Chapter 15 Tribological Performance of Surface Textured Automotive Components: A Review



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Abstract Presently, worldwide research is focused on energy efficiency to meet the requirement of greenhouse gas emission and economic reasons. The substantial amount of energy is consumed mainly in industrial, transportation and power generation sectors. The leading source of energy consumption is friction. Friction causes wear of the machine components and their replacement. Therefore, to minimize energy consumption the friction reduction is a foremost objective. Potential mechanisms for the reduction of friction are modified lubricants, coatings, and surface texture. In comparison with other mechanisms, surface texturing is a feasible, promising and well-established technique to improve the tribological performance of machine components for more than two decades. Surface texture decreases an area of contact, act as a reservoir and improves hydrodynamic effect in dry, mixed and hydrodynamic lubrication regime respectively. All of this contributes to reducing the friction coefficient. In addition to this, the surface texture improves the load capacity, dynamic stability and noise intensity of the bearings. It develops additional hydrodynamic effect and minimizing fluid leakage in oil and gas seals. Also, in automobile components such as piston rings, cylinder liner and wet clutch, the texture geometry is considered to be micro-pocket. Lubricant retained in the micro-pocket can be released to surrounding areas of texture to improve the tribological performance. This article outlines the recent advancement in texture design for different automotive components, their mechanisms, key findings and future roadmap. Also, the challenges in the fabrication of surface texture for automotive components are discussed in detail.

Keywords Surface texture · Lubrication regimes · Frction and wear · Micro-pocket

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15.1 Introduction

The growing global population, societal needs, and enlargement in automotive (industrial and transportation) sectors are increasing the energy demand rapidly. In addition, limited oil resource, and environmental pollution are nowadays serious issues. At present, about one-third of the global energy resource is consuming to overcome the friction (Holmberg and Erdemir 2017). Especially in automotive engines, approximately 50–60% frictional loss comes from piston ring-cylinder system (Zhan and Yang 2014). This causes millions of tons of CO_2 emission per year (Braun et al. 2014). Additionally, friction causes wear and tear which leads to failure of the automotive components. Reducing friction in the automotive system can therefore improve the life of components, energy consumption, fuel efficiency, and hence gas emission.

In general, two surfaces in relative motion cause friction. It is desirable in generalized applications such as holding an object, walking on the floor, applying brakes, etc. However, in automotive components such as bearings, piston-cylinder arrangement, etc. friction is undesirable. As friction is associated with wear and lubrication, tribology plays an important role. Tribology is basically a science evolving friction, wear, and lubrication. To improve the tribological performance of automotive components the majority of research consisting of modification in (1) lubrication, and (2) surface engineering as follows (Blau and Qu 2004):

- 1. Lubrication
 - Lubricants
 - Lubricant filtration and supply
- 2. Surface engineering
 - Surface treatment: Coating
 - Surface topography: Surface texturing.

However, the modification in surface engineering specifically surface topography is found to be the most effective and reasonable technique. Surface topography includes surface texturing on the surface. The intentional creation of well defined identical features on the surface is called surface texturing. Surface texture on the surface is not a new concept to improve the tribological characteristics. In 1966, Hamilton et al. (1966) have proposed that surface micro-irregularities can reduce the friction between parallel surfaces. But in the following three decades the little attention was paid by the researchers on the textured surface. The new momentum was gained by Etsion's group in 1996. Since then extensive research on surface texturing is going on. At present, it is a well-established and promising technique to improve the tribological performance of automotive components such as bearings, seals, piston ring, wet clutch, cylinder liner, etc. Adopting surface texture on these components can improve the load carrying capacity, fluid film stiffness, friction, wear, leakage, and noise intensity.

In recent times few review articles on the advancement of surface texture in bearings (Gropper et al. 2016; Wang 2014), piston-cylinder systems and mechanical seals (Ahmed et al. 2016) have been published. The theoretical modeling of these components has been summarized by Etsion (2013). Etsion (2005) reviewed the effect of surface texture on bearing sliders under different lubrication condition. Whereas Shamsul Baharin et al. (2016) have presented the mechanism of surface texture to improve the tribological characteristics at different lubrication regimes. In addition to this, advancement in the fabrication of surface texture is nowadays a hot topic of research (Ibatan et al. 2015; Gachot et al. 2017; Coblas et al. 2015). While special attention is paid on laser surface texturing technique (Arslan et al. 2016; Patel et al. 2018). However, the influence of texture design parameters on the tribological performance of automotive components has not been discussed in detail so far. It is therefore essential to summarize the texture behavior in automotive components that affects energy consumption and environmental pollution. With this consideration, in the present review article, the influence of texture design parameters on the tribological performance of automotive components is presented. The recent advancement in texture design is mainly discussed. Moreover, modern texture fabrication techniques (both conventional and non-conventional), and their pros and cons are discussed. Also, key findings and future roadmap to improve the tribological performance of automotive components is presented.

15.2 Texture Design

Designing of surface texture is an essential part of surface topography. Extensive research work on texture design characteristics is carried out. Still, there is no fixed rule for the selection of texture shape under the given operating condition for a particular application. Though surface texturing has the potential to improve the tribological characteristics, it may show an adverse effect if designed wrong. Therefore, a clear understanding of texture design is necessary. Here, the texture design is majorly classified based on geometrical characteristics and positional features as shown in Fig. 15.1.

15.2.1 Texture Geometry

Texture geometrical characteristics such as shape (top profile and bottom profile), and parameters are discussed in detailed as follows:

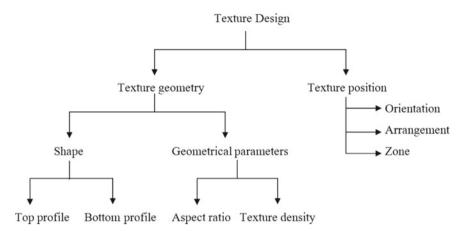
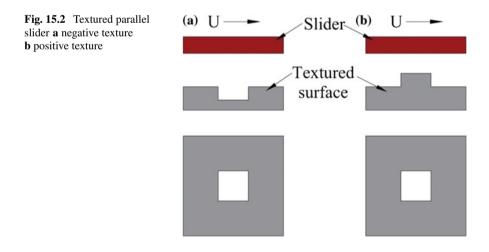


Fig. 15.1 Texture design features

15.2.1.1 Texture Shape

Both negative (recessed) and positive (protruded) texture shapes can be generated as shown in Fig. 15.2. While negative texture is majorly accepted due to advantages in terms of lubrication and ease of manufacturing. Under hydrodynamic lubrication, the negative texture shape is better for friction reduction. But from a leakage point of view, the positive texture showed good results (Siripuram and Stephens 2004). Texture shape is characterized by its top and bottom profile.



Top Profile

An extensive research work has been conducted on standard texture profiles. Siripuram and Stephens (2004) considered the standard texture profiles viz. circle, square, diamond (square-oriented), hexagonal, and triangular for studying the effect of geometry on the hydrodynamic lubrication. They found that the friction coefficient is insensitive to these texture profiles.

However, Yu et al. (2010, 2011) investigated that elliptical texture profile gives better performance than square and circular profile for friction reduction. The commonly used texture top profiles are shown in Fig. 15.3.

Motivated by triangular profile, Uddin and Liu (2016) have observed that star like (series of triangular spikes around the center) texture profile performs better than standard top profiles both in terms of film pressure and friction coefficient. Under mixed lubrication condition, Ren et al. (2007) developed a numerical model for non-standard texture profiles: triangular grooves, sinusoidal waves, fishbone with grooves, fishbone with dimples, and honeycombs. The short grooves and sinusoidal waves showed the strongest load carrying capacity. Galda et al. (2009) have investigated the tribological performance of newly developed texture profiles viz. short drop, and a long drop. They found that spherical and long drop shapes are superior to short drop profile. For the maximization of load capacity and minimization of friction coefficient in grease lubricated textured bearing, Yu et al. (2016) have

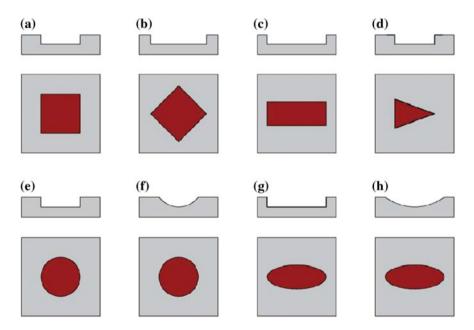


Fig. 15.3 Standard textured profiles a square b diamond c rectangle d triangle e cylinder f sphere g elliptical h ellipsoidal

found that triangular profile performs better than the sphere, spheroid, and T shaped grooved profile. In micro-turbine bearings, Bompos et al. (2012) adopted the egg-shaped texture profile. In comparison with smooth bearing a substantial improvement in hydrodynamic characteristics was observed for textured bearing. For the gas lubricated parallel slider, Qiu et al. (2012, 2013) have found that texture shape with a round edge (sphere, ellipsoidal and circle) performs better than straight-edged profiles (triangle and chevron) to improve the hydrodynamic pressure, friction coefficient, and bearing stiffness. From the above discussion, it is observed that modified (non-standard) texture profiles have better tribological characteristics than predefined simple geometries.

To further improve the results, the texture profile needs to be optimum. Rahmani et al. (2010a), Rahmani and Rahnejat (2018) have optimized the standard texture profiles: square, rectangle and isosceles triangle using an analytical approach for maximization of load capacity and minimization of friction coefficient. For the maximization of load capacity, Fesanghary and Khonsari (2012) have optimized the hydrodynamic film in sectorial-shape thrust bearing using sequential quadratic programming technique. The optimally designed bearing showed more than 100% increase in load capacity than the conventional bearing. Fesanghary and Khonsari (2013) developed the optimization model for grooved texture. The obtained "heartlike" texture profile was validated experimentally. Shen and Khonsari (2015) have presented the numerical optimization of texture top profile using sequential quadratic programming (SOP) for maximization of load capacity. Under unidirectional and bidirectional sliding, the optimum texture profiles were obtained as chevron-type and trapezoid-like respectively. Also, Zhang et al. (2017) have optimized the irregular texture shape for minimum friction coefficient and maximum load capacity using a genetic algorithm (GA) optimization technique. The optimized fusiform texture profile showed a better result than elliptical and circular profiles. Wang et al. (2018a) presented the numerical optimization of chevron-shaped grooved texture to maximize the load capacity of thrust bearing using hybrid (GA-SQP) technique. Experimentally they confirmed that optimum chevron-shaped grooved texture reduces the friction coefficient and temperature rise of the specimen. For gas mechanical seals, Wang et al. (2018b) have modified the spiral grooved texture using multi-objective optimization. The optimized grooved profile showed better result than conventional spiral grooved texture. From the above literature, it is confirmed that the optimized non-standard texture profiles performs better than simple profiles.

Bottom Profile

Texture bottom profile also significantly influences the tribological characteristics. The commonly used texture bottom profiles are flat, curved, and asymmetric wedge (see Fig. 15.4). With the cavitation effect, Nanbu et al. (2008) have analyzed the influence of wedge-shaped texture bottom profiles on the hydrodynamic characteristics. They found that texture bottom profile strongly influences the pressure distribution and film thickness. Shen and Khonsari (2013) have observed that cavitation occurs

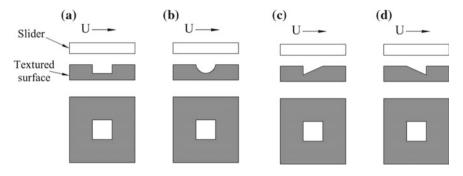


Fig. 15.4 Commonly used texture bottom profiles a flat b curved c convergent wedge d divergent wedge

in the divergent region of the texture, this leads to an asymmetric distribution of pressure. For this mechanism, the load capacity generated by the rectangular bottom profile is larger than the triangular (oblique and isosceles) bottom profiles. Wang et al. (2018c) compared the four different bottom profiles of grooved texture. They observed that the location of cavitation region, the volume fraction of vapor phase and the shape of vortex are influenced by the bottom shape of the groove texture in thrust bearing.

Meng et al. (2015) have observed that the flat (rectangle) and curved (spherical) bottom profile are the most commonly used texture bottom profile to improve the tribological characteristics. For taking the advantages of both rectangle and spherical profiles, they combined these profiles to form the structure characterized by two-layer pores defined as compound texture. In which, first and second layers are rectangle and spherical, respectively. Compound texture showed improved load carrying capacity and friction coefficient than smooth and simple (rectangle) textured hydrodynamic journal bearing due to its twice hydrodynamic effect. Further, they developed the compound groove textures that reduce the acoustic power level of the journal bearing than that of smooth and simple grooved bearings (Meng and Zhang 2018).

Without a cavitation effect, Han et al. (2011) have found that divergence and convergence clearance in the symmetrical texture bottom profile is balanced. However, an asymmetrical bottom profile showed more potential to enhance hydrodynamic characteristics. Similarly, Schuh and Ewoldt (2016) investigated that asymmetric bottom profile generates the normal force, and reduces the shear stress and friction coefficient. Further, Lee et al. (2017) have obtained the significant improvement in hydrodynamic performance with arbitrarily shaped texture bottom profile. Adopting an optimization approach, Lin et al. (2017) have optimized the arbitrary texture bottom profile using sequential linear programming (SQL) for increasing normal force and decreasing friction force. Wang et al. (2017) used the hybrid (GA-SQP) optimization technique for the grooved bottom profile. The optimized profile showed better load capacity than common profiles such as micro wedge and step.

From the above discussion, it is observed that asymmetrical bottom profiles perform better than symmetrical bottom profiles. For top profile also it is confirmed that the modified profile is preferable over simple profiles. However, the simultaneous modification of both top and bottom profile using the optimization technique can appreciably improve the result is still lagging. This approach can be adopted in different machine components including automotive components.

15.2.1.2 Texture Parameter

The surface texture has a certain size along the length, width, and depth. But it is essential to form parameters that include all geometrical aspect of the texture. For this, the non-dimensional parameters: texture density and the aspect ratio are widely used in the literature. These texture parameters are sensitive to tribological characteristics in most automotive components (Gropper et al. 2016).

Aspect Ratio

Aspect ratio is defined as the ratio of texture depth (h_b) and the characteristics length (*l*) (see Fig. 15.5).

Aspect ratio,
$$Ar = \frac{h_b}{l}$$
 (15.1)

Aspect ratio is described by (i) varying the texture depth and keeping length constant and (ii) varying texture length and keeping depth constant. The former way to define the aspect ratio is directly related to depth, and hence more popular. In some literatures, the aspect ratio is also named as depth ratio.

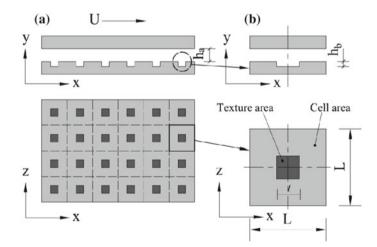


Fig. 15.5 a Textured parallel slider and b square textured unit cell

Texture Density

Texture density is defined as the ratio of texture area (l^2) and the cell area (L^2) as shown in Fig. 15.5.

Texture density,
$$T_d = \frac{l^2}{L^2}$$
 (15.2)

For entire surface, the texture density is expressed as the N times texture area to the total surface area (see Fig. 15.5a). In some literature texture density is also known to be area ratio.

15.2.2 Texture Position

Although texture shape and its geometrical characteristics influence the tribological characteristics, the texture distribution pattern influences the result. This is because the volume and position of lubricating oil accumulate generates the hydrodynamic effect.

15.2.2.1 Texture Orientation

Texture shape can be oriented at different angles along the direction of motion. Yu et al. (2010) have investigated that elliptical texture perpendicular to the direction of sliding gives a better result for friction reduction than oriented along the direction of sliding. However, Ismail and Sarangi (2013) considered both positive and negative textures for elliptical and triangular profiles. They found that the angular orientation of negative texture is uncertain on hydrodynamic performance. But the positive textures: 60° triangular orientations. In textured gas seals, Bai et al. (2010) have investigated the influence of elliptical texture orientation on hydrodynamic characteristics such as load capacity and slide leakage. They investigated that texture oriented at 0° and 90° in the direction of flow has less leakage, but the hydrodynamic effect is not significant. However, at 40° – 70° , the significant improvement in load capacity is obtained. But leakage is also found to be high.

15.2.2.2 Texture Arrangement

In most of the surface texture analysis as discussed above, the linear texture arrangement has been adopted. In journal bearing, Lu et al. (2014) have modified the conventional texture arrangement to the phyllotactic patterned. Experimentally they found 20–30% less frictional coefficient than the conventional linear textured arrangement. Braun et al. (2014) have investigated the tribological behavior of sliding pairs with spherical textures arranged in a hexagonal pattern in the mixed lubrication regime of the pin-on-disk experiment resulted in decreasing friction coefficient. Also, Schneider et al. (2017) examined the different texture arrangements viz. hexagonal, cubical and random for friction reduction. In comparison with plane surface, 82% friction reduction occurred in a hexagonal arrangement. For cubical and random arrangement, respectively 50 and 48% reduction in friction coefficient than plane surface was achieved.

15.2.2.3 Texture Zone

Surface texturing on the entire bearing surface does not necessarily improve the hydrodynamic characteristics. Surface texture may show adverse results if designed wrong. Kango et al. (2012) have observed that surface texture at different locations of the bearing surface helps in enhancing bearing the bearing performance. Brizmer and Kligerman (2012) have investigated that texturing on the entire surface is meaningless for both short and infinite long oil lubricated bearings. Kango et al. (2014) have found that surface texture (both dimple and groove) in the converging zone of the bearing is a preferable arrangement. At variable process parameters such as eccentricity ratio, clearance, journal speed and oil viscosity, Shinde and Pawar (2017a) found that grooved texture in 90°-180° zone gives maximum load capacity. However, the minimum frictional torque is achieved at 90°–360° texture zone. But for multi-objective optimization, textured zone 90°-175° showed appropriate result (Shinde and Pawar 2017b). Similarly, Zhang et al. (2018a) have done optimization of texture position in the convergent zone of the journal bearing. The optimized semi-elliptical shaped texture arrangement showed better bearing load and friction coefficient than half-textured and plane surfaces. They also optimized the texture position numerically in bearing sliders for maximization of load capacity and minimization of friction coefficient. Experimentally they verified that optimized texture position shows best performance than plane, fully textured, and conventionally partial textured surfaces (Zhang et al. 2018b). Contrarily, Tala-Ighil et al. (2011) have investigated that surface texturing in the divergent zone $(185^{\circ}-230^{\circ})$ improves the bearing performance. Further, they observed found that texture arrangement is sensitive to journal speed (Tala-Ighil and Fillon 2015). At lower operating speed, 0°-180° textured position showed best result. Upon further increasing the speed, 185°-230° texture zone obtained best performance. Also in piston ring, partial texturing shows better results than full texturing Kligerman et al. (2005). Etison and Sher (2009) have found that partial laser texture (symmetrical type) has obtained about 4% reductions in fuel consumption. However, it is found to be insignificant for the effect of exhaust gas. Also, noticeable difference in engine oil temperature and crank case pressure is observed than plane surface.

From the above discussion, it is clear that texture shape, geometrical parameters, position, zone, etc. have significant influence on tribological characteristics. But the

detailed analysis by considering every parameter is time-consuming. Also, it is difficult to predict the exact behavior of texture design parameters on output responses. However, response surface methodology has the potential to address this issue. A large number of parameters can be considered in a single design model for different output responses. This approach can significantly reduce the computation time and experimental cost. Fu and Untaroiu (2017) have investigated the tribological performance of surface textured thrust bearing using a design of experiment followed by multi-objective optimization for optimal partially texture geometry with an elliptical dimple. The output responses aimed to maximize the load capacity and minimize the friction torque. The geometrical characteristics such as length of major and minor axes, dimple depth, radial and circumferential space between two dimples, and the radial and circumferential extent were parameterized and analyzed using response surface methodology based central composite design of experiment technique. Using this, the proper combinations of parameter were obtained. Further, they (Fu and Untaroiu 2018) compare the rectangle and elliptical texture using a design of experiment. Using the parametric study, a model was developed. Based on this, they found that elliptical textures have more potential to improve tribological characteristics than rectangular texture.

15.3 Surface Texturing in Automotive Components

Surface texture has significant influence on tribological performance of automotive components. In fluid film bearings, surface texture increases the film thickness. The increase in film thickness at the convergent zone generates additional hydrodynamic pressure to further separate the surfaces. This resulted in increasing load capacity and decreasing local shear stress and hence friction (Gropper et al. 2016). Surface texture on one of the mating faces in circumferential gas seal generates considerable hydrodynamic effect which maintains the small clearance between shaft and seal ring. This prevents the rubbing and hence decreases in friction and wear (Kligerman and Etsion 2008). To improve the lubricant quantity in automobile components such as piston rings and wet clutch, the texture geometry is act as micro-pocket. Lubricant retained in the micro-pocket can be released to surrounding areas of texture to improve the tribological performance (Kligerman et al. 2005; Yagi et al. 2015). Adopting surface texture on the rake face of the cutting tool decreases the tool-chip contact area. Also, the debris trapped in the texture and reduction of stiction at the tool-chip interface takes place. This result in a decrease in friction, wear and hence lesser heat generated in the cutting tool (Jesudass Thomas and Kalaichelvan 2018). In case of an artificial implant, adopting surface texture decrease surface contact under boundary lubrication and acts as a lubricant reservoir in elastohydrodynamic lubrication (Ghosh and Abanteriba 2016). Surface texture has widespread applications in automotive components such as piston ring, cylinder liner, clutches etc. as discussed in detail as follows.

15.3.1 Cylinder Liner

A cylinder liner is serves as the inner wall of cylinder to be fitted into an engine block. It forms sliding surface for the piston ring while retaining lubricant within. The main function of cylinder liner is to control the friction loss between piston ring and cylinder liner, wear of piston ring and cylinder liner, consumption of lubricant, emission of gases and fuel efficiency. To improve this, one of the feasible and possible techniques is to modify topography of cylinder liner by honing scratches/grooves. Dimkovski et al. (2012) have optimized the cylinder liner surface finish for base honing pressure and plateau honing time. But, axial scratches occurred due to abrasive wear are undesirable. Also, they studied on undesirable scratches on the liner which increases the oil consumption and gas emission (Dimkovski et al. 2011). However, Guo et al. (2013) have investigated that the lubrication performance of the cylinder liner is sensitive to surface texture. Grabon et al. (2017) have considered different surface textures were of cylinder liner. They found that increasing honing angle of cylinder liner causes higher frictional resistance. However, at lower honing angle the reduction of liner height was significant. Kim et al. (2018) have found that valley generated by honing functions is undesirable for the formation of dynamic pressure on interacting surfaces. The different mark of plateau honing was compared for friction and wear with those of randomly ground surfaces. They found that deep-grooved honing marks produce larger amounts of wear than shallow-grooved honing. From the tests with the deep-grooved honing marks, it was found that the severe interactions due to asperity contacts and the formation of relatively thin films produced larger amounts of wear volumes than the test with the shallow-grooved ones.

15.3.2 Wet Clutch

In automatic transmission of automobile wet clutches are used. In wet clutch, paperbased friction materials are used as sliding member. It requires high and stable friction at variable speed, wear resistant and heat resistant. To achieve this, researchers have worked on material characteristics or modifications of paper-based wet clutch such as optimization of material composition, different additives in the automatic transmission of fluid and pore characteristics of the paper (Kitahara and Matsumoto 1994; Matsumoto 1996; Kugimiya 2000). However, limited work has been done on surface modification of paper to improve the tribological characteristics. Yagi et al. (2015) formed microscale texture in the form of microgrooves at variable depth and pitch on the paper-based friction material using laser surface texturing. At low temperature and low surface pressure conditions, the dynamic friction coefficient was improved due to microgroove. Also, they found that the dynamic friction coefficient is depends upon depth and pitch of the microgrooves. It can be controlled by properly designing of microgroove pattern. However, adopting microgrooves was found to be insignificant to improve static friction coefficient.

15.3.3 Piston Ring

Piston ring is the major source of engine friction. Reducing friction occurred by piston ring is an effective way to decreases fuel consumption and hence exhaust gas emission. To improve the tribological characteristics of the piston ring by adopting surface texture is largely studied by Etison and co-authors (Arslan et al. 2016; Ronen et al. 2001; Ryk et al. 2002). They found that laser surface texturing is feasible and most promising technique to improve the tribological characteristics of the piston ring. Kligerman et al. (2005) developed the analytical model for partial laser surface textured piston ring for friction reduction. This is verified experimentally by Ryk et al. (2005). For symmetrical type partial laser texture, Etsion and Sher (2009) have found that approximately 4% reduction in fuel consumption is obtained than smooth surface. However, the effect on exhaust gas is found to be insignificant. Ahmed et al. (2016) reviews the effect of texture geometrical parameters on piston ring. They found that (i) aspect ratio is the most significant parameter to improve the tribological characteristics of piston ring (ii) for better performance maximum possible texture density should be selected. (iii) Under starved lubrication condition higher texture depth shows detrimental results. Also, optimum texture depth is depends on the operating conditions. (iv) Partial texturing shows better result for friction reduction than full texturing. Recently, Bifeng et al. (2018) have proposed a novel texturing scheme on piston ring by modifying barrel-shaped ring scheme in the form of variable texture depth. They found that average friction power for novel texture pattern is 10.04% lesser than normal texturing. While, in comparison with barrel shape, it has obtained 16.85% lesser average friction power.

15.3.4 Engine Bearings

The plane and textured hydrodynamic journal bearing as shown in Fig. 15.6 is well established. Although, limited research is carried out on engine bearings with texture effect. Surface texture adopted in engine bearings can decrease the friction coefficient, noise intensity, fuel consumption and increase in load carrying capacity. This ultimately reduces energy consumption to cause a decrease in fuel consumption and engine emissions (Ligier and Noel 2015).

In modern times, surface texture analysis of tires in automobiles can be studied to control the friction for safe driving at various road conditions. Furthermore, surface texture can also be adopted in automotive components such as piston pin, cam and follower, brake to improve the performance.

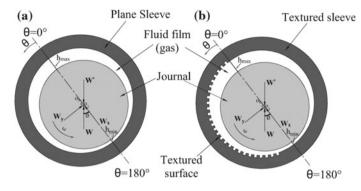


Fig. 15.6 Hydrodynamic lubricated a plane bearing and b textured bearing

15.4 Texture Fabrication Techniques

Texture quality is largely affected by fabricating technique (Ibatan et al. 2015). Surface textures can be fabricated by several conventional and non-conventional machining processes (Ahmed et al. 2016). It is therefore essential to have a detailed discussion on each fabricating technique along with pros and cons.

The conventional machining processes such as vibro-rolling and abrasive machining can be used to create grooves (Rahmani et al. 2010b). While embossing technique is capable of generating surface texture on plastically deformable materials (Pettersson and Jacobson 2006). But this technique is not suitable for hard and brittle materials. Jesudass Thomas and Kalaichelvan (2018) fabricated the surface texture on the rake face of a single point cutting tool using surface indentation techniques viz. Rockwell hardness tester, Vickers hardness tester, and diamond dresser. They found that surface texturing using Vickers hardness tester has obtained better tribological performance than other techniques. Also, standard texture profiles can be generated using micro-grinding process. It is a simple, productive and inexpensive technique to create surface texture on the surface (Stepien 2008). Moreover, Cannon and King (2009) fabricated the surface-texture using micro-casting which is a relatively new technique. This technique can create holes of about 10-100 µm diameter and aspect ratio of 2:1. A vibromechanical texturing technique which is based on single point turning process can produce surface texture. In the conventional turning process, the controlled motion of cutting tool along the X-axis (rotation about spindle axis) and Yaxis (across the length of the workpiece) can generate continuous spiral grooves along the circumference of the workpiece. Adopting controlled motion (tunable oscillation of the tool path) along the Z-axis, a variety of texture sizes can be machined. This method is cost-effective, produces less surface damage, and is very precise (Greco et al. 2009). To investigate the influence of texture bottom profile in thrust bearing, Schuh and Ewoldt (2016) developed the asymmetric-depth-profile angled along the normal surface. The asymmetric depth was achieved by tilting the workpiece with respect to the end mill axis of rotation. Overall it is observed that surface texture using conventional/traditional machining techniques are easier to fabricate. However, its accuracy is affected.

Presently, non-conventional machining processes are most widely used to fabricate surface texture. Wakuda et al. (2003) have considered abrasive jet machining to fabricate the micro-texture. The textured surface showed an improved friction coefficient. However, the bottom profile was affected. Different forms of etching can also generate texture on the surface. Lu and Khonsari (2007) generated cylindrical and elliptical texture shapes on the surface of the journal bearing using chemical etching. The process includes imprinting of the pattern followed by etching. The texture depth was controlled by etching time. Although etching is flexible to generate complex texture geometries, it is a time-consuming process. Moreover, the profile features of the texture such as round shape cannot be controlled. The other etching techniques such as reactive ion etching, UV lithography, photo-etching, physical vapour deposition and UV photo-lithography can also produce a variety of texture shapes (Rahmani et al. 2010b). In electrochemical machining (ECM) process, the material removal is taking place by electrolysis process. ECM generates a very smooth surface without heat affected layer with higher efficiency and at low cost. These advantages attract the researchers to fabricate surface texture using this technique. Lu et al. (2014) developed the phyllotactic patterned surface texture on hydrodynamic lubricated bearing surface using ECM. In comparison with conventional linear textured arrangement the significant improvement in friction reduction (about 20-30%) was obtained. Using ultrasonic machining (USM), different texture shapes such as circle, rectangle, and spherical texture shapes can be produced with high accuracy. Shin et al. (2015) have created the surface texture on bearing steel surface using USM. They found that the reduction in friction coefficient is taken place at appropriate density and depth. But this technique is limited to brittle material and it is not cost effective. Electric discharge machining (EDM) is based on the mechanism: melting and vaporization of the material. Using this technique the complex texture geometry with smooth surface finish can be achieved with no mechanical stress on the workpiece (Pal and Choudhury 2016). However, the material removal rate is low and heat affected zone influences the tribological performance. Yamaguchi et al. (2016) developed the whirling electrical discharge texturing (WEDT) to create microtextures on the inner surface of a cylinder using single-pulse discharge. They confirmed that it is an effective technique to reduce the friction coefficient. Moreover, they found that based on lubrication condition, the optimal texture-area ratio exists. These techniques can be employed for surface texturing but laser surface texturing (LST) is the most efficient technique (Ahmed et al. 2016). LST is a promising technique based on its flexibility and speed (Arslan et al. 2016). It is clean to the environment and provides excellent control of texture size. Earlier, LST was unable to produce complex texture geometry. To achieve this goal, the direct laser interference patterning technique is employed which use multiple laser beam interference. However, burrs or bulges are created after machining is a limitation of this process.

15.5 Concluding Remarks

In the present research review first the texture design is presented in details for automotive components. Then fabrication techniques for automotive components are discussed in details. It is concluded that,

- Texture design parameters have a significant influence on the tribological performance of automotive components.
- To improve the tribological performance, the parametric analysis of texture design parameter should be carried out for qualitative analysis. This significantly reduces the computation time and cost of the experiment without sacrificing the accuracy.
- Optimized texture parameters perform better than simple/standard texture shapes to improve the tribological characteristics of automotive components. However, the optimum process parameters depend on operating conditions and specific application.
- In the automotive system, majority of the research is focused on adopting surface texture on piston-ring cylinder. It is essential to investigate the tribological performance of surface textured crank case bearings, wet clutch, piston pin, cam and follower and tires,
- The conventional machining processes are cheaper and easier to generate the texture. But they are less accurate and it is very difficult to produce complex texture. While a variety of non-conventional texturing techniques can easily generate complex texture geometries with accuracy. Especially laser surface texturing is most famous, widely used and it provides excellent control on texture size.

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