ORIGINAL

Performance, emission and combustion study on neat biodiesel and water blends fuelled research diesel engine

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Abstract

This work investigates the experimental study to examine the operation characteristics of an orange peel oil biodiesel (BD100) along with water as an oxygenated additive in various volume fractions. Hard water is mixed into 100% of BD100 at 4 and 8% volume. Span 80 was employed as a nonionic surfactant, which emulsifies water into biodiesel. The experimental investigation on diesel engine reveals that the addition of water to BD100 resulted in enhancement of brake thermal efficiency (BTE) with a reduction in brake specific fuel consumption (BSFC). The tested emission parameters such as CO, HC, NOx and smoke were decreased by 11.4, 9.89, 3.36 and 5.14% for BD94W4S2 (94% volume of biodiesel) + (4% volume of water) + (2% volume of surfactant) and 12.22 , 10.11 , 4.85 and 6.18% for BD90W8S2 (90% volume of biodiesel) + (8% volume) + (2% volume) respectively when compared to BD100. Further, the cylinder pressure and heat release rate and were enhanced by adding water to base fuel (BD100). Overall, improvement in ignition characteristics of the engine was finer by adding, 4 and 8% volume of water to BD100 on account of its chemical properties.

1 Introduction

Diesel engines are governing the transportation, agricultural and power sectors down to their enhanced fuel economy, higher durability, and longer reliability than petrol engine. The great challenge of employing diesel fuel is the pollution caused by their use in the engines. Clean and sustainable fuels for diesel engines are extremely vital. In this context, biodiesel (BD) as a supplementary diesel engine fuel seems suitable to minimize the drawback of diesel fuel $[1-3]$ $[1-3]$ $[1-3]$. Nontoxic nature, biodegradability are the major benefits of biodiesel [\[3](#page-7-0), [4](#page-7-0)]. Currently, many countries are utilizing the biodiesel mixing its blend with pure diesel in their engines. Many authors have

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agreed to the reductions in pollutant emissions and comparable performance when diesel engines run by BD or its blends with raise in NO_x [\[5](#page-7-0)–[8\]](#page-7-0). Many investigations have employed using orange peel oil biodiesel as a prospective candidate as a possible option fuel to diesel. Deep [[9\]](#page-7-0) investigated the engine output parameters of a diesel engine using orange peel oil biodiesel and they found that 100% of palm biodiesel can be fed as fuel. Purushothaman [[10\]](#page-7-0) investigated the engine output parameters using various blends of orange peel oil biodiesel and their results have shown a significant improvement in thermal efficiency with lower fuel consumption. Further CO, HC, and NOx are reduced by blending orange peel oil biodiesel with diesel. From the comprehensive literature reports, it is observed that no significant study has been experimentally carried out on the water - orange peel oil biodiesel blends at different proportions to analyze of its combustion, emission and performance characteristics.

Many investigations have also proved that by adding oxygenated additives such as alcohol, water to biodiesel facilitates as a catalyst in boosting the engine combustion and perfor-mance and improve its fuel properties [\[11](#page-7-0)–[18](#page-7-0)]. Oxygenated additives alter the chemical composition of biodiesel and enhance the oxidation characteristics. The oxygenated additive also aids the engine combustion and performance characteristics of fuel $[17]$ $[17]$. Vellaiyan $[11]$ $[11]$ formulated the water-

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emulsified diesel and conducted the experiments in a research engine. The results show that an increase in water concentration with diesel directly reduces the NO_x emissions due to latent heat absorption of water particles during the combustion process. In another study, Vellaiyan [[12](#page-7-0)] tried the water emulsification method with diesel. Senthilkumar and Jaikumar [\[13\]](#page-7-0), Sadhik Basha and Anand [\[14\]](#page-7-0) attempted the water emulsification process with waste cooking oil and jatropha methyl ester oil blends respectively and found a significant reduction in NO_X emissions. Koc and Abdullah [\[15](#page-7-0)] conducted a research work on the water to diesel-biodiesel blends in a CI engine. Results showed a fuel shows a significant drop in NO_X , HC, CO and smoke emission by adding water to diesel-biodiesel blends. Lthnin [[16](#page-7-0)] conducted a research work on adding water to diesel. Significant reduction of HC, CO, NO, smoke emissions were obtained by adding water to diesel. Selim and Ghannam [[17\]](#page-7-0) investigated the effect of oxygenated-additives (water) in diesel and found a considerable reduction in all the emissions. An in detail review of the previous work has shown a space in accomplishing the addition of BD100 with water in CI engine application. Hence, the present study aims to analyze the emission, performance and combustion characteristics of BD100, water emulsified BD100 (BD94W4S2, and BD90W8S2) and compared with diesel.

2 Materials and methods

2.1 Base fuel preparations

The steam distillation plant was used for the extraction of orange peel oil. The plant consists of a steam preparation chamber and orange peel heating distillation chamber. Figure 1 shows the photographic view of distillation Set-up. The steam distillation process was employed to extract the

orange peel oil from the orange peels by using steam. 1.2 kg of orange peels are placed in the steam chamber and heated to 110 °C. The fumes consisting of orange essence and steam vapor are routed to a condensing chamber for cooling purpose. The mixture of liquid water and orange peel oil was collected in a collection tank. Orange peel oil was separated from the mixture due to its density difference. 1.2 kg of orange peel yielded 700 ml of orange peel oil. A 500 g sample of orange peel oil in the reactor was heated to the temperature of 65 °C. The molar ratio of 5:1 (methanol to orange peel oil) and catalysts of 0.3% (wt/wt) to orange peel oil was used in transesterification process. The mixture was allowed to cool in the vessel yielding two distinct layers of ester and glycerol. Orange oil biodiesel was separated and washed with water. The emulsion preparation was carried out by varying the concentration of orange oil biodiesel, surfactants (Span 80), and water. In the emulsion preparation process, an electromagnetic stirrer along with ultrasonicator was used to mix the water with oil in presence of surfactants [\[19](#page-7-0)].

2.2 Properties of test fuels

The physicochemical properties of BD90W8S2, BD94W4S2, BD100, and diesel are listed in Table [1](#page-2-0). The density of BD100 is 5.7% higher than diesel due to its molecular structure and weight. An inclusion of water in BD100 marginally increases its density. The viscosity of BD100 is 31% higher than diesel that may affect the fuel atomization process in the diesel engines [[20\]](#page-7-0). However, upon adding the water to biodiesel (BD90W8S2, BD94W4S2) exhibits 13.2% reduction in viscosity. Lower viscous fuel, as in the case of BD90W8S2, BD94W4S2 will result in proper mixing with air and end up with better combustion than BD100 [[20](#page-7-0)]. The calorific value of BD100 is 9.5% lower than diesel, which indicates that for the, BD100 fuelled diesel engine requires more fuel for same engine power output. An inclusion of water further (BD90W8S2, BD94W4S2) reduces the heating value of fuel due to latent heat absorption of water. However, the supplementary oxygen present in BD90W8S2, BD94W4S2 can reimburse the fuel consumption $[21]$. The cetane index of diesel is higher than biofuels due to its shorter chain length. The fuel with higher cetane index (BD100, BD90W8S2, and BD94W4S2) will promote smoother and complete combustion [[21](#page-7-0)]. The flash point of BD100, BD90W8S2, and BD94W4S2 is higher than diesel. Higher flash points are desirable for safer handling of fuel for transportation and storage.

2.3 Experimental setup

The experimental investigation was carried out on a naturally aspirated, 2-cylinder, four-stroke, and DI (Direct injection) diesel engine with a rated power of 9 kW at 2400 rpm. The Fig. 1 Photographic view of distillation set-up engine technical specifications of the experimental setup are

Table 1 Properties of fuels

Properties	BD90W8S2	BD94W4S2	BD100	Diesel
Density @ 18 \degree C (gm/cc)	0.859	0.866	0.871	0.820
Kinematic viscosity @35 °C (mm ² /s)	3.3	3.5	3.8	2.5
Calorific value (kJ/kg)	38,257	38,576	38,887	42,957
Cetane index (CI)	52	53	54	47
Flash point	138	142	145	60

tabulated in Table 2. The experimental setup consisted of an engine, a dynamometer, fuel supply system, data acquisition unit, emission analyzer and a smoke opacimeter. AVL emission analyzer was employed to measure HC, NOx, and CO emissions. AVL 437C model opacimeter was employed to measure smoke opacity. The engine is made to warm up for 15 min after feeding each fuel blend so as to attain the static operating conditions. Uncertainty of all measured values was evaluated by square root method and shown in Table [3](#page-3-0).

3 Results and discussion

Table 2

3.1 Carbon monoxide emissions (BSCO)

Variation of BSCO with Brake Power (BP) for Diesel, BD100, BD94W4S2, and BD90W8S2 is presented in Fig. [2](#page-3-0). At lower BP, the formation of CO emissions from tested fuels is in lower range, which is due to the lower fuel intake and lean mixture. By increasing the BP on the engine, the quantity of fuel intake is increased causing incomplete combustion and higher CO emission [\[22,](#page-8-0) [23\]](#page-8-0). CO emission from BD100 is lesser than diesel at all conditions. This is down to inherent oxygen in BD100 [\[24](#page-8-0), [25](#page-8-0)]. Adding water reduces the CO emission from BD100. Addition of water reduces the temperature of fuel during combustion owing to its higher heat ab-sorption, which reduces BSCO emissions [\[11,](#page-7-0) [13\]](#page-7-0). The BSCO emissions of BD100 is further reduced by increasing the water from 4 to 8% vol. Higher volume of water augments the rate

of combustion and also provides excess oxygen and reduces BSCO emissions [\[14\]](#page-7-0). These result matches to the many literatures [\[11,](#page-7-0) [13](#page-7-0)] that reported lower BSCO emissions of diesel engine powered by biodiesel, diesel and water mixture. CO emissions at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 3.7, 3.1, 2.3 and 2.1 g/kW-hr respectively.

3.2 Unburned hydrocarbon emissions (BSHC)

Variation of BSHC emission with BP for Diesel, BD100, BD94W4S2, and BD90W8S2 is presented in Fig. [3](#page-4-0). The observation is similar to BSCO emissions. At lower brake power, the formation of BSHC emissions from tested fuels is in lower range, which is due to the lean mixture [\[26\]](#page-8-0). By increasing the BP on the engine, the magnitude of fuel is amplified which results in rich mixture causing incomplete combustion and higher BSHC emissions [[22,](#page-8-0) [27](#page-8-0)]. BSHC emission from BD100 is lesser than diesel. The inherent oxygen content of BD100 promotes combustion process and reduces BSHC emissions. Adding water reduces the BSHC emission from BD100. Addition of water reduces the temperature of fuel during combustion owing to its higher heat absorption, which reduces BSHC emissions [\[11](#page-7-0)]. The BSHC emissions of BD100 is further reduced by increasing the water from 4 to 8% vol. Water particles present in the mixture provides the additional oxygen for oxidation reaction of BSHC emissions. The other possible reason is due increase in concentration of hydroxyl with addition of water particles in BD100 [\[13\]](#page-7-0). The presence of water particles in the BD100 accelerates the heat sink effect which in turn lower the BSHC emission. These result matches to the many literatures [\[11](#page-7-0), [13,](#page-7-0) [15\]](#page-7-0) that reported lower BSHC emissions of diesel engine powered by biodiesel, diesel and water particle mixture. BSHC emissions at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 0.42, 0.33, 0.25 and 0.23 g/kW-hr respectively.

3.3 Oxides of nitrogen emissions ($BSNO_X$)

Variation of $BSNO_x$ emission with BP for Diesel, BD100, BD94W4S2, and BD90W8S2 is presented in Fig. [4.](#page-4-0) BSNOx emission increases with BP for all tested fuels. This is owing to the increasing cylinder temperature during com-bustion [[28](#page-8-0)]. BD100 has shown higher NO_x emission

Table 3 Gas analyzer range, accuracy and uncertainties details $\frac{M}{\sqrt{2}}$

Model of the gas analyzer	AVL gas analyzer			
Measured quantity	Range	Accuracy	Uncertainties	
$\rm CO$	$0 - 5000$ ppm	0.02%	± 0.5 (%)	
HС	$0 - 19,999$	± 10 ppm	± 0.1 (%)	
NOx	$0 - 4999$ ppm	± 10 ppm	± 0.3 (%)	
Smoke	AVL 437 Smoke meter	0.01%	± 1.0 (%)	
Cylinder pressure	$0 - 250$ bar	± 0.1 bar	± 0.1	
Crank angle	$0 - 360^{\circ}$	$\pm 1^{\circ}$	± 0.2	
Performance			Absolute	Relative
BTE			$\pm 0.01\%$	± 1

compared to diesel. The inherent oxygen content of BD100 is the chief motive for higher NO_x generation at all engine loads [\[29,](#page-8-0) [30](#page-8-0)]. However, $BSNO_x$ emission is significantly lower for BD100 with addition of water at all BP. Water present in BD100 initiates the combustion by reducing the combustion temperature and delay period $[14–16]$ $[14–16]$ $[14–16]$ $[14–16]$. NO_x emissions are further reduced with increase in water content from from 4 to 8% vol to POBD. The shorter ignition delay period with addition of water particles to POBD reduces the combustion duration and NO_x emission [\[17](#page-7-0)]. In addition, drop in local combustion temperature due high latent heat absorption of inner phase water particles during the combustion results in low level of BSNO_x emission [\[18\]](#page-7-0). BSNO_x emissions at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 10.9, 12.1, 10.1 and 10.7 g/kW-hr respectively.

3.4 Smoke opacity

Smoke opacity of Diesel, BD100, BD94W4S2, and BD90W8S2 is presented in Fig. [5.](#page-5-0) At lower BP, the formation of smoke emissions from tested fuels is in lower range, which is due to the lean mixture. With rise in BP, the quantity of fuel

Fig. 2 The difference in CO emissions with brake power

supplied is increased, which results in rich mixture causing incomplete combustion and higher smoke emissions. Smoke emission from BD100 is lesser than diesel. The inherent oxygen content of BD100 promotes combustion process and reduces smoke emissions [[19,](#page-7-0) [22,](#page-8-0) [30\]](#page-8-0). Adding water reduces the smoke emission from BD100. Addition of water reduces the temperature of fuel during combustion owing to its higher heat absorption, which reduces smoke emissions. The endothermic reaction of water associated with BD94W4S2, and BD90W8S2 shows low magnitude of smoke emission at all BP when compared with BD100 and diesel. These result matches to the many literatures [[31](#page-8-0)–[34](#page-8-0)] that reported lower smoke emissions of diesel engine powered by biodiesel, diesel and water mixture. Smoke emissions at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 1.5, 1.3, 1.1 and 0.8% respectively.

3.5 Brake thermal efficiency

BTE of Diesel, BD100, BD94W4S2, and BD90W8S2 are presented in Fig. [6.](#page-5-0) BTE of all tested fuels is improved with an increase in BP of the engine. The BTE for BD100,

BD94W4S2, and BD90W8S2 is lower than diesel at all engine loads.This is due to the higher density, lower calorific value, and higher viscosity of the biofuels [[12](#page-7-0), [20](#page-7-0), [21](#page-7-0)]. Adding water increases the BTE of BD100. This may be due to the latent heat absorption of water particles present in emulsion fuel and pro-longed ignition period [[4](#page-7-0), [5\]](#page-7-0). The other possible reason could be due to vigorous micro-explosion of water in BD100, which in turn increases the air and fuel mixing and results in higher BTE [[4,](#page-7-0) [5](#page-7-0)]. These result matches to the many literatures [\[31](#page-8-0)–[34\]](#page-8-0) that reported lower BTE of diesel engine powered by diesel, biodiesel and water mixture. BTE at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 29.8, 24.3, 25.1 and 26.3% respectively.

3.6 Brake specific energy consumption

BSFC of Diesel, BD100, BD94W4S2, and BD90W8S2 are presented in Fig. [7.](#page-6-0) With the increase in BP, the total quantity of BSFC increases but the brake power increases more effectively after overcoming some inevitable engine operating losses, for which the BSFC decreases [\[35,](#page-8-0) [36\]](#page-8-0). The BSFC for BD100, BD94W4S2, and BD90W8S2 is higher than diesel at all conditions. Lower heating value of the BD100, BD94W4S2, and BD90W8S2 increases the fuel consumption rate in all circumstances [\[11](#page-7-0), [17](#page-7-0)]. In addition, this is also due to the higher density, lower calorific value, and higher viscosity of the biofuels [\[12,](#page-7-0) [20](#page-7-0), [21\]](#page-7-0). Adding water lowers the BSFC of BD100. Water present in BD94W4S2 and BD90W8S2 accelerates the combustion process owing to improved micro-explosion behaviour [\[37\]](#page-8-0). The other possible reason for this behaviour is the water content in the BD94W4S2 and BD90W8S2 which converts into superheated steam and produce more power which in turn reduce the fuel consumption rate at all engine load [[7,](#page-7-0) [8](#page-7-0)]. This result matches to the many kinds of literature [[31](#page-8-0)–[34](#page-8-0)] that reported lower BSFC of diesel engine powered by biodiesel, diesel and water mixture. BSFC at peak conditions for Diesel, BD100, BD94W4S2, and BD90W8S2 are 0.21, 0.43, 0.29 and 0.24 kg/kW-hr respectively.

Fig. 4 The difference in NO_x emissions with brake power

3.7 In-cylinder pressure variation

The in-cylinder pressure variations for the BD100, BD94W4S2, and BD90W8S2 and diesel are depicted in Fig. [8](#page-6-0). The diesel engine produces similar in-cylinder pressure pattern for all test fuels. The peak pressure for BD100, BD94W4S2, and BD90W8S2 and Diesel is 66.2, 63.2, 61.8 and 57.11 bar. Since the viscosity of diesel is lower than BD100, BD94W4S2, and BD90W8S2, the ignition delay of diesel is shortened and results in uniform combustion and lower peak pressure [[22](#page-8-0), [24\]](#page-8-0). The other reason for lower peak pressure for diesel is due to its higher calorific value [\[38](#page-8-0)]. Fuel with higher calorific value requires a lesser quantity of fuel during combustion $[22, 25]$ $[22, 25]$ $[22, 25]$ $[22, 25]$. It is observed that adding water to BD100 observe a considerable increase in peak pressure. Water present in BD94W4S2 and BD90W8S2 accelerates the combustion process owing to improved micro-explosion behaviour [[20](#page-7-0), [21](#page-7-0)]. Hence the combustion is initiated early causing higher peak pressure. Further, the combustion starts early for BD100 with an increase in the content of water

particles [\[26,](#page-8-0) [39](#page-8-0)]. This is due be due to vigorous microexplosion of water in BD100, which in turn increases the air and fuel mixing and results in higher peak pressure [\[26\]](#page-8-0). This result matches to the many kinds of literature [\[22](#page-8-0)–[26\]](#page-8-0) that reported similar pressure variation of diesel engine powered by biodiesel, diesel and water mixture.

3.8 Heat release rate

The variation in heat release rate for the tested fuels is shown in Fig. [3.](#page-4-0) The possible reason for higher heat release rate for diesel is due to its higher calorific value. Fuel with higher calorific value produces more quantity of heat during combus-tion [\[22](#page-8-0)]. By adding 4 and 8 vol% of water particles to BD100 observe a considerable increase in heat release rate. This is due to the higher surface area to volume ratio of water particles which results in improved combustion, reduced delay period and higher heat release rate [[23](#page-8-0), [24](#page-8-0)]. Further, the HRR raises for BD100 with an increase in the content of water particles. This is due to vigorous micro-explosion of water in BD100,

with brake power

Fig. 7 The difference in BSFC with brake power

Fig. 8 Variation of in-cylinder pressure at 100% engine load across the crank angle

Fig. 9 Variation of Heat release rate at 100% engine load across the crank angle

which in turn increases the air and fuel mixing, accelerates the combustion, and releases the maximum HRR [[24](#page-8-0), [25\]](#page-8-0). This result matches to the many kinds of literature [\[22](#page-8-0)–[26\]](#page-8-0) that reported similar HRR variation of diesel engine powered by biodiesel, diesel, and water mixture Fig. [9.](#page-6-0)

4 Conclusion

The ignition characteristics of orange peel oil biodiesel (BD100) and various concentration of water (4 and 8% vol) added in orange peel oil biodiesel are detailed in the present study and the results are compared with diesel. The study conclusions are as follows:

- The dosage of ppm of 4 and 8% vol of water improves the combustion attributes such as heat release rate and cylinder pressure owing to the improved thermal conductivity of orange peel oil biodiesel.
- & Brake Thermal Efficiency for orange peel oil biodiesel is lesser than Diesel at all conditions. However, the water inclusion with BD100 effectively enhanced the efficiency. In addition, the BSFC values of BD100 reduced with the inclusion of water particles across the entire engine operating conditions
- The BD100 releases more amount of NO_x emission compared to diesel. However, the water inclusion with BD100 significantly reduces the NO_x emission of BD100 fuelled diesel engine.
- The BD100 releases considerably less amount of HC, Smoke and CO emission than diesel. Additionally, the water inclusion with BD100 further reduces the HC, Smoke and CO emission of BD100 fuelled diesel engine.

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