ORIGINAL ARTICLE

# **Optimization of loading path in hydroforming T-shape using fuzzy control algorithm**

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Received: 18 February 2012 / Accepted: 24 May 2013 / Published online: 11 June 2013 © Springer-Verlag London 2013

Abstract The loading path is crucial to the quality of forming parts in the process of tube hydroforming, and thus the design and optimization of loading path is an important issue for tube hydroforming. Wrinkling is a catastrophic defect for thinwalled tube hydroforming. In order to avoid wrinkling, an adaptive simulation approach integrated with a fuzzy control algorithm is used to optimize the loading path of hydroforming a T-shaped tube. The tubular material used is stainless steel and has an outer diameter of 103 mm and the wall thickness of 1.5 mm. The controlled variables are the axial feeding, the counterpunch displacement, and the internal pressure. A code is developed to make the optimization automatically, which works together with LS-DYNA. Six evaluation functions are adopted for identifying geometrical shape and quality of Tshape. Failure indicators obtained from the simulation results are used as the input of the fuzzy control, and then process parameters are adjusted according to the expert experiences in the fuzzy controller. In this way, a reasonable loading path for producing a sound T-shape is obtained, and also a T-shaped product is successfully hydroformed by experiment. The result shows that the fuzzy control algorithm can provide an adequately reliable loading path for hydroforming T-shaped tubes.

Keywords T-shape  $\cdot$  Hydroforming  $\cdot$  Fuzzy control  $\cdot$  Loading path  $\cdot$  Optimization

### **1** Introduction

Manufacturing of complicated components with high quality induce that the hydroforming process is becoming one of the

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National Key Laboratory for Precision Hot Processing of Metals, Harbin Institute of Technology, P.O. Box 435, 92 West Da-Zhi Street, Harbin 150001, China e-mail: bgteng@hit.edu.cn main interest for researchers and manufactures. The development of hydroforming technology has also led to a wide range of application in industry, especially for manufacturing high-quality lightweight components for the automobile, aviation, and aerospace industry [1–3]. Compared with conventional stamping and welding processes, the advantages of tube hydroforming include part consolidation on assemblies, weight reduction, improved structural strength and stiffness, excellent material utilization, fewer secondary operations, and improved part quality [4–7].

There are many different variables in this process that have significant influences on the final quality of products, hence, manufacturing of different components without any defects such as bursting or wrinkling encounters many difficulties. Currently, finite element simulation is generally used to reduce the product design time of hydroforming, and process design phases should also be expedited and shortened [8]. However, manufacturing engineers usually have to rely on their trial-and-error method in process design due to a lack of comprehensive system knowledge.

Many researchers have concentrated on the design and optimization of load paths in the hydroforming process. Ray et al. [9, 10] investigated the manufacturing of T- and Xshaped joints by using an intelligent load control algorithm, which can determine the optimal load paths for X- and Tbranch tube hydroforming processes by maximizing the part expansion and simultaneously maintaining the wall thickness, forming stresses, and plastic strains within the allowable limits. Aydemir et al. [11] used ABAQUS and the fuzzy adaptive approach to simulate a T-shaped tube hydroforming process. The wrinkling and bursting criterion based on energy and the forming limit curve are, respectively, employed. However, in above works, load path of controlling the counterpunch for a branch was not considered. Imaninejad et al. [12] used LS-DYNA and the optimization software LS-OPT to optimize the internal pressure and axial feed loading paths for a T-joint design, in which the minimum thickness

**Fig. 1** Scheme of the adaptive simulation procedure



variation is chosen as the design objective, while keeping the maximum effective stress below the material ultimate strength. Jansson et al. [13] proposed an adaptive optimization method based on the use of response surface methods. The objective of the optimization is to maximize the minimum thickness, while keeping the maximum wrinkling at an acceptable level. Lorenzo et al. [14] developed a gradientbased approach to optimize the process of Y-shaped tube hydroforming, and both the internal pressure path and the counterpunch action were taken into account. Abedrabbo et al. [15] used genetic algorithm search method to optimize the process parameters and determine the best loading paths for tube hydroforming process, in which FLSD is chosen as the failure criterion. In addition, Mirzaali et al. [16] used simulated annealing optimization algorithm for optimizing internal hydraulic pressure to obtain the maximum formability of axisymmetric tubes under a failure criteria based on material's FLD. Manabe et al. [17] developed a fuzzy control system to optimize the loading conditions of hydroforming an aluminum alloy T-shaped tube with a counterpunch. Two evaluation functions are used for identifying geometrical shape and quality of T-shaped tube, i.e., wavy buckling deformation of the branch and the contact length between the counterpunch and the top of the branch.

This paper deals mainly with the effect of loading parameters on the T-shape hydroforming process. A fuzzy logicbased load control algorithm is developed for the control of the internal pressure and the axial feeding. An adaptive system is proposed to obtain adequate process parameters for hydroforming T-shaped tubes. Six evaluation functions are considered for identifying the geometrical shape and the quality of Tshaped tubes. The counterpunch control algorithm is also considered in this study. To demonstrate the effectiveness of this fuzzy adaptive process control system, experiments of hydroforming a stainless steel T-shape have been carried out. By comparing the experimental results with the simulated results, the validity of the fuzzy control system for hydroforming T-shaped tubes with a counterpunch is confirmed.

## 2 Adaptive simulation approach by using fuzzy control algorithm

In the process of hydroforming T-shaped tubes, there is normally a tendency of wrinkling, if the axial feeding is too large with respect to the applied internal pressure. On the other hand, if the internal pressure is too high with respect to the axial feeding, then there is a chance of bursting of the tube due to the excess wall thinning. Thus, the forming load parameters have to be properly adjusted and tuned in order to obtain a sound component. An adaptive approach is based on the ability to early detect the onset and growth of defects or



**Fig. 2** Wrinkle location for hydroforming T-shapes due to unreasonable load path

### **Fig. 3** Diagram of slope evaluation function



The min slope

the occurrence of unwanted situations during the process and promptly react to them. In this work, winkling of the tube is considered as the main process failure criteria, and based on this failure criterion, the fuzzy adaptive process control is developed. The general strategy of the proposed method is that, under the condition of no wrinkling, as far as possible more axial feeding and less internal pressure will be provided to ensure the minimum thinning. The scheme of the adaptive simulation procedure is shown as Fig. 1.

The total simulation of the process is performed in many discrete steps simulations. In each step, the pressure, the axial feeding, and the counterpunch action are adjusted by the fuzzy load control algorithm as per requirement, which are calculated on the basis of values of evaluation functions. Evaluation functions are adopted for identifying current geometrical shape and quality of T-shape, and their values are processed into fuzzy language variables for fuzzy controller as input values. The process parameters of the next step are calculated by the optimized fuzzy control algorithm and are adjusted according to fuzzy control rules. Using these parameters, preprocess program rebuilds the finite element analysis (FEA) model and delivers the model to LS-DYNA solver. The load control algorithm acts as a closed-loop control system controlling the pressure, the axial feeding, and the counterpunch action to avoid wrinkling of the tube. The final optimal loading path is the superposition of the process parameters in each step. An adaptive system is developed to make the optimization automatically, which works together with LS-DYNA/explicit.

#### 2.1 Evaluation functions

Symmetry line

 $P_i^{\circ}P_{i+1}$ 

Tube

Wrinkling prediction in FEA is generally based on three main methods, i.e., plastic bifurcation theory, energy method, and geometry method. The geometry method is employed in this paper to indicate the onset of wrinkles.

There are several geometry-based wrinkles due to unreasonable load parameters during hydroforming T-shapes. Wrinkles usually encounter at the top of branch, the middle of main tube, and the corner between main tube and branch, which are illustrated in Fig. 2.

In the initial stage, excessive axial feeding can result in the wrinkle at the top of branch, as shown in Fig. 2a. This can be indicated through the slope variation of nodes of interest. Figure 3 is the schematic of the onset of wrinkles indicated by the slope variation. The slope  $k_{ij}$  are simply calculated from any two nodal coordinates between points *i* and *j* along the symmetry line.

$$k_{ij} = \left(y_i - y_j\right) / (x_i - x_j), i = 1, 2, \dots, j - 1; j = i, i + 1, \dots, n$$
(1)

where  $(x_i, y_i)$  is the nodal coordinates of point *i* along the symmetry line, and *n* is the number of node.

**Fig. 4** Corner wrinkle in the transition zone



Middle layer of tube

**Fig. 5** Diagram of the evaluation function  $F_C$  for middle wrinkle



The maximum slope  $F_{A\text{max}}$  and the minimum one  $F_{A\text{min}}$  between any two nodes along the symmetry line can be defined as follows:

$$F_{A\max} = \max\{k_{12}, k_{13}, \dots, k_{1n}, k_{23}, \dots, k_{(n-1)n}\}$$
(2)

$$F_{A\min} = -\min\{k_{12}, k_{13}, \dots, k_{1n}, k_{23}, \dots, k_{(n-1)n}\}$$
(3)

The slope variation of the wrinkle-free part does not change the sign between positive and negative; whereas the sign will change during the onset of wrinkling. The changed sign of slope and the minimum slope  $F_{A\min}$  can be used to indicate an existence of wrinkles in the forming process, and whether the wrinkle is an unstable folding due to the excessive axial feeding or the lower internal pressure.

Corner wrinkle between main tube and branch shown in Fig. 2b is another kind of defects in the T-shape forming process. The maximum gap  $F_B$  between tube and die along the symmetry line can be used to indicate the existence of wrinkles. It can be seen in Fig. 4.

 $F_B = \max \{d_1, d_2..., d_1\}$ , where  $d_i$  is the gap between die and any node in the corner region along the symmetry line.

**Fig. 6** Diagram of contact length evaluation functions (**a**) and diagram of the forming process (**b**). Contact zone between the branch and counterpunch





**Fig.** 7 Membership functions of input variables **a**  $F_{Amax}$ ,  $F_{Amin}$ ,  $F_B$ ,  $F_C$  **b**  $F_{Dx}$ , and  $F_{Dy}$ 



Fig. 8 Membership functions of output variables

Middle wrinkle shown in Fig. 2c can be detected by the maximum gap  $F_C$  between tube and die, as shown in Fig. 5, which is adopted as an evaluation function to measure this defect.  $F_C = \max\{d_1, d_2, ..., d_1\}$ , where  $d_i$  is the gap between any

 $T_C$  = max $\{a_1, a_2, ..., a_1\}$ , where  $a_i$  is the gap between any node of tube and side wall of the die.

The counterpunch imposing reaction force during the process is an effective method to avoid excessive thinning

 Table 2
 Mechanical properties of tube used from uniaxial tensile test

$\sigma_s$ (MPa)	$\sigma_b$ (MPa)	n	K(MPa)	r	v	δ(%)
325	637	0.385	1173	0.8	0.28	49.5

of formed branch. Figure 6 is the scheme of the branch contacting with the counterpunch during forming process, it can be seen that the contact length is varied in different contact direction. The contact zone shown in Fig. 6b is an approximate ellipse, and the length of major axis  $F_{Dx}$  and the length of minor axis  $F_{Dy}$  are adopted as evaluation functions to measure the area of the contact zone.

### 2.2 Design of fuzzy algorithm controller

The geometry of interest location of T-shape during forming can be described by the values of these six evaluation functions mentioned above, i.e., the maximum and minimum slope  $F_{Amax}$  and  $F_{Amin}$ , the maximum gap  $F_B$  and  $F_C$ , the length of major and minor axis of contact ellipse  $F_{Dx}$  and  $F_{Dy}$ . In this paper, the fuzzy algorithm controller is developed to decide hydroforming parameters according to the different status of the tube. The values of evaluation functions should be fuzzed as the input variables of the controller. The evaluation functions  $F_{Amax}$ ,  $F_{Amin}$ ,  $F_B$ , and  $F_C$  are fuzzed into four sets (zero, small, medium, and large), whose membership functions are shown in Fig. 7a. Similarly,  $F_{Dx}$  and  $F_{Dy}$  are fuzzed into six sets (zero, very small, small, medium, large, and very large), whose membership functions are shown in Fig. 7b.

Similarly, the output variables of the fuzzy controller for the axial feeding  $S_1$  and the counterpunch displacement  $S_2$ were divided into four sets, i.e., zero, small, medium, and large. The membership functions are shown in Fig. 8.

In this adaptive simulation system, T-shape hydroforming process was divided into three stages (initial, middle, and final stages) in order to simplify the fuzzy control rules. In the initial stage, the counter punch is fixed, while the axial punches impose the axial feed, and the liquid pressure is added into the tube at a fast speed simultaneously. At this stage, the protrusion dose not contact with the counter punch, it is in free-bulging state. In the initial stage,  $F_{Amax}$ ,  $F_{Amin}$ ,  $F_B$ , and  $F_C$  are considered as the input variables in the fuzzy control of the axial feeding. Because each evolution function has four sets (zero, small, medium, and large), 256 combinations should be considered by the axial feeding fuzzy

Table	1 Chemical c	composition
of the	1Cr18Ni9Ti (	China)

Element	С	Si	Mn	S	Р	Cr	Ni	Ti
Wt.%	0.037	0.56	1.51	0.0019	0.028	17.23	9.57	0.5





control rules in this stage. The middle stage of forming starts from the time of contact between the top of the protrusion and the counter punch to the time of basic affixation. In the final stage, the counter punch begins to retract, but maintains contact with the top of the protrusion, and imposes counter punch force on top of the protrusion to avoid excessive thinning at the top of the protrusion. In the middle and final stage,  $F_{Dx}$ ,  $F_{Dy}$ ,  $F_B$ , and  $F_C$  are regarded as the input variables for the counterpunch displacement fuzzy control, and  $F_B$ and  $F_C$  are used as the input variables in the fuzzy control of the axial feeding. Evolution function  $F_{Dx}$  and  $F_{Dy}$  are fuzzed into six sets (zero, very small, small, medium, large, and very large). So in the middle and final stage, 576 combinations should be considered by the counterpunch displacement fuzzy control rules and also 16 combinations by the axial feeding fuzzy control rules.

So a total of 848 fuzzy control rules with different combination of evaluation functions are used to decide axial feeding and counterpunch displacement. For example, for the control of the axial feeding, If  $F_{Amax}$  is L (large),  $F_{Amin}$  is 0(zero),  $F_B$  is 0(zero),  $F_C$  is 0(zero), then axial feeding is L (large). The fuzzy control rule of the counterpunch displacement can be expressed if  $F_{Dx}$  is VL (very large),  $F_{Dy}$  is S (small),  $F_B$  is 0 (zero),  $F_C$  is 0 (zero), then the counterpunch displacement is S (small).



Fig. 10 The FEM model of T-shape hydroforming

## **3** Adaptive simulation and experiment of T-shaped tube hydroforming

To confirm the validity of adaptive simulation system based on the fuzzy control algorithm for T-shaped tube hydroforming, the adaptive simulation and experiment have been carried out. The tube material is 1Cr18Ni9Ti (China); to determine the mechanical properties of the tube, tensile specimens are cut from the tube and tested. The chemical composition and material properties of the 1Cr18Ni9Ti stainless steel are presented in Tables 1 and 2, respectively.

The special hydroforming press is used for T-shape hydroforming. This hydroforming press consists of a hydraulic press for closing the die, three horizontal cylinders, a pressure intensifier, a hydraulic drive system, and a computer control system. The major parameters are the maximum internal pressure 400 MPa, the closing force 3,150 kN, the left and right horizontal axial force 800 kN, and the middle horizontal axial force 500 kN, which provided the support for the counterpunch. Figure 9 presents the lower die used for experiment and the dimension of the T-shape to be formed.



Fig. 11 Optimized loading curve from adaptive simulation



Fig. 12 Results of a adaptive simulation and b experiment

### 3.1 FEM model

An adaptive simulation code based upon fuzzy control algorithm and evaluation functions mentioned above was developed to optimize loading path for T-shape hydroforming. Working together with LS-DYNA, the software can automatically design optimized loading path instead of traditional trail-and-error FEA approach. Wrinkle indicators obtained

**Fig. 13** The thickness distribution of formed tube

from the simulation results are used as the input of the fuzzy control, and then process parameters are adjusted according to the expert experiences in the fuzzy controller.

A blank tube with outer diameter of 103 mm, initial length of 340 mm, and wall thickness of 1.5 mm is used. The diameter of the branch tube is equal to that of the main tube, and the radius between main tube and the branch is 25 mm. The initial distance between the counterpunch and the tube is 0 mm. The lubricant is formulated with  $MoS_2$  (spray type) and is applied to the die and outer tube surface.

In the FE model, only half part of the tube is actually analyzed due to the symmetry of the problem, as shown in Fig. 10. The tube is discreted to BT shell elements; the number of elements is 3,400, all of which are quadrilateral elements. The die is discreted to rigid elements; the total amount is 1,984; 1,835 of which are quadrilateral elements, and 149 of which are triangular elements. The leftward and rightward punches are discreted to rigid elements; there are 324 elements, 312 of which are quadrilateral elements, and 12 are triangular elements. The counter punch is also discreted to rigid elements, there are 1,077 of them, 1,040 of which are quadrilateral elements.

In the simulation, the work hardening law of material is  $\sigma$ =  $K\varepsilon^n$ . Strength coefficient value K=1,173 and work hardening exponent n=0.385, which is obtained from uniaxial tensile test. Other parameters are: density 7.83 g/cm<sup>3</sup>, Young's modulus 207GPa, Poisson's ratio 0.28. Contact between the tube and die is modeled using a penalty-based contact algorithm, and the friction coefficient of 0.05 is applied to the contact surface. Pressurized liquid is imposed by uniform pressure, which is added on the internal face of the tube.

The adaptive simulation parameters are set as follows: final branch height is 55 mm, final top radius of the branch



is less than 15 mm, and the internal pressure increment for each loop is 0.5 MPa.

#### 3.2 Adaptive simulation and experimental results

The computation is performed using a low-cost PC (Pentium 4, 2.40 GHz, 512 MB RAM) running Windows XP. After 112 times loop of adaptive simulation, which costs about 31 h, the program ended normally, and the requirement of tube shape was satisfied. The process parameters of every single-step are calculated by the optimized fuzzy control algorithm, which is adjusted according to the experience. The final optimized loading path is the superposition of every single step. The optimized loading curve shown in Fig. 11 is generated by the adaptive simulation using the fuzzy control algorithm. The final internal pressure is 56 MPa, and the final axial feeding is 47.6 mm (unilateral). The simulation result of tube wall thickness distribution is shown in Fig. 12a, which indicated that the minimum thickness at the top corner of branch tube is 1.08 mm (the thinning rate 28 %), and the maximum thickness at the transition region between the main and branch tube is 1.94 mm (the thickening rate 29.3 %). The experiment of T-shape hydroforming using the optimized loading curve generated by the adaptive simulation is carried out, and the experimental result is shown in Fig. 12b.

The experimental product is cut linearly along axial direction to measure thickness at the designated points (see Fig. 13). The values in brackets are simulation results. The minimum thickness at the top corner of branch tube is 1.03 mm (the thinning rate 31.3 %), the error is 3.3 % compared with the adaptive simulation result. The maximum thickness at the rounded transition region between the main and branch tube is 1.90 mm (the thickening rate 26.7 %), which is less than 2.6 % compared with the adaptive simulation one.

### **4** Conclusion

A T-shape hydroforming adaptive simulation system, worked together with LS-DYNA, based on the fuzzy control algorithm is developed in this paper. Six evaluation functions are used to identify geometrical shape of forming tube and to indicate the onset of wrinkling. The adaptive system can generate the optimized loading curve automatically, and a stainless steel thin-walled T-shaped product was successfully hydroformed using the optimized loading path, which was obtained from this system. The experimental result shows that the maximum thinning and the thickening ratio of the T-shape are 31.3 and 26.7 %, respectively. Comparing with the simulation results, the error of the maximum or minimum thickness is less than 3.3 %. Experimental results confirm the

validity of the fuzzy control system for hydroforming Tshapes with a counterpunch.

**Acknowledgments** This work was financially supported by the National Key Technology R&D Program, Development of Advanced Forming Technologies of High Strength Steel and Integration Applications in Target Car, and the National Natural Science Foundation of China (project number: 50875060). The authors would like to take this opportunity to express their sincere appreciation.

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