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A novel installation parameter optimization design method of forming tool for screw rotor

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Abstract

Characterized by a complex contour profle involving arc, cycloid, and involute, the screw rotor is usually manufactured by a forming tool. The finished surface quality and efficiency of the screw rotor are determined by the cutting performance of the forming tool. However, the machining performance of the forming tool is closely related to the structure shape of cutting edge, which is then determined by the installation parameters of the forming tool. Therefore, to make the cutting performance of forming tool controllable, it is essential to investigate the relationship between the cutting performance of the forming tool and its installation parameters at the design stage. In this paper, a novel installation parameter optimization design method of forming tool for screw rotor is presented. A parametric optimization program is designed to fnalize the range of installation parameters satisfying the spatial meshing relation and machining equipment parameters. The profle characteristics of forming tool under diferent center distances and mounting angles have been investigated. For validation, several screw rotors were ground in experiments and the resulted profle errors were analyzed. The results show that the cost of precision grinding of screw rotor can be signifcantly reduced, without compromise of machining quality. As such, the proposed design method could serve as a promising platform to facilitate screw rotor manufacture.

Keywords Screw rotor · Forming tool · Cutting performance · Design method

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1 Introduction

The screw rotor is the core component of screw machinery (e.g., screw pump/compressor/vacuum pump/expander), and its manufacturing accuracy has a direct infuence on the comprehensive performance of screw machinery [\[1](#page-14-0)]. Undesired profle errors will lead to a wide series of degraded performance including poor sealing, excessive noise and vibration, and reduced wear resistance. Therefore, in real production, the tooth profle accuracy of the screw rotor is required to be extremely high. As such, forming processing has become the most commonly used processing method in screw rotor manufacture, due to its controllable processing accuracy and convenience in profle modifcation. The mounting parameters of the forming tool can directly determine the attitude of the spatial contact line between the forming tool and the screw rotor, thus further afecting the cutting performance of the forming tool. Therefore, to obtain reasonable mounting parameters becomes the prerequisite in the design of forming tool. Unfortunately, in traditional screw rotor forming tool design, the mounting parameters are usually obtained by empirical methods. Such methods

often lead to poor cutting performance of the forming tool, which will further affect the machining accuracy and efficiency in screw rotor manufacture. In order to achieve robust manufacture of screw rotor, it is essential to optimize the cutting performance of the forming tool by selecting appropriate mounting parameters.

The design principle and method of forming tools for screw rotors have been described in detail in previous literatures [[2,](#page-14-1) [3](#page-15-0)]. These studies have provided the solution method and the solution step for the screw rotor forming tool profle, and valuable references for the design of screw rotor forming tools are provided. Furthermore, some scholars proposed diferent screw rotor profle forming tool design methods and realized screw rotor forming on various machine tools. Typically, Tang et al. [\[4](#page-15-1)] proposed a new method of screw rotor forming tool design, which managed to overcome the technical difficulty incurred by discontinuous one-medium derivative of screw rotor profle curve. Wu et al. [\[5\]](#page-15-2) proposed a radial ray method to design the screw rotor forming grinding wheel. This method frstly solved the forming wheel profile from the given screw rotor profile and then fgured out the screw rotor profle from the forming wheel profle, thus realizing the screw rotor profle precision forming grinding simulation. Li [[6\]](#page-15-3) proposed a novel calculation process based on the end milling cutter spiral groove machining principle to compute the grinding wheel profle with a known groove model and grinding wheel axis setting parameters. Hoang et al. [[7](#page-15-4)] established a general mathematical model for internal-meshing honing for screw rotors. Furthermore, the proposed mathematical model has been verifed to hone screw rotors with constant lead and variable lead. Bizzarri and Bartoň [[8\]](#page-15-5) proposed a method for machining screw rotors with double-fank milling on the fve-axis machine tool and verifed the feasibility of this method on several existing screw rotor profles. While these studies have provided valuable information and useful reference on forming tool design methods, they have not fully addressed the infuence of forming tool mounting parameters on the screw rotor profle errors.

In contrast, the infuence of screw rotor forming tool mounting parameters on screw rotor machining profle errors has been revealed in another group of studies. Stosic [[9](#page-15-6)], for example, proposed a method for calculating the wear amount of a screw rotor forming milling tool and obtained the infuence of the tool wear on the profle error of the screw rotor through the method of transforming the space coordinate system. In this study, considering the tool and the screw rotor have diferent relative motion speeds at different contact points, the cutting process would result in different tool wear levels. In a subsequent study, Stosic [[10\]](#page-15-7) illustrated the infuence of installation angle, axial deviation, and center distance on the profle error of the screw rotor. Similarly, Tao et al. [[11\]](#page-15-8) proposed a method to evaluate the screw rotor profle error caused by the installation parameter error of forming grinding wheel, including the installation angle error, center distance error, and axial displacement error. Both single factor and coupling factors were comprehensively investigated in this method. Furthermore, Zhao et al. [[12\]](#page-15-9), who investigated the infuence of multiple factors on screw rotor profle error in CNC precision forming grinding of screw rotor, proposed a new CNC grinding wheel segmentation dressing method to improve screw rotor grinding precision and efficiency. In addition, Hoang and Wu [[13\]](#page-15-10) established a general coordinate system for simulating the machining of single-thread screw rotor with end milling cutter on a multi-axis CNC milling machine. The machining accuracy of the rotor can be improved by using diferent combinations of end mill tool installation parameters or tool profle corrections. These studies were verifed either by machining experiments or simulation experiments.

The aforementioned research is mainly focusing on the mechanism of the screw rotor profle error; meanwhile, other researchers have also proposed the compensation method to reduce the error [[14–](#page-15-11)[16\]](#page-15-12). Representatively, on the basis of studying the generation of profle error of the screw rotor, Liu et al. [[17\]](#page-15-13) proposed a profle error compensation method to address the wear of the forming wheel. This method was realized by adjusting the installation parameters of the forming wheel, which improved the precision forming grinding efficiency. These researchers have attempted to improve the machining accuracy of the screw rotor by controlling the mounting parameters of the forming tool. On the one hand, the reported models are helpful to improve the profle accuracy of the screw rotor to some extent. On the other hand, these models are established based on a common assumption that the forming tool mounting parameters and profle have been pre-determined, which is often not the case in real production. Furthermore, the design method of mounting parameters of the screw rotor forming tool has not been well described in these studies. In order to overcome the problem of interference between tool and rotor profle during machining of concave rotor profle, Zhang and Fong [[18\]](#page-15-14) proposed a novel tilt form grinding (TFG) method. With a proposed mounting parameter setting method, the undercutting and secondary enveloping can be efectively avoided in grinding the screw rotor of a vacuum pump with concave profle. Subsequently, Deng and Shu [\[19\]](#page-15-15) used space envelopment theory to design the profle of forming tool and introduced the design method of mounting parameters. However, such methods can only ensure no interference between the forming tool and the screw rotor. There is still a lack in research to correlate the setting parameters with the cutting performance of the forming tool. Practical experience shows that reasonable installation parameters can not only reduce the manufacturing cost of the forming tool but also improve the cutting performance of the forming tool, thereby improving

Accordingly, this paper proposes a method to design mounting parameters of screw rotor forming tools based on their machinability, in order to guarantee that the designed forming tool have excellent cutting performance. First, the design method of forming tool and the associated forming grinding method are introduced, which provide a theoretical basis for subsequent research. Subsequently, a parametric optimization design program has been designed to solve the forming tool installation parameters satisfying the spatial meshing principle, and the cutting performance of the forming tool with various installation parameters has been analyzed to determine the range of reasonable mounting parameters. Finally, the screw rotor forming grinding experiments have been performed with the proposed approach, and the experimental results are in good agreement with the theoretical results, clearly demonstrating the efectiveness of the proposed screw rotor forming tool installation parameter design method. The main contribution of this paper is to provide theoretical support for the selection of screw rotor forming tool installation parameters, so as to avoid downgrade of machining accuracy, efficiency or increase of machining cost caused by empirical methods, or trial cutting method.

2 Theoretical background

2.1 The coordinate system relationship

It is a prerequisite for accurate design of screw rotor forming tool profle to fgure out installation parameter range

Fig. 1 Forming tool section diagram

satisfying the spatial meshing relationship and forming process requirements. To achieve this, the analytical relationship between the coordinate systems of screw rotor and forming tool needs to be established first. With (x_t, y_t) denoting a point in the shaft section of the screw rotor, the three-dimensional (3D) model representation can be described as:

$$
[X, Y, Z] = [xt \cos \theta - yt \sin \theta, xt \sin \theta + yt \cos \theta, p\theta]
$$
(1)

where *p* is the screw parameter of the screw rotor determined by $p = S/2\pi$, with *S* representing the lead of the screw rotor.

Similarly, with (Z_c, R_t) denoting a point in the crosssection of the forming tool shaft, as shown in Fig. [1,](#page-2-0) the 3D structure of the forming tool can be described by the following equation [[3\]](#page-15-0):

$$
[X_c, Y_c, Z_c] = [R_t \cos \phi, R_t \sin \phi, f(R_t)]
$$
\n(2)

where X_c, Y_c , and Z_c are the 3D representation of the forming tool; R_t is the radius when the width of the forming tool is Z_c ; ϕ is the angle between R_t and the plane $X_c O_c Z_c$; and the positive direction is defined as from X_c to Y_c .

During the forming process, the screw rotor profle is generated through space meshing motion between the forming tool and the screw rotor. The geometric relationship between a screw rotor and a forming tool is illustrated in Fig. [2.](#page-3-0) The rotating shafts of the screw rotor and the forming tool are spatially crossed, generating a setting angle ω . *T* is called the center distance, representing the distance between the forming tool axis and the screw rotor axis. *M* is a point on the space contact line between the screw rotor and the forming tool. *O* − *XYZ* is the space coordinate system of the screw

Fig. 2 Spatial position relationship between forming tool and screw rotor

rotor, while $O - X_c Y_c Z_c$ is the space coordinate system of the forming tool.

The mutual transformation relationship between the space coordinate system $O - X_c Y_c Z_c$ of the forming tool and the space coordinate system *O* − *XYZ* of the screw rotor is as follows:

$$
[X, Y, Z] = [X_c \cos\omega + Z_c \sin\omega, Y_c + T, Z_c \cos\omega - X_c \sin\omega]
$$
\n(3)

$$
\left[\vec{i}, \vec{j}, \vec{k}\right] = \left[\cos\omega \vec{i_c} + \sin\omega \vec{k_c}, \vec{j_c}, \cos\omega \vec{k_c} - \sin\omega \vec{i_c}\right] \tag{4}
$$

$$
[X_c, Y_c, Z_c] = [X\cos\omega - Z\sin\omega, Y - T, X\sin\omega + Z\cos\omega]
$$
 (5)

$$
\left[\vec{i_c}, \vec{j_c}, \vec{k_c}\right] = \left[cos\omega\vec{i} - sin\omega\vec{k}, \vec{j}, \cos\omega\vec{k} + sin\omega\vec{i}\right]
$$
 (6)

where $\vec{i_c}$, $\vec{j_c}$, and $\vec{k_c}$ are the unit vectors in X_c , Y_c , and Z_c directions; \vec{i} , \vec{j} , and \vec{k} are the unit vectors in *X*, *Y*, and *Z* directions.

2.2 Forming tool profile computation model

When the 3D model of the screw rotor is given, the profle equation of the forming tool can be obtained through a series of mathematical operations. The spatial contact line equation between the forming tool and the screw rotor can be expressed by the following equation [[3](#page-15-0)]:

$$
(\vec{k}_c \times \vec{R}) \cdot \vec{n} = 0 \tag{7}
$$

where $\vec{R} = \vec{O}$ \vec{M} is the radial vector of point *M* in spatial coordinate system $O - X_c Y_c Z_c$ and \vec{n} is the normal vector at point *M* in spatial coordinate system *O* − *XYZ*. The normal vector \vec{n} in spatial coordinate system $O - XYZ$ can be solved from the following equation:

$$
\vec{n} = \frac{\partial \vec{r}}{\partial t} \times \frac{\partial \vec{r}}{\partial \theta} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ \frac{\partial x}{\partial t} & \frac{\partial y}{\partial t} & \frac{\partial z}{\partial t} \\ \frac{\partial x}{\partial \theta} & \frac{\partial y}{\partial \theta} & \frac{\partial z}{\partial \theta} \end{vmatrix}
$$
(8)

By taking the partial derivative of each component in Eq. [\(1\)](#page-2-1) with regard to t and θ , the following equation can be established:

$$
\begin{cases}\n\frac{\partial X}{\partial t} = \frac{dx_t}{dt} \cos\theta - \frac{dy_t}{dt} \sin\theta \\
\frac{\partial Y}{\partial t} = \frac{dx_t}{dt} \sin\theta + \frac{dy_t}{dt} \cos\theta \\
\frac{\partial Z}{\partial t} = 0\n\end{cases}
$$
\n(9)

$$
\begin{cases}\n\frac{\partial X}{\partial \theta} = -x_t \sin \theta - y_t \cos \theta \\
\frac{\partial Y}{\partial \theta} = x_t \cos \theta - y_t \sin \theta \\
\frac{\partial Z}{\partial \theta} = p\n\end{cases}
$$
\n(10)

By substituting Eqs. (9) (9) and (10) (10) into Eq. (8) (8) , the components of the normal vector \vec{n} can be solved:

$$
\begin{cases}\nn_x = p\left(\frac{dx_i}{dt}\sin\theta + \frac{dy_i}{dt}\cos\theta\right) \\
n_y = -p\left(\frac{dx_i}{dt}\cos\theta - \frac{dy_i}{dt}\sin\theta\right) \\
n_z = \left(\frac{dx_i}{dt}\cos\theta - \frac{dy_i}{dt}\sin\theta\right)(x_i\cos\theta - y_i\sin\theta) - (-x_i\sin\theta - y_i\cos\theta)\left(\frac{dx_i}{dt}\sin\theta + \frac{dy_i}{dt}\cos\theta\right)\n\end{cases} (11)
$$

Substituting Eqs. (5) (5) , (6) (6) and (11) (11) into Eq. (7) (7) yields:

$$
\begin{aligned}\n\left[(x_t \cos\theta - y_t \sin\theta)(\cos\theta - K \sin\theta) + (x_t \sin\theta + y_t \cos\theta)(\sin\theta + K \cos\theta) \right] \\
\left[Y - T - p \cot\omega \right] + \left[p(\cos\theta - K \sin\theta) \right] \cdot p\theta \\
+ \left[p(\sin\theta + K \cos\theta) \right] \cdot T \cot\omega = 0\n\end{aligned}
$$
\n(12)

where *K* is the first derivative of y_t with regard to x_t . As can be seen, angle θ is the only unknown variable, and it can be solved when T , ω , and the profile equation of the screw rotor are given. Thus, the spatial contact line can be deduced, and the profle equation of the forming tool can be solved.

2.3 Rotor profile computation model

Similarly, when the 3D model of the forming tool is given, the profile equation of the screw rotor can be obtained through a series of mathematical operations. The spatial contact line equation between the screw rotor and the forming tool can be expressed by the following equation [\[3](#page-15-0)]:

$$
(\vec{k} \times \vec{r} + p\vec{k}) \cdot \vec{n} = 0 \tag{13}
$$

where $\vec{r} = \vec{OM}$ is the radial vector of point *M* in spatial coordinate system $O - XYZ$. In the forming tool spatial coordinate system $O − X_cY_cZ_c$, the vector \vec{r} can be expressed in the following equation:

$$
\vec{r} = \vec{R} + T\vec{j_c} = R_t \cos\phi \vec{i_c} + R_t \sin\phi \vec{j_c} + f(R_t) \vec{k_c} + T\vec{j_c}
$$
 (14)

By introducing Eq. (6) (6) into Eq. (14) (14) , in the spatial coordinate system $O - XYZ$ of screw rotor, the vector \vec{r} can be represented using the following equation:

$$
\vec{r} = [R_t \cos\phi \cos\omega + f(R_t) \sin\omega]\vec{i} + (R_t \sin\phi + T)\vec{j} + [f(R_t) \cos\omega - R_t \cos\phi \sin\omega]\vec{k}
$$
\n(15)

The term $\vec{k} \times \vec{r} + p\vec{k}$ can be deduced as follows:

$$
\vec{k} \times \vec{r} + p\vec{k} = \left[R_t \cos\phi \cos\omega + f(R_t) \sin\omega \right] \vec{j} - (R_t \sin\phi + T)\vec{i} + p\vec{k}
$$
\n(16)

The normal vector \vec{n} can be deduced according to the following equation:

$$
\vec{n} = \frac{\partial \vec{r}}{\partial R_t} \times \frac{\partial \vec{r}}{\partial \phi} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ \frac{\partial X}{\partial R_t} & \frac{\partial Y}{\partial R_t} & \frac{\partial Z}{\partial R_t} \\ \frac{\partial X}{\partial \phi} & \frac{\partial Y}{\partial \phi} & \frac{\partial Z}{\partial \phi} \end{vmatrix}
$$
(17)

By taking the partial derivative of each item in Eq. ([3\)](#page-3-8) with regard to R_t and Z_c , the following equations can be established:

 \sim *X*

$$
\begin{cases}\n\frac{\partial X}{\partial R_i} = \cos\phi\cos\omega + f'(R_t)\sin\omega \\
\frac{\partial Y}{\partial R_i} = \sin\phi \\
\frac{\partial Z}{\partial R_i} = f'(R_t)\cos\omega - \cos\phi\sin\omega\n\end{cases}
$$
\n(18)

$$
\begin{cases}\n\frac{\partial X}{\partial \phi} = -R_t \sin \phi \cos \omega \\
\frac{\partial Y}{\partial \phi} = R_t \cos \phi \\
\frac{\partial Z}{\partial \phi} = R_t \sin \phi \sin \omega\n\end{cases}
$$
\n(19)

By substituting Eqs. (18) (18) and (19) (19) into Eq. (17) (17) , the components of the normal vector \vec{n} can be solved:

$$
\begin{cases}\nn_x = R_t \sin\omega - f'(R_t) \cos\omega R_t \cos\phi \\
n_y = -f'(R_t) R_t \sin\phi \\
n_z = R_t \cos\omega + f'(R_t) \sin\omega R_t \cos\phi\n\end{cases}
$$
\n(20)

Therefore, the following relationship can be obtained by substituting Eqs. (16) (16) and (20) into Eq. (13) (13) :

$$
(T + p \tan \omega) \cos \phi - [f(R_t) + \frac{1}{f'(R_t)} R_t] \tan \omega \sin \phi + \frac{1}{f'(R_t)} (\rho - T \tan \omega) = 0 \tag{21}
$$

where $f'(R_t)$ is the first derivative of Z_c with respect to R_t . Since each R_t corresponds to two different values of Z_c , the R_t value can be considered as a function of Z_c , as shown in Fig. [1](#page-2-0). The $\text{term } 1/f'(R_t)$ is deduced by the derivative rule for inverses:

$$
\frac{1}{f'(R_t)} = [f^{-1}(Z_c)]'
$$
\n(22)

Equation (21) (21) shows that angle ϕ is the only unknown variable; angle ϕ can be deduced according to T , ω , and the profle equation of the forming tool. Thus, the section profle of the screw rotor can be fgured out.

2.4 Profile error calculation method

The defnition of screw rotor profle error is the basis of studying the infuence of mounting parameters on screw rotor profle error. The axial section of screw rotor profle is shown in Fig. [3](#page-5-0). By comparing the diference between the machined profle and the theoretical profle, the machined profle error can be obtained, as shown in Fig. [4.](#page-5-1) The coordinates of point C_i on the machined profile are denoted as (x_i, y_i) , where $i = \{1, 2, ..., m\}$ with *m* representing the number of points determined by the geometric size of the screw rotor and the measuring accuracy of the measuring equipment. In this paper, the profile error, E_i , is defined as the minimum distance between the point C_i and the initial theoretical profile. The profile error E_i is negative when point C_i is inside the theoretical profle, and vice versa.

In order to facilitate the calculation of the profle error, the theoretical profle of the rotor can be ftted by cubic parameter splines as follows:

$$
y = S(x) \tag{23}
$$

Therefore, E_i can be obtained by the following equation:

$$
E_i = \pm \sqrt{(x_i - x_k)^2 + (y_i - S(x_k))^2}
$$
 (24)

where points $(x_k, S(x_k))$ satisfy Eq. ([23\)](#page-4-8). If $(x_i^2 + y_i^2)$ $(x_k^2 + S(x_k)^2)$, the machined profile is outside the theoretical profle and ''+'' is chosen; otherwise, ''-'' is chosen. Using this method, the profle error of any point on the machined profle can be accurately obtained.

3 Mounting parameter optimization decision method

In the screw rotor precision forming process, installation parameters determine both the precision of forming tool profile and the machining performance. In view of the

Fig. 3 Theoretical profle of screw rotor

importance of mounting parameters to screw rotor forming tool, it is worthy to investigate the selection strategy of mounting parameters. In Sect. [2,](#page-2-2) the mutual transformation

Fig. 4 Schematic diagram of profle error

relation between screw rotor and forming tool and the calculation method of screw rotor profle error have been explained in detail. Based on such knowledge, two sequential tasks are proposed in this section to optimize the design of installation parameters. Firstly, the range of installation parameters of the forming grinding wheel satisfying the spatial meshing characteristics is fgured out. Subsequently, the evaluation criterion of cutting performance of forming grinding wheel is established, and a decent range of installation parameters is determined according to the actual processing requirements. Thus, a feasible strategy is provided for the precise design and selection of installation parameters of screw rotor forming tool. In this paper, the screw rotor precision forming grinding was investigated to demonstrate the efectiveness of the proposed approach.

3.1 Mounting angle range optimization decision method

In order to solve the mounting angle range satisfying the machining requirements, the mounting angle optimization design program is illustrated in Fig. [5](#page-6-0). During the actual grinding process of the screw rotor, the forming wheel radius becomes smaller and smaller with the increase of grinding time. In order to make full use of the abrasive on the forming wheel, the minimum installation center distance satisfying the process requirements is taken as the initial center distance parameter. Taking a certain type of male rotor for example, its geometric parameters are shown in the Table [1.](#page-6-1)

The steps of the program for optimization of grinding wheel installation angle of screw rotor are described as follows:

- Step 1. Input initial parameter, including discrete data points of screw rotor section profle, pitch circle lead angle, screw parameter, center distance, and cycle-index.
- Step 2. Set initial center distance and mounting angle.
- Step 3. Generate forming tool profile.
- Step 4. Generate screw rotor simulation profile.
- Step 5. Go to the next step if cycle-index *n* is no larger than 3; otherwise, return to the frst step.
- Step 6. Evaluate the profile error of simulation profile of screw rotor. When the maximum value of the profle error \leq threshold Δ , the current mounting angle value would be recorded; otherwise, return to step 2.

In the initial parameter setting, the range of the installation angle is limited to [λ -5, λ +5], and the judgment condition is set as $E_{i\text{max}} \leq \Delta$. Through three cycles of optimization calculations, the precise mounting angle range satisfying the manufacturing conditions can be obtained. In order to guarantee the machining accuracy of the screw rotor, a higher

Fig. 5 Flowchart of installation angle optimization design

precision screw rotor forming tool is required. Therefore, in this paper, the threshold Δ is set to be a higher order of precision than the screw rotor profle. For example, when

Table 1 Geometrical parameters of the female rotor

Items	Value
Tip diameter (mm)	54.94–54.95
Root diameter (mm)	33.00 - 33.01
Lead (mm)	108.00
Pitch diameter (mm)	33.00
Lead angle $(°)$	46.171
Profile tolerance (mm)	$+0.01$
Screw direction	Right-handed

the screw rotor profile tolerance is \pm 0.01 mm, the threshold should be defned as 0.001 mm.

3.2 Center distance range optimization decision method

Similarly, in order to solve the center distance range that meets the process conditions, the center distance optimization design program is designed as shown in Fig. [6](#page-7-0). In the actual screw rotor grinding process, the installation center distance is usually determined by the structure of the grinding machine and the size of the forming grinding wheel. Therefore, center distance should be searched within the interval $[T_{\min}, T_{\max}]$, as it satisfies the structural parameters of machine tool and cutting tool.

The steps of the program for optimization of grinding wheel installation center distance of screw rotor are as follows:

- Step 1. Input initial parameter, including discrete data points of screw rotor section profle, pitch circle lead angle, center distance, and screw parameter.
- Step 2. Set initial center distance and mounting angle.
- Step 3. Generate forming tool profile.
- Step 4. Generate screw rotor simulation profile.
- Step 5. Evaluate the profile error of simulation profile of screw rotor. When the maximum value of the profle error \leq threshold Δ , record the current mounting angle; otherwise, prompt an error and end the program.

The mounting angle and center distance of the forming tool before the screw rotor forming process can be optimized by the method mentioned in this paper. The mounting parameters of the forming tool of the female screw rotor mentioned in Table [1](#page-6-1) are shown in Fig. [7](#page-7-1).

3.3 Profile characteristic analysis of forming tool

As shown in Fig. [8](#page-7-2), as long as the rotary surface of the forming tool moves relative to the screw surface of the screw rotor, there is always a tangent contact line between the two surfaces, which is the most essential feature of forming grinding. In the process of screw rotor form grinding, contact line can be considered as the actual grinding blade. The forming tool only keeps rotating at high speed around its own axis without any translation. Meanwhile, the screw rotor performs linear and rotary composite motion, equivalent to rotary cutting by the grinding wheel along the screw groove of the screw rotor, so as to remove excessive workpiece material.

The forming tool profle is the projection of the contact line on the cross section of the grinding wheel shaft. As such, the shape and spatial pose of the contact line determine

Fig. 6 Flowchart of center distance optimization design

not only the spatial contact relationship between the grinding wheel and the screw rotor but also the profle of the grinding wheel. Therefore, the precision grinding quality

Fig. 7 Mounting parameters range of forming tool

Fig. 8 The shape and pose of space contact line

of screw rotor is closely related to the shape and position of the spatial contact line.

In order to obtain the mounting parameters that match the processing requirements, it is necessary to carry out further in-depth research on the profle characteristics of the forming tool under diferent mounting parameters. Furthermore, to evaluate the cutting performance of the forming tool under diferent mounting parameters, the cutting performance evaluation system of the forming tool should be established frst, as shown in Fig. [9](#page-8-0).

Profle characteristics of the forming tool under diferent installation parameters are visualized in Fig. [10.](#page-9-0) The following information can be derived from Fig. [10.](#page-9-0) When the center distance between the screw rotor and the forming tool remains constant, the maximum profle curvature of the forming tool decreases slightly with the increase of the mounting angle of the grinding wheel, and the maximum curvature appears at the transition place between the outer circle and the side of the grinding wheel. In contrast, when the mounting angle between the forming tool and the screw rotor remains stable, the profle curvature of the forming tool remains stable even if the center distance between the screw rotor and the formed grinding wheel changes. According to practical experience, the larger the curvature of forming grinding profle is, the faster the grinding wheel will wear. In addition, from the perspective of manufacturing, a smaller curvature of shaped wheel profle usually indicates a lower manufacturing cost.

The wear rate k , expressed in as Eq. (25) (25) , is defined as the ratio of the grinding wheel wear volume to the material volume removed from the workpiece [\[20](#page-15-16)].

$$
k = \frac{V_w}{V_s} = \frac{2\pi R_w S_w}{L_s S_s} \tag{25}
$$

where R_w and L_s are, respectively, the radius of grinding wheel and the length of rotor helix at the contact point between forming tool and workpiece. S_w and S_s are the wear unit area of grinding wheel and workpiece at their contact point.

When the grinding parameters (grinding speed, cutting depth, cooling conditions) remain unchanged, k , S_s , and L_s remain constant. Therefore, when the grinding wheel radius R_w is smaller, the grinding wheel wear unit area S_w is larger. As a result, the smaller the grinding wheel radius is, the faster the grinding wheel wear will be. In addition, the grinding wheel needs to be dressed more frequently, leading to a downgrade of grinding efficiency.

On the other hand, when the center distance remains unchanged, the grinding width of the grinding wheel decreases as the mounting angle increases. As can be seen from Fig. [10,](#page-9-0) when the mounting angle is 43.171°, the maximum grinding width of the grinding wheel is 16.1 mm. However, when the mounting angle is 46.171°, the maximum grinding width of the grinding wheel is 13.9 mm, and the corresponding abrasive of the grinding wheel is saved by more than 13%, which in turn will help reduce the processing cost.

In summary, through the analysis of screw rotor profle characteristics under diferent mounting parameters, the following conclusions can be known. Diferent mounting parameters between the screw rotor and the grinding wheel correspond to diferent profle characteristics of the forming wheel and may lead to diferent grinding performance. Compared with the center distance, the mounting angle has a more signifcant efect on the profle characteristics of the forming tool. In order to make full use of the grinding wheel, the mounting angle should be determined in priority. Diferent mounting angles will lead to diferent widths of grinding wheel, so that the screw rotor manufacturing costs are also diferent. When the above factors are taken into consideration, a smaller mounting angle should be selected to achieve precision form processing of the screw rotor with a lower cost.

4 Experiment and results

4.1 Experimental setup

In order to verify the efectiveness of the proposed mounting parameter decision method of screw rotor forming grinding wheel, a set of experiments for screw rotor

Fig. 10 Profle characteristics of forming tool. **a** Center distance 126.50 mm. **b** Center distance 136.50 mm. **c** Center distance 146.50 mm. **d** Center distance 156.50 mm. **e** Center distance 166.50 mm. **f** Center distance 176.50 mm

Fig. 10 (continued)

forming grinding were designed. The experiment setup is shown in in Fig. [11,](#page-10-0) with equipments listed in Table [2](#page-10-1). The screw rotor material in the experiment is Y40Mn and the hardness is HB190-210, which is in accordance with the actual application. The screw rotor profle error is measured by the fully automatic CNC-controlled P26 precision measuring center as shown in Fig. [12,](#page-11-0) and its measurement accuracy is 1 μm. The mounting parameters of the forming grinding wheel in the experiment are described in Table [3.](#page-11-1) To minimize the effects of grinding wheel wear

(1) Worktable. (2) Diamond dressing wheel. (3) Grinding wheel. (4) Screw rotor. (5) Auxiliary supporting

Fig. 11 Grinding experimental setup. 1 Worktable. 2 Diamond dressing wheel. 3 Grinding wheel. 4 Screw rotor. 5 Auxiliary supporting

on the accuracy of experimental results, the grinding wheel was modifed before each grinding experiment. The profle error of the screw rotor was measured in the middle position.

4.2 Results and discussion

In the experiment, the proposed optimization design program in Sects. [3.1](#page-5-2) and [3.2](#page-6-2) is applied to determine the screw rotor forming tool installation parameters range. Subsequently, Eq. ([25\)](#page-8-1) is implemented to determine the wear resistance of formed grinding wheel. The results of the six experiments are shown in the "Appendix Fig. [13](#page-12-0)," where the actual profle measured from experiment and the theoretical profle are compared. Compared with the theoretical profle, the error of most positions of each helical groove are within the tolerance band \pm 0.01 mm. This means that the design method of mounting angle and center distance in this paper is accurate and reliable. In addition, the experimental results show that the error distribution trend of screw rotor is consistent with the

Table 2 Experimental equipment

No.	Setup	Brand	Model
	Machine tool	Star SU	G 500 H Profile Grinding Machine
$\mathbf{2}$	Grinding wheel	NORTON	3NO60-H12VSP
3	Cutting fluid	Variocut	G600HC
4	Measuring equipment	Klingelnberg	P ₂₆

1. Printing mechanism 2. Clamping device 3. Screw rotor 4. Measuring head 5. Display instrument

Fig. 12 The fully automatic CNC-controlled P 26 precision measuring center. 1 Printing mechanism 2 Clamping device. 3 Screw rotor. 4 Measuring head. 5 Display instrument

curvature distribution trend of forming grinding wheel. The largest error of profile appears at the transition between arc segment (root) and cycloid segment (side), which is also the position with the largest curvature on the forming grinding wheel. It is also observed that as the radius of grinding wheel becomes smaller, the actual machining profle of screw rotor tends to be larger. The reason behind this is fewer abrasive particles will distribute on the surface as the radius of the grinding wheel gets smaller, thus leading to faster wear of abrasive particles. In conclusion, the experimental results show that the obtained range of installation parameters can achieve desired grinding accuracy of the screw rotor profle, which shows the efectiveness of the proposed method.

Table 3 Mounting parameters

5 Conclusion

A novel design method for the mounting parameters of screw rotor forming grinding wheel considering the cutting performance of forming grinding wheel was proposed in this paper. An optimum design program of mounting angle and center distance has been developed. Based on the established optimum design program, the range of mounting angle and center distance satisfying the spatial meshing relationship was solved. Furthermore, the profle characteristics (cutting performance) of the grinding wheel under diferent mounting parameters have been investigated. The numerical cases showed that the mounting angle had a signifcant efect on the profle characteristics (cutting performance) of the forming grinding wheel. In contrast, the center distance had little efect on the profle characteristics (cutting performance) of the forming wheel. Nonetheless, when the center distance becomes smaller, the grinding wheel radius becomes smaller, and the grinding wheel wear becomes faster. Grinding experiments for male rotor with diferent mounting parameters were performed to validate the results of the numerical cases. Some important conclusions are drawn as follows:

- 1. A novel model for calculating the mounting parameters of screw rotor forming grinding wheel has been established based on the spatial engagement principle. The range of mounting parameters satisfying the meshing principle and grinding equipment process parameters were obtained.
- 2. The profile characteristics (cutting performance) of forming grinding wheel under different mounting parameters were investigated. The relationship between the curvature, width of the shaft section profle of the grinding wheel, and the mounting parameters of forming grinding wheel were clarifed. Compared with center distance, the mounting angle which has a more signifcant infuence on the profle characteristics of the forming grinding wheel should be controlled in priority.
- 3. When optimizing grinding efficiency, a smaller mounting angle should be chosen because a larger grinding wheel width can improve the stability of the grinding system. When optimizing grinding quality and economy, a larger mounting angle should be selected, because a larger mounting angle can reduce the curvature of the grinding wheel profle, while reducing the width of the grinding wheel to save the cost of the grinding wheel.

The above conclusions show that the mounting parameter design method is accurate and reliable for screw rotor forming grinding. In this paper, a method based on the cutting

extendibility, which can be applied to the forming of spiral

To promote the proposed approach in actual production, future work is planned to visualize the proposed approach with augmented reality and virtual reality technology to indicate quantitative analytics with consideration of actual/dynamic manufacturing requirements/constraints. Thus, the usability and generalizability of the method can be further improved.

groove parts such as worm, helical gear, etc.

performance of the tool is proposed for the design of tool mounting parameters in screw rotor forming machining, which provides valuable theoretical basis for the selection of installation parameters and avoids the unreliability of empirical method. The proposed optimization design method can be easily integrated into the forming machine system to improve its machining quality and efficiency. In addition, the research results of this paper have good universality and

Appendix

a) Center distance 176.500mm, mounting angle 43.171

b) Center distance 176.500mm, mounting angle 44.171°

Fig. 13 The results of the experiments. **a** Center distance 176.500 mm, mounting angle 43.171°. **b** Center distance 176.500 mm, mounting angle 44.171°. **c** Center distance 176.500 mm, mounting angle 45.171°. **d** Center distance 176.500 mm, mounting angle 46.171°. **e** Center distance 136.500 mm, mounting angle 46.171°. **f** Center distance 156.500 mm, mounting angle 46.171°

c) Center distance 176.500mm, mounting angle 45.171°

d) Center distance 176.500mm, mounting angle 46.171°

Fig. 13 (continued)

d) Center distance 136.500mm, mounting angle 46.171°

Fig. 13 (continued)

 \overline{Date}

 35.0 200

 25.0

 20.0

 15.0

 10.0

 5.0

 0.0 -5.0

 -10.0

 -15.00

Pre-depth

Form

Author contribution All authors contributed to the study conception and design. Zongmin Liu performed the data analyses and wrote the manuscript. Jirui Wang performed the experiment. Ning Liu contributed signifcantly to analysis and manuscript preparation. Qian Tang and Bin Xing helped perform the analysis with constructive discussions. All authors read and approved the fnal manuscript.

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Availability of data and material The authors confrm that the data supporting the fndings of this study are available within the article.

f) Center distance 156.500mm, mounting angle 46.171°

Declarations

Ethics approval Not applicable.

Consent to participate Not applicable.

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