condition on the road and tire surfaces, the condition is enforced weakly on the part of the road surface coinciding with the SI.

#### 11.4 Road roughness

The road roughness is added in a random fashion, with  $R_{\rm a}=0.21~{\rm mm}$  and  $S_{\rm m}=17.68~{\rm mm}$ , where  $R_{\rm a}$  and  $S_{\rm m}$  are  $L_1$  norm of the surface position relative to the average surface position and mean spacing between the peaks. Figure 28 shows the road roughness. The dents are then projected to the tire surface. In addition, when a dent is isolated, we create a gap with a depth of 0.1 mm for the control point, serving as a path that deisolates the dent.

#### 11.5 Mesh and computational conditions

The mesh, shown in Fig. 29, consists of 706,872 control points and 342,296 elements. The method is the ST-SUPS, and the stabilization parameters are those given by Eqs. (4)–(9) in [1]. There are 1440 time steps per rotation, which is



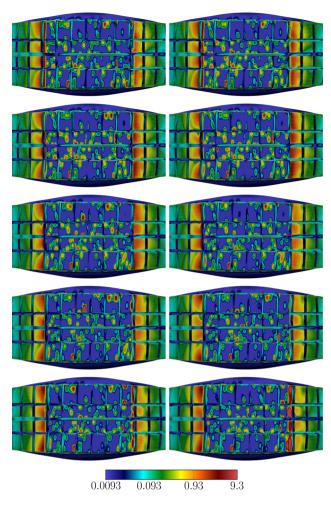
**Fig. 30** Tire aerodynamics with actual tire geometry and road roughness. Isosurfaces corresponding to a positive value of the second invariant of the velocity gradient tensor, colored by the velocity magnitude (km/h), viewed from the bottom. The *gray* zones are the full-contact areas. (Color figure online)

equivalent to a time-step size of  $8.09 \times 10^{-5}$  s. The number of nonlinear iterations per time step is 3, and the number of GMRES iterations per nonlinear iteration is 300.

#### 11.6 Results

Figure 30 shows the isosurfaces corresponding to a positive value of the second invariant of the velocity gradient tensor, colored by the velocity magnitude. Figure 31 shows positional-averaged shear stress acting on the tire surface, which is the fluid friction. As can be seen from the vectors included in the figure, there is flow in the deisolation gaps, to and from the dents, depending on the tire deformation pattern.





**Fig. 31** Tire aerodynamics with actual tire geometry and road roughness. Positional-averaged shear stress (Pa) acting on the tire surface. The vectors included indicate only the direction of the shear stress

#### 12 Concluding remarks

Extending the work that addressed the challenges associated with computational analysis of tire aerodynamics with actual tire geometry, road contact and tire deformation, in this article we included and addressed the computational challenges associated with the road roughness and the fluid film between the tire and the road. The challenges include (i) the complexity of an actual tire geometry with longitudinal and transverse grooves, (ii) the spin of the tire, (iii) maintaining accurate representation of the boundary layers near the tire while being able to deal with the flow-domain topology change created by the road contact, (iv) the turbulent nature of the flow, (v) dealing with the fluid trapped between the tire and road surfaces, (vi) maintaining the cost effectiveness and generality of the computational method as we bring solution to fluid films embedded in the flow domain, and (vii) representing the road roughness.

The ST computational method ST-SI-TC-IGA does most of the heavy lifting in overcoming the computational challenges. The core component of the ST-SI-TC-IGA is the ST-VMS, and the other key components are the ST-SI, ST-TC and ST-IGA. The challenge created by the turbulent nature of the flow is addressed with the VMS feature of the ST-VMS. The moving-mesh feature of the ST framework enables highresolution flow computation near the air-tire interfaces as the tire rotates. These two features are enhanced with the higher-order accuracy of the ST framework. With the ST-SI, we are able to do moving-mesh computations with the tire spinning. The mesh covering the tire spins with it, and the SI between the spinning mesh and the rest of the mesh accurately connects the two sides of the solution. With the ST-TC, we are able to do moving-mesh computations even with the TC created by the contact between the tire and the road. This enables dealing with the contact while maintaining high-resolution flow representation near the tire. Integration of the ST-SI and ST-TC enables high-resolution flow representation even though parts of the SI are coinciding with the tire and road surfaces. It also enables dealing with the tire-road contact location change and contact sliding. Integration of the ST-IGA with the ST-SI and ST-TC not only enables a more accurate representation of the tire geometry and increased accuracy in the flow solution, but also keeps the element density in the tire grooves and in the narrow spaces near the contact areas at a reasonable level.

The additional methods include a method for creating deisolation paths for the trapped fluid, a method for reducing the number of control points as a space occupied by the fluid shrinks down to a narrow gap, and a method for representing the road roughness without increasing the mesh resolution on the road surface.

We first presented a 2D test computation for a straight channel. This test showed that mesh transitions between the zones with multiple elements and a single element across a gap are handled well. Then we presented, in the context of a simple 2D tire model, test computations with different contact representations. These tests demonstrated the significance of the deisolation paths. The last computation was with a 3D tire model with actual tire geometry and road roughness. This computation showed that the integrated set of ST methods are fully capable of dealing with essentially all the computations presented show the effectiveness of the integrated set of ST methods targeting tire aerodynamics.

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