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A study on helical surface generated by the primary peripheral surfaces of ring tool

S. Berbinschi · V. Teodor · N. Oancea

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Abstract Often in the engineering practice, cutting tools bounded by primary peripheral surfaces of revolution are used because of their effectiveness. Among these, ring and tangential tools can be used for the generation of constant pitch cylindrical helical surfaces. In this paper, we present an algorithm for the profiling of these types of tools. The algorithm is based on the topological representation of the tool's primary peripheral surface. The main goal is to devise a methodology for the profiling of tools whose surfaces are reciprocally enveloping with cylindrical helical surfaces. We present a numerical example for the numerical determination of the axial section form for this type of tools. The application method for this algorithm was developed in the CATIA graphical design environment within which the procedure is developed as a vertical application. In addition, we present a solution for the shape correction of the tool's axial crosssection by considering the existence of singular points on the profile of the helical surface to be generated where multiple normals to the surface exist.

Keywords Ring tool · Helical surfaces generation · Topological representation

1 Introduction

The ring and tangential tools are tools bounded by a revolution primary peripheral surface. The ring tools are

tools designated for the generation of the cylindrical helical surfaces with constant pitch (threads) on specialized machine tools or using specialized technological equipments for longitudinal turning machines.

The ring tool is frequently made as an enwrapping milling tool. The advantage of this technological solution is the increased productivity of this process. Although the tools of this type generate in the cutting motion a revolving surface, the issue of profiling the primary peripheral surface of this surfaces reciprocally enveloping with cylindrical helical surfaces with constant pitch, is a problem different from the profiling of the side mill [1–4].

The profiling method of this type of tool uses the fundamental theorems of the surfaces generation [1, 5] or the complementary methods as "the minimum distance method" [1], "the in-plane generating trajectories method" [6]. Also, the development of the graphical design environment allows solving these problems using 3D design environment [7–10] or using solid modeling [11, 12].

2 Ring tool's profiling algorithm

The basic idea behind the proposed algorithm is that a helical movement, with \vec{V} axis and p helical parameter, can be decomposed in two rotations conjugated with this relative motion as shown in Fig. 1.

The three movements, as shown in Fig. 1, are:

I is the translation movement correlated with the rotation movement II, in order to produce a helical motion with the same helical parameter p as of the surface to be generated. In most cases, this motion is executed by the workpiece;

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Fig. 1 The decomposition of the helical motion into two rotation motions

- II the rotation movement of blank around its own axis, \overline{B} . In the most practical cases, this motion is uniform; and
- III the rotation motion of tool around its own axis, the axis \vec{A} . This axis is positioned regarding the axis \overline{B} at *a* distance and inclined with angle α regarding the *Z* axis.

One can represent these relative motions as

$$\left(\vec{V},p\right) \approx \left(\vec{A},\omega_{A}\right) + \left(\vec{B},\omega_{B}\right)$$
 (1)

where

$$p = a \cdot tan(\beta) = b \cdot tan(\alpha) \tag{2}$$

with p is the helical parameter.

In this way, it is possible to choose as revolution surface's axis, the *B* axis, which may be established as the axis of the ring surface. Also, it is possible to arbitrarily choose the values *b* and β according to the dimension of the helical surface and the diameter of the ring tool (see Eqs. 1 and 2).

The Nikolaev's theorem [5] for the determination of the characteristic curve between the Σ helical surface, Fig. 2, and the primary peripheral surface of ring tool allows to determine the geometrical locus of points which belongs to the Σ surface where the condition is accomplished:

$$\left(\vec{B}, \vec{N}_{\Sigma}, \vec{r}_2\right) = 0 \tag{3}$$

where, \vec{B} is the vector that overlapped the ring tool's axis; \vec{N}_{Σ} is the normal to the helical surface; and \vec{r}_2 is the position vector of the current point that belongs to the helical surface, regarding the O_2 origin of the reference system joined with the axis X_2 .



Fig. 2 Ring surface reciprocally enveloping with helical surface

The condition 3 has the geometrical significance, the fact that the three vectors are in the same plane. Moreover, the contact points (the tangency points) between the Σ helical surface and the primary peripheral surface of the ring tool defines the characteristic of Σ surface, in the rotation movement of this around the \vec{B} axis. The characteristic curve represents the geometric locus of intersection points between the normal draws from the points belongs to the \vec{B} axis to the surface to be generated (so, the projection of the \vec{B} axis to the Σ surface).

In this way, for the Σ surface known in the $X_2Y_2Z_2$ reference system by equations form:

$$\Sigma \left| \vec{r}_2 = X_2(u, v) \vec{i} + Y_2(u, v) \vec{j} + Z_2(u, v) \vec{k}, \right|$$
(4)

with u and v as independent variable parameters. In the $X_2Y_2Z_2$ reference system, it is defined the normal to the surface,

$$\vec{N}_{\Sigma} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ \dot{X}_{2u} & \dot{Y}_{2u} & \dot{Z}_{2u} \\ \dot{X}_{2v} & \dot{Y}_{2v} & \dot{Z}_{2v} \end{vmatrix}.$$
(5)

The axis of the future ring tool has directrix parameters

$$\vec{B} = \vec{i}.$$
 (6)

After developing, the condition 3 will be,

$$\begin{vmatrix} X_2(u,v) & Y_2(u,v) & Z_2(u,v) \\ N_{X2} & N_{Y2} & N_{Z2} \\ 1 & 0 & 0 \end{vmatrix} = 0$$
(7)

equivalent with a link between the variable parameters,

$$v = v(u) \tag{8}$$

In this way, the characteristic curve on the Σ surface is given by the 4 and 9 equations assembly:

$$C_{\Sigma}|X_2 = X_2(u); Y_2 = Y_2(u); Z_2 = Z_2(u).$$
(9)

By revolving the characteristic curve 10 around the X_2 axis, the axis overlapped to \vec{B} is generated the primary peripheral surface of the ring tool— S_B . The axial section of the primary peripheral surface of the ring tool, $(C_{\Sigma})_{X_2Y_2Z_2}$ is determined as:

$$(S_B)_{X_2Y_2Z_2} \begin{vmatrix} H = X_2(u); \\ R = \sqrt{Y_2^2(u) + Z_2^2(u)}. \end{cases}$$
(10)

2.1 Profiling of tool in CATIA design environment

An application of the presented algorithm for the profiling of the ring tool for the generation of a trapezoidal thread was presented (see Fig. 2). The 3D method to profile the ring tool—*HSGT* (helical surface generating tool)—is grounded on the generative shape design environment facilities. The worked piece (in fact, the generated surface) is 3D modeled, as it can be observed in Fig. 3. The worked piece reference system, *XYZ*, and the ring tool reference system, $X_2Y_2Z_2$, the last one as Euler system, are created (see Fig. 2).

By giving the "projection" command, the ring tool axis projection onto the Σ surface is realized; thus, the characteristic curve is determined. By subsequently using the "revolve" command, the tool primary peripheral surface (S) results after rotating the characteristic curve around Z_2 axis.

The ring tool axial section is then obtained as an intersection between the surface S and a plain which includes the Z_2 (\vec{A}) axis—by applying the "intersection" command. The coordinates of points that defined the generating profile of the thread and the distance between axes are given in Table 1. In Fig. 3, the 3D model of the

Table 1 Input parameters of the trapezoidal thread

Symbol	Description	Z (mm)	X (mm)
A	Initial point on thread crest	50	20
В	Initial point on thread flank	50	0
С	Initial point on thread bottom	40	-10
D	Final point on thread bottom	40	-20
Ε	Final point on thread flank	50	-30
$S_{\rm e}$	Flute sense	Right	
Pe	Helix pitch	50	
b	Distance between tool's axis and thread axis	100	

thread axial profile was shown. In Fig. 4, the HSGT–visual basic application (VBA) was presented, where the profile's elements and the tool's type are given. Figure 5 presented the relative position of the primary peripheral surface for the ring tool, its axis, the characteristic curve, and the axial section of the surface. Because the axial section of the helical surface has singular points, the points C and D will result in discontinuity points on the surfaces in enveloping (see Fig. 5).

2.2 Numerical results-ring tool for trapezoidal thread

In Table 2 and Fig. 6, we present the coordinates of the characteristic curves and of the axial cross-section of ring tool. The existence of the singular points (see Fig. 5, the points C' and D') leads to intersection points for the characteristic curves on adjacent flanks, which impose to take a decision regarding the form of the axial section. The simplest solution is to eliminate the portions from the characteristic curve, in points C' and D', if there are acceptable modifications of the generated surfaces at the thread bottom. In Fig. 6, the intersection zone of the profiles that forms the axial section and eliminates further in the profile board construction for the ring tool was shown.



Fig. 3 Generating profile of the trapezoidal thread

			Thread			
- Threa	ad Param	eters -	- Ring Tool			_
zA	50	zB	50	zC		40
xA	20	xВ	0	xC		-10
zD	40	zE	50	хE		-30
xD	-20	b	100	Pe		50
Ttype Rin	ngTool 🔻		Modify	Se	Right	-

Fig. 4 The HGST-VBA application



Fig. 5 The primary peripheral surface for the ring tool, the axis, the characteristic curve, and the axial section of the surface

2.3 Ring tool for ball thread

Figure 7 showed the model of the ball thread and its axial section (an assembly of circle's arc, filleted), as so as the following reference systems:

is the reference system associated
with the ball thread;
reference system associated with
the ring tangential tool's primary
peripheral surface; and
additional reference systems.

The generation movement assembly, the movements I, II, and III, has the significances given in Fig. 1. In Table 3, the input parameters correlated with the HGST-VBA application. Figures 7 and 8 present the forms and the coordinates of the characteristic curve and the axial section for the helical surfaces assembly that composes the ball thread flute. Obviously, in this case, singular points on the profile do not exist.



Fig. 6 The axial section of the primary peripheral surface for ring tool

3 Tangential tool

The tangential tool is a tool bounded by a revolution primary peripheral surface. The tangential tools are tools designated for the generation of the cylindrical helical surfaces with constant pitch (threads) on specialized machine tools or using specialized technological equipments for longitudinal lather machines. The tangential tool may used on a grinder machine.

3.1 Ring tangential tool: algorithm

The problem of profiling the ring tangential tool, Fig. 9, is, in principle, similar with the known problem of the side mill tool's profiling. In principle, the Nikolaev [5] condition for the determination of the characteristic curve—the tangency curve between a cylindrical helical surface with constant pitch, Σ , and a revolution surface with \vec{A} axis, with position known in the reference system of the Σ surface—is:

$$\left(\vec{A}, \vec{N}_{\Sigma}, \vec{r}_1\right) = 0,\tag{11}$$

Crt. no.	X (mm)	<i>Y</i> (mm)	Z (mm)	Crt. no.	$X (\mathrm{mm})$	Y (mm)	Z (mm)
1	-3.7081	39.8280	-23.2389	26	-1.3370	44.4147	-37.1742
2	-3.4432	40.9627	-22.0605	27	-0.9506	43.3140	-35.9989
3	-3.1696	42.0962	-20.8829	28	-0.5724	42.2112	-34.8229
4	-2.5961	44.3593	-18.5303	29	-0.2025	41.1065	-33.6461
÷	:	÷	:	30	0.1589	39.9999	-32.4685
	<i>R</i> (mm)		H (mm)		<i>R</i> (mm)		$H (\mathrm{mm})$
1	150.0197		12.3447	26	145.7095		37.5908
2	148.9044		13.4541	27	146.8372		38.6877
3	147.7900		14.5646	28	147.9658		39.7835
4	146.6766		15.6760	29	149.0955		40.8784
÷	:		:	30	150.2260		41.9723
	Crt. no. 1 2 3 4 : 1 2 3 4 : 1 2 3 4 : 1 2 3 4 : 1 2 3 4 : 1 2 3 4 : : 1 2 3 4 : : : : : : : : :	Crt. no.X (mm)1 -3.7081 2 -3.4432 3 -3.1696 4 -2.5961 ::R (mm)1150.01972148.90443147.79004146.6766::	Crt. no.X (mm)Y (mm)1 -3.7081 39.8280 2 -3.4432 40.9627 3 -3.1696 42.0962 4 -2.5961 44.3593 :::R (mm):1 150.0197 2 148.9044 3 147.7900 4 146.6766 ::	Crt. no.X (mm)Y (mm)Z (mm)1 -3.7081 39.8280 -23.2389 2 -3.4432 40.9627 -22.0605 3 -3.1696 42.0962 -20.8829 4 -2.5961 44.3593 -18.5303 ::::R (mm)H (mm)1 150.0197 12.3447 2 148.9044 13.4541 3 147.7900 14.5646 4 146.6766 15.6760 :::	Crt. no.X (mm)Y (mm)Z (mm)Crt. no.1 -3.7081 39.8280 -23.2389 26 2 -3.4432 40.9627 -22.0605 27 3 -3.1696 42.0962 -20.8829 28 4 -2.5961 44.3593 -18.5303 29 ::::30R (mm)H (mm)1 150.0197 12.3447 26 2 148.9044 13.4541 27 3 147.7900 14.5646 28 4 146.6766 15.6760 29 :::: 30	Crt. no.X (mm)Y (mm)Z (mm)Crt. no.X (mm)1 -3.7081 39.8280 -23.2389 26 -1.3370 2 -3.4432 40.9627 -22.0605 27 -0.9506 3 -3.1696 42.0962 -20.8829 28 -0.5724 4 -2.5961 44.3593 -18.5303 29 -0.2025 \vdots \vdots \vdots \vdots 30 0.1589 R (mm)H (mm)R (mm)1 150.0197 12.3447 26 145.7095 2 148.9044 13.4541 27 146.8372 3 147.7900 14.5646 28 147.9658 4 146.6766 15.6760 29 149.0955 \vdots \vdots \vdots 30 150.2260	Crt. no.X (mm)Y (mm)Z (mm)Crt. no.X (mm)Y (mm)1 -3.7081 39.8280 -23.2389 26 -1.3370 44.4147 2 -3.4432 40.9627 -22.0605 27 -0.9506 43.3140 3 -3.1696 42.0962 -20.8829 28 -0.5724 42.2112 4 -2.5961 44.3593 -18.5303 29 -0.2025 41.1065 \vdots \vdots \vdots \vdots 30 0.1589 39.9999 R (mm) H (mm) R (mm)1 150.0197 12.3447 26 145.7095 2 148.9044 13.4541 27 146.8372 3 147.7900 14.5646 28 147.9658 4 146.6766 15.6760 29 149.0955 \vdots \vdots \vdots 30 150.2260

Table 2 Coordinates of points on the characteristic curves and the axial section of ring tool



Fig. 7 Characteristic curve and axial section of the ring tool

where \vec{A} is the versor of the rotation axis of the tool bounded by a revolution surface;

- \vec{N}_{Σ} is the normal at the helical surface; and
- \vec{r}_1 is the vector which link the current point onto the Σ surface with a point of the \vec{A} axis (frequently, the origin of the reference system joined with this axis, here $X_1Y_1Z_1$).

Condition 11 is equivalent with the statement: the characteristic curve of a cylindrical helical surface with constant pitch, Σ , in the rotation motion around a fixed axis, \vec{A} , is composed by all the points belongs to the Σ surface, which represent the projection of the \vec{A} axis to the Σ surface. The specific problem is that the tool's axis position is deferent regarding the position of the side mill, regarding the blank.

The generation process kinematics presumes the following motions:

- I is the rotation motion of the blank,
- II translation motion of the blank correlated with the motion I, and

Table 3 Input parameters of the ball	thread
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Symbol	Description	Value
p	Helical parameter	2.5464 mm
r	Flank radius	5.4 mm
е	Half distance between the centers of circles with radius r	0.155 mm
$D_{\rm j}$	Diameter of centers cylinder of the axial profile	49 mm
h	Distance between the D_j diameter and the center of circle with radius r	0.17 mm
D	External diameter of thread	48 mm
r_0	Fillet radius	1 mm
$S_{\rm e}$	Helix sense	Right
D_{axis}	Distance between axis	150 mm
β	Tool's angle in plane XZ	10°
α	Tool's angle in plane ZX	6.0566°



Fig. 8 Axial section of ring tool

III the rotation movements of the ring tangential tool (the cutting motion).

The assembly of motions I and II defines a helical motion with axis and helical parameter identical with the axis and the helical parameter of the surface to be generated.

They are defined the reference systems:

- *xyz* is the reference system where is defined the helical surface (the *Z* axis is the axis of the helical surface).
- $X_1Y_1Z_1$ reference system joined with the ring tangential tool (the X_1 axis is the axis of the ring tangential tool).

If, in the XYZ reference system, it is defined the Σ helical surface:

$$\Sigma: \overrightarrow{r} = x(u,v) \cdot \overrightarrow{i} + y(u,v) \cdot \overrightarrow{j} + z(u,v) \cdot \overrightarrow{k}$$
(12)

with u and v variable parameters, then, by the coordinates transformation, see Fig. 9,

$$\begin{vmatrix} X_1 \\ Y_1 \\ Z_1 \end{vmatrix} = \begin{vmatrix} \cos \beta & 0 & -\sin \beta \\ 0 & 1 & 0 \\ \sin \beta & 0 & \cos \beta \end{vmatrix} \cdot \begin{vmatrix} x+a \\ y+b \\ z-c \end{vmatrix}$$
(13)

the helical surface Σ refers to the reference system $X_1Y_1Z_1$, by equations:

$$\Sigma \begin{vmatrix} X_{1} = [x(u,v) - a]\cos\beta - [z(u,v) - c]\sin\beta; \\ Y_{1} = y(u,v) - b; \\ Z_{1} = [x(u,v) - a]\sin\beta + [z(u,v) - c]\cos\beta, \end{vmatrix}$$
(14)

with a, b, and c technological constants.



Fig. 9 The generation process kinematics with the ring tangential tool

In the condition for the determination of the characteristic curve 11 defining:

 $\vec{A} = \vec{i}$ the versor of the ring tool;

 \vec{N}_{Σ} the normal to the Σ surface, in the reference system $X_1Y_1Z_1$

$$\vec{r}_1 = X_1(u,v) \cdot \vec{i} + Y_1(u,v) \cdot \vec{j} + Z_1(u,v) \cdot \vec{k},$$
(15)

the current vector on the Σ surface, in the reference system $X_I Y_I Z_I$, Eq. 14.

The Eqs. 11 and 14 assembly represents the characteristic curve, in principle, in form:

$$(C_{\Sigma})_{X_1Y_1Z_1} \begin{vmatrix} X_1 = X_1(u); \\ Y_1 = Y_1(u); \\ Z_1 = Z_1(u). \end{cases}$$
(16)

By revolving, the characteristic curve around the X_1 axis is determined the primary peripheral surface of the ring tangential tool. The constants *a*, *b*, *c*, and β are determined from the condition that the trajectory of the *S* point, corresponding to the external diameter of the Σ surface, to be tangent at the helix (see Figs. 9 and 10).

Also, the projection of the helix corresponding to the R_e blank radius, in the same plane *yz*, is a curve with form:

$$L_{\rm E} \begin{vmatrix} y = R_{\rm e} \sin \varphi; \\ z = p \cdot \varphi, \end{cases}$$
(17)

with φ variable and p helical parameter, $p = \frac{p_e}{2\pi}$.



Fig. 10 The ring tangential tool's axis position

From the condition that the two curves 16 and 17 to be tangents in the *M* point, is determined the equations assembly:

- the condition of common point:

$$R_{\rm S}\cos\theta + b = R_{\rm e}\sin\phi; \tag{18}$$

$$R_S \cos\beta \sin\theta + c = p \cdot \varphi; \tag{19}$$

the condition of common tangent:

$$-R_{\rm S}\,\sin\,\theta = R_{\rm e}\,\cos\,\varphi;\tag{20}$$

$$R_{\rm S}\cos\beta\cos\theta = p; \tag{21}$$

The 18, 19, 20, and 21 equations assembly determine the values b, c, φ , and θ (the linear value a and the angle β have to be accepted from a constructive point of view, $a=R_{\rm e}$).

3.2 Applications

3.2.1 Ring tangential tool for trapezoidal thread

In the following, an application of the proposed algorithm for the determination of the primary peripheral surface of the ring tangential tool, for generation of a trapezoidal thread, with generatrix of helical surface presented in Fig. 3

S.G.S.E VBA - Dis	c, Ge	ear Shap	ed, Shaping	Tool ?	×	
Initial Data ——		x	Y	7		
Thread Ori	qin	0	0	0		
Helix Start Po	oint	50	0	0		
Helix Pi	tch	50	- 	nolv I		
Helix He	ight	70	- Input	Data		
1 1						
		Com	posed Prof			
Tool's Profile		Х	Y	Z		
Line Segment -	Pi	0	40	-10		
Add Profile	Pf	0	50	0		
Edit Profile						
Delete Profile						
Line.1						
Tool's Type			h			
Tan Ring Tool 👻		a ∢ →				
Add Tool	Os	-50	-32	150		
Edit Tool	Ue	0	-18	0		
Delete Tool	Rota	ation Ax	is Distance			
	An	gle: 🗖	Zn rotation	0		
Existent Tools:			2p rotation.	4 >		
Tangential Ring Tool Line.1						
Export Tool's Profile Points 0						
	_					

Fig. 11 HSGT application-ring tangential tool, trapezoidal thread

is presented. The method is the same with those described in paragraph 2.1.

The input data for the profile of the thread, the helix pitch, and the distance between the tool's axis and the thread's axis are inserted in the HSGT application, presented in Fig. 11, according to Table 4. Figure 12 represented the surfaces of the trapezoidal thread's flank, characteristic curves on the thread's flanks, primary peripheral surfaces of the ring tangential tool, and the axial section.

The form of the axial section of the ring tangential tool (the plane X_1Y_1) is represented in Fig. 13. We have to notice that the axial tool's profile is asymmetric. Obviously, in the points *B* and *C* (see Fig. 12) on the composed profile of the tool, emerged discontinuities that may be solved by link this zones and accepting a

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Symbol	Description	Y (mm)	Z (mm)
A	Initial point of thread head	50	20
В	Initial point of thread flank	50	0
С	Initial point of thread bottom	40	-10
D	Final point of thread bottom	40	-20
Ε	Final point of thread flank	50	-30
Se	Helix sense	Right	
p _e	Helix pitch	50 mm	
а	x coordinate of tool's origin	-50 mm	
b	y coordinate of tool's origin	-32 mm	
с	z coordinate of tool's origin	150 mm	
$U_{\rm e}$	Tool's rotation around Y_1 axis	-18°	

 Table 4 Input parameters of the trapezoidal thread (straight lined segments)

deformation of the thread bottom, according to a required target.

3.2.2 Ring tangential tool for ball thread

Figure 14 presented the model of the ball thread and its axial section, an assembly of circle's arc, filleted, as so as the reference systems:

Xyz	is the reference system associated
	with the ball thread;
$X_1 Y_1 Z_1$	reference system associated with
	the ring tangential tool's primary
	peripheral surface; and
$x_0 y_0 z_0$ and $x_0' y_0' z_0'$	additional reference systems (see
	Fig. 15).

The generation movement assembly, the movements I, II, and III, has the significances given in Fig. 9.

In this way, the helix belongs to the ball thread flute and situated onto the cylinder with radius R_e :

$$\begin{aligned} x &= R_{\rm e} \cos \varphi; \\ y &= R_{\rm e} \sin \varphi; \\ z &= -p \varphi, \end{aligned}$$
 (22)

is transferred, by coordinates transformation:

$$\begin{vmatrix} X_1 \\ Y_1 \\ Z_1 \end{vmatrix} = \begin{vmatrix} 1 & 0 & 0 \\ 0 & \cos \beta & \sin \beta \\ 0 & -\sin \beta & \cos \beta \end{vmatrix} \cdot \left[\begin{vmatrix} \cos \alpha & 0 & -\sin \alpha \\ 0 & 1 & 0 \\ \sin \alpha & 0 & \cos \alpha \end{vmatrix} \cdot \begin{vmatrix} R_e \cos \varphi \\ R_e \sin \varphi \\ p\varphi \end{vmatrix} - \begin{vmatrix} 0 \\ R_e \\ 0 \end{vmatrix} \right] - \begin{vmatrix} 0 \\ R_e \\ -R_s \end{vmatrix}$$
(23)



Fig. 12 3D model of the helical surface; 3D model of the ring tangential tool's primary peripheral surface

The helix, in the reference system $X_1Y_1Z_1$, associated with the ring tangential tool with the circle:

$$X_1 = R_S \cos \theta;$$

$$Y_1 = 0;$$

$$Z_1 = R_S \sin \theta,$$

(24)

of the ring tangential tool's primary peripheral surface, allow to determine the parameters: α , β , θ , and φ .

Other solution may be obtained by knowing the angle of helix for the cylinder with radius R_{e} ,

$$\alpha = \operatorname{arctg} \frac{p}{R_{\rm e}} \tag{25}$$

and the normal plane to the helix in the point O_0 (see Fig. 15).

The plane of the circle R_S is revolved around the axis x_0 (x_0), with the angle β determined from constructive point



Fig. 13 Axial section of the ring tangential tool's primary peripheral surface



Fig. 14 Ball thread; axial profile, and reference systems

of view from the condition to avoid the interference between the tool and the opposite flank, see Fig. 14. The x axis is symmetrical with the arcs with radius r. In Table 5, the input parameters correlated with the application HSGT.

In Figs. 16 and 17, we present the forms and the coordinates of the characteristic curve and the axial section for the helical surfaces assembly that composes the ball thread flute. Obviously, in this case, it is not possible to define singular points on the profile.

4 Conclusions

This paper presents algorithms and numerical applications for the profiling of ring tools for the generation of cylindrical helical surfaces with constant pitch, based on the topological representation of the tool's primary peripheral surface. The proposed method uses the capabilities of the CATIA graphical design environment. This method



Fig. 15 Ball thread; axial profile, and reference systems

Table 5 Input parameters of the ball thr
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Symbol	Description	Value
p	Helical parameter	2.5464 mm
r	Flank radius	5.4 mm
е	Half distance between the centers of circles with radius <i>r</i>	0.155 mm
$D_{\rm j}$	Diameter of centers cylinder of the axial profile	49 mm
h	Distance between the D_j diameter and the center of circle with radius r	0.17 mm
D	External diameter of thread	48 mm
r_0	Fillet radius	1 mm
$S_{\rm e}$	Helix sense	Right
Daxis	Distance between axis	150 mm
β	Tool's angle in plane zy	10°
α	Tool's angle in plane xz	6.0566°

allows to determine the composed characteristic curves of the helical surfaces (the case of the helical flutes of the motion threads) as so as, the highlighting of the singular points on the tool's profile, including modalities for the

solving of the inherent discontinuities by the method of virtual extending of the profiles. The results obtained in graphical and numerical form

confirm the method quality. Based on this method, an original software, in VBA, was created.

The profiling of the ring tangential tool is similar to the profiling of the side mill tool. The particular position of tool's axis may limit the length of the machined thread.

The specific application HSGT allows the determination of the characteristic curve (in particular for composed characteristic curves for complex surfaces) and allows the solving of problems due of the singular points. The profile, in the tool's axial section, is rigorously determined in the specifically HSGT application.

The proposed method, developed in the CATIA graphical design environment, for the profiling of the ring tangential tool's primary peripheral surfaces allows determining the characteristic curves and the axial



Fig. 16 Characteristic curve and axial section of ring tangential tool



Fig. 17 Axial section of ring tangential tool

section. The HSGT application is based on the decomposition of the helical movement—the self-generating movement of the surface to be generated—a cylindrical helical surface with constant pitch. They are presented with analytical and graphical solutions for the determination of the constructive parameters of the generating tool. Also, four numerical applications for cylindrical helical surface with constant pitch used in machine part's construction were presented.

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References

- Oancea N (2004) Surface generation through winding. Volume 2. complementary theorems. "Dunărea de Jos" University Publishing House, Galați, ISBN 973-627-106-4, ISBN 973-627-170-6
- Veliko I, Gentcho N (1998) Profiling of rotation tools for forming of helical surfaces. Int J Mach Tool Manuf 38:1125–1148
- Rodin RP (1990) Osnovy proektirovania rezhushchikh instrumentov (Basics of design of Cutting Tools). Vishcha Shkola, Kiev
- Shalamanov VG, Smetanin SD (2007) Shaping of helical surfaces by profiling circles, Russian engineering research, vol. 27, No. 7, pp. 470-473, ISSN 1068-798X
- Litvin FL (1984) Theory of gearing. NASA Scientific and Technical Information Division, Washington, reference publications 1212
- Teodor V (2011) Contributions to the elaboration of a method for profiling tools—tools which generate by enwrapping. Lambert, Saarbrucken, ISBN 978-3-8433-8261-8
- Berbinschi S, Teodor V, Oancea N (2010) Contributions to the elaboration of a graphical method for profiling of tools which generate by enveloping. I. Algorithms, Buletinul Institutului Politehnic din Iaşi, tomul LVI (LX), fasc. 2, secția Construcții de Maşini, ISSN 1011-2855, pp. 41-48
- Berbinschi S, Teodor V, Oancea N (2010) Contributions to the elaboration of a graphical method for profiling of tools which generate by enveloping. II. Applications for rack-gear profiling, Buletinul Institutului Politehnic din Iaşi, tomul LVI (LX), fasc. 2, secția Construcții de Maşini, ISSN 1011-2855, pp. 49-56

- Berbinschi S, Teodor V, Oancea N (2010) Kinematical method for rack-gear tool's profiling in CATIA design environment. Proceedings of the 14 International Conference ModTech, ISSN 2066-3513, pp. 119-122
- Berbinschi S, Teodor V, Oancea N (2010) Comparisons between CAD method and analytical method—rack-gear tool's profiling. The annals of the "Dunărea de Jos" University of

Galați, fasc. V, Technologies in machine building, ISSN 1221-4566, pp. 57-64

- Baicu I, Oancea N (2002) Profilarea sculelor prin modelare solidă, Editura Tehnica-Info, Chişinău, ISBN 9975-63-172-X
- Ivanov V, Nankov G, Kirov V (1998) CAD orientated mathematical model for determination of profile helical surfaces. Int J Mach Tool Manuf 38(8):1001–1015