# ORIGINAL ARTICLE

# Thermal error modeling and compensation for a high-speed motorized spindle

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Abstract To improve the precision of CNC machine tools, a motorized spindle thermal error model based on least square support vector machine (LS-SVM) was proposed. A thermal error compensation method was implemented, which takes the length of cutting tools and thermal tilt angles into account. A five-point method was applied to measure radial thermal declinations and axial expansion of the spindle with eddycurrent sensors. This resolves a problem arising out of the three-point thermal error measurement, where the radial thermal-induced angle errors cannot be obtained. Variables sensitive to thermal error were selected by grouping and optimizing temperature variables using a combined fuzzy cluster and correlation analysis. LS-SVM models were established for axial elongation and radial thermal yaw and pitch angle errors. Moreover, a method to test the goodness of prediction for the results based on the model is discussed. The results indicated that the LS-SVM has high predictive ability based on fuzzy cluster grouping, and prediction accuracy reached up to 90 %. In addition, the axial accuracy was improved by 82.6 % after error compensation, and the axial maximum error decreased from 39 to 8  $\mu$ m. Moreover, the X/Y direction accuracy can reach up to 77.4 and 86 %, respectively, which demonstrated that the proposed methodology of measurement, modeling, and compensation was effective.

Keywords Motorized spindle  $\cdot$  Thermal tilt angle  $\cdot$  LS-SVM  $\cdot$  Error compensation  $\cdot$  Fuzzy cluster

# **1** Introduction

Precision CNC jig boring machines are typically used for processing complex box-type components. Heat generated during the fabrication of these components gives rise to thermal errors. These thermal errors account for a larger proportion of the total error as the machine tools become more sophisticated. As a result, the accuracy of the tool decreases and dimension deviates from the initial design value. This is particularly an issue when the machine is used for longer time periods. One of the factors resulting in the decreased accuracy over time related to usage and machine age is inadequate maintenance of the tool. The reduced accuracy due to the thermal errors accounts for 70 % of the total errors arising from various error sources [1]. Research presented in literature related machine precision such as the one by Donmez et al. also points to temperature variations resulting in manufacturing errors, thereby reducing machine precision [2]. Nonuniform temperature distribution in CNC machine tools varies with time, becoming non-linear and non-stationary. Moreover, the motorized spindles have complex, dynamic, non-stationary, and speed-dependent thermal characteristics in comparison to conventional spindles [3].

In recent years, finite element method (FEM) has been used to analyze temperature distribution and thermal deformation for machine tools. Creighton et al. used FEM to analyze temperature distribution characteristics for a high-speed micro-milling spindle. An exponential model for the axial thermal error was constructed and correlated to spindle speed and run time [4]. Zhao et al. proposed a method to calculate the thermal conductivity coefficient for the spindle surface. Simulations were used to aid in the analysis of the temperature field variation along with thermal deformation of the spindle [5]. However, the tool error for the precision CNC machine tool is a mutually coupled problem with many complex factors, which are in turn governed by numerous variables. This

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makes it extremely difficult to establish a theoretical equation taking into account thermoelasticity and heat transfer. Yang et al. [6] used artificial neural networks (ANNS) to establish a relationship between temperature of the spindle and the resulting thermal errors. Ouafi et al. constructed an artificial neural network model for spindle thermal error with the temperature drawing on statistical distribution, which effectively improves the machining accuracy [7]. The grey neural network was proposed to predict the thermal error, and experiments on the axial thermal deformation of the spindle in a five-axis machining center are conducted to build and validate the proposed models [8]. Vissiere et al. measured thermal drifts for a spindle with a new method, which allows measurement accuracy with nanometer resolution [9]. Vyroubal et al. focused on compensation methods for the thermal deformation in spindle axis direction, which was based on decomposition analysis. Such a method was found to be low cost in actuality and an effective strategy to reduce the thermal error [10]. Hong et al. studied thermal characteristics of a rotary axis on a five-axis machine and analyzed the relation between thermal and motion error for the rotary axis [11]. Huang et al. proposed a combined thermal error model for the high-speed spindle in a machine tool and used the five-point method to measure the thermal drifts, and a genetic algorithm (GA) was introduced to optimize the BP network's initial weights and thresholds, solving the global minimum searching problem [12].

In addition, the temperature sensor selection for the thermal error modeling of the machine tool is important. The fuzzy cmeans clustering method and the ISODATA method are used to group the data of thermal sensors, which are effective for thermal sensor selection [13]. The direct criterion method and indirect grouping method based on the synthetic grey correlation theory were presented to optimize the selection of a minimum number of temperature sensors for thermal error compensation on a machine tool. After optimization, the number of thermal points reduced from 16 to 4 [14].

If thermal errors propagate through the measurement and modeling stage, they can be alleviated by employing compensation methods. Zhang et al. proposed one such compensation technique to improve machine precision, which is based on the use of the external machine zero point shift function. Ethernet data communication protocol was used for the machine tools [15]. Fu and Miao et al. built the spindle axial thermal error model using a multivariate linear regression method [16, 17]. The use of axial thermal error compensation method as a method for improving the machine precision has also been reported in literature by other research groups [18, 19]. Pajor et al. presented a method for supervising the feed screw thermal elongation. This method reduced ball screw thermal errors [20]. Wu et al. has successfully achieved realtime compensation for axial expansion on a vertical machine tool by using a multiple regression model [18]. Liu et al. compensated the thermal drift of milling and boring machines along the Z direction [21]. Ouafi et al. presented an integrated and comprehensive modeling approach for real-time thermal error compensation based on multiple temperature measurements. After the use of this compensation strategy, spindle errors were reduced from 19  $\mu$ m to less than 1  $\mu$ m [7]. Gebhardt et al. described a high-precision grey box model for compensating thermal errors for a five-axis machine. Using this method, the thermal errors for rotation/swiveling were reduced by a factor of 85 % [22]. Wang et al. proposed another prediction model for axial thermal deformation and applied the model to compensate the error for a CNC machine [23].

The methods reported in literature mainly discuss measurement and modeling methods for spindle axial thermal elongation. However, the existing methods fail to take into account errors resulting from the radial thermal angle error. Spindle thermal deformation for a CNC machine tool is usually expressed as deviation from the spatial position and gestures, i.e., the drift in geometry and spatial phase, which affects machining precision. Here, we consider the jig boring machine, particularly the thermal expansion of the spindle axial of the tool, which affects the geometry of the bore. In addition, errors due to the radial thermal angle could affect the geometry and surface roughness of the bore. Due to this, it is critical to measure both axial and radial thermal errors simultaneously. To realize error compensation, the error due to the spindle radial thermal angle must be translated to the linear coordinate axis. The compensated components of the thermal errors are closely related to spindle radial thermal inclination angle errors and handle length. Using a three-point method, the absolute thermal deformation along a radial direction can be measured and it does not reflect the variations in radial thermal deformation for the spindle. The thermal inclination angle errors cannot be obtained using this method and results in a thermal error compensation model with reduced accuracy.

To improve the accuracy and overcome the disadvantages associated with the three-point method, a five-point method is proposed. In comparison to other methods, the five-point method has the advantage of allowing simultaneous measurement of axial and radial thermal drifts for the motorized spindle system. As a result, the variation in the spindle position and orientation can be analyzed. This overcomes the challenge in measurement of inclined angles for the spindle radial thermal errors often associated with the three-point method. Moreover, an integrated thermal error model provides a more accurate mathematical model for the thermal error compensation, which includes spindle thermal elongation, radial thermal pitch angle error, and yaw angle errors based on the five-point method.

The support vector machine is a new machine learning theory, which has the advantage of using a simple and versatile algorithm. It depends on using training data with good generalization and global optimization characteristics, which can be used for parameters with non-linear relationships. Lin and Zhao et al. [24, 25] established a spindle thermal error model based on the least square support vector machine theory. This model was found to have perfect robustness. However, as in the previous cases, their model also ignored radial declination angle errors.

To the best of our knowledge, a method based on the least square support vector machine and fuzzy clustering to model thermal elongation and declination angles has not been presented in current literature. The proposed model can be used to predict axial and radial errors with high accuracy. Thermal drifts using this method were translated into coordinate offsets and were used to establish mathematical equations for final compensation along three directions, allowing improvement in the machine tool accuracy.

The current study focuses on a spindle system, which is used in a box-type precision CNC jig boring machine. Thermal balance experiments were performed using the five-point method. Least square support vector machine (LS-SVM) models are established for spindle axial thermal elongation and radial thermal declinations, using the fuzzy clustering regression analysis method to optimize the temperature variables. Subsequently, thermal error offset equations were derived and the compensation was carried out.

### 2 Thermal error modeling

2.1 Hierarchical clustering method to group temperature variables

A variety of fuzzy clustering analysis methods based on the fuzzy graph theory have been proposed in literature. Among them, the biggest tree method based on the fuzzy graph theory is the most popular [26]. The fuzzy c-means clustering method is applied to identify the temperatures, and the representative as an independent variable is selected; meanwhile, it eliminates coupling among the variables [27]. In this paper, temperature variables for 11 measuring points were grouped using fuzzy clustering. Statistical correlation was then applied to optimize the measurement points. The correlation coefficient between each variable temperature and thermal error were calculated. Finally, the measurement points with the largest value for the correlation coefficient in each group were taken as the typical temperature variables. The number of variables for temperature is small, so system cluster analysis was used. The variable packet flows are shown in Fig. 1.

Assume the temperature variable  $T = \{T_1, T_2, ..., T_m\}$  is the object carried out using fuzzy clustering analysis. Each object in *T* is denoted as  $T_k$  (k=1, 2, ..., m), whose characteristics are described by finite values. A corresponding vector  $P(T_k) = (-T_{k1}, T_{k2}, ..., T_{ks})$  is related to  $T_k$ .  $T_{kj}(j=1, 2, ..., s)$  is the *j*th

characteristics value of  $T_k$ .  $P(T_k)$  is the eigenvector for  $T_k$ . Fuzzy clustering analysis divides the sample T into c fuzzy subsets  $\tilde{T}_1$ ,  $\tilde{T}_2$ ,...,  $\tilde{T}_c$ , according to the similarity between the feature vectors.

Cluster analysis, also known as hierarchical clustering analysis, gradually clusters based on feature vector distance criteria. The classification moves from more to less, until it reaches the desired classification. The following are the general steps used for system clustering:

- 1. Initialize data, assuming that the sample set *T* contains *m* subsets  $T_1^{(0)}, T_2^{(0)}, \ldots, T_m^{(0)}$ , which form one class. Then, distance is calculated between each subset to obtain a  $m \times m$  dimensional distance matrix  $D^{(b)}$ .
- 2. Next, the smallest element in the distance matrix  $D^{(b)}$  (except diagonal elements) is determined. If the minimum element is the distance between  $T_i^{(b)}$  and  $T_j^{(b)}$ , then the two elements are merged into  $T_{ij}^{(b+1)}$ . Finally, a new classification  $T_1^{(b+1)}$ ,  $T_2^{(b+1)}$ ,...,  $T_{m-1}^{(b+1)}$  is obtained.
- 3. After this, distances between the new categories after obtaining merging cluster to get the distance matrix  $D^{(b+1)}$ .
- 4. Step 2 is repeated until the classification meets the requirements.

## 2.2 Least square support vector machine

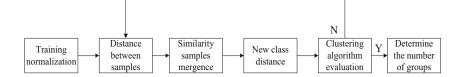
The support vector machine (SVM) is an approach based on statistical leaning and minimized structural risk. Vapnik [28] suggested that it is dependent on training data to a small degree with good generalization and global optimization characteristic and can be used to handle parameters with non-linear relationship. Suykens et al. proposed a modification by using a sum of squared error terms in the standard SVM objective function. This new sum was used as the loss function. The new method is referred to as the least square (LS)-SVM [29].

The modeling process for LS-SVM, which is used to solve the problem, is described as [29]

$$\min J(\omega, \xi_i) = \frac{1}{2} \|\omega\|^2 + \gamma \frac{1}{2} \sum_{i=1}^{l} \xi_i^2$$
  
s.t.  $y_i = \omega^T \varphi(x_i) + b + \xi_i$   $i = 1, ..., l$  (1)

where *J* is the function with structural minimized risk, and  $\omega$  is a weight vector,  $\omega \in \mathbb{R}^n$ .  $\xi_i$  is the error variable, and  $\gamma$  is an adjustable parameter.  $x_i$  is the input,  $x_i \in \mathbb{R}^n$ .  $y_i$  is the target volume.  $\varphi(\cdot)$  is a mapping function, where a low-dimensional space can be mapped to the *n*-dimensional kernel space. *b* is the deviation, and *l* is the number of inputs.

## Fig. 1 Fuzzy clustering grouping



To build the Lagrange function using Eq. (1),

$$L(\omega,\xi_{i},b,\alpha_{i}) = \frac{1}{2} \|\omega\|^{2} + \gamma \frac{1}{2} \sum_{i=1}^{l} \xi_{i}^{2} - \sum_{i=1}^{l} \alpha_{i} \left( \omega^{T} \varphi(x_{i}) + b + \xi_{i} - y_{i} \right)$$
(2)

where the parameter  $\alpha_i(i=1,\dots,l)$  is a Lagrange multiplier. Based on the necessary conditions for extreme value, the following equations can be induced:

$$\begin{cases} \frac{\partial L}{\partial \omega} = 0 \Rightarrow \omega = \sum_{i=1}^{l} \alpha_{i} \varphi(x_{i}) \\ \frac{\partial L}{\partial \xi_{i}} = 0 \Rightarrow \alpha_{i} = \gamma \xi_{i} \\ \frac{\partial L}{\partial b} = 0 \Rightarrow \sum_{i=1}^{l} \alpha_{i} = 0 \\ \frac{\partial L}{\partial \alpha_{i}} = 0 \Rightarrow y_{i} = \omega^{T} \varphi(x_{i}) + b + \xi_{i} \end{cases}$$
(3)

and  $i=1, \dots, l$ ; then,  $\omega$  and  $\xi_i$  are eliminated from Eq. (3), and the matrix equation can be obtained.

$$\begin{pmatrix} 0 & 1 & \cdots & 1\\ 1 & K(x_1, x_1) + 1/\gamma & \cdots & K(x_1, x_l)\\ \vdots & \vdots & & \vdots\\ 1 & K(x_l, x_1) & \cdots & K(x_l, x_l) + 1/\gamma \end{pmatrix} \begin{pmatrix} b\\ \alpha_1\\ \vdots\\ \alpha_l \end{pmatrix} = \begin{pmatrix} 0\\ y_1\\ \vdots\\ y_l \end{pmatrix} \quad (4)$$

According to Mercer conditions [22], the LS-SVM regression analytic formula can be induced by applying the kernel function  $K(x,x_i)$ .

$$f(x) = \sum_{i=1}^{l} \alpha_i K(x, x_i) + b$$
 (5)

In Eq. (5),  $\alpha_i$  and *b* are calculated using Eq. (4). The kernel function  $K(x_i, x_j)$  is any symmetric function which satisfies the Mercer conditions. Here, the radial basis function (RBF) is chosen as the kernel function and is given by

$$K(x_i, x_j) = \exp\left(-\frac{\|x_i - x_j\|^2}{2\sigma^2}\right)$$
(6)

#### 3 Spindle thermal characterization experiment

#### 3.1 Experimental setup

The experimental system is shown in Fig. 2, which shows the spindle of a precision CNC jig boring machine. A synchronous acquisition system was used for measurements and to determine the temperature along with the associated thermal deformation. This system uses Pt100 precision magnetic temperature sensors to measure temperature for the spindle system. High-precision eddy-current sensors were used to measure thermal drifts in the spindle. Temperature sensors were located on front bearing (T<sub>6</sub>, T<sub>7</sub>), rear bearing (T<sub>1</sub>), motor (T<sub>8</sub>, T<sub>11</sub>), ambient temperature (T<sub>5</sub>), spindle base (T<sub>2</sub>), the cooling fluid inlet (T<sub>9</sub>), bearing cooling out (T<sub>3</sub>), front bearing coolant out (T<sub>4</sub>), and the motor cooling out (T<sub>10</sub>).

#### 3.2 Measurement principle

The spindle thermal drifts were measured using a five-point method [30]. A displacement sensor measurement setup is shown in Fig. 3. The spindle is parallel to the Z-axis, and the axial thermal expansion can be obtained by  $S_5$ . Radial thermal yaw  $\theta_x$  along the partial X direction was measured by sensors  $S_1$  and  $S_3$ , whereas radial thermal pitch  $\theta_y$  along the partial Y direction was measured using sensors  $S_2$  and  $S_4$ .

After the spindle was run for long time periods, thermal expansions occur along the axial direction and thermal angle inclination occurs along the radial direction. This results from the uneven temperature gradient distribution, which is shown in Fig. 4, and the thermal yaw angle is

$$\Delta L_3 = L_3^i - L_3^0 \tag{7}$$

$$\Delta L_1 = L_1^i - L_1^0 \tag{8}$$

$$\Delta L = \Delta L_3 - \Delta L_1 \tag{9}$$

$$\tan\theta_x = \frac{\Delta L}{D_s} \tag{10}$$

where i denotes the number of measurements. The thermal yaw angle is negligible for this experiment, that is **Fig. 2** Experimental setup showing the spindle of a precision CNC jig boring machine



$$\theta_x \rightarrow 0$$
, so:

$$\theta_x \operatorname{\tilde{tan}} \theta_x$$
 (11)

As shown in Eq. (12), the thermal yaw can be obtained by applying Eqs. (7)–(11).

$$\theta_x = \frac{\left(L_3^i - L_1^i\right) - \left(L_3^0 - L_1^0\right)}{D_s} \tag{12}$$

where  $L_3^0$  and  $L_1^0$  are the radial displacement between the sensor probe and the spindle measured by  $S_3$  and  $S_1$ , respectively.  $L_3^i$  and  $L_1^i$  are the transient displacement during operation.  $D_s$  is the distance between  $S_1$  and  $S_3$  and  $S_2$  and  $S_4$ , and  $D_s=120$  mm.

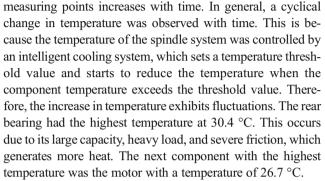
Similarly, the thermal pitch angle in the *Y* direction can be obtained:

$$\theta_y = \frac{\left(L_4^i - L_2^i\right) - \left(L_4^0 - L_2^0\right)}{D_s} \tag{13}$$

## 3.3 Results and analysis

The spindle speed affects the temperature field distribution and the magnitude of thermal drifts. In order to simulate actual spindle speed during processing, the specific speed distribution is shown in Fig. 5.

The spindle system temperature variation is shown in Fig. 6. The overall trend for the temperatures at all the



The measurements shown in Fig. 7 indicate that the displacement trends gradually increase over time, eventually reaching thermal equilibrium. Z-axis axial thermal elongation increases with time. The elongation direction is negative, which indicates spindle thermal expansion to the negative direction on the Z-axis. It takes approximately 385 min to reach thermal equilibrium, and the maximum elongation was 39.6 µm. Thermal error on X-axis direction is positive, which indicates that during the heating process, the spindle is away from the displacement sensors  $S_1/S_3$ . It deviates from the Z-axis, and the spindle swings to the negative direction along the X-axis on the XZ plane. Thermal yaw angle to the Z-axis in this case is  $\theta_x$ , and the maximum heat offset error was 35.8  $\mu$ m. Thermal error in the Y direction is negative, which indicates that during operation the spindle is closer to the displacement sensors  $S_2/S_4$ . It deviates from the Z-axis, and the spindle in the YZ plane pitches to the negative direction on the Y-axis. The thermal pitch angle to the Z-axis in this case is  $\theta_{\nu}$  and the maximum thermal offset was 20.2  $\mu$ m.

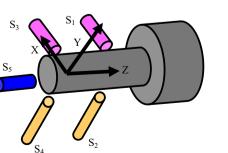


Fig. 3 Spindle five-spot installation diagram

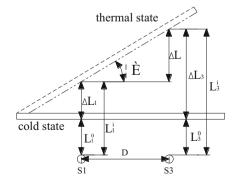


Fig. 4 The spindle thermal inclination sketch

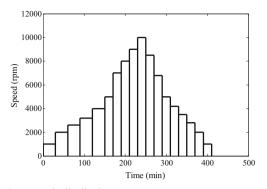


Fig. 5 Step speeds distribution

## 4 Thermal error prediction and compensation

## 4.1 Parameter identification and model training

Optimization of thermal key points, m=11, sets the number of packets C=4, after calculating, and the combination of Euclidean-centroid clustering algorithm obtained the optimal grouping; the cluster groupings shown in Fig. 8 divide the temperature variables into groups  $\{T_1\}$ ,  $\{T_5\}$ ,  $\{T_2, T_3, T_4, T_6, T_9, T_{10}\}$ , and  $\{T_7, T_8, T_{11}\}$ .

Based on the results for the groups, correlation coefficients between axial thermal error E and the temperature  $T_i$  can be calculated as follows:

$$\rho_{T_iE} = \frac{\sum_{j=1}^n \left(T_{ij} - \overline{T_i}\right) \left(E_j - \overline{E_j}\right)}{\sqrt{\sum_{j=1}^n \left(T_{ij} - \overline{T_i}\right)^2} \sqrt{\sum_{j=1}^n \left(E_j - \overline{E_j}\right)^2}}$$
(14)

In the above equation, i=1, 2, ..., m refers to the temperature measurement points and j=1, 2, ..., n refers to the number of measurements.  $T_{ij}$  is the temperature of the measuring point,  $E_j$ is the thermal elongation,  $\overline{T}_i$  is the average temperature of the *i*th measurement point, and  $\overline{E_j}$  is the average thermal elongation. Correlation coefficients are listed in Table 1. The

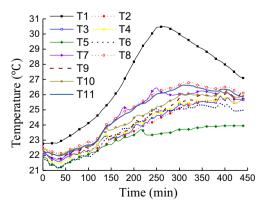


Fig. 6 Temperatures of the spindle

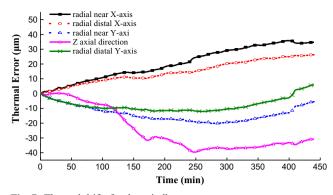


Fig. 7 Thermal drifts for the spindle

temperature variable with the highest coefficient was selected as a typical variable in each cluster.  $T_{10}$  is the outlet liquid temperature of the motor coolant, and its temperature has a significant influence on motor temperature. Thus,  $T_{10}$  is reserved as a key variable. Finally,  $T_1$ ,  $T_5$ ,  $T_6$ ,  $T_7$ , and  $T_{10}$  were chosen as the typical temperature variables.

Using the five identified temperature variables as the input, namely  $x=[T_1, T_5, T_6, T_7, T_{10}]$ , and  $x_j=[T_1^j, T_5^j, T_6^j, T_7^j, T_{10}^j]$  as temperature vector of the *j*th measurement,  $j=1, \dots, l$ and l=89. Lagrange coefficients of thermal elongation *E*, thermal yaw angle  $\theta_x$ , and thermal pitch angle  $\theta_y$  are  $\alpha_i, \beta_i, \eta_i$ , respectively, and the corresponding deviations are  $b_1, b_2, b_3$ . Then, the ranges of values are:

$$\begin{cases} \gamma_n = 5n, \quad n = 1, 2, \cdots, 20, \text{ and } \gamma_0 = 1\\ \sigma_k^2 = 1 + 0.5k, \quad k = 0, 1, \cdots, 18 \end{cases}$$
(15)

Through the use of the cross-validation method (CV) for solving  $\gamma$  and  $\sigma$ , Lagrange coefficients  $\alpha_i, \beta_i, \eta_i$  and deviation  $b_1, b_2, b_3$  were calculated by Eqs. (4) and (6). Shown in Table 2, the LS-SVM consists of 89 vector machines.

## 4.2 Model prediction

The sample size for the data is 89. LS-SVM is used to predict thermal drifts for the spindle. The curve fitting and actual measurements are compared in Figs. 9, 10, and 11.

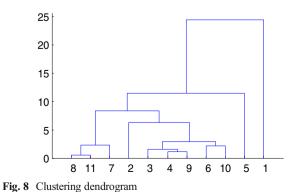


 Table 1 Correlation coefficients between temperature and axial thermal error

Temperature	$T_{I}$	$T_2$	$T_3$	$T_4$	$T_5$	$T_6$	$T_7$	$T_8$	$T_9$	<i>T</i> <sub>10</sub>	$T_{11}$
Cluster T	4	2	2	2	3	2	1	1	2	2	1
ρ	0.9651	0.8593	0.9054	0.9242	0.9546	0.9706	0.9902	0.9739	0.8948	0.9344	0.9737

The evaluation criteria for fitting the model need to be established, assuming the absolute value of the residual error is  $|e_i|$ , minimum as  $|e_i|_{min}$ , maximum as  $|e_i|_{max}$ , and mean values as  $\overline{|e_i|}$ . Root mean square error is RMSE, the determination coefficient is  $R^2$ , and the predictive ability is  $\eta$ .

$$\text{RMSE} = \sqrt{\frac{\sum_{i=1}^{n} (y_i - \widetilde{y}_i)^2}{n}}$$
(16)

$$R^{2} = 1 - \frac{\sum_{i=1}^{n} \left( y_{i} - \widetilde{y}_{i} \right)^{2}}{\sum_{i=1}^{n} \left( y_{i} - \overline{y}_{i} \right)^{2}}$$
(17)

$$\eta = 1 - \frac{\frac{1}{n} \sum_{i=1}^{n} \left| y_{i} - \widetilde{y}_{i} \right|}{\frac{1}{n} \sum_{i=1}^{n} \left| y_{i} \right|} = 1 - \frac{\sum_{i=1}^{n} \left| y_{i} - \widetilde{y}_{i} \right|}{\sum_{i=1}^{n} \left| y_{i} \right|}$$
(18)

where  $y_i$  is the measurement value, i is the predicted value by the thermal error model,  $\overline{y}_i$  is the average value of the measurement,  $i=1, \dots, n$ , and n is the number of data points. The fitting performance parameters of the least square support vector machine are shown in Table 3.

The absolute mean value of the residual error is small, the RMSE are close to zero, and the coefficient of determination  $R^2$  is close to 1. In addition, the predictive ability of the model in the three different directions was more than 90 %, which indicates that the LS-SVM model has higher prediction accuracy.

Table 2The parametersfor LS-SVM

Output	$\gamma$	$\sigma^2$	b	l
Ε	50	1	0.2390	89
$\theta_x$	5	2	0.1930	89
$\theta_y$	75	8	-0.1217	89

#### 4.3 Thermal error compensation

## 4.3.1 Compensation equation of thermal drifts

Figure 12 describes the spatial pose of the spindle thermal drift on *XOZ*, and the point *P* is the deflexion center. Through axial elongation *E* and radial inclination  $\theta_x$ , the spindle declined from  $\overrightarrow{PO}$  to  $\overrightarrow{PO'}$ , so the thermal offset component in the *X* direction can be written as

$$\Delta O_x = (D_{0x} + D_t + \Delta D) sin\theta_x \tag{19}$$

where the offset in the X direction is  $\Delta O_x$ ,  $D_{0x}$  is the distance between deflexion center and spindle nose,  $D_t$  is the length of the tool, and  $\Delta D$  is the axial elongation E.

The thermal offset in the Z direction is  $\Delta O_z$ :

$$\Delta O_z = \Delta D - \Delta O_D = \Delta D - (D_{0x} + D_t + \Delta D)(1 - \cos\theta_x) (20)$$

Because the axial elongation is less than the length of the tool, that is

$$\Delta D << D_{0x} + D, \quad \text{and} \quad \theta_x \to 0 \tag{21}$$

so

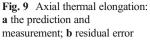
$$\frac{\sin\theta_x \to \theta_x}{\cos\theta_x \to 1} \tag{22}$$

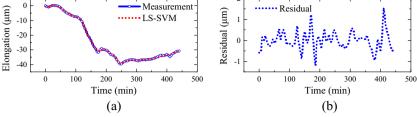
Equations (21) and (22) were substituted into Eqs. (19)–(20), then thermal offsets in the X and Z directions can be obtained.

$$\Delta O_x = (D_{0x} + D_t)\theta_x \tag{23}$$

$$\Delta O_z = \Delta D \tag{24}$$

This indicates that the thermal offset in the Z direction has no relationship with the tool length, while the X directional thermal offset is closely related to the tool length.





Similarly, the thermal error offset  $\Delta O_y$  in the Y direction can be obtained:

$$\Delta O_y = (D_{0y} + D_t)\theta_y \tag{25}$$

where  $D_{0y}$  is the distance between the deflexion center and spindle nose.

Assuming the distance between the deflexion center and spindle nose is  $D_{0x}$ ,  $D_{0y}$  in the X and Y directions, respectively, as is shown in Fig. 12b, there is

$$D_{0x} = \frac{\Delta L_1}{\tan \theta_x} - D_{L1} = 548.659 \text{ mm}$$
(26)

$$D_{0y} = \frac{\Delta L_2}{\tan \theta_y} - D_{L2} = 508.706 \text{ mm}$$
(27)

In Eq. (26),  $\Delta L_1 = L_1^i - L_1^0$  and the value is measured by the displacement sensor  $S_1$ , which is shown in Fig. 7.  $\tan \theta_x \sim \theta_x$ , the  $\theta_x$  is shown in Fig. 10a calculated by Eq. (12).  $\Delta L_1$  and  $\theta_x$  are dynamic changed state variables, and the ratio of them is also dynamic changed, but the fluctuation is small and the ratio is close to 791.817 mm, so treat  $\frac{\Delta L_1}{\tan \theta_x} = 791.817$  mm. In a similar way, in Eq. (27),  $\Delta L_2 = L_2^0 - L_2^0$  and the value is acquired by the displacement sensor  $S_2$ , which is also shown in Fig. 7. The computed result of  $\theta_y$  is shown in Fig. 11a, and treat  $\frac{\Delta L_1}{\tan \theta_x} = 751.864$  mm.

The distance between displacement sensors  $S_1$ ,  $S_2$  and the spindle nose is  $D_{L1}$ ,  $D_{L2}$ , and  $D_{L1}=D_{L2}=243.158$  mm. The thermal offsets for the coordinate can be obtained by applying Eqs. (5) and (23)–(27).

**Fig. 10** Radial thermal yaw angle: **a** the prediction and measurement; **b** residual error

$$\Delta O_x = (D_t + 548.66) \left[ b_2 + \sum_{i=1}^l \beta_i K(x, x_i) \right]$$
(28)

$$\Delta O_y = (D_t + 508.71) \left[ b_3 + \sum_{i=1}^l \eta_i K(x, x_i) \right]$$
(29)

$$\Delta O_z = b_1 + \sum_{i=1}^{l} \alpha_i K(x, x_i)$$
(30)

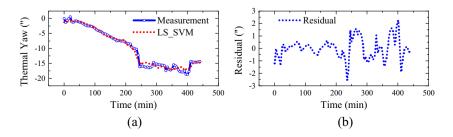
where  $x = [T_1, T_5, T_6, T_7, T_{10}]$ ,  $\alpha_i, \beta_i, \eta_i$ , and  $b_1, b_2, b_3$  are as shown in Table 2.

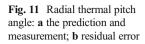
# 4.3.2 The principle of thermal error compensation

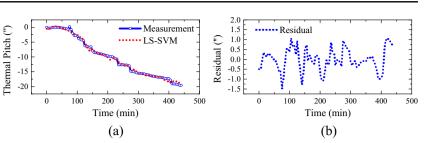
The motorized spindle system of the precision jig boring machine deflects its ideal position space because of the thermal error, and it can be learned that the thermal offsets on the three directions are all negative through experimental analysis, as shown in Figs. 9a, 10a, and 11a. Define the thermal deviation vector of the cutting tool as  $\vec{OO'}$ :

$$\overrightarrow{OO'} = \left\{ -\Delta O_x, -\Delta O_y, -\Delta O_z \right\}$$
(31)

The accuracy of the machine tool is determined by the relative displacement between the cutter and workpiece in machining. Based on the analysis of Eq. (31), the thermal error leads the spindle system to generate thermal offsets on three coordinate directions and reduces the machining







accuracy. In order to eliminate the impact of the thermal offsets, the extra compensated amount must be added on the coordinate of the workpiece, so the essence of the thermal error compensation is to move the kinematic pair of the machine tool to make the cutter and the workpiece generate a relative motion in the opposite direction of the machine tool thermal offset, and compensates the error resulting from the thermal deformation. As shown in Fig. 13, the compensation signal of the thermal error is inserted into the CNC system and superimposed with the machining coordinate values and the feedback signal of the encoder. Then, the new synthetic coordinates are utilized to control the machine tool motion, thus realizing the final compensation of the thermal errors.

In order to improve the machining accuracy, eliminating the effect of thermal drifts on the spindle, the direction of the thermal error compensation component is opposite to the tool thermal offsets vector, and the amount of them should be equal:

$$\Delta H_s = - \overrightarrow{OO'}$$
(32)

where  $\Delta H_s$  is the final compensation vector of the spindle system thermal errors.

Assuming that  $OW(P_x, P_y, P_z)$  is the coordinate of a point W on the workpiece, after conducting the thermal error compensation, the new coordinates of the point W becomes  $OW'(P_x', P_y', P_z')$ , and satisfies

$$OW' = OW + \Delta H_s \tag{33}$$

Substitute Eqs. (31) and (32) into Eq. (33):

$$\begin{cases} P'_{x} = P_{x} + (D_{ox} + D_{t})\theta_{x} \\ P'_{y} = P_{y} + (D_{0y} + D_{t})\theta_{y} \\ P'_{z} = P_{z} + E \end{cases}$$
(34)

 Table 3
 The fitting performance parameters for LS-SVM

Output	$ e_i _{\min}$	$ e_i _{\max}$	$ e_i $	RMSE	$R^2$	$\eta$ %
	μm	μm	μm			
Ε	0.002	1.556	0.313	0.427	0.999	98.7
$\theta_x$	0.008	2.552	0.645	0.846	0.980	93.6
$\theta_y$	0.002	1.489	0.447	0.565	0.992	95.6

The mathematical models of the axial thermal elongation E, the radial thermal yaw angle error  $\theta_x$ , and the thermal pitch angle error  $\theta_y$  have been established in Section 4.2, substituting these prediction models into Eq. (34) to get the final coordinates of the workpiece, and the new coordinates being compensated are as follows:

$$\begin{cases} P'_{x} = P_{x} + \Delta O_{x} = P_{x} + (D_{t} + 548.66) \left[ b_{2} + \sum_{i=1}^{l} \beta_{i} K(x, x_{i}) \right] \\ P'_{y} = P_{y} + \Delta O_{y} = P_{y} + (D_{t} + 508.71) \left[ b_{3} + \sum_{i=1}^{l} \eta_{i} K(x, x_{i}) \right] \\ P'_{z} = P_{z} + \Delta O_{z} = P_{z} + b_{1} + \sum_{i=1}^{l} \alpha_{i} K(x, x_{i}) \end{cases}$$
(35)

The compensation components of the thermal errors on three coordinate axes are

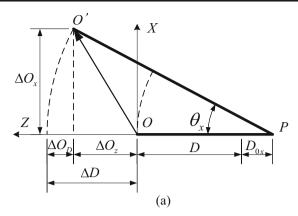
Ζ

$$\Delta H_{s} = \begin{pmatrix} \Delta x \\ \Delta y \\ \Delta z \end{pmatrix}$$

$$= \begin{pmatrix} (D_{t} + 548.66) \left[ b_{2} + \sum_{i=1}^{l} \beta_{i} K(x, x_{i}) \right] \\ (D_{t} + 508.71) \left[ b_{3} + \sum_{i=1}^{l} \eta_{i} K(x, x_{i}) \right] K(x, x_{i}) \\ b_{1} + \sum_{i=1}^{l} \alpha_{i} \end{pmatrix}$$
(36)

where  $D_t$  is the length of the cutting tools,  $\Delta x$ ,  $\Delta y$ ,  $\Delta z$  are the compensation values in the *X*, *Y*, and *Z* directions, respectively, and they are compensated by the CNC controller.

Figure 13 is a schematic diagram of the setup for the spindle thermal error compensation. The Siemens 840D CNC system is used for the experiment. The temperature module acquires signal from PT100 and sends them to the CNC system using RS-232 communication. A thermal error compensation module is embedded into CNC based on secondary development of 840D. It can receive error compensation parameters and then passes them on to the PLC. Finally, thermal error offset was calculated and sent to the CNC to achieve compensation using PLC. While the thermal yaw error and pitch error were translated into the components of



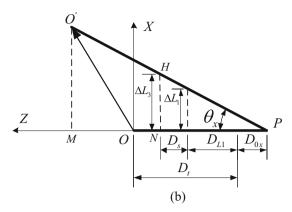


Fig. 12 The geometric principle of the spindle thermal error compensation

the coordinate axis, three components were compensated by the principle described of this compensation system.

## 4.3.3 Results and comparison

In order to compare with the compensation effect of the fivepoint method, the compensation algorithm based on the threepoint method is established. There are two displacement sensors in the radial direction based on the three-point method, but each of the radial planes has only one sensor. As shown in Fig. 12b, it has only one sensor,  $S_3$  or  $S_1$ , in the *XOZ* plane. The end of the spindle is the point *O* and  $S_3$  is closer to it, and the measured thermal yaw angle error is larger and the compensation effect is better than  $S_1$ , so the measured data of  $S_3$  is chosen to build the compensated model and compared with the five-point method. In the cool state, sign the measured point of  $S_3$  on the spindle as *N*. When the spindle is at the thermal state, the point is *H*. In order to verify the effect of the thermal error compensation, it is needed to compare the measured values of  $S_3$  before/after compensation.

Before the compensation, the measured thermal yaw angle error was  $\vec{NH}$  :

$$NH = \Delta L_3 = L_3^i - L_3^0 \tag{37}$$

For the motorized spindle system, the thermal offsets of the end of the tool are compensated by the CNC system, that is, the compensation values of the thermal errors are added to the coordinates of the tool end point O'. Before compensation, the thermal offset of  $S_3$  is the measured value  $\vec{NH}$ , and the threepoint method treats  $\vec{NH}$  as the compensation component in the X direction, which means that the CNC system compensates  $\vec{NH}$  at the point O'. But the actual distribution value at

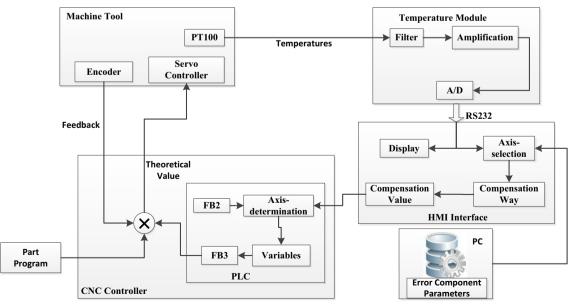


Fig. 13 Thermal error compensation control

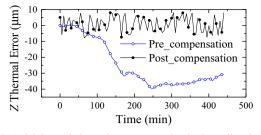


Fig. 14 Axial thermal elongation compensation in the Z direction

point H of the compensation value is  $NH_3$ . For  $\Delta O'MP \cong \Delta HNP$ , so there is the following proportional:

$$\frac{\overline{NH_3}}{\overline{O'M}} = \frac{D_{ox} + D_{L1} + D_s}{D_{ox} + D_t + \Delta O_z}$$
(38)

The actual compensated component of the thermal yaw angle error at point O' is  $\overrightarrow{NH}$  based on the three-point method, that is:

$$O'M \sim NH$$
 (39)

Applying the three-point method, the actual compensation value of the measured point H at the spindle is  $NH_3$ . It can be obtained by applying Eqs. (38) and (39):

$$\overset{\frown}{NH_3} = \frac{D_{ox} + D_{L1} + D_s}{D_{ox} + D_t + \Delta O_z} \overset{\frown}{NH}$$
(40)

Equation (40) illustrates that, before compensation, the measurement  $\vec{NH}$  decreases with the increase in the distance between the measuring point N of S<sub>3</sub> with the spindle end; the compensated value could reduce and the compensation effect may become bad.

While the actual compensated component of the thermal yaw angle error at point O' is O'M based on the five-point method, that is:

Fig. 15 Radial thermal error

compensation in the X direction

$$O'M = \Delta O_x = (D_{ox} + D_t)\theta_x \tag{41}$$

The actual compensation value of the measured point H at the spindle is  $NH_5$ . It can be induced by Eq. (41):

$$\vec{NH}_{5} = \frac{D_{ox} + D_{L1} + D_{s}}{D_{ox} + D_{t} + \Delta O_{z}} (D_{ox} + D_{t})\theta_{x}$$
(42)

It is obvious that

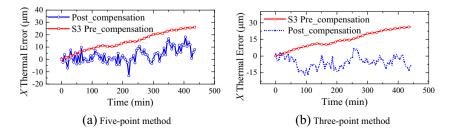
of the three-point method.

$$\stackrel{\sim}{NH_3} < \begin{pmatrix} \stackrel{\sim}{NH} \approx \stackrel{\sim}{NH_5} \end{pmatrix}$$
(43)

The compensation principle of the thermal pitch angle error on the *YOZ* plane and the measured values of  $S_4$  before/after compensation are compared.

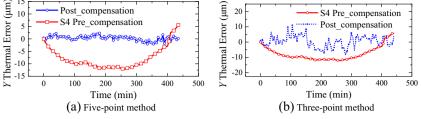
The spindle system moves in the opposite directions of the thermal offsets when the compensation is carried out. Therefore, the measurement of the displacement sensor  $S_3$  will decrease after compensation; the more it decreases, the more the compensated amount is, and the better the compensated amount effect is. Inequality (43) presents that the compensated amount in the five-point method is larger than that of the three-point method, which is closer to the measurement  $\vec{NH}$  before compensation, and its compensation effect is better than that

After the compensation for thermal error, the errors in axial and radial directions reduced significantly and are shown in Figs. 14, 15, and 16. Applying the five-point method, the axial maximum error decreased from 39 to 8  $\mu$ m, and the average error reduced from 24.6 to 4.3  $\mu$ m, namely the average offset is about 20  $\mu$ m. Axial accuracy improved by 82.6 %, which demonstrates that the method based on the proposed measurement and modeling is effective. The absolute average value of radial *X* direction thermal error *S*<sub>3</sub> reduced from 14.6 to 4.9  $\mu$ m, and accuracy improved by 66 %. Meanwhile, the absolute maximum value of radial *Y* direction thermal error *S*<sub>4</sub> reduced from 12.1 into 2.3  $\mu$ m with accuracy improving by 90 %, while the axial and radial accuracy were improved by 82.6, 43, and 30.6 %, respectively, based on the three-point





**Fig. 16** Radial thermal error compensation in the *Y* direction



method. So the five-point method is better than three-point method.

Characteristics of the CNC system determine that the spindle thermal error compensation can only be conducted at the end of its coordinates. And defects of the three-point method to measure the thermal drifts are as follows:

- Due to the installation problem and other factors, the thermal deformation of the spindle terminal position is not easy to be directly measured by the displacement sensors. Generally speaking, the sensors are not assembled at the end of the spindle. Therefore, the measurement results do not accurately reflect thermal drifts of the end on the spindle, making a rough compensation model. While the distance between the end of the tool and the displacement sensor increases, the effect of the compensation based on the three-point method will be worse. Although the five-point method cannot measure the end thermal drifts also, the thermal error model proposed in this paper can calculate the compensated amounts of thermal offsets of the tool end, achieving a more accurate compensation.
- 2. Based on the three-point method, if the length of the cutting tool is changed, the thermal error model is no longer applicable because the influence of the tool length is ignored, and extra experiments are needed to establish a new model, which needs for better cost. While the compensation model of the thermal error does not need to be modified and has a perfect generalization based on the five-point method, considering the length of the cutting

tools. In summary, the five-point method is better than the three-point method.

When the thermal offsets of the spindle are compensated on three directions based on the five-point method, the thermal inclination angle errors were reduced, and the result is shown in Fig. 17. After compensation, the maximum measurement of the thermal yaw angle error is 5", and the average of the absolute value of the residuals is 1.2". While the maximum thermal pitch angle error is 4.5'', the average of the absolute value of the residuals is 1.4". In fact, the thermal inclination angle errors of the spindle system are not reduced, but by performing the thermal error compensation, the feed system moves the extra distances in the opposite direction of the thermal offsets. After compensation, the measured values of the thermal offsets reduced. And the  $\theta_x$  and  $\theta_y$  decreased calculated by Eqs. (12) and (13); therefore, the proposed method of the thermal error compensation can improve the terminal machining accuracy of the CNC machine tool.

# **5** Conclusions

Spindle thermal error modeling with axial elongation and radial thermal angle errors are suitable for actual practical conditions. This is due to the fact that it accurately describes thermal deformation space-pose and can consequently be used to improve machining accuracy. Radial thermal-induced angle errors were ignored in methods discussed in literature as most

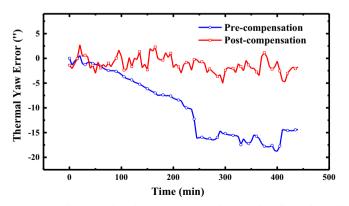
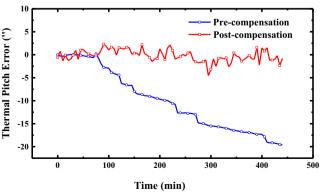


Fig. 17 Radial thermal angle errors compensation based on five-point method



methods are based on three-point measurement of the spindle thermal errors. To solve this problem, a five-point method was applied to measure the spindle thermal drifts. The use of a least square support vector machine model to incorporate thermal elongation and declination angles was proposed. The methods were based on fuzzy cluster with a high predictive accuracy. Moreover, the method combining fuzzy cluster and correlation analysis was proposed to group and optimize the temperature variables, reducing the multicollinearity of the temperature variables and the improving stability of the model. In addition, equations for thermal error offset were derived, which take into account the tilt angles and length of the cutting tools. As a result a real-time compensation was implemented. Experimental results demonstrate that the axial (in the X and Y directions) and radial accuracy were improved by 82.6, 66, and 90 %, respectively, which demonstrated that the proposed method of measurement, modeling, and compensation was effective.

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**Conflict of interest** We declare that we have no financial and personal relationships with other people or organizations that can inappropriately influence our work; there is no professional or other personal interest of any nature or kind in any product or company that could be construed as influencing the position presented in, or the review of, the manuscript.

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