Knowledge-Based System for Automatic Design of Bending Dies

Deepak Panghal, Shailendra Kumar and Hussein M.A. Hussein

1 Introduction

Sheet metal parts are widely used in automobiles, electrical and electronic equipments, computer hardware, home and office furniture, kitchen utensils, and other similar products. Sheet metal operations are used to produce these parts. Sheet metal operations are performed using punch and die setup, commonly known as die or press tool, in a press machine. A die has a complex structure consisting of several components. Die block, punch(es), stripper plate, punch plate, back plate, die-set, etc., are some major components of a die. To manufacture good quality parts, die design is an important activity in tool design department of sheet metal industries.

Bending shapes are obtained using bending process. Bending is one of the most common sheet metal forming processes. It is used to form the various shapes like L, U, V, and Z from sheet metal blanks. It improves the stiffness of blank by increasing its moment of inertia. Bending process has wide applications in electrical, electronic, automobile, aircraft, aerospace, pharmaceutical, refrigeration, and air-conditioning industries. Figure 1 shows a typical bending process. The downward movement of punch applies force on the blank which results in bending. Thereafter, the punch moves back in upward direction and the bent part is extracted from the die. The blank undergoes compressive stress on the concave side and

D. Panghal (\boxtimes)

S. Kumar

H.M.A. Hussein Mechanical Engineering Department, Helwan University, Cairo, Egypt e-mail: hmahuss@hotmail.com

National Institute of Fashion Technology, New Delhi, India e-mail: panghaldeepak@gmail.com

Mechanical Engineering Department, S.V. National Institute of Technology, Surat, Gujarat, India e-mail: skbudhwar@med.svnit.ac.in

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tensile stress on convex side during bending operation. The inner radius of bent part depends on the radius of die opening.

Bending process is performed using a suitable type of bending die mounted in a press machine. Bending dies are generally classified into three types (Boljanovic 2004):

- (i) Wiping die,
- (ii) U-bending die, and
- (iii) V-bending die.

Wiping die is used to perform edge bending as shown in Fig. 1. U-bending die is used to produce U shaped bending parts. Similarly, V-bending die is used to form V-shaped bending parts. The sheet metal blank is bent between a V-shaped punch and die in V-bending die.

1.1 Design of Bending Dies

Die design is a highly specialized area and therefore usually performed by experienced and highly skilled die designers in sheet metal industries (Singh and Rao 1997; Kumar and Singh 2004, 2007; Lin and Hsu 2008; Narayanan 2010). In die design process, die designer needs to recognize design features of sheet metal part, check the features from manufacturability point of view, determine process planning parameters, select proper size of die components, and lastly prepare the drawings of die components and die assembly. Manual process of design of bending die is complex, time consuming, error prone, and skill-intensive (Cheok and Nee 1998a, b; Kumar and Singh 2011). Nowadays, some commercial softwares (AutoCAD, Inventor, CATIA, Pro-E, IDEA, Solid Edge, Unigraphics NX, CADCEUS, etc.) are being used in sheet metal industries for die design. These softwares assist the die designer in simple calculations, drafting, and storage and retrieval of data and drawings. But these systems are not capable to automate the die design process and need experienced persons to operate and interpret the results. Few software packages are developed specifically for the design of bending dies. For example, a dedicated tool "act/unfold" for unfolding of bending parts is developed by M/s Alma CAM of USA. The tool is unfolding software that uses table approach as well as neutral axis method. Since the tool is independently developed so integration approach is missing. M/s Autodesk's Inventor software has a sheet metal modeling environment in which sheet metal bending parts can be modeled. M/s Dassualt System's software CATIA has a tool named as generative sheet metal Design. The main features of the software are associative and dedicated sheet metal feature-based modeling, concurrent engineering between the unfolded or folded part representations, access to predeveloped standards tables, dedicated drawing capability including unfolded view and specific setting. The specialized CAD software "MetaBEND" is a product of M/s Metamation of United States. Key features of the CAD system are bend sequencing and collision checking optimized tool selection.

1.2 Knowledge-Based System

Knowledge-based system (KBS) is an important technique of artificial intelligence (AI) (Poole and Mackworth 2010). It is an intelligent computer program that uses knowledge and inference procedures to solve problems that are difficult enough to require significant human expertise for their solution (Feigenbaum 1981). As depicted in Fig. 2, a KBS has three main elements—knowledge base, inference mechanism, and user interface. The knowledge base contains domain knowledge, which may be expressed as any combination of IF-THEN rules, factual statements, frames, objects, procedures and cases. The inference mechanism allows manipulating the stored knowledge for solving problems. User interface is a platform of interaction between user and system. The use of KBS is proliferating in many domains, where their applications are proving to be critical in the process of



decision support and problem solving. KBS has wide applications in engineering domain such as state transition analysis, production planning, advisory system, electronic power planning, automobile process planning, system development, knowledge verification/validation, knowledge base maintenance, scheduling strategy, communication system fault diagnosis, material processing design, resource utilization, probabilistic fault diagnosis, load scheduling, geoscience, sensor control, etc. (Kusiak 1990; Meziane et al. 2000; Liao 2005).

2 Literature Review

In the past sheet metal forming was heavily dependent on skills of workers. However, with increased production demands, this procedure was replaced by automated stamping, which today is one of the most widely used manufacturing process to plastically deform sheet metals into desired shapes. With the advancement in the area of CAD/CAM and artificial intelligence (AI), some researchers (Karima and Richardson 1987; Shpitalni and Saddan 1994; Cheok and Foong 1996; Huang et al. 1996; Singh and Sekhon 1998; Duflou et al. 1999; Cakir et al. 2005; Kim et al. 2006; Kumar and Singh 2011) started to exploit these techniques for the design of stamping dies.

Major published work related to the applications of CAD and KBS in the area of process planning of bending parts, and bending die design is discussed in this section.

2.1 Process Planning of Bending Parts

Worldwide researchers have applied research efforts to ease the complexity of process planning of sheet metal parts. For example, Hoffmann et al. (1992) developed a bending sequence generator and tool selection as part of a computerized process planning tool labeled as MANICAP. Shpitalni and Saddan (1994) proposed a technique for automatic determination of bending sequences in sheet metal parts. In this technique, the problem of automatic tool selection and bending sequence determination are formulated as a graph search problem. Vin et al. (1994) reported a bend sequencing approach which provides emphasis on the accuracy aspect. They used integrated approach of bend sequencing with other process planning parameters like planning and selection of tools, and blank size. Huang and Leu (1998) studied the effects of process variables in V-bending of steel sheet. Gupta et al. (1998) developed a process planning system for sheet metal parts for generation of possible bending sequences and manufacturing costs, selection of punches and dies, interference checking, gripper selection and robot motion plan, and near optimal plan development. Duflou et al. (1999) derived an algorithm for design verification and automatic process planning for bent sheet metal parts. Later

they (Duflou et al. 2005) presented a tool selection methodology to be integrated in the automatic bend sequencing system. Shigeru and Koguchi (2002) proposed a method, which generates bending sequence for a sheet metal part handled by a robot. Rico et al. (2003) developed a system to obtain valid bending sequences automatically according to the possible tool-part collisions and tolerances. Sousa et al. (2006) presented an optimization method applied to the design of V and U-bending sheet metal processes. Mkaddem and Saidane (2007) examined springback in wiping die bending process using experimental approach and response surface methodology (RSM). Bozdemir and Golcu (2008) defined the springback angle with minimum error using the best reliable ANN training algorithm. Fu et al. (2010) optimized the process of multiple-step incremental air-bending forming of sheet metal by using genetic algorithm-back propagation neural network prediction and finite-element model simulation. Kontolatis and Vosniakos (2010) applied sheet metal bending processes in a multitude of mechanical parts. Thipprakmas and Phanitwong (2011) described process parameters of spring back in V-bending process using Taguchi technique. Lin (2012) proposed an approach of bending sequence for process planning of bending parts.

2.2 Bending Die Design

Since its emergence in the 1950s, AI has provided several techniques that have been used in efforts to solve some complex and narrow domain industrial problems. The KBS/expert system (ES), neural networks (NN), fuzzy logic (FL), multi agents (MA), genetic algorithms (GA), and simulated annealing (SA) are important AI techniques. Among these techniques, KBS is one of the most promising developments (Ismail et al. 1995). The KBS approach is being used in sheet metal industries for manufacturability assessment, process planning, strip-layout design, die design, die material selection, etc., Karima and Richardson (1987) conceptualized application of KBS in sheet metal forming industry. The system supports in converting the knowledge acquired from relevant sources into facts, procedures, judgment and control. Karima (1989) proposed a hybrid system for process planning in sheet metal forming. System considers the overall process of stamping engineering from the micro and the macro perspectives. Shpitalni and Saddan (1994) presented an application "BEND" of the graph search method for the generation of bending sequences. Lin and Chang (1996) developed an expert system for the selection of sheet metal bending tool. Ong et al. (1997) reported to develop a fuzzy set system to determine the sequence of bending operations in press brake machines. Gupta et al. (1998) proposed a process planning system for sheet metal parts. They applied a greedy algorithm to determine the bending sequence. Duflou et al. (1999) suggested a penalty function method and the traveling salesman problem method to determine bending sequences in press brake machines. Cakir et al. (2005) proposed an expert system for die and mold making operations. System requires the input from the user for part geometry and material, tool condition, and operation type. Kim et al. (2006) developed a fuzzy

set theory method for determining the sequence of bending operations. Farsi and Arezoo (2009) proposed a procedure for bend sequencing determination with Fuzzy sets. Kumar and Singh (2011) developed an intelligent progressive die design system named as "INTPDIE" using KBS approach. Naranje and Kumar (2014) proposed a KBS for automatic design of deep drawing dies. Nasr et al. (2014) reported an expert system for design of sheet metal blanking dies.

The reviewed literature reveals the growing interest of worldwide researchers and technocrats in the area of computer aided die design for sheet metal work. Commercial CAD softwares being used in sheet metal industries assist die designer to reduce the design time to some extent and hence increase productivity. But these are unable to fully automate the die design process. Further, die designer needs to use multiple software packages for various activities of die design process and faces problems pertaining to nonexistence of connectivity between these softwares. Also these systems are costly and hence unaffordable for small scale industries, especially in developing countries. Further well trained, competent and experienced persons are required to operate these softwares and interpret the results. Some researchers have applied efforts to develop KBSs for die design. But very less effort is applied to automate the design of bending dies. Further, the systems reported in the literature for bending dies are not able to accomplish all design tasks and not fully automatic. Also domain experts are required to operate and accomplish the die design process.

The present chapter describes a KBS developed for automatic design of bending dies.

3 Proposed KBS for Automatic Design of Bending Dies

As the design task of bending die involves various activities, therefore, the proposed system namely ASDBD (Automatic System for Design of Bending Dies) has been organized into various subsystems and modules as depicted in Fig. 3. The subsystems and modules are briefly described as under (Panghal and Kumar 2013).

Fig. 3 Organization of the proposed system	Automati	c System for Design (ASDBD)	of Bending Die
	РРВР	BDCOMP	AUTOBDMOD
	SPRINGBABALOWBSIZEBSEQBFORCE	 BDBLOCK STPP PUNCH PPBPLT DSET FASTNR 	 DBMOD STPPMOD PUNCHMOD PPMOD BPMOD DSETMOD DBAMOD DTAMOD

3.1 Subsystem PPBP

The subsystem namely PPBP (*P*rocess *P*lanning of *B*ending *P*arts) is developed for process planning of sheet metal parts produced on bending die. System is structured in form of five modules as listed below.

- (i) Module SPRINGBA for determination of springback angle,
- (ii) Module BALOW for calculation of bend allowance,
- (iii) Module BSIZE for determination of blank size,
- (iv) Module BSEQ for determination of bending sequence, and
- (v) Module BFORCE for calculation of bend force.

(i) Module SPRINGBA

This module is developed for determination of springback angle. The Springback angle depends on sheet material, sheet thickness, and bend radius (Boljanovic 2004). It decreases with decrease in bend radius, sheet thickness, hardness of sheet material, and bend angle. There are certain parameters like punch radius, die radius, and clearance between punch and die which are used to control springback angle. Most of the die designers in sheet metal industries determine springback angle using their own thumb rules. After detailed discussion with domain experts, production rules are framed to develop KBS module for determining springback angle. The knowledge base of this module consists of more than 200 production rules coded in VB 6.0 language. A sample of production rules incorporated in the module is given in Table 1. The execution of this module is shown in Fig. 4. Module takes its inputs automatically from part data files PART.DAT and FE.DAT (generated during execution of computer aided system for feature extraction as discussed in Chap. 3). Finally it displays springback angle and stores this output in a data file namely SBA.DAT.

(ii) Module BALOW

Bend allowance is length of arc of neutral axis of a bend angle (Fig. 5). For a larger bend radius neutral axis is located approximately at the half of sheet thickness of bending part. The die designer needs to determine bend allowance for bending parts before designing the die (Boljanovic 2004). The general equation for the bend allowance (B.A.) at the neutral axis is given as:

$$B.A. = \frac{\pi\varphi}{180}R_n \tag{1}$$

where

 φ = bend angle R_n = bend allowance radius at neutral axis (Fig. 4)

S. No.	IF	THEN
1	Material = Steel Annealed; and $R_f/t \le 1$; and $1.5 <$ Sheet thickness in mm ≤ 2.0 ; and Bend angle $\le 15^{\circ}$	Springback angle $(\Delta \alpha) = 0.5^{\circ}$
2	Material = Steel Annealed; and $R_f/t \le 2$; and $1.5 <$ Sheet thickness in mm ≤ 2.0 ; and $15^\circ \le$ Bend angle $\le 30^\circ$	Springback angle $(\Delta \alpha) = 0.7^{\circ}$
3	Material = Steel Low Carbon Hardened; and $R_f/t \le 2$; and 1.0 < Sheet thickness in mm ≤ 1.5 ; and $45^\circ \le$ Bend angle $(\varphi_f) \le 60^\circ$	Springback angle $(\Delta \alpha) = 2.2^{\circ}$
4	Material = Steel Low Carbon Hardened; and $R_f/t \le 2$; and 1.0 < Sheet thickness in mm ≤ 1.5 ; and $60^\circ \le$ Bend angle $(\varphi_f) \le 90^\circ$	Springback angle $(\Delta \alpha) = 2.7^{\circ}$
5	Material = Cu (ETP); and $R_f/t \le 2$; and 2.0 \le Sheet thickness in mm ≤ 2.0 ; and Bend angle (φ_f) $\le 15^\circ$	Springback angle $(\Delta \alpha) = 0.75^{\circ}$
6	Material = Cu (ETP); and $R_f/t \le 2$; and 2.0 \le Sheet thickness in mm ≤ 2.0 ; and 15° $<$ Bend angle (φ_f) $\le 30^\circ$	Springback angle $(\Delta \alpha) = 1.0^{\circ}$
7	Material = Cu (ETP); and $R_f/t \le 2$; and 2.0 \le Sheet thickness in mm \le 2.0; and 30° $<$ Bend angle (φ_f) \le 45°	Springback angle $(\Delta \alpha) = 1.3^{\circ}$

Table 1 A sample of production rule for SPRINGBA







Fig. 5 Bend terminology

Bend allowance radius
$$(R_n) = \xi \times t + R_i$$
 (2)

where

 ξ = bend factor R_i = inner radius

Therefore, from Eqs. (1) and (2),

$$B.A. = \frac{\pi\varphi}{180} (\xi t + R_i) \tag{3}$$

The value of bend factor ($\underline{\zeta}$) corresponding to R_i/t ratio is given in Table 2 (Boljanovic 2004).

In the proposed system, a data base file for values of ' ζ ' with respect to R_i/t ratio is constructed to determine bend allowance. The execution of the module BALOW is shown in Fig. 6. The module recalls the data file FE.DAT (as discussed in Chap. 3) to read sheet thickness. Then the system searches the value of ζ corresponding to R_i/t ratio in the data file. The intermediate values of ' ζ ', which are not available in the database are determined by interpolation method. Finally, the system calculates bend allowance and displays it for the user. The value of bend allowance is stored automatically in a data file namely BALOW.DAT which is used further for determining blank size.

(iii) Module BSIZE

Blank size of a bending part depends on part shape (bend width, length of legs, bend angle, and bend radius) and bend allowance. A module namely BSIZE is developed for determination of blank size of part. Execution of the proposed module is depicted in Fig. 7. The module recalls data file FE.DAT, to read part

Table 2 Values of ξ corresponding to R_i/t

R_i/t	00.1	00.2	00.3	00.4	00.5	00.8	11.0	11.5	22.0	33.0	44.0	55.0	110.0
ξ	00.23	00.29	00.32	00.35	00.37	00.40	00.41	00.44	00.45	00.46	00.47	00.48	00.5



Fig. 6 Execution of module BALOW



Fig. 7 Execution of module BSIZE

detail. Bend allowance is taken from the data file BALOW.DAT generated during execution of previous module. Next, module unfolds all bends one by one using built-in API (application programming interface) namely 'UNFOLD' of Inventor software for determination of blank size. This API uses the matrix transformation approach for unfolding the bends. Finally module displays the three-dimensional (3-D) and two-dimensional (2-D) drawings of blank of bending part for the user. The 3-D drawing of blank is stored in a drawing file labeled as BSIZE.IPT.

(iv) Module BSEQ

For determination of bending sequence, a module namely BSEQ (*Bending Sequence*) is developed (Panghal et al. 2016). There are certain process variables like bend angle, position of a bend, angle with feed direction, etc., that play a vital role in deciding bend sequence. By considering these parameters, a procedure is developed to determine the bend sequence of a complex bending part. The steps to determine bend sequence are briefly described as under.

(a) Identification of base/mother plane

The base plane is most important plane in a bending part. All the bends are formed around it as it remains stationary during bending operation. Base plane is identified on the basis of following criteria:

- It must be centrally located, i.e., base plane must be surrounded by maximum number of bends, and It must be the largest plane.
- If more than one plane qualifies on the basis of above criteria, then selection of base plane is done on the basis of minimum distance between the centers of plane to the farthest bend axis among the qualifying planes.

To implement the above criteria, all planes of bending part are sorted area wise. The plane of highest area is awarded highest marks (i.e., 100 marks) and other planes are awarded marks as per their proportionate area with reference to the plane having highest marks. Thereafter, the planes are sorted as per number of surrounding planes. The plane with highest number of surrounded planes is awarded highest marks and other planes are awarded marks as per their proportionate number of surrounding planes. The plane having highest total marks (i.e., out of 200 marks) awarded as per the above two criteria is selected as base/mother plane. In case of tie of marks among the planes, then selection of base plane is done on the basis of minimum distance between the centers of plane to the farthest bend axis among the qualifying planes. The proposed procedure for identification of base plane is shown in Fig. 8.

(b) Categorization of Bends

Bends are required to be categorized in such a manner that they have highest probability of getting grouped together. Feed direction plays a vital role in categorization of bends. Bends can be categorized on the basis of bend axis position with reference to the feed direction, i.e., parallel, perpendicular, or inclined.



Fig. 8 Procedure for selection of base/mother plane





(c) Grouping of Bends

The bends from each category is required to be grouped together. The purpose of the grouping is to identify the bends which can be formed together. There are many





variables on the basis of which bends can be grouped together. After detailed discussion with the domain experts and review of published literature, the following rules have been identified which have much impact on grouping of bends.

Rule1: Bends whose bend axis is coplanar and having the same direction (either up or down) can be grouped in the same group.

For example, in a typical bending part shown in Fig. 9 bends B1 and B2 are grouped in same group.

Rule 2: Bends that are equal number of bends away from the base plane and also have the same direction (either up or down) can be grouped in the same group.

For example, in a typical bending part as shown in Fig. 10, bends B1 and B2 should be grouped in the same group.

The above two rules are applied to all categories of bends individually. The procedure for grouping of bends is shown in Fig. 11.

(d) Prioritization of Bend Groups

There are number of variables that contribute in prioritization of groups. On reviewing published literature and detailed discussion with domain experts, the rules that are most significant in deciding the prioritization of groups are identified. All these rules do not have equal impact on deciding the sequence of groups. A significance factor for each rule is identified. The groups are awarded marks as per rules and further multiplied by individual significance factor. The total marks for individual groups are calculated. The groups are further sorted in descending order as per total marks obtained to decide the sequence of groups. The rules with their significance factor are tabulated in Table 3.

The above procedure is coded in VB 6.0 programing language. User can interact with the system through GUI. The system is parametric and automatic in nature. Execution of the proposed module is depicted in Fig. 12. The module reads 3-D drawing of blank from the file BSIZE.IPT generated during execution of BSIZE module. The module also recalls data file FE.DAT to take inputs such as sheet thickness, number of bends, bend angle, and bend direction. Thereafter, module sequentially performs the activities includes identification of base plane, bend categorisation, bend grouping, bend group prioritization, and finally displays the bending sequence through 3-D drawings of bending part. The output of the module in form of group sequence (along with detail of bends in each group) is stored in a data file namely BSEQ.DAT.



Fig. 11 Procedure for grouping of bends

 Table 3 Rules for deciding prioritization of groups

Rule	Description	Significance factor
R1	The distance in terms of number of planes in between a bend and base plane have the maximum impact on bend sequencing. Bend having maximum number of planes away from the base plane should be addressed earlier	0.6
R2	The group having more number of bends, later it should be addressed	0.4
R3	The bend angle more than 90° should be addressed in two stages	0.3
R4	The bends that has perpendicular bend axis to feed direction should be addressed first. Thereafter, inclined and in the last bend with parallel bend axis should be considered	0.1



Fig. 12 Execution of module BSEQ



Fig. 13 Schematic diagram for bending force calculation (Boljanovic 2004)

(v) Module BFORCE

Bending force depends on the strength of sheet material, size and thickness of bending part, and clearance between punch and die. Bending force is required to be calculated for different types of bends. A schematic diagram for bending force calculation is depicted in Fig. 13.

The formula used for force calculation is as follows:

$$Force(F) = K \times Swt^2/L$$
(4)

where

- K bend Constant (values are listed in Table 4)
- *S* Ultimate tensile strength of material
- w Sheet metal width at bend
- t Sheet thickness

$$L = r_1 + c + r_2 \tag{5}$$

where

- r_1 punch corner radius
- c die clearance
- r_2 die corner radius

Table 4	Value of constant	S. No.
K (Boljand	ovic 2004)	1

S. No.	Bending type	Κ
1	Wiping	0.333
2	U-die	0.667
3	V-die	1.33

Fig. 14 Execution of module BFORCE





Fig. 15 Execution of subsystem PPBP

A module namely BFORCE is developed to calculate the bend force required to bend a part. The execution of this module is shown in Fig. 14. Module BFORCE reads essential input data from data files PART.DAT and FE.DAT. Then the module calculates the bend force and displays it for the user.

The Execution of subsystem PPBP is shown in Fig. 15. System initially takes input from PART.DAT file. Further, the required inputs related to design features of bending part are taken from another data file namely FE.DAT. The output of SPRINGBA module in the form of springback angle is stored in the data file namely SBA.DAT. The module namely BALOW is capable to determine bend allowance for all bend and the output of it is stored in the data file BA.DAT. The next module namely BSIZE is developed for determination of blank size of bend

part. Bend allowance is taken from the data file BA.DAT generated during execution of previous module. The module recalls a built-in API (application programming interface) namely "UNFOLD" of Inventor software for determination of blank size. This API uses the matrix transformation approach for unfolding the bends. Finally module displays the three-dimensional (3-D) and two-dimensional (2-D) drawings of blank of bending part for the user. The outputs of the module stores in data file BSIZE.DAT. Next module namely BSEO is used to determine the bend sequencing of bend part. The module reads 3-D drawing of blank from the file BSIZE.DAT generated during execution of BSIZE module. The module also recalls data file FE.DAT to take inputs such as sheet thickness, number of bends, bend angle, and bend direction. Thereafter module sequentially performs the activities including identification of base plane, bends categorization, bend grouping, bend group prioritization, and finally displays the bending sequence through 3-D drawings of bending part. The output of the module in form of group sequence (along with detail of bends in each group) is stored in a data file namely BSEQ. DAT. Finally, a module namely BFORCE is developed for determining the bend force. A module namely BFORCE developed to calculate the bend force required to bend a part. Module BFORCE reads essential input data from data files PART.DAT and BFE.DAT. The module calculates the bend force and displays it for the user.

3.2 Subsystem BDCOMP

A subsystem namely BDCOMP(*Bending Die Components*) is developed for selection of type and size of bending die components. It is structured in form of six KBS modules namely:

- (i) Module BDBLOCK for selection of die block,
- (ii) Module STPP for selection of stripper plate/pressure plate,
- (iii) Module PUNCH for selection of punch(es),
- (iv) Module PPBPLT for selection of punch plate and back plate,
- (v) Module DSET for selection of die-set, and
- (vi) Module FASTNR for selection of fasteners (bolts and dowels).

The knowledge base of proposed subsystem consists of more than 700 production rules coded in VB 6.0 language. A sample of production rules is given in Table 5. Execution of the system is shown in Fig. 16. For required inputs, the system reads data files PART.DAT and BSEQ.DAT generated during execution of process planning subsystem. Various modules of the system impart expert advices for selection of bending die components including die block, stripper plate/pressure plate, punch(es), punch plate, back plate, die-set, and fasteners. The module namely BDBLOCK imparts expert advice on the selection of die block. The outputs of modules are stored in different data files which can be recalled for automatic die modeling.

1	Sheet thickness (mm) > 0.5; and Sheet thickness (mm) \leq 1.6; and Die material = Tool steel [SERVERKER 21]	Select die block thickness (mm) = 24.0
2	Sheet thickness (mm) > 1.6; and Sheet thickness (mm) \leq 3.2; and Die material = Tool steel [SERVERKER 21]	Select die block thickness (mm) = 29.0
3	Sheet thickness in mm > 3.2; and Sheet thickness in mm \leq 4.5; and Die material = Tool steel [SERVERKER 21]	Select thickness of die block (mm) = 35.0
4	Size of blank length (mm) \ge 150.0; and Size of blank length (mm) \le 280; and Sheet thickness (mm) \ge 6.4; and Die material = Tool steel [SERVERKER 21]	Select width of die block (mm) = (blank width + 72.0); and Select length of die block (mm) = (blank length + 100.0)
5	Type of the die = wiping; and Sheet thickness in mm > 2.5; and Sheet thickness in mm \leq 3.0; and Bending force \leq 100,000 N	Select stripper plate thickness in mm = 25; Width and length of stripper plate in mm = length and width of die block; and Select four medium pressure springs with 50 % deflection; free length of each spring in mm = 25; preload of spring in mm = 3.75; outer diameter of spring in mm = 12.0
6	Type of the die = wiping; and Sheet thickness in mm \ge 3.0; and Bending force \le 100,000 N	Select stripper plate thickness in mm = 28; Width and length of stripper plate in mm = length and width of die block; and Select four medium pressure springs with 50 % deflection; free length of each spring in mm = 25; preload of spring in mm = 3.75; outer diameter of spring in mm = 12.0
7	Type of die = wiping die; and Sheet thickness in mm > 2.5; and Sheet thickness in mm \leq 3; and Bending force > 10,000 N; and Bending force \leq 15,000	Select punch type = wiping; and Punch length in mm = 70; and Punch width in mm = (bend width + 4.0); and Punch thickness in mm = 30; and Punch corner radius in mm = two times of sheet thickness
8	Type of die = wiping die; and Sheet thickness in mm > 2.5; and Sheet thickness in mm \leq 3; and Bending force > 15,000 N	Select punch type = wiping; and Punch length in mm = 70; and Punch width in mm = (bend width + 4.0); and Punch thickness in mm = 40; and Punch corner radius in mm = two times of sheet thickness
9	Punch thickness in mm > 30; and Sheet material = steel annealed hardened	Select thickness of punch plate in mm = 30; and Length of punch plate = die block length; and Width of punch plate = die block width

Table 5 A sample of production rules incorporated in subsystem BDCOMP

(continued)

10	Sheet thickness in $mm < 3.0$; and Sheet material = steel annealed hardened	Select thickness of back plate in mm = 9.5; and Back plate length = die block length; and Back plate width = die block width
11	Production quantity > 30,000; and Production quantity \leq 45,000; and Length of die in mm > 200; and Length of die in mm \leq 250; and Width of die in mm > 80; and Width of die in mm \leq 100; and Dimensional tolerance in mm < 0.1	Select center pillar die-set with dimensions in mm as: Thickness of punch holder plate = 25; Thickness of die holder plate = 32; Diameter of pillar = 32; Length of die-set = (length of die + 140); Width of die-set = (width of die +140); and Shank diameter = 24
12	Production quantity > 30,000; and Production quantity \leq 45,000; and Length of die in mm > 250; and Length of die in mm \leq 315; and Width of die in mm > 100; and Width of die in mm \leq 125; and Dimensional tolerance in mm < 0.1	Select center pillar die-set with dimensions in mm as - Thickness of punch holder plate = 32; Thickness of die holder plate = 40; Diameter of pillar = 32; Length of die-set = (length of die + 140); Width of die-set = (width of die +140); and Shank diameter = 24
13	Die area > 5000.0 mm ² ; and Die area $\leq 10,000.0$ mm ² ; and Bending force $\leq 50T$	Use four Allen bolts of size M6 and two dowels of diameter 6 mm
14	Die area > 10,000.0 mm ² ; and Die area \leq 15,000.0 mm ² ; and Bending force \leq 50T	Use four Allen bolts of size M8 and two dowels of diameter 8 mm

Table 5 (continued)

3.3 Subsystem AUTOBDMOD

A subsystem labeled as AUTOBDMOD (*Automatic Bending Die Modeling*) is developed for automatic modeling of die components and die assembly of bending die. This system works in conjunction with KBS modules developed for selection of die components of bending die. The outputs of various modules of subsystem BDCOMP are stored in different data files. These data files are recalled automatically for die modeling. The users need not to enter any input to the system. For development of the proposed subsystem, Visual Basic (VB 6.0) program is written to invoke commands of Inventor software such as EXTRUDE, DRAFT, CHAMFER, etc., for 3-D modeling of die components and die assembly. To generate 2-D drawings, a computer code in VB 6.0 language is written to open 3-D files into drawing view of Inventor software. The developed computer code also ensures the proper scaling and automatic dimensioning of 2-D drawings. Graphic user interface (GUI) of the proposed system is developed using VB 6.0 and is interfaced with Inventor software. The subsystem AUTOBDMOD is structured into eight modules namely:



Fig. 16 Execution of system BDCOMP

- (i) Module DBMOD for modeling of die block,
- (ii) Module STPPMOD for modeling of stripper plate and pressure plate,
- (iii) Module PUNCHMOD for modeling of punch(es),
- (iv) Module PPMOD for modeling of punch plate,
- (v) Module BPMOD for modeling of back plate,
- (vi) Module DSMOD for modeling of die-set,
- (vii) Module DBAMOD for modeling of die bottom assembly, and
- (viii) Module DTAMOD for modeling of die top assembly.

Execution of the proposed subsystem AUTOBDMOD is shown in Fig. 17. The first module namely BDMOD is developed for automatic modeling of die block of bending die. This module takes required inputs from data files DBLOCK.DAT and FASTNR.DAT generated during the execution of KBS modules developed respectively for selection of die block and fasteners. Finally, the module generates 2-D and 3-D drawings of die block in the drawing editor of Inventor software. 3-D drawing is saved automatically as a global block namely GDBMOD. The next module labeled as STPPMOD is constructed for automatic modeling of stripper plate/pressure plate. Inputs in form of size of stripper plate/pressure plate and fasteners are read automatically from the data files namely STPP.DAT and FASTNR.DAT generated during execution of previously developed KBS modules of subsystem BDCOMP. Finally, the module generates 2-D and 3-D drawings of stripper plate/pressure plate. The output in form of 3-D drawing is stored automatically in a global block namely GSTPPMOD. The module namely PUNCHMOD is developed for automatic modeling of punch(es) of bending die. The module takes input automatically from data file PUNCH.DAT. The module generates 2-D and 3-D drawing of punch(s). The 3-D drawing is saved automatically as global block namely GPUNCHMOD. The module labeled as PPMOD is developed to generate drawings of punch plate of bending die. The module takes inputs in the form of size of punch plate from the data file PPBLT.DAT and number and size of fasteners from the data file FASTNR.DAT. Output in form of 3-D drawing of punch plate is saved as a global block namely GPPMOD. The module namely BPMOD takes required inputs from data files PPBLT.DAT and FASTNR.DAT. Finally it generates 3-D and 2-D drawings of back plate. The 3-D drawing of back plate is automatically saved as a global block namely GBPMOD. The module DSMOD is constructed for modeling of die-set of bending die. The module takes required input from data file DSET.DAT and FASTNR.DAT. The 3-D drawing of die-set is automatically saved as global block namely GDSETMOD. The modules labeled as DBAMOD and DTAMOD are developed, respectively for automatic modeling of bottom die assembly and top die assembly of bending die. Bottom die assembly consists of die holder plate of die-set, die block, and pressure plate (in case of U-die only). Top die assembly is a unit of punch holder plate of die-set, back plate, punch plate, punch(es) and stripper plate. The proposed modules compute reference point on die-set for inserting global blocks of respective die components to generate 3-D drawings of bottom die assembly and top die assembly. The 2-D drawings are generated automatically from 3-D files.



Fig. 17 Execution of the proposed system AUTOBDMOD

4 Validation of System ASDBD

The developed system ASDBD has been tested successfully for different sheet metal parts taken from stamping industries. A sample run of the system for one sheet metal part (Fig. 18) of stamping industry namely M/s Kochar Agro Industries Pvt. Ltd.





Faridabad, India is depicted in Figs. 19, 20, 21, 22, 23, 24, 25, 26, 27, 28, 29, 30, 31, 32, 33, 34, 35, 36, 37, 38, 39, 40, 41, 42, 43, 44, 45, 46, 47, 48, 49, 50, 51, 52, 53, 54, 55, 56, 57, 58 and 59. The recommendations obtained by the system were found very similar to those actually used in the sheet metal industries for the example part. In sheet metal industries, experienced die designers use various commercial CAD softwares for drafting of die components. These softwares are not only costly (cost of a single software is more than 4 Lakhs Indian Rupees) but also not capable to automate die design. The developed system executes the tedious task of design of bending die in less than an hour and gives final outputs in form of drawings of die components and die assembly which can be used directly for die manufacturing.

benu						
Feature Extract	ion					
✓ Iner Radiu	s 📝 Outer Radius	Thickness	/ Bend Angle 🗹 B	end Line Width 🔽	Bend Direction	
Bend Number	Inner Radius(mm)	Outer radius(mm)	Bend Angle(degr	Bend Width(mm)	Thickness(mm)	Bend Direction
Bend1	1	2.5	90.00	25.2	1.5	Down
Bend2	1	2.5	90.00	25.2	1.5	Down
Bend3	1	2.5	85.00	11.5	1.5	Down
Bend4	1	2.5	85.00	11.5	1.5	Down



Mate	rial	Steel L	ow Carbon Hardeneo	ł		
Shee	t Thk	1.5				
	Bend	No.	Bend Radius (mm)	Bend Angle (degree)	Spring back Angle (degree)	
•	1		1	90.00	2.7	
	2		1	90.00	2.7	
	3		1	85.00	2.7	
	4		1	85.00	2.7	

Fig. 20 Output of module SPRINGBA

Mate	rial	Steel	Low Carbon Hardened	d		
Shee	t Thk	1.5				
	Bend	No.	Bend Radius (mm)	Bend Angle (degree)	Bend Allowance (mm)	
+	1		1	90.00	2.483	
	2		1	90.00	2.483	
	3		1	85.00	2.345	
	4		1	85.00	2.345	

Fig. 21 Output of module BALOW



Fig. 22 Output of module BSIZE



Fig. 23 Mother/base plane selection

Sr No	BendName	Direction of bend to feed Direction	Bend Direction	Bend Angle (Degrees)	Bend Level			(2) Create	Grouping		
1	Bend1	Perp	Down	90.00	1			(3) Apply	Rule		
2	Bend2	Perp	Down	90.00	1						
3	Bend4	Parallel	Down	85.00	2			(d) Rend	Casuante		
4	Bend3	Parallel	Down	85.00	2			(4) 0010	Jequerce		
								R1 factor 0.6	R2 factor	R3 factor	R4 factor
GroupName	Bend in Group	Bend	Group	Min be ande	nd 81	R2 Rule	R3 Rule	R1 factor 0.6	R2factor 0.4 R4 tule	R3 factor 0.2 Total marks	R4factor 0.1
GroupName Group-1	Bend in Group Bend 1 Bend2	Bend Level	Group Type Perp	Min be angle 50	nd R1	R2 Rule	R3 Rule	R1 factor 0.6	R2 factor 0.4 R4 tule	R3 factor 0.2 Total marks	R4 factor 0.1

Fig. 24 Output of module BSEQ



Fig. 25 Bending sequence

Mater	ial	Steel	Low Carbon	Hardened					
Sheel	Thk	1.5				Calculate Bend F	orce		
	Bend	No.	Bend Name	Bend Radius (mm)	Bend Angle (degree)	Type of Bending		BendWidth (mm)	Bend Force (N
+	1		Bend1	1	90.00	U-die	-	25.2	3,479.34
	2		Bend2	1	90.00	U-die	-	25.2	3,479.34
	3		Bend3	1	85.00	Wiping	-	11.5	792.71
	4		Bend4	1	85.00	Wiping	-	11.5	792.71

Fig. 26 Output of module BFORCE

Even a novice can perform the die design using this system. Therefore, it eliminates dependency on domain experts. The system is implemented on a PC having Inventor software (Cost is approximately 3 Lakhs Indian Rupees). Therefore, its low cost of implementation makes it affordable even for small scale industries.

🖳 Die	Block					
Die Blo	ock Ma	terial SEVERKER	21 🗸			
Shee	t Thick	mess 1.5	[Die Block Details		
			User	Inputs		
		Stage(s)	Blank Length (mm)	Blank Width (mm)	Advice	
	•	Stage 1	120	35	Advice	
		Stage 2	120	35	Advice	
				Please select Lengt Thickness of Die Bl	h of Die Block 168m ock 24 mm.	nm , Width of Die Block 83 mm , and

Fig. 27 Output of module BDBLOCKfor die block (stage 1)

heet Thic	kness 1.5		Die Block Details		
		User	Inputs		
	Stage(s)	Blank Length (mm)	Blank Width (mm)	Advice	
	Stage 1	120	35	Advice	
•	Stage 2	120	35	Advice]
			Please select Length Thickness of Die Blo	of Die Block 168mr ck 24 mm.	m , Width of Die Block 83 mm , and

Fig. 28 Output of module BDBLOCKfor die block (stage 2)

She	et Thickness		Stripper P	late / Pressure Pad de	tai		
	1		Read Force	Type Of bend	Rend internal		
	Stage(s)	BendName	(N)	type	distance	Advice to User	
	Stage 1	.Bend3.Bend4	792.71	Wiping		Advice	
	Stage 2	,Bend1,Bend2	3479.34	U-die	46	Advice	
			select Len thickness the strippe	gth of stripper plate of stripper plate =20 er bolt quantity=4 an	=168mm , width of mm, and select d Springs quantity:	Stripper plate = 83mn =4	n,
			-			F	OK

Fig. 29 Output of module STPP for stripper plate (stage 1)

Shee	et Thickness 1.5						
			Stripper P	late / Pressure Pad de	tail		
	Stage(s)	BendName	Bend Force (N)	Type Of bend	Bend internal distance	Advice to User	
	Stage 1	.Bend3,Bend4	792.71	Wiping		Advice	
	Stage 2	.Bend1.Bend2	3479.34	U-die	46	Advice	
			select Leng thickness of	th of Pressure pad =: f pressure pad =10m	29.2mm , width of m, Springs quantit	Pressure pad = 49m y=1	m,
							ОК

Fig. 30 Output of module STPP for pressure plate (stage 2)

	Punch Deta	ails					
	Stag	e 1					
	e	Bend Radius (mm)	Bend Width (mm)	Bend Force (N)	Type of Bending	U Bend Internal Distance	Advice to User
۲		1	11.5	792.71	Wiping		Advice
		1	11.5	792.71	Wiping		Advice
4			Sel =20	ect Wiping Punch L 0 mm , Punch Radi	.ength =60 mm , F us =3 mm ,	Punch Width=15.5 r	nm , Punch Thickness

Fig. 31 Output of module PUNCH for punch (stage 1)

_	Punch Deta	ails					
	Stag	e 2					
	e	Bend Radius (mm)	Bend Width (mm)	Bend Force (N)	Type of Bending	U Bend Internal Distance	Advice to User
•		1	25.2	3,479.34	U-die	46	Advice
	1	1	25.2	3,479.34	U-die	46	Advice
			Sel =4	ect U-die Punch Le 6 mm - Punch Radi	ngth =60 mm , P	unch Width=29.2 m	m , Punch Thickness

Fig. 32 Output of module PUNCH for punch (stage 2)

Punch	Plate			
eet Th	ickness 1.5			
Punc	h Plate Details			
	Stage No	BendName	Advice to User	
•	Stage 1 Stage 2	.Bend4,Bend3 ,Bend1,Bend2	Advice Advice	
			Select Punch Thk =20mm	Plate Length = 168mm , PunchPlate Width =83mm, Punch Plate
			-	OK

Fig. 33 Output of module PPBPLT for punch plate (stage 1)

Punch	Plate			
eet Th	ickness 1.5			
Punc	ch Plate Details			
	Stage No	BendName	Advice to User	
	Stage 1 Stage 2	.Bend4,Bend3 .Bend1,Bend2	Advice Advice	
			Select Punc Thk =20mm	h Plate Length = 168mm , PunchPlate Width =83mm, Punch Plate 1
	_	_		ОК

Fig. 34 Output of module PPBPLT for punch plate (stage 2)

Punch	Back Plate			
neet Th	ickness 1.5			
Punc	ch Back			
Plate	Uetails			
	Stage No	BendName	Advice to User	
+	Stage 1	.Bend4,Bend3	Advice	
	Stage 2	.Bend1,Bend2	Advice	
			Select Punch Back Pla Punch Back Plate Thio	te Length = 168 mm , Punch Back Plate Width =83 mm, kness =9.5 mm
	_			ОК

Fig. 35 Output of module PPBPLT for back plate (stage 1)

Punch	Back Plate			
eet Thi	ickness 1.5			
Punc	th Back			
Piate	Details			
	Stage No	BendName	Advice to User	
	Stage 1	.Bend4.Bend3	Advice	
	Stage 2	,Bend1,Bend2	Advice	
			Select Punch Back Plate Length = 168 Punch Back Plate Thickness =9.5 mm	mm , Punch Back Plate Width =83 mm,
				ОК

Fig. 36 Output of module PPBPLT for back plate (stage 2)

Die	Set Selection							3 8 9
					deta	ail of die set	Save	7
	Stages	BendName	Die length	Die Width	Type of Die set	Advice to User		
•	1	.Bend4,Bend3	168	83	Diagonal Pillar	Advice		
	2	,Bend1,Bend2	168	83	Diagonal Pillar	Advice		
			S	et Punch Holo Ind diameter Select Punch I Select Die holo Select shank d	der Plate thickness of pillar =25 mm holder plate length der plate length= 2 iameter =24 mm	=25 mm and d a= 268 punch ho 268 die holder p	ie holder plate thic older plate width =: late width =183	kness = 32 mm 183
	-	_						ОК

Fig. 37 Output of module DSET for die-set (stage 1)

Die	Set Selection							
					deta	al of die set	Save	
	Stages	BendName	Die length	Die Width	Type of Die set	Advice to User		
	1	,Bend4,Bend3	168	83	Diagonal Pillar	Advice		
	2	,Bend1,Bend2	168	83	Diagonal Pillar	Advice		
				set Punch Hold and diameter o Select Punch h Select Die hold Select shank dia	er Plate thickness : f pillar =25 mm older plate length= er plate length= 26 ameter =24 mm	=25 mm and die = 268 punch hol 88 die holder pla	holder plate thicl der plate width =1 te width =183	kness = 32 mm 183
	_	_						ОК

Fig. 38 Output of module DSET for die-set (stage 2)

Stages	Bends In stage	Die Block length	Die block Width	Die Area (mm2)	User Advice
1	,Bend4,Bend3	168	83	13,944.00	Advice
2	,Bend1,Bend2	168	83	13,944.00	Advice
		Select 4	nos Allen bolts of s	ize M8 and 2 nos d	owels of diameter 8 m

Fig. 39 Output of module FASTNR for fasteners (stage 1)



Fig. 40 Output of module DBMOD—3-D drawing of die block (stage 1)



	1 1		
	1 1	1111	
	1 1	11 11	8
	1 1	1111	4
	1 1	1111	
	1 1	1 1 11	

Fig. 41 Output of module DBMOD—2-D drawing of die block (stage 1)



Fig. 42 Output of module DBMOD—3-D drawing of die block (stage 2)



	_
	24,0

Fig. 43 Output of module DBMOD—2-D drawing of die block (stage 2)



Fig. 44 Output of module DBMOD—3-D drawing of die block (stage 2)



Fig. 45 Output of module DBMOD—2-D drawing of die block (stage 2)



Fig. 46 Output of module STPPMOD—3-D drawing of stripper plate (stage 1)





Fig. 47 Output of module STPPMOD—2-D drawing of stripper plate (stage 1)





Fig. 49 Output of module PUNCHMOD—2-D drawing of punch (stage 1)





Fig. 51 Output of module PUNCHMOD—2-D drawing of punch (stage 2)



Fig. 52 Output of module PPMOD—3-D drawing of punch plate (stage 1)



Fig. 53 Output of module PPMOD—2-D drawing of punch plate (stage 1)



Fig. 54 Output of module BPMOD—3-D drawing of back plate (stage 1)



Fig. 55 Output of module BPMOD—2-D drawing of back plate (stage 1)



Fig. 56 Output of module DSETMOD—3-D drawing of punch holder plate (stage 1)



Fig. 57 Output of module DSETMOD-2-D drawing of punch holder plate (stage 1)



Fig. 58 Output of module DSETMOD—3-D drawing of die holder plate (stage 1)



Fig. 59 Output of module DSETMOD-2-D drawing of die holder plate (stage 1)

5 Conclusions

The developed system automates all major activities of design of bending dies including process planning, selection of type and size of die components, and modeling of die components and die assembly. The system takes most of the required inputs directly from the output data files generated during execution of CAD system of automatic feature extraction and KBS of manufacturability assessment as discussed in Chap. 3. Thereafter, the system modules select various process parameters like bend allowance, part blank size, bend sequencing, and bend force. Next, the system selects the type and size of various die components including die block, stripper plate/pressure plate, punch(es), punch plate, back plate, die-set, and fasteners. Finally, the system generates 2-D and 3-D drawings of all major die components, and bottom and top die assembly of bending die. These drawings can be directly used for die manufacturing. The developed system has been tested successfully on various types of sheet metal parts taken from stamping industries. The system is integrated and parametric in nature.

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