

# Chapter 6

## Heat and Mass Transfer of a Rotating Disk for Large Prandtl and Schmidt Numbers

### 6.1 Laminar Flow

Convective heat and mass transfer over a single disk rotating in fluid with high Prandtl or Schmidt numbers can be found in many practical and research applications. For instance, in electrochemistry, where the Schmidt numbers are several orders of magnitude larger than unity, rotating disk electrode is involved in measurements of the convective diffusion coefficient [1–14]. Another example is naphthalene sublimation technique often used to measure mass transfer coefficients  $\alpha_m$  [15–29].

The differential Eq. (1.28) of convective diffusion, including the time-averaged fluctuating components, is analogous to the energy Eq. (2.5), provided that the temperature  $T$  and the thermal diffusivity  $a$  are replaced by the concentration  $C$  and the diffusion coefficient  $D_m$ , respectively. The Navier–Stokes and continuity equations hold, if constant fluid properties are assumed.

If the Schmidt number  $Sc$  replaces the Prandtl number, and the nondimensional function  $\theta$  is written as

$$\theta = (C - C_\infty)/(C_w - C_\infty), \tag{6.1}$$

then the self-similar Eqs. (2.32)–(2.36) for steady-state axisymmetric laminar flow become valid for convective mass transfer.

Surface concentration on the disk does not vary; thus  $C_w = \text{const}$ . Therefore, rewritten convective diffusion Eq. (2.36) reduces to

$$\theta'' - ScH\theta' = 0. \tag{6.2}$$

The following equations (analogous to Eq. (3.4)) can be used for estimation of the Sherwood number

$$Sh = K_1 Re_\omega^{n_R}, \quad Sh_{av} = K_2 Re_\varphi^{n_R}. \quad (6.3)$$

The constants  $K_1$  and  $K_2$  in Eq. (6.3) are affected by the boundary conditions, flow type (laminar, transitional, or turbulent) and the Schmidt numbers. The exponent  $n_R$  is affected by the flow type, whereas  $K_1 = K_2$  and  $n_R = 1/2$  in a laminar flow regime.

Thus, the aforementioned analogy between convective heat and mass transfer, enables the use of theoretical solutions or empirical experimental equations simply via replacing  $C$ ,  $Sc$  and  $Sh$  with of  $T$ ,  $Pr$  and  $Nu$  (or vice versa), accordingly.

For laminar flow, Eqs. (2.32)–(2.36) for  $Pr > 1$  and  $Sc > 1$  at  $N = 0$  and  $\beta = 0$  were solved numerically using Mathcad [30]. Table 6.1 shows that the calculated coefficient  $K_1$  is increasing with growing  $Pr$  or  $Sc$  numbers.

For the same Prandtl number, the constant  $K_1$  is an increasing function of the exponent  $n_*$ : the value of  $K_1$  at  $Pr = 0.71$ , 2.0 and  $10^6$  becomes 3.3, 2.73 and 2.2 times larger, respectively, if the constant  $n_*$  changes from  $-1$  to 3. Thus, at increased Prandtl numbers, the influence of the exponent  $n_*$  on the constant  $K_1$  gets less pronounced.

The approximate Eq. (3.6) for the coefficient  $K_1$  for the boundary condition (2.30),  $Pr \geq 1$  and nonzero values  $n_*$  was derived by Dorfman [31]. Values of  $K_1$  by Eq. (3.6) surpass the exact solution. Equation (3.6) deviates from the exact solution at  $n_* \leq 0$  by 16–40 % even for  $Pr = 1$ . For  $n_* = 0$  and  $Pr = 1-3$ , this deviation reaches 10–11 %. For larger exponents  $n_*$  and  $Pr = 1-3$ , the deviation of

**Table 6.1** Values of the constant  $K_1$ , exact solution of Eqs. (2.32)–(2.36) for  $Pr > 1$  [30]

$Pr$ ( $Sc$ )	$n_* = -2$	$n_* = -1.5$	$n_* = -1$	$n_* = -0.5$	$n_* = 0$	$n_* = 1$	$n_* = 2$	$n_* = 3$	$n_* = 4$
1.0	0.0	0.1305	0.2352	0.3221	0.3963	0.5180	0.6159	0.6982	0.7693
1.5	0.0	0.1682	0.2979	0.4028	0.4906	0.6324	0.7450	0.8389	0.9199
2.0	0.0	0.1989	0.3482	0.4669	0.5653	0.7226	0.8466	0.9498	1.0386
2.28	0.0	0.2140	0.3728	0.4982	0.6016	0.7663	0.8960	1.0036	1.0963
2.5	0.0	0.2251	0.3907	0.5209	0.6280	0.7982	0.9319	1.0428	1.1383
3.0	0.0	0.2480	0.4279	0.5680	0.6826	0.8640	1.0061	1.1238	1.2251
5.0	0.0	0.3206	0.5445	0.7153	0.8533	1.0697	1.2382	1.3774	1.4971
10.0	0.0	0.4410	0.7368	0.9577	1.1341	1.4083	1.6206	1.7957	1.9460
15.0	0.0	0.5254	0.8710	1.1268	1.3300	1.6446	1.8877	2.0880	2.2599
20.0	0.0	0.5924	0.9776	1.2610	1.4855	1.8323	2.0999	2.3203	2.5095
50	0.0	0.8536	1.3925	1.7835	2.0909	2.5635	2.9269	3.2260	3.4825
100	0.0	1.1108	1.8009	2.2979	2.6871	3.2840	3.7422	4.1190	4.4421
500	0.0	1.9943	3.2033	4.0644	4.7351	5.7596	6.5442	7.1888	7.7413
1000	0.0	2.5467	4.0802	5.1691	6.0162	7.3083	8.2972	9.1096	9.8057
$10^4$	0.0	5.6363	8.9846	11.348	13.181	15.971	18.104	19.855	21.356
$10^5$	0.0	12.291	19.548	24.657	28.613	34.632	39.230	43.003	46.236
$10^6$	0.0	26.626	42.304	53.328	61.860	74.834	84.742	92.873	99.838

Eq. (3.6) from the exact solution is smaller (1–6 %). However, at  $Pr \rightarrow \infty$ , the inaccuracy of Eq. (3.6) abruptly increases [30].

The functional dependence of the constant  $K_1$  on the Schmidt (or Prandtl) number according to Dorfman’s Eq. (3.6) and the exact solution for  $n_* = 0$  ( $T_w = \text{const.}$  or  $C_w = \text{const.}$ ) is depicted in Fig. 6.1. The inaccuracies of Eq. (3.6) make it unusable already for  $Sc = 1\text{--}3$ .

Equations (3.7) and (3.8) can be rewritten for mass transfer for  $Sc = 0\text{--}\infty$ , respectively

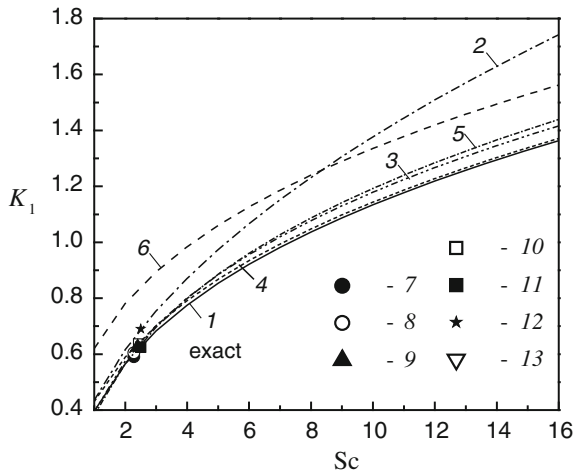
$$K_1 = 0.6109Sc / (0.5301 + 0.3996Sc^{1/2} + Sc)^{2/3}, \tag{6.4}$$

$$K_1 = 0.6Sc / (0.56 + 0.26Sc^{1/2} + Sc)^{2/3}. \tag{6.5}$$

Equations (6.4) and (6.5) result nearly in the same values. Maximal deviation of them from the exact solution is 4 and 5 %, respectively, for  $Sc = 5\text{--}20$  (Fig. 6.1). For higher Schmidt numbers, inaccuracies of Eqs. (6.4) and (6.5) tend to zero (Table 6.2).

Another expression was derived in the work [13]

$$K_1 = 0.621Sc / (1 + 0.298Sc^{-1/3} + 0.14514Sc^{-2/3}). \tag{6.6}$$



**Fig. 6.1** Constant  $K_1$  in Eq. (6.3), laminar flow at  $C_w = \text{const.}$  [30]. 1—Exact solution; 2—Eq. (3.6) for  $n_* = 0$ ; 3—Eqs. (6.4) and (6.5); 4—Eq. (6.6); 5—Eq. (6.7); 6—Eq. (6.8). Experiments: 7— $K_1 = 0.59$ ,  $Sc = 2.28$  [15]; 8— $K_1 = 0.604$ ,  $Sc = 2.28$  [17]; 9— $K_1 = 0.625$ ,  $Sc = 2.4$  [20, 21, 26]; 10— $K_1 = 0.636$ ,  $Sc = 2.44$  [19]; 11— $K_1 = 0.625$ ,  $Sc = 2.5$  [25]; 12— $K_1 = 0.69$ ,  $Sc = 2.5$  [24]; 13— $K_1 = 0.628$ ,  $Sc = 2.5$  [22]

**Table 6.2** Constant  $K_1$  by Eqs. (6.4)–(6.8), a rotating disk for  $C_w = \text{const.}$  or  $T_w = \text{const.}$  [30]

$Pr (Sc)$	Exact	(6.4) [33]	(6.5) [34]	(6.6) [13]	(6.7) [2]	(6.8) [11]
1.0	0.3963	0.3941	0.4025	0.4303	0.3827	0.62
2.0	0.5653	0.5753	0.5864	0.5892	0.5664	0.7812
2.28	0.6016	0.6144	0.6257	0.6238	0.6065	0.816
2.5	0.6280	0.6430	0.6543	0.6491	0.6358	0.8415
5.0	0.8533	0.8839	0.8946	0.8676	0.8855	1.0602
20.0	1.4855	1.5414	1.5414	1.4924	1.5688	1.6829
50	2.0909	2.1552	2.1424	2.0958	2.2019	2.2841
100	2.6871	2.753	2.7278	2.6915	2.8147	2.8778
500	4.7351	4.7885	4.7222	4.7400	4.8890	4.9209
1000	6.0162	6.056	5.9651	6.0218	6.1773	6.2000
$10^4$	13.181	13.126	12.904	13.192	13.3565	13.358
$10^5$	28.613	28.332	27.834	28.639	28.7954	28.7779
$10^6$	61.860	61.074	59.990	61.915	62.0461	62.000

At the expense of a larger deviation from the exact solution for  $Sc = 1$ – $2$  (8 % at  $Sc = 1$  and 4 % already at  $Sc = 2$ ) Eq. (6.6) ensures only deviations of less than 1–3 % at higher Prandtl or Schmidt numbers (Fig. 6.1; Table 6.2).

For  $Sc \rightarrow 0$ , Eqs. (6.4)–(6.6) reduce to the asymptotic relation  $K_1/Sc = 0.885$  [32]. For  $Sc \rightarrow \infty$ , they ensure agreement with another asymptotic  $K_1 = 0.62Sc^{1/3}$  [11, 32].

One more relation for  $K_1$  for  $Pr = 0$ – $\infty$  was designed as a combination of asymptotic solutions for the cases  $Pr \rightarrow 0$  and  $Pr \rightarrow \infty$  [2]. Rewritten, using  $Sc$  number, this results in

$$K_1 = \left[ (0.88447Sc)^{-1.077} + (0.62048Sc^{1/3})^{-1.077} \right]^{-1/1.077}. \quad (6.7)$$

For  $Sc = 2$ , Eq. (6.7) merges with the self-similar solution. Deviation of Eq. (6.7) from the exact solution grows up to 3.2 % at  $Sc = 2.5$ , exhibits a maximum of 5.6 % at  $Sc \approx 20$  and, for larger Schmidt numbers, diminishes to 2.7 % at  $Sc = 1000$  and 0.6 % at  $Sc = 10^5$ .

Over the range of  $Sc < 2$ , deviation of Eq. (6.7) from the exact solution changes its sign, and increases in absolute values being 3.4 % at  $Sc = 1$  and 8.2 % at  $Sc = 0.1$  (Table 6.2).

To conclude, preference should be rendered to that of Eqs. (6.4)–(6.7) that ensures the lowest inaccuracy at the Schmidt numbers specific for the problem is to be solved.

**Application to electrochemistry problems.** Levich [11] derived an asymptotic solution for convective diffusion for very large Schmidt numbers  $Sc \gg 1$

$$K_1 = 0.62Sc^{1/3}. \quad (6.8)$$

It coincides with the asymptotic solution for heat transfer for  $Pr \gg 1$  given in [32].

Table 6.2 elucidates that Eq. (6.8) correlates well with the exact solution at  $Sc > 500$  (deviation for  $Sc = 500$  is 3.9 % and reduces to zero for  $Sc \rightarrow \infty$ ). Equation (6.8) overruns the exact solution by 7.1 % at  $Sc = 100$  and by 56.7 % at  $Sc = 1$ . Levich's Eq. (6.8) was successfully validated in experimental studies [4, 5, 7, 8, 12, 14] at high Schmidt numbers.

Rotating disk electrodes are intensively employed in experimental electrochemical investigations [1, 11]. Convective diffusion, which displays itself as the diffusion of the electrical current on the electrode, is modeled by Eq. (1.28).

For this case, Eq. (6.3) for laminar flow in view of Eq. (6.8) is usually rewritten as [1, 11]

$$i_L = 0.62nFC_F C_\infty D_m^{2/3} \nu^{-1/6} \omega^{1/2}, \quad (6.9)$$

where  $i_L$  is the limiting diffusion current of electrons to the surface of a rotating disk electrode;  $n$  is the number of electrons that are involved in the current;  $F$  is the disk area;  $C_F$  is the Faraday constant (96,485 C/mol);  $C_\infty$  is the concentration at infinity, mol/m<sup>3</sup>. Based on this, one can ascertain that the mass transfer coefficient can be written as  $\alpha_m = i_L / (nFC_F C_0)$ , while Eq. (6.9) translates into Eq. (6.8).

In practice, the following tasks are actual: (1) searching a functional dependence of  $i_L$  on  $\omega$ ; (2) finding the diffusion coefficient  $D_m$ , whereas the value of  $i_L$  is measured; and (3) measurements of Volt–Ampere characteristics using a rotating disk electrode.

***Naphthalene sublimation technique for experimental determination of the mass and heat transfer coefficients.*** Convective heat transfer from a surface to air is analogous to convective mass transfer in naphthalene sublimation to air. Naphthalene sublimation has been often employed to measure the average mass transfer of an *entire* disk weighted before and after the measurement to determine the amount of naphthalene lost by the disk as a result of experiments [19–21, 25–27]. Currently, accurate instrumentation is available for local pointwise scanning of the naphthalene layer thickness and subsequent calculation of local mass transfer coefficients for laminar, transitional, and turbulent flow [15–18, 22, 24, 28]. In frames of the analogy between the surface heat and mass transfer, constants  $K_1$  in Eq. (3.4) for the Nusselt number and Eq. (6.3) for the Sherwood number can be expressed as [23]

$$K_1 = C Pr^{m_p}, \quad (6.10)$$

$$K_1 = C Sc^{m_p}, \quad (6.11)$$

where the coefficient  $C$  is identical in both equations. The effects of the Prandtl and Schmidt numbers are described by respective multipliers in Eqs. (6.10) and (6.11).

Equations (6.10) and (6.11) are used for the Prandtl and Schmidt numbers moderately diverging from unity:  $Pr = 0.7$ – $0.74$  for air, whereas  $Sc = 2.28$ – $2.5$  for naphthalene sublimation in air. Therefore, the constant  $C$  is assigned to be equal to the coefficient  $K_1$  at  $Sc = 1$  and  $Pr = 1$  at  $T_w = \text{const.}$  or  $C_w = \text{const.}$  (see Table 6.1), i.e.,  $C = 0.3963$ .

Authors [15–19, 24, 28] used the naphthalene sublimation technique to measure rotating disk mass transfer and set the exponent  $m_p$  to be the same for all values of  $Pr$  and  $Sc$ , which yields a relation between the  $Nu$  and  $Sh$  numbers

$$Nu/Sh = (Pr/Sc)^{m_p}. \quad (6.12)$$

The scatter of the values of the exponent  $m_p$  in the literature amounted up to 45 %:  $m_p = 1/3$  [17],  $m_p = 0.4$  [15, 17, 18, 20],  $m_p = 0.53$  [19], and  $m_p = 0.58$  [24].

Erroneous values  $m_p$  entail fallacious results of post-processing of the experimental data from the naphthalene sublimation technique aimed at estimation of heat transfer in air. An analysis and recommendation of the proper value  $m_p$  were made by the author [23].

Exponent  $m_p$  can be detected from the self-similar solution of the problem (Tables 3.1 and 6.1). Table 6.3 lists exponents  $m_p$  for the Prandtl/Schmidt numbers moderately deviating from unity [23]. It is evident from here that the function  $m_p(Pr)$  exhibits a decreasing trend and varies from  $m_p = 0.5723$  to  $m_p = 0.5024$ , if the Prandtl/Schmidt numbers grow from 0.7 up to 2.5. Consequently, the effective exponent  $m_p = 0.53$  suggested in [19] is practically the average  $m_p$  value weighted over the range  $Pr = 0.7$ – $2.5$ .

Figure 6.1 depicts different experimental data for the constant  $K_1$  in naphthalene sublimation in air. These data agree well with the self-similar solution (see Table 6.2); only the too large value  $K_1 = 0.69$  for  $Sc = 2.5$  [24] falls out from the overall picture.

**Table 6.3** Value  $m_p$  in Eqs. (6.10), (6.11) and (6.12) based on the exact solution of Eqs. (2.32)–(2.36) for laminar flow [23, 30]

$Pr (Sc)$	0.5	0.6	0.7	0.71	0.72	0.8	0.9	0.95	0.99
$m_p$	0.5954	0.5827	0.5723	0.5714	0.5705	0.5638	0.5571	0.5551	0.5632
$Pr (Sc)$	1.05	1.1	1.5	2	2.28	2.4	2.5	3	4
$m_p$	0.5438	0.5424	0.5264	0.5123	0.5064	0.5041	0.5024	0.4949	0.4841
$Pr (Sc)$	5	10	20	50					
$m_p$	0.4765	0.4566	0.4411	0.4251					

Post-processing [23] of the measured results using Eq. (6.12) at  $m_p = 0.53$  to reduce them to conditions of heat transfer at  $Pr = 0.71$  yields the values  $K_1 = 0.325$  [17],  $K_1 = 0.328$  [20, 21, 26],  $K_1 = 0.331$  [19],  $K_1 = 0.321$  [25],  $K_1 = 0.322$  [22] that agree well with the exact value  $K_1 = 0.326$  for  $Pr = 0.71$  and  $T_w = \text{const.}$  (see Table 3.1) and reliable experimental data (see Chap. 3). Falling out of the overall good conformance are (a) the constant  $K_1 = 0.318$  resulting from the low value  $K_1 = 0.59$  in naphthalene sublimation at  $Sc = 2.28$  measured in [15], and (b) the constant  $K_1 = 0.354$  stemming from the high experimental value  $K_1 = 0.69$  in naphthalene sublimation obtained in [24] (see Fig. 6.1).

The use of the value  $m_p = 0.4$  suggested in [15, 17, 18, 20] and widely used throughout the literature brings for the heat transfer in air at  $Pr = 0.71$  [23]:  $K_1 = 0.37$  [15];  $K_1 = 0.379$  [17];  $K_1 = 0.384$  [20, 21, 26];  $K_1 = 0.388$  [19];  $K_1 = 0.378$  [25];  $K_1 = 0.380$  [22];  $K_1 = 0.417$  [24]. All these recalculated data are too large as compared to the exact value  $K_1 = 0.326$ .

Taking the exponent  $m_p = 1/3$  [17], one can obtain for  $Pr = 0.71$  [23] the constants  $K_1 = 0.4$  [15];  $K_1 = 0.409$  [17];  $K_1 = 0.416$  [20, 21, 26];  $K_1 = 0.421$  [19];  $K_1 = 0.411$  [25];  $K_1 = 0.413$  [22];  $K_1 = 0.454$  [24]. They surpass the exact value  $K_1 = 0.326$  to an even larger extent.

Involvement of the exponent  $m_p = 0.58$  [24] yields for  $Pr = 0.71$  the values  $K_1 = 0.3$  [15],  $K_1 = 0.307$  [17],  $K_1 = 0.308$  [20, 21, 26],  $K_1 = 0.311$  [19],  $K_1 = 0.301$  [25],  $K_1 = 0.303$  [22], that are too small [23]. Only the value  $K_1 = 0.332$  [24] is acceptable, which is due to the high value  $m_p = 0.58$  chosen by the authors [24] to agree with the exact solution  $K_1 = 0.326$ . However, it is clear that the too large exponent  $m_p = 0.58$  results from the too large value  $K_1 = 0.69$  in naphthalene sublimation measured in [24], which is discordant with the measurements of the other researchers.

Authors [25] rearranged Eq. (6.12) in the following way

$$Nu/Sh_{Sc=2.5} = f(Pr)Pr^{1/3}. \quad (6.13)$$

The value  $K_1 = 0.625$  at  $Sc = 2.5$  and function  $f(Pr) = 0.576, 0.634, 0.737, 0.842$  and  $0.926$  at  $Pr = 0.1, 1, 2.5, 10$  and  $100$ , respectively, yield jointly the values  $K_1 = 0.321, 0.396, 0.625, 1.134$  and  $2.686$  at the Prandtl numbers mentioned above. This is fully consistent with the self-similar solution at  $T_w = \text{const.}$  (Table 6.1,  $n^* = 0$ ). The correction function  $f(Pr)$  can be recast to incorporate the multiplier  $Pr^{1/3}$ . Use of Eq. (6.13) ensures higher accuracy than that conveyed by approaches operating with a single value of  $m_p$ , though Eq. (6.13) is less practical as Eq. (6.12), because of the involvement of a tabulated function.

To conclude, Eq. (6.12) with the exponent  $m_p = 0.53$  [19] can be suggested as the most accurate and practical one for post-processing of the measured laminar mass transfer coefficients of a rotating disk in naphthalene sublimation in air in order to recalculate it to laminar heat transfer in air. As an alternative, Eq. (6.13) (or its modification) can be used.

## 6.2 Transitional and Turbulent Flow for the Prandtl and Schmidt Numbers Moderately Different from Unity

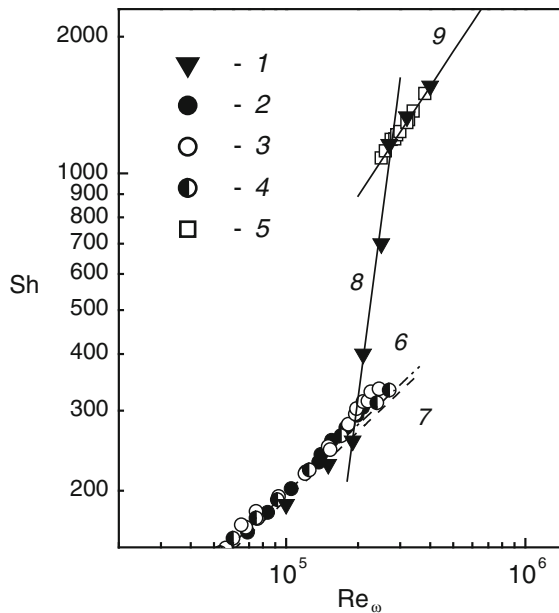
Values of  $Pr \leq 5$  and  $Sc \leq 5$  are considered here as those moderately deviating from unity. The objective is again a validation of the experimental technique dealing with sublimation of naphthalene from a rotating disk in air at  $Sc = 2.28$ – $2.5$  [23].

Local Sherwood numbers in naphthalene sublimation experiments in air in transitional and turbulent flow obtained in the recent works [15, 18] together with the data for laminar flow and different empirical approximations are depicted in Fig. 6.2. Recast Eq. (3.13) [15] for transitional flow (corrected range of validity) and empirical equations [15, 18] for turbulent flow look as follows [15, 18]

$$Sh = 2.0 \times 10^{-19} Re_\omega^4 \quad \text{for} \quad Re_\omega = (1.9\text{--}2.75) \times 10^5 \quad (\text{Ref. [15]}), \quad (6.14)$$

$$Sh = 0.0512 Re_\omega^{0.8} \quad \text{for} \quad Re_\omega \geq 2.75 \times 10^5 \quad (\text{Ref. [15]}), \quad (6.15)$$

$$Sh = 0.0518 Re_\omega^{0.8} \quad \text{for} \quad Re_\omega \geq 2.5 \times 10^5 \quad (\text{Ref. [18]}). \quad (6.16)$$



**Fig. 6.2** Local Sherwood numbers for naphthalene sublimation in air [23, 30]. Experiments: 1— $Sc = 2.28$  [15]; 2— $Sc = 2.4$  [21]; 3— $Sc = 2.4$  [26]; 4— $Sc = 2.44$  [19]; 5— $Sc$  not mentioned [18]. Empirical approximations, Eq. (6.3): 6—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.625$  [20, 21, 25, 26]; 7—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.604$  [17]; 8—transitional flow,  $n_R = 4$ ,  $K_1 = 2 \times 10^{-19}$ , Eq. (6.14) [15]; 9—turbulent flow,  $n_R = 0.8$ ,  $K_1 = 0.0512$ , Eq. (6.15) [15]



In practice, one often needs to estimate average Sherwood numbers  $Sh_{av}$  (or average Nusselt numbers  $Nu_{av}$ ) of an *entire* disk, where areas occupied by laminar/transitional flow or laminar/transitional/turbulent flow emerge at the same time. For instance, only surface-averaged mass transfer coefficients of an *entire* disk were measured in [19–21, 25, 26].

Measurements of the average Sherwood number over an *entire* disk covered with areas of laminar, transitional and turbulent flow were performed by [19–21, 25, 26]. Reynolds analogy between mass transfer and fluid flow was involved to derive a quite inconvenient theoretical solution for  $Sh_{av}$  for an entire disk [21, 26] incorporating parameters, which were rather difficult to determine by means of the used approach. More promising is the model for  $Sh_{av}$  first used in the paper [7] and further generalized by the author of the present work [23, 30], which enables verifications of the recent measurements of the local Sherwood numbers by means of comparisons with the vast database for the average Sherwood numbers for an *entire* disk.

The author [7] assumed that laminar-turbulent transition sets on instantly at a radial coordinate  $r_{tr}$  corresponding to the Reynolds number  $Re_{\omega, tr}$ . Subsequently, the value  $Sh_{av}$  for an entire disk can be found using the following integral

$$Sh_{av} = \frac{2}{b} \left[ \int_0^{r_{tr}} Sh_{lam} dr + \int_{r_{tr}}^b Sh_{turb} dr \right]. \quad (6.17)$$

Sherwood numbers are presented by Eq. (6.3) accompanied with the constants  $K_{1, lam}$  and  $n_R = 1/2$  for laminar flow, and  $K_{1, turb}$  and  $n_R = 0.8$  for turbulent flow.

An integration of Eq. (6.17) yields

$$Sh_{av} = K_{1, lam} Re_{\omega, tr}^{1/2} \left( \frac{Re_{\omega, tr}}{Re_{\varphi}} \right)^{1/2} + \frac{2}{2n_R + 1} K_{1, turb} Re_{\varphi}^{n_R} \left[ 1 - \left( \frac{Re_{\omega, tr}}{Re_{\varphi}} \right)^{n_R + 1/2} \right]. \quad (6.18)$$

If  $Re_{\varphi} < Re_{\omega, tr}$ , the second summand in Eq. (6.18) must be discarded. Asymptotically at  $Re_{\varphi} \gg Re_{\omega, tr}$ , Eq. (6.18) degenerates to Eq. (6.3) for turbulent flow with

$$K_{2, turb} = \frac{2}{2n_R + 1} K_{1, turb}. \quad (6.19)$$

Given  $n_* = 0$ , which effectively means  $T_w = \text{const.}$  and  $C_w = \text{const.}$ , Equation (6.18) translates into Eq. (3.25), while Eq. (6.19) turns to Eq. (3.35) in view of the relation  $2n_R = 1 + m$  resulting from Eqs. (2.78) and (3.31).

In [12] it is suggested taking into account regions of laminar, transitional and turbulent flow separately. If the transition sets on at the radial location  $r_{tr1}$  (or at

$Re_{\omega, \text{tr}1}$ ) and ends at the radial location  $r_{\text{tr}2}$  (or  $Re_{\omega, \text{tr}2}$ ), a definite integral for  $Sh_{\text{av}}$  can be written as

$$Sh_{\text{av}} = \frac{2}{b} \left[ \int_0^{r_{\text{tr}1}} Sh_{\text{lam}} dr + \int_{r_{\text{tr}1}}^{r_{\text{tr}2}} Sh_{\text{tran}} dr + \int_{r_{\text{tr}2}}^b Sh_{\text{turb}} dr \right]. \quad (6.20)$$

The transitional Sherwood number  $Sh_{\text{tran}}$  is specified by the first of Eq. (6.3) complemented with experimental values of  $K_{1, \text{tran}}$  and  $n_{\text{R}, \text{tran}}$  for transitional flow. Integration of Eq. (6.20) results in

$$\begin{aligned} Sh_{\text{av}} = & K_{1, \text{lam}} Re_{\omega, \text{tr}1}^{1/2} \left( \frac{Re_{\omega, \text{tr}1}}{Re_{\varphi}} \right)^{1/2} + \frac{2}{2n_{\text{R}, \text{tran}} + 1} K_{1, \text{tran}} Re_{\omega, \text{tr}2}^{n_{\text{R}, \text{tran}}} \left( \frac{Re_{\omega, \text{tr}2}}{Re_{\varphi}} \right)^{1/2} \\ & \times \left[ 1 - \left( \frac{Re_{\omega, \text{tr}1}}{Re_{\omega, \text{tr}2}} \right)^{n_{\text{R}, \text{tran}} + 1/2} \right] + \frac{2}{2n_{\text{R}} + 1} K_{1, \text{turb}} Re_{\varphi}^{n_{\text{R}}} \left[ 1 - \left( \frac{Re_{\omega, \text{tr}2}}{Re_{\varphi}} \right)^{n_{\text{R}} + 1/2} \right]. \end{aligned} \quad (6.21)$$

Equation (6.21) holds at  $Re_{\varphi} \geq Re_{\omega, \text{tr}2}$ . If  $Re_{\varphi} < Re_{\omega, \text{tr}2}$ , the last term in Eq. (6.21) is discarded, whereas the second summand turns to

$$\begin{aligned} Sh_{\text{av}} = & K_{1, \text{lam}} Re_{\omega, \text{tr}1}^{1/2} \left( \frac{Re_{\omega, \text{tr}1}}{Re_{\varphi}} \right)^{1/2} \\ & + \frac{2}{2n_{\text{R}, \text{tran}} + 1} K_{1, \text{tran}} Re_{\varphi}^{n_{\text{R}, \text{tran}}} \left[ 1 - \left( \frac{Re_{\omega, \text{tr}1}}{Re_{\varphi}} \right)^{n_{\text{R}, \text{tran}} + 1/2} \right]. \end{aligned} \quad (6.22)$$

Asymptotically for  $Re_{\varphi} \gg Re_{\omega, \text{tr}2}$ , Eq. (6.21) transforms to the second of Eq. (6.3), whereas the constant  $K_{2, \text{turb}}$  is given by Eq. (6.19). A solution derived in [12] is a particular case of Eq. (6.21), whose empirical constants resulting from experiments [12] at high  $Sc$  numbers are fixed numerical values. Hence, the solution [12] as it is can not be used to describe the experimental data for naphthalene sublimation.

Substitution of numerical values of the constants resulting from measurements at naphthalene sublimation in air [15] (see Eqs. (6.14), (6.15) and caption to Fig. 6.1) into the general Eqs. (6.18)–(6.22) yields

(a) applied to Eq. (6.18)

$$Sh_{\text{av}} = 0.59 Re_{\omega, \text{tr}}^{1/2} \left( \frac{Re_{\omega, \text{tr}}}{Re_{\varphi}} \right)^{1/2} + \frac{2}{2.6} 0.512 Re_{\varphi}^{0.8} \left[ 1 - \left( \frac{Re_{\omega, \text{tr}}}{Re_{\varphi}} \right)^{1.3} \right], \quad (6.23)$$

(b) applied to Eq. (6.19)

$$K_{2,\text{turb}} = \frac{2}{2.6} K_{1,\text{turb}} = 0.0394, \quad (6.24)$$

(c) applied to Eq. (6.21)

$$\begin{aligned} Sh_{\text{av}} &= 0.59 \times 1.9 \times 10^5 \times Re_{\varphi}^{-1/2} + \frac{4}{9} 10^{-19} (2.75 \times 10^5)^{4.5} \\ &\quad \times Re_{\varphi}^{-1/2} \left[ 1 - \left( \frac{1.9 \times 10^5}{2.75 \times 10^5} \right)^{4.5} \right] \\ &\quad + 0.0394 Re_{\varphi}^{0.8} \left[ 1 - \left( \frac{2.75 \times 10^5}{Re_{\varphi}} \right)^{1.3} \right], \quad Re_{\varphi} \geq 2.75 \times 10^5, \quad (6.25) \end{aligned}$$

(d) applied to Eq. (6.22)

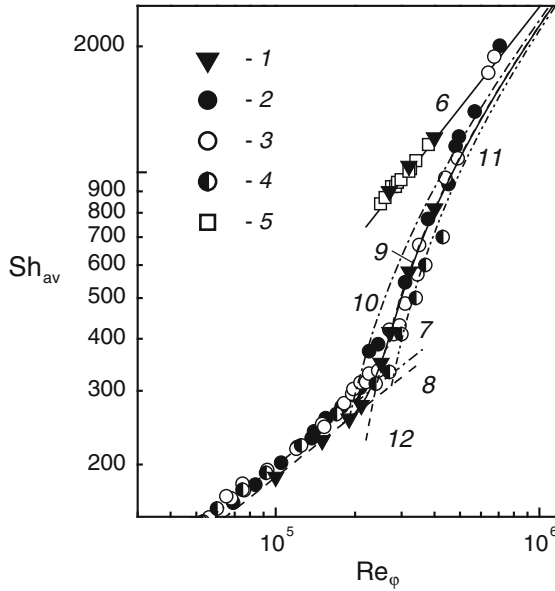
$$\begin{aligned} Sh_{\text{av}} &= 0.59 \times 1.9 \times 10^5 \times Re_{\varphi}^{-1/2} + \frac{4}{9} 10^{-19} Re_{\varphi}^4 \left[ 1 - \left( \frac{1.9 \times 10^5}{Re_{\varphi}} \right)^{4.5} \right], \\ Re_{\varphi} &= (1.9-2.75) \times 10^5. \end{aligned} \quad (6.26)$$

The Reynolds number  $Re_{\omega,\text{tr}}$  (instant transition to turbulence) in Eq. (6.23) remains a free parameter to be tuned for a better agreement with particular experiments.

Figure 6.3 shows validations of Eqs. (6.23)–(6.28) by comparison with experimental data. Experimental data 1, 5 and curve 6 for  $Sh_{\text{av}}$  for purely turbulent flow stem from the works [23, 30] and result from reprocessing of the measured data [15, 18] and Eq. (6.15) using Eq. (6.24). For laminar flow, we have  $K_{2,\text{lam}} = K_{1,\text{lam}}$  (curves 7 and 8). Curve 9 combining Eqs. (6.25), (6.26) and incorporating boundaries of transitional flow conforms to experiments [19, 21, 26] for  $Sh_{\text{av}}$  for an *entire* disk depicted in Fig. 6.3.

In Fig. 6.3, experimental data 1 for  $Sh_{\text{av}}$  for an *entire* disk were calculated in [23] using Eqs. (6.25), (6.26) and measurements [15] for laminar, transitional and turbulent flow. These data points go beyond curve 9 at respective values of the argument  $Re_{\varphi}$ .

The replacement of the Reynolds number  $Re_{\omega,\text{tr}}$  in Eq. (6.23) (instant transition to turbulence) with its values at the onset and end of transition (i.e.,  $1.9 \times 10^5$  and  $2.75 \times 10^5$ ) yields curves 10 and 11 lying above and below curve 9, respectively. Reynolds number of the instant transition to turbulence  $Re_{\omega,\text{tr}} = 2.35 \times 10^5$ , an arithmetic mean of values  $Re_{\omega,\text{tr}1}$  and  $Re_{\omega,\text{tr}2}$ , substituted into Eq. (6.23) conveyed curve 12, which agrees with curve 9.



**Fig. 6.3** Average Sherwood numbers, naphthalene sublimation in air [23, 30]. Experiments: 1— $Sc = 2.28$  [15]; 2— $Sc = 2.4$  [21]; 3— $Sc = 2.4$  [26]; 4— $Sc = 2.44$  [19]; 5— $Sc$  not mentioned [18]. Calculation, Eq. (6.3): 6—turbulent flow,  $n_R = 0.8$ ,  $K_2 = 0.0394$ , Eq. (6.24) [15]; 7—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.625$  [20, 21, 25, 26]; 8—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.59$  [15]. Calculation of  $Sh_{av}$  for the *entire* disk: 9—Eqs. (6.25) and (6.26); 10—Eq. (6.23) at  $Re_{\omega, tr} = 1.9 \times 10^5$ ; 11—Eq. (6.23) at  $Re_{\omega, tr} = 2.75 \times 10^5$ ; 12—Eq. (6.23) at  $Re_{\omega, tr} = 2.35 \times 10^5$

In the asymptotic case of  $Re_\phi \rightarrow \infty$ , lines 9–12 coincide with curve 6 valid for purely turbulent flow.

Thus, Eqs. (6.21) and (6.22) incorporating terms accounting for the coexistence of laminar, transitional and turbulent flow areas ensure the best agreement with experiments for the  $Sh_{av}$  number for an *entire* disk. Equation (6.18) resulting from a simpler model [7] ensures the efficiency similar to that of Eqs. (6.21) and (6.22), if an “effective” Reynolds number  $Re_{\omega, tr}$  of the instant transition to turbulent flow is chosen correctly.

**Application to the naphthalene sublimation technique.** Again, a recalculation of the mass transfer to heat transfer data is performed using Eq. (3.4) and (6.3), with the constants  $K_1$  defined in Eqs. (6.10) and (6.11), accordingly [23]. The factor  $C$  is equal to the constant  $K_1$  for  $Sc = 1$ ,  $Pr = 1$  under conditions  $T_w = \text{const.}$  or  $C_w = \text{const.}$

Authors [15, 18] used the constant  $m_p = 0.4$  for a turbulent flow regime and  $Pr = Sc = 0.7\text{--}2.5$ . Equation (6.12) at  $m_p = 0.4$  yields the value  $K_1 = 0.0323$  for heat transfer in air at  $T_w = \text{const.}$  and  $Pr = 0.72$ , starting from the values  $K_1 = 0.0512\text{--}0.0518$  (see Eqs. (6.15) and (6.16)) for  $Sc = 2.28$  as a base for the recalculation. But, in reality, experiments [35–37] (see Table 3.5) conveyed the value of the constant  $K_1 = 0.0188$  at  $T_w = \text{const.}$  and  $Pr = 0.72$ . The theoretical model [38, 39] gave the

value  $K_1 = 0.0187$  for the same conditions. Thus, also for turbulent flow, an erroneous value  $m_p$  leads to fallacious translation of the naphthalene sublimation data to heat transfer in air.

Equations (6.10), (6.11) are to be used for the Prandtl and Schmidt numbers moderately diverging from unity:  $Pr = 0.7\text{--}0.74$  for air;  $Sc = 2.28\text{--}2.5$  for naphthalene sublimation. Hence, the constant  $C$  in Eqs. (6.10), (6.11) must be equal to the coefficient  $K_1 = 0.0232$  in turbulent flow at  $Sc = 1$ ,  $Pr = 1$  and conditions  $T_w = \text{const.}$ ,  $C_w = \text{const.}$  (Table 3.7).

Detecting of the exponent  $m_p$  for turbulent flow is performed using experimental data. Only experimental Eqs. (6.15) and (6.16) [15, 18] can serve for this purpose. Based on Eq. (6.15) as well as the values  $C = 0.0232$ ,  $K_1 = 0.0188$  (at  $T_w = \text{const.}$  and  $Pr = 0.72$ ) [35–37], one can transform Eqs. (6.10) and (6.11) as follows

$$K_1 = 0.0232Pr^{0.64} \quad \text{for } Pr \leq 1, \quad (6.27)$$

$$K_1 = 0.0232Sc^{0.96} \quad \text{for } Sc \geq 1. \quad (6.28)$$

This means that the exponent  $m_p$  for turbulent flow is not universal being a function of the Prandtl and Schmidt numbers, which apparently results from different effects of the  $Pr$  or  $Sc$  larger and smaller than unity. Equations (6.27) and (6.28) yield as a result

$$Nu/Sh = Pr^{0.64}/Sc^{0.96}. \quad (6.29)$$

Using the idea of a correction function, Eq. (6.13), one can transform Eq. (6.29) as

$$Nu/Sh_{Sc=2.28} = f(Pr). \quad (6.30)$$

At  $Pr = 0.72$ , the correction function  $f(Pr)$  takes the value  $f(Pr) = 0.367$ .

As an alternative, one can use an effective value of the exponent  $m_p$  so that

$$Nu/Sh = (Pr/Sc)^{0.87}. \quad (6.31)$$

The exponent  $m_p = 0.87$  in Eq. (6.31) is more than twice larger than the value 0.4 mistakenly recommended in [15, 18]. Nevertheless, the value  $m_p = 0.87$  must not be used in Eqs. (6.27) and (6.28) to avoid significant errors in predictions of the constant  $K_1$ .

The empirical Eq. (3.10) [37] for transitional flow at  $T_w = \text{const.}$  and  $Pr = 0.72$  is the most appropriate to be used jointly with Eq. (6.14) for transitional flow at  $C_w = \text{const.}$  Thus, Eq. (6.12) should be recast in view of Eqs. (3.10) and (6.14) as follows

$$Nu/Sh = (Pr/Sc)^{0.6}. \quad (6.32)$$

Equation (3.10) is valid for  $Re_\omega = 1.95 \times 10^5 - 2.5 \times 10^5$ , while Eq. (6.14) holds at  $Re_\omega = 1.9 \times 10^5 - 2.75 \times 10^5$ . These differences are though rather insignificant.

To conclude, Eqs. (6.29)–(6.31) should be employed to recalculate the data for turbulent mass transfer for naphthalene sublimation in air to the conditions of heat transfer in air. Equation (6.32) should be applied for transitional flow for the same purpose [23].

### 6.3 Transitional and Turbulent Flow at High Schmidt Numbers

High values of the Schmidt numbers can be encountered in electrochemistry problems:  $Sc = 34 - 10,320$  [4, 5, 7, 8, 12]. Main objectives of this section are validation and development of recommendations for the further use of the experimental and theoretical data of different authors [40].

Experimental data [7] for average Sherwood numbers for an entire disk at  $Re_\phi = 0.278 \times 10^6 - 1.8 \times 10^6$ ,  $Sc = 930 - 10,320$  were described by a relation

$$Sh_{av} = Sc^{1/3} Re_\phi^{-1/2} [0.62 Re_{\omega, tr} + 1.08 \times 10^{-2} (Re_\phi^{1.37} - Re_{\omega, tr}^{10.37})]. \quad (6.33)$$

Here, Eq. (6.18) at  $K_{1, lam} = 0.62 Sc^{1/3}$ ,  $K_{1, turb} = 0.0148 Sc^{1/3}$ ,  $Re_{\omega, tr} = 2.78 \times 10^5$  and  $n_R = 0.87$  was taken into account. In the transitional region at  $Re_\omega = 2.3 \times 10^5 - 2.9 \times 10^5$ , Eq. (6.33) lies below the experimental data [7] (in analogy to curve 12 in Fig. 6.3), which results from simplifications incorporated in model (6.18) and mentioned in Sect. 6.2.

A reduced form of Eq. (6.33) for purely turbulent flow [7] and an equation for the local Sherwood numbers derived in [30, 40] have the following form

$$Sh_{av} = 1.08 \times 10^{-2} Re_\phi^{0.87} Sc^{1/3}, \quad (6.34)$$

$$Sh = 1.48 \times 10^{-2} Re_\omega^{0.87} Sc^{1/3}. \quad (6.35)$$

Measurements [4] of the average Sherwood numbers for an entire disk performed at  $Re_\phi = 5 \times 10^4 - 1.8 \times 10^6$ ,  $Sc = 345 - 6450$  (transition at  $Re_\phi = 2.3 \times 10^5 - 2.9 \times 10^5$ ) for the region of turbulent flow were described by the relation

$$Sh_{av} = 0.0725 Re_\phi^{0.9} Sc^{0.33}. \quad (6.36)$$

The authors [5] measured local  $Sh$  and average  $Sh_{av}$  numbers at laminar, transitional, and turbulent flow for  $Re_\omega = 4 \times 10^4 - 2.2 \times 10^6$ ,  $Sc = 680 - 7200$  (transition at  $Re_\omega = 2.2 \times 10^5 - 3.0 \times 10^5$ ). Sherwood numbers for the turbulent flow [5] and average values for an entire disk (approximated in [30]) are given by the following relations, respectively

$$Sh = 1.09 \times 10^{-2} Re_{\omega}^{0.91} Sc^{1/3}, \quad (6.37)$$

$$Sh_{av} = 7.67 \times 10^{-3} Re_{\phi}^{0.91} Sc^{1/3}, \quad (6.38)$$

$$Sh_{av} = Sc^{1/3} Re_{\phi}^{-1/2} [0.62 Re_{\omega, tr} + 7.67 \times 10^{-3} (Re_{\phi}^{1.41} - Re_{\omega, tr}^{1.41})], \quad (6.39)$$

where the Reynolds number of the abrupt transition was  $Re_{\omega, tr} = 2.78 \times 10^5$  [7].

Experiments [8] for  $Sh_{av}$  for an entire disk were performed at  $Re_{\phi} = 10^4 - 1.18 \times 10^7$ ,  $Sc = 34 - 1400$ . For purely turbulent flow at  $Re_{\phi} = 8.9 \times 10^5 - 1.18 \times 10^7$ , authors [8] obtained

$$Sh_{av} = 1.17 \times 10^{-2} Re_{\phi}^{0.896} Sc^{0.249}. \quad (6.40)$$

Experiments for the local Sherwood numbers in transitional flow at  $Re_{\omega} = 2.0 \times 10^5 - 3.0 \times 10^5$  and  $Sc = 1192 - 2465$  were described by the empirical Eq. (3.14) [12].

The authors [12] deduced empirical equations for  $Sh_{av}$  for turbulent flow (based on experiments [4]), simultaneous existence of laminar and transitional flow, as well as simultaneous existence of laminar, transitional and turbulent flow, respectively

$$Sh_{av} = 7.8 \times 10^{-3} Re_{\phi}^{0.9} Sc^{1/3}, \quad (6.41)$$

$$Sh_{av} = Sc^{1/3} Re_{\phi}^{-1/2} [0.89 \times 10^5 + 9.7 \times 10^{-15} Re_{\phi}^{3.5}], \quad (6.42)$$

$$Sh_{av} = Sc^{1/3} Re_{\phi}^{-1/2} [7.8 \times 10^{-3} Re_{\phi}^{1.4} - 1.3 \times 10^5]. \quad (6.43)$$

Equations (6.42) and (6.43) are particular cases of Eqs. (6.22) and (6.21), accordingly, with Eq. (6.8) used for laminar, Eq. (3.14) for transitional and Eq. (6.41) for turbulent flow.

Theoretical solutions for the local turbulent Sherwood numbers at high Schmidt numbers derived in [9, 41] can be presented as follows, respectively,

$$Sh_{av} = 7.07 \times 10^{-3} Re_{\phi}^{0.9} Sc^{1/3}, \quad (6.44)$$

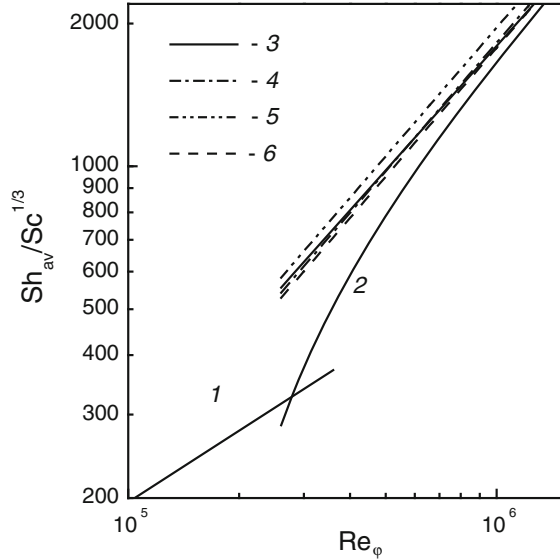
$$Sh_{av} = 5.93 \times 10^{-3} Re_{\phi}^{0.91} Sc^{0.34}. \quad (6.45)$$

A theoretical solution obtained in [10] coincides with Eq. (6.41). The solution obtained in [42] has a form of Eq. (6.41) with the coefficient changed to  $6.43 \times 10^{-3}$ .

In [43] a theoretical solution claimed to be valid for  $Sc = 0.72 - \infty$  has been proposed, however, as demonstrated in [30, 40], this relation is inaccurate.

Some of the theoretical and empirical relations for the Sherwood numbers for purely turbulent flow, as well as for average  $Sh_{av}$  numbers for an entire disk

**Fig. 6.4** Average Sherwood numbers at high Schmidt numbers [30]. Approximation of experiments: 1—laminar flow, Levich's Eq. (6.8); 2—Eq. (6.33) for an entire disk [7]; 3—Eq. (6.34) [7]; 4—Eq. (6.36) [4]; 5—Eq. (6.41) [10, 12]. Theoretical solution: 6—Eq. (6.44) [9]



simultaneously occupied by laminar, transitional, and turbulent flow areas agree well with each other. Curves by Eqs. (6.34), (6.36), (6.44), and (6.45) practically merge (see Fig. 6.4). Equation (6.41) significantly surpasses original experiments [4]; corrected coefficient  $6.43 \times 10^{-3}$  [42] shifts predictions by Eq. (6.41) 9 % below those by Eq. (6.44). Empirical Eqs. (6.34) and (6.36) practically coincide, which corroborates the reliability of these experiments.

Equation (6.38) for turbulent flow and Eq. (6.39) for an entire disk significantly surpass Eqs. (6.33) and (6.34), respectively (see Fig. 6.5). Only in Eq. (6.40) [8], exponent 0.249 at the Schmidt number is not equal to 1/3. The large scatter of experiments around the approximation curve [8] is rather an evidence that the exponent 0.249 is erroneous. As demonstrated in [30, 40], differences between the curves by Eq. (6.40) plotted in the relation  $Sh_{av}/Sc^{1/3}$  versus  $Re_\phi$  for different  $Sc$  values revealed in experiments [8] is rather significant. Hence, Eq. (6.40) should be discarded as too inaccurate.

The exponent for the Reynolds number  $Re_\phi$  in Eq. (6.35) diverges from those in Eqs. (6.15) and (6.16). Equation (6.35) can be recast to make the exponent for  $Re_\phi$  equal to 0.8. This yields for the entire disk [30, 40]

$$Sh_{av}Sc^{-1/3} = 0.62Re_{\omega,tr}^{1/2} \left( \frac{Re_{\omega,tr}}{Re_\phi} \right)^{1/2} + \frac{2}{20.6} 0.0365Re_\phi^{0.8} \left[ 1 - \left( \frac{Re_{\omega,tr}}{Re_\phi} \right)^{1.3} \right]. \quad (6.46)$$



**Fig. 6.5** Average Sherwood numbers at high Schmidt numbers, approximation of experiments [30]. 1—laminar flow, Levich’s Eq. (6.8); 2—Eq. (6.33) for an *entire* disk [7]; 3—Eq. (6.39), *entire* disk; 4—Eq. (6.46), *entire* disk; 5—Eq. (6.34) [7]; 6—Eq. (6.38) [5]

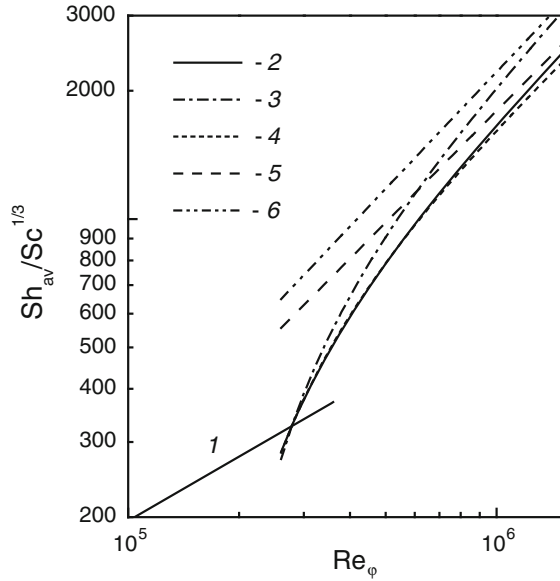


Figure 6.5 depicts curves 2 and 4 plotted by Eqs. (6.33) and (6.46), respectively. Here again  $Re_{\omega, tr} = 2.78 \times 10^5$ , like in Eq. (6.33). Curves 2 and 4 in fact merge for  $Re_\phi \leq 9.0 \times 10^5$ ; deviations start to become visible for  $Re_\phi > 9.0 \times 10^5$ .

For purely turbulent flow, Eq. (6.46) reduces asymptotically to the relations

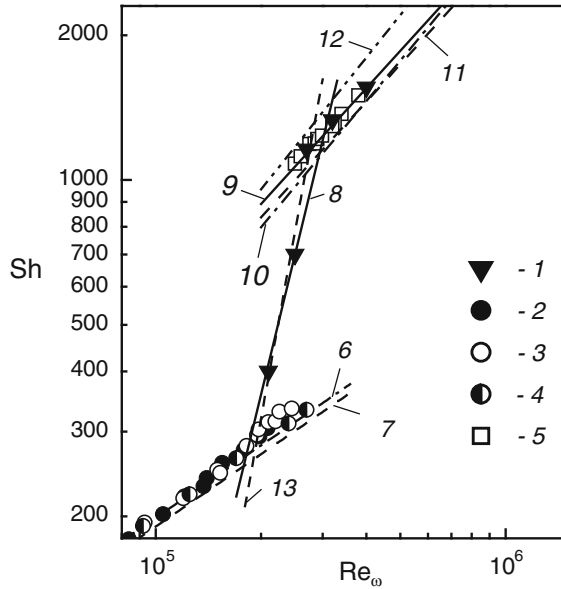
$$Sh = 3.65 \times 10^{-2} Re_\omega^{0.8} Sc^{1/3}, \tag{6.47}$$

$$Sh_{av} = 3.65 \times 10^{-2} \frac{2}{2.6} Re_\phi^{0.8} Sc^{1/3} = 2.81 \times 10^{-2} Re_\phi^{0.8} Sc^{1/3}. \tag{6.48}$$

Figure 6.6 demonstrates that Eq. (6.35) [7] used at  $Sc = 2.28$  predicts Sherwood numbers close to the experiments [15, 18] for naphthalene sublimation in air and their approximation Eq. (6.15). Equations (6.35) (curve 10) and (6.15) (curve 9) correlate well at larger Reynolds numbers  $Re_\omega = 0.6 \times 10^6 - 2.0 \times 10^6$ . Curve 11, Eq. (6.47), lies in the vicinity of curve 10 at smaller Reynolds numbers  $Re_\omega \leq 0.7 \times 10^6$ . Equation (6.47) yields  $K_1 = 0.048$  at  $Sc = 2.28$ , which is only 6.7 % below the value  $K_1 = 0.0512$  in Eq. (6.15). Curve 12 by Eq. (6.37) goes noticeably beyond experiments and approximation curve 9 in Fig. 6.6.

Dependence 13 in Fig. 6.6 plotted by experimental Eq. (3.14) [12] for transitional flow at  $Sc = 2.28$  conforms well to Eq. (6.14) and experiments [15].

To conclude, the most reliable empirical relations for developed turbulent flow and an *entire* disk relying on the analysis made above are Eqs. (6.33)–(6.36).



**Fig. 6.6** Local Sherwood numbers for naphthalene sublimation [30]. Experiments: 1— $Sc = 2.28$  [15]; 2— $Sc = 2.4$  [21]; 3— $Sc = 2.4$  [26]; 4— $Sc = 2.44$  [19]; 5— $Sc$  not mentioned [18]. Empirical Eq. (6.3): 6—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.625$  [20, 25, 26]; 7—laminar flow,  $n_R = 1/2$ ,  $K_1 = 0.604$  [17]; 8—transitional flow,  $n_R = 4$ ,  $K_1 = 2 \times 10^{-19}$  [15]; 9—turbulent flow,  $n_R = 0.8$ ,  $K_1 = 0.0512$  [15]. Developed turbulent flow,  $Sc = 2.28$ : 10—Eq. (6.35) [7]; 11—Eq. (6.47); 12—Eq. (6.37) [5]. Transitional flow,  $Sc = 2.28$ : 13—Eq. (3.14) [12]

### 6.4 An Integral Method for $Pr$ and $Sc$ Numbers Much Larger Than Unity

**Model with a constant value**  $\Delta \ll 1$ . The thickness of the thermal (or diffusion) boundary layer at very high  $Pr$  or  $Sc$  numbers is much smaller than the thickness of the velocity boundary layer (i.e.,  $\Delta \ll 1$ ). Hence, in Eq. (3.40) obtained for  $\Delta = \text{const.}$  and  $T^+ \equiv T^+(y^+)$ , all summands in the parentheses in its left-hand part but  $a_*$  tend to zero

$$\Delta^{2n+1} a_* = \frac{4 + m}{4 + m + n_*} (a_* - 2b_* + c_*) Pr^{-n_p}. \tag{6.49}$$

Relying on Eq. (6.49), one can derive analytical solutions for constants  $\Delta$  and  $K_1$

$$\Delta = \left[ \frac{4 + m}{4 + m + n_*} \left( 1 - \frac{2D_3}{C_2} \right) \right]^{\frac{1}{2n+1}} Pr^{-\frac{n_p}{2n+1}}, \tag{6.50}$$

$$K_1 = K_3 \left[ \frac{4 + m}{4 + m + n_*} \left( 1 - \frac{2D_3}{C_2} \right) \right]^{\frac{-n}{2n+1}} Pr^{1+n_p \left( \frac{n}{2n+1} - 1 \right)}, \quad (6.51)$$

where the coefficients  $C_2$  and  $D_3$  are described in the comments to Eqs. (2.68) and (2.69).

The cumulative exponent at the Prandtl number in Eq. (6.51) for  $Pr \gg 1$  must be equal to  $1/3$  (see Sect. 6.3), which yields the following expression for  $n_p$

$$n_p = \frac{2}{3} \cdot \frac{2n + 1}{n + 1}. \quad (6.52)$$

As a result, the constants  $K_1$  and  $K_2$  in view of Eq. (3.35) can be written as

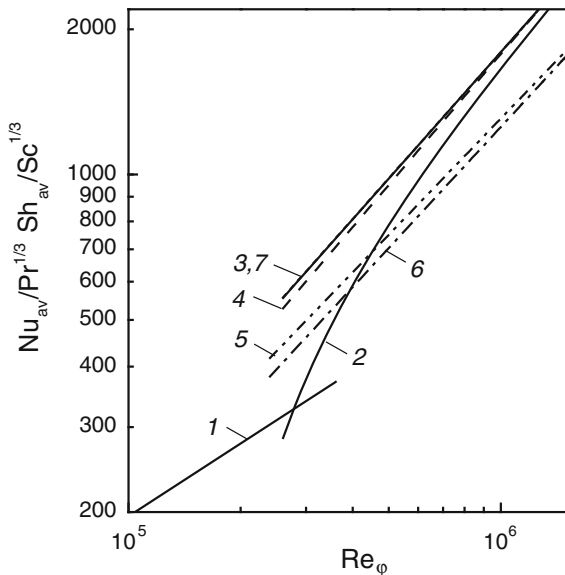
$$K_1 = K_3 \left[ \frac{4 + m}{4 + m + n_*} \left( 1 - \frac{2D_3}{C_2} \right) \right]^{\frac{-n}{2n+1}} Pr^{1/3}, \quad (6.53)$$

$$K_2 = K_3 \left[ \frac{4 + m}{4 + m + n_*} \left( 1 - \frac{2D_3}{C_2} \right) \right]^{\frac{-n}{2n+1}} \frac{n_* + 2}{2 + n_* + m} Pr^{1/3}. \quad (6.54)$$

To enable validations against electrochemical experiments, let us further treat the Sherwood numbers rather than the Nusselt numbers and replace  $Pr$  with  $Sc$ .

In Fig. 6.7, Eq. (6.54) for  $Sh_{av}$  (at  $n_* = 0$ ) is validated against the empirical Eq. (6.34) [7] and theoretical Eq. (6.44) [9]. Curves 5 and 6 predicted by Eq. (6.54) at  $n = 1/7$  and  $1/9$  lie 20–30 % below the curves 3 and 4 suggested by Eqs. (6.34) and (6.44), accordingly. Such a discrepancy between theory and measurements is

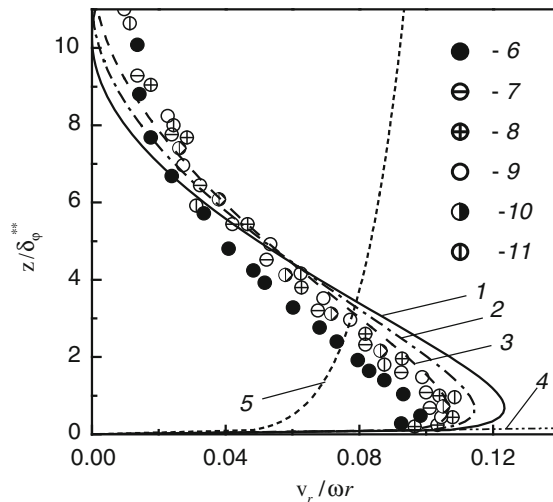
**Fig. 6.7** Average Sherwood numbers for high Schmidt numbers [30]. Approximation of experiments: 1—laminar flow, Levich’s Eq. (6.8) [11]; 2—Eq. (6.33) for an entire disk [7]; 3—Eq. (6.34) [7] Theoretical solutions: 4—Eq. (6.44) [9]; 5—Eq. (6.54) for  $n = 1/7$ ; 6—Eq. (6.54) for  $n = 1/9$ ; 7—Eq. (6.73)



too high. In addition, the slope of the curves 5 and 6 (exponents at  $Re_\phi$  being 0.8 and 0.833, constants  $K_2$  being 0.0207 and 0.0126, accordingly) distinctly deviates from the slope of curves 3 and 4 (exponents at  $Re_\phi$  being 0.87 and 0.9, constants  $K_2$  being 0.0207 and 0.126, accordingly). Therefore, some model approaches incorporated in the present integral method partially fail at high  $Pr$  and  $Sc$  numbers and need to be improved. In Eq. (3.32) for the coefficient  $K_1$ , the total exponent at the Reynolds number can be increased, provided that the relative thickness  $\Delta$  is assigned to be a decreasing function of the local Reynolds number  $Re_\omega$ .

**Model with a variable value of  $\Delta$ .** The present integral method incorporates a model, in frames of which a boundary layer consists two parts. In the vicinity of the wall, a viscous and heat conduction sub-layers emerge, where the velocity and temperature profiles are described by Eq. (2.62). In the main part of the boundary layer (outside of the viscous sub-layer), velocity components are described by the power-law functions (see Chaps. 2 and 3). If Prandtl and Schmidt numbers are slightly different from unity, the thermal/diffusion and velocity boundary layers have a thickness of the same order of magnitude [30, 40]. Hence, integration of Eq. (2.23) for the thermal boundary layer has been performed over the *entire* velocity boundary layer. Viscous and heat conduction sub-layers are not taken into account in this integration, because they are negligibly thin in comparison with the overall boundary layer thickness. Velocity profiles in Eqs. (2.17)–(2.19) are integrated in the same way [31, 38, 39, 44, 45].

At very high Prandtl and Schmidt numbers, the boundary layer structure changes drastically. A very thin thermal/diffusion boundary layer is fully incorporated inside



**Fig. 6.8** Radial velocity profiles in the turbulent boundary layer over a free rotating disk [30]. 1— $n = 1/7$ ; 2— $1/8$ ; 3— $1/9$ . Equation (2.41), [44]; 4—Eq. (6.55) at  $Re_\omega = 1.0 \times 10^6$ ; 5—Eq. (3.27) at  $\sigma = 0$ ,  $n = 1/7$  (see Fig. 3.7). Experiments: 6— $Re_\omega = 0.4 \times 10^6$ ; 7— $0.65 \times 10^6$ ; 8— $0.94 \times 10^6$ ; 9— $1.6 \times 10^6$  [46]; 10— $0.6 \times 10^6$ ; 11— $1.0 \times 10^6$  [47]

the viscous sub-layer of the velocity boundary layer; here the radial velocity profile varies linearly depending on the coordinate  $z$  (curve 4 in Fig. 6.8). This fact is taken into account in theoretical models [5, 9–11, 32, 42] for large  $Pr$  and  $Sc$  numbers.

Next to the wall, the radial velocity  $v_r$  varies as a linear function

$$v_r = \frac{\tau_{wr}}{\mu} z = \frac{\tau_w \alpha}{\mu(1 + \alpha^2)^{1/2}} z = \frac{\rho V_*^2 \alpha}{\mu(1 + \alpha^2)^{1/2}} \frac{c_f}{2} z = \alpha(1 + \alpha^2)^{1/2} \omega A_c Re_\omega^{n_R} z. \quad (6.55)$$

Here the constant  $n_R$  is defined in Eq. (3.31).

The coordinate of the boundary of the viscous sub-layer  $z_1^+$ , where the linear model (6.55) holds, can be written as

$$\frac{z_1}{\delta} = \frac{z_1^+}{\gamma(1 + \alpha^2)^{1/2} A_c^{1/2} Re_\omega^{1/(1+3n)}}, \quad (6.56)$$

where  $z_1^+ = 12.54; 13.44; 14.23$  and  $15.09$  for  $n = 1/7; 1/8; 1/9$  and  $1/10$ , respectively (see Chap. 2). According to Eq. (6.56), this corresponds to  $z_1/\delta = 0.01\text{--}0.02$ . Figure 6.8 confirms the validity of the model (6.55) up to  $z/\delta_\phi^{**} = 0.2$ ,  $z/\delta = 0.02$ , or  $\Delta = \delta_T/\delta = 0.02$ .

In the power-law model, the Stanton number is given by Eq. (2.64). In Sect. 2.4.3, the model assumption  $(z_{1T}^+/z_1^+)^{n_T-1} Pr^{-n_T} = Pr^{-n_p}$  completes Eq. (2.64), whereas validations of this model against experiments deliver the value of the exponent  $n_p$ .

At high Prandtl or Schmidt numbers, the entire thermal/diffusion boundary layer is included inside the viscous sub-layer of the velocity boundary layer. Hence, one can assume that the relation between the coordinates  $z_1^+$  and  $z_{1T}^+$  (viscous and heat conduction sub-layer) can be recast as

$$(z_{1T}^+/z_1^+)^{n_T-1} Pr^{-n_T} = K_\alpha Pr^{-n_p}. \quad (6.57)$$

Validations of the model (6.57) against experiments for the  $Nu$  or  $Sh$  numbers enable finding the coefficient  $K_\alpha$  and exponent  $n_p$ . Consequently, given  $n = n_T$ , Eqs. (2.66) and (2.67) turn to

$$St = (c_f/2) \Delta^{-n} Pr^{-n_p} K_\alpha = A_c Re_\omega^{-2n/(3n+1)} \Delta^{-n} Pr^{-n_p} K_\alpha, \quad (6.58)$$

$$Nu = St Re_\omega Pr (1 + \alpha^2)^{1/2} = A_c (1 + \alpha^2)^{1/2} Re_\omega^{n_R} \Delta^{-n} Pr^{1-n_p} K_\alpha. \quad (6.59)$$

Substituting Eqs. (2.53), (6.55), (6.58) and (6.59) into Eq. (2.20), one can transform the latter to the following notation:

$$\frac{n}{2(n+2)} \alpha \omega \frac{d}{dr} [r \delta^2 \Delta^2 Re_\omega^{n_R} \Delta T] = K_\alpha \Delta^{-n} Pr^{-n_p} Re_\omega^{n_R} \nu \Delta T, \quad (6.60)$$

where Eq. (2.77) determines the boundary layer thickness  $\delta$ , while  $\Delta T = T_w - T_\infty$ .

The condition  $\Delta = \text{const.}$  is inapplicable to Eq. (6.60), otherwise the exponents at the variable  $r$  on the left- and right-hand sides of Eq. (6.60) are not equal to each other.

Let us assume the parameter  $\Delta$  to be a power-law function

$$\Delta(r) = C_{\Delta} r^k. \quad (6.61)$$

Substituting Eq. (6.61) into Eq. (6.60) and keeping in mind Eqs. (2.30), (2.77), (2.78) and (3.31), one can finally obtain

$$\Delta = C_{\Delta^*} Re_{\omega}^{k/2}, \quad (6.62)$$

$$C_{\Delta^*} = C_{\Delta^{**}} Pr^{-n_p/(2+n)}, \quad (6.63)$$

$$C_{\Delta^{**}} = \left[ \frac{K_2 2(n+2)/n}{\alpha \gamma^2 (1 - nk + n_* + 2n_R)} \right]^{1/(n+2)}, \quad (6.64)$$

$$k = -2m/(2+n). \quad (6.65)$$

Equation (6.59) for the  $Nu$  number and the expression for  $Nu_{av}$  can be written as follows:

$$Nu = K_1 Re_{\omega}^{n_{R^*}}, \quad (6.66)$$

$$Nu_{av} = K_2 Re_{\varphi}^{n_{R^*}}, \quad (6.67)$$

$$n_{R^*} = n_R + mn/(2+n), \quad (6.68)$$

$$K_1 = K_2 K_3 C_{\Delta^{**}}^{-n} Pr^{1/3}, \quad (6.69)$$

$$K_2 = 2K_1/(2n_{R^*} + 1), \quad (6.70)$$

$$n_p = (2+n)/3. \quad (6.71)$$

Equation (6.71) takes into account the fact that the total exponent at the  $Pr$  number in Eq. (6.69) must be equal to 1/3.

Thus, in Eqs. (6.66) and (6.67), the total exponent  $n_{R^*}$  at the Reynolds number is larger than that in Eq. (3.31) due to the additional term  $mn/(2+n)$  (see Eq. (6.68)). This summand emerges as a result of the model with the *variable parameter*  $\Delta$  being a subsiding function of the coordinate  $r$  or, in other words, local  $Re_{\omega}$  (see Eq. (6.62)).

The values of the exponent  $n_{R^*}$  are:  $n_{R^*} = 0.84$  at  $n = 1/7$ , and  $n_{R^*} = 0.868$  at  $n = 1/9$ . The latter agrees well with the exponent 0.87 at the  $Re_{\varphi}$  number in the experiment-based Eq. (6.35) [7]. To bring Eq. (8.73) at  $n_{R^*} = 0.868$  into agreement with Eq. (6.35), the constant  $K_{\alpha}$  must be equal to  $K_{\alpha} = 1.254$ , which yields at  $n_* = 0$

$$Nu = 1.52 \times 10^{-2} Re_{\omega}^{0.868} Pr^{1/3}, \quad (6.72)$$

$$Nu_{av} = 1.11 \times 10^{-2} Re_{\varphi}^{0.868} Pr^{1/3}, \quad (6.73)$$

$$\Delta = 18.31 Re_{\omega}^{-0.3158} Pr^{-1/3}. \quad (6.74)$$

In Fig. 6.7, curve 7 by Eq. (6.73) and curve 3 by Eq. (6.34) merge. Equations (6.72) and (6.35) are also practically identical.

The parameter range in experiments [7] is  $Re_{\varphi} = 0.278 \times 10^6 - 1.8 \times 10^6$ ,  $Sc = 930 - 10,320$ . At minimal values  $Sc = 930$  and  $Re_{\omega} = 0.278 \times 10^6$  [7], Eq. (6.74) yields  $\Delta = 0.036$ . Parameter  $\Delta$  is a decreasing function of the Schmidt and Reynolds numbers. Thus  $\Delta = 0.015$  at  $Sc = 10,320$  and  $Re_{\omega} = 0.278 \times 10^6$ , whereas  $\Delta = 0.02$  at  $Sc = 930$  and  $Re_{\omega} = 1.8 \times 10^6$ . This conforms to the limit  $\Delta \leq 0.02$  restricting validity of the linear model of the radial velocity profile.

**Model with variable  $\Delta$  and profile  $T^+$  depending on  $Re_{\omega}$ .** In the theoretical works [9–11, 42], the Nusselt number at high  $Pr$  values is described by a relation

$$Nu = K_N (1 + \alpha^2)^{1/2} (c_f/2)^{1/2} Re_{\omega} Pr^{1/3}, \quad (6.75)$$

where  $K_N$  is an empirical constant; Eq. (2.82) at  $n = 1/7$  was used for  $c_f/2$ . Local and average Nusselt numbers take a form of Eqs. (6.66) and (6.67), respectively, with

$$K_1 = K_N (1 + \alpha^2)^{1/2} A_c^{1/2} Pr^{1/3}, \quad (6.76)$$

$$n_{R^*} = (2n + 1)/(3n + 1), \quad (6.77)$$

while the constants  $K_1$  and  $K_2$  are related with Eq. (6.70). At  $n = 1/7$ , Eq. (6.77) brings  $n_{R^*} = 0.9$ .

Matching Eqs. (6.44) and (6.75) in view of Eq. (6.70), one can obtain  $K_N = 0.05986$ .

Substitution of Eq. (6.61) into the thermal boundary layer equation yields again Eq. (6.62) for  $\Delta$  with

$$C_{\Delta^*} = C_{\Delta^{**}} Pr^{-1/3}, \quad (6.78)$$

$$C_{\Delta^{**}} = \left[ \frac{K_N 2(n+2)/n}{\alpha \gamma^2 (1 + 2m + 2k + n_* + 2n_R) A_c^{1/2}} \right]^{1/(n+2)}, \quad (6.79)$$

$$k = (2n - 1)/(3n + 1). \quad (6.80)$$

Setting the values  $n = 1/7$  and  $n_* = 0$  into Eqs. (6.62), (6.78)–(6.80) one can obtain

$$\Delta = 12.54Re_{\omega}^{-1/4}Pr^{-1/3}. \quad (6.81)$$

At the lower experimental limit of  $Sc = 930$  and  $Re_{\omega} = 0.278 \times 10^6$  [7], the value of  $\Delta$  in view of Eq. (6.81) reduces to  $\Delta = 0.037$ . For the conditions  $Sc = 10,320$  and  $Re_{\omega} = 0.278 \times 10^6$ :  $\Delta = 0.016$ . For  $Sc = 930$  and  $Re_{\omega} = 1.8 \times 10^6$ :  $\Delta = 0.023$ . These values for  $\Delta$  conform to the data obtained by Eq. (6.74) and the upper limiting boundary  $\Delta \leq 0.02$  of the validity of the linear model for the radial velocity.

According to the *models with a constant and variable value of  $\Delta$* , the function  $T^+$  in wall coordinates, defined by the power-law Eq. (2.23) at  $n = n_T$  does not depend on the Reynolds number, which is consistent with the results presented in [48, 49].

Model incorporating Eq. (6.75) results in the profile of  $T^+$  being a function of  $Re_{\omega}$

$$T^+ = (z^+)^n (1 + \alpha^2)^{-n/2} \gamma^{-n} A_c^{-n/2} C_{\Delta^*}^{-n} K_N^{-1} Pr^{2/3} Re_{\omega}^{-0.5(2n^2+n)/(3n+1)}. \quad (6.82)$$

To conclude, a novel methodology for simulations of temperature/concentration profiles for the values of  $Pr$  and  $Sc$  much larger than unity was outlined in this section. *An original integral method enabled evaluating a relative thickness  $\Delta$  of the thermal/diffusion boundary layers that has not been attained by the other investigators. It was demonstrated that the model with a subsiding function  $\Delta(r)$  yields a new summand in the expression for the exponent at the Reynolds number, which determines functional dependence of  $Nu$  or  $Sh$  numbers on the local radius  $r$ .* Consequently, theoretical relations obtained for Nusselt and Sherwood numbers are in a good consistency with the selected empirical equations.

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